

**OPTIMAL ELECTRO-HYDRAULIC  
CONTROL OF MINING BOOMS**

TO MY PARENTS

# OPTIMAL ELECTRO-HYDRAULIC CONTROL OF MINING BOOMS

by

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## SUMMARY

Mining boom rippers are used in the UK coal industry to drive tunnels, which provide ventilation and access to the coalseams. The present tunnelling machines are open-loop controlled; a better positional control can be obtained by introducing feedback, and a microprocessor based system provides automatic operation.

A dual microprocessor system based on the 8085A 8-bit microprocessor has been built and tested on a boom ripper. Poppet valves are used in this system for their ease in interfacing to the microprocessor and their high dirt tolerance. A bang-bang (on-off) control philosophy is adopted to provide the optimal positional control: the valves are fully opened to give the boom the maximum acceleration moving towards the desired position and they will be closed completely to provide the maximum deceleration when a pre-determined criterion has been satisfied.

Two methods in providing the switching signals have been studied: the signals are determined by simply comparing the desired and the current boom positions in one method, or they can be determined by evaluating the switching function which accounts for the valve characteristics and other system properties. Both methods give a positional accuracy limited by the ADC (approximately  $\pm 1$  cm at the cutter end,  $\pm 0.4\%$  of the full range). However, the switching function evaluation method shortens the overall time to change the boom positions at high speed, while increasing the complexity in the circuit and software design.

It has been demonstrated that an APU can increase the speed and simplify the software in handling mathematical operations. An APU board has been developed, although it is not used in the simple comparison method. The APU will be particularly useful if the system is going to compensate for the machine chassis movement when the signals for the machine's position and orientation can be obtained.

Keywords.

Microprocessor Application.  
Mining Boom Ripper Positioning.  
Bang-Bang (On-Off) Control.  
Optimal Switching.

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NOTATION :

The following notations are used throughout the thesis. If they are only used in the local text then the notations will be re-defined.

$A_1$	Effective area of the ram in chamber 1.
$A_2$	Effective area of the ram in chamber 2.
$a$	Area ratio = $\frac{A_2}{A_1}$ .
$B$	Effective external viscous damping of the system.
$b_1$	Effective width of valve 1.
$b_2$	Effective width of valve 2.
$C_{D_1}$	Flow discharge coefficient for valve 1.
$C_{D_2}$	Flow discharge coefficient for valve 2.
$C_L$	Internal leakage coefficient across the ram.
$F$	External applied force to resist positive movement of the ram.
$K$	Effective stiffness of the system other than the hydraulic stiffness.
$L_r$	Reference length.
$\mathcal{L}$	Laplace transform operator.
$M$	Effective inertial mass of the actuator.
$P_s$	System supply pressure.
$P_e$	System exhaust pressure.
$P_1$	Pressure in chamber 1.
$P_2$	Pressure in chamber 2.



Notation (cont)

$P'_1$	Pressure 'external' to chamber 1. $\left\{ \begin{array}{ll} = P_s & \text{when } X_{1v} > 0 \\ = P_e & \text{when } X_{1v} < 0 \end{array} \right.$
$P'_2$	Pressure 'external' to chamber 2. $\left\{ \begin{array}{ll} = P_e & \text{when } X_{2v} > 0 \\ = P_s & \text{when } X_{2v} < 0 \end{array} \right.$
$P_1, P'_1, P_2, P'_2$	Non-dimensional pressures for $P_1, P'_1, P_2, P'_2$
$Q_1$	Fluid flow rate into chamber 1.
$Q_2$	Fluid flow rate into chamber 2.
$Q_r$	Reference fluid flow rate.
$Q_1, Q_2$	Non-dimensional fluid flow rates $Q_1$ and $Q_2$ respectively.
$S_1$	Hydraulic stiffness of the fluid column in chamber 1.
$S_2$	Hydraulic stiffness of the fluid column in chamber 2.
$T$	Time.
$t$	Non-dimensional time.
$T_r$	Reference time.
$t_d$	Non-dimensional time delay for On-Off valves.
$t_c$	Non-dimensional time for valve operation (linear type).
$V_1$	Volume of the fluid between the ram face and the valve in chamber 1.
$V_2$	Volume of the fluid between the ram face and the valve in chamber 2.

Notation (cont)

$V_T$	Total volume of fluid in the circuit when the ram is at its reference position.
$V_{1r}$	Volume of the fluid between the ram face and the valve in chamber 1 when the ram is at its reference position.
$V_{2r}$	Volume of the fluid between the ram face and the valve in chamber 2 when the ram is at its reference position.
$V_{1L}, V_{2L}$	Volume of the fluid in line on side 1 and 2 respectively.
$V_{1C}, V_{2C}$	Clearance volume of the fluid in chamber 1 and 2 respectively.
$X_V$	Maximum valve opening from the close position.
$X_{1V}$	Displacement of valve 1 from its close position.
$X_{2V}$	Displacement of valve 2 from its close position.
$x_{1V}, x_{2V}$	Non-dimensional quantities for $X_{1V}$ and $X_{2V}$ respectively.
$Y$	Ram displacement from its reference position.
$y$	Non-dimensional quantity for $Y$ .
$Y_{st}$	Piston stroke measured between the two extremes.
$\ddot{Y}_O, \dot{Y}_O, Y_O$	Non-dimensional acceleration, velocity and displacement of the ram at the application of the switching signal to the valves.
$\beta$	Bulk modulus of the fluid
$\rho$	Density of the fluid.
$\nu$	Ratio = $\frac{\beta}{P_s}$
$\omega_r$	Natural hydraulic frequency at the reference position

## Notation (cont)

$$\zeta \quad \text{Damping factor} = \frac{BT_r}{2M}$$

$$\xi \quad \text{Constant} = \frac{1}{Q_r} C_{D_1} b_1 X_v \sqrt{\frac{P_s}{\rho}}$$

## Subscripts

The following subscripts normally have these meanings unless specifically defined.

s	{ System supply when used with pressure. Spool valve when used with displacement.
r	Reference.
T	Total.
1	Upstream or chamber without piston rod.
2	Downstream or chamber with piston rod.
L	Leakage.
C	Clearance.
e	{ System exhaust when used with pressure. Equilibrium when used with other quantities.
v	Valve.
st	Stroke.

NOTE ON THE LAYOUT OF THESIS

All figures and tables are included in the text to facilitate reference to them. They are put on the pages facing the text making reference to them whenever possible. Except that the figures referred to in Section 3.8 are grouped together and placed at the end of Chapter 3.

**CHAPTER 1**

**INTRODUCTION**

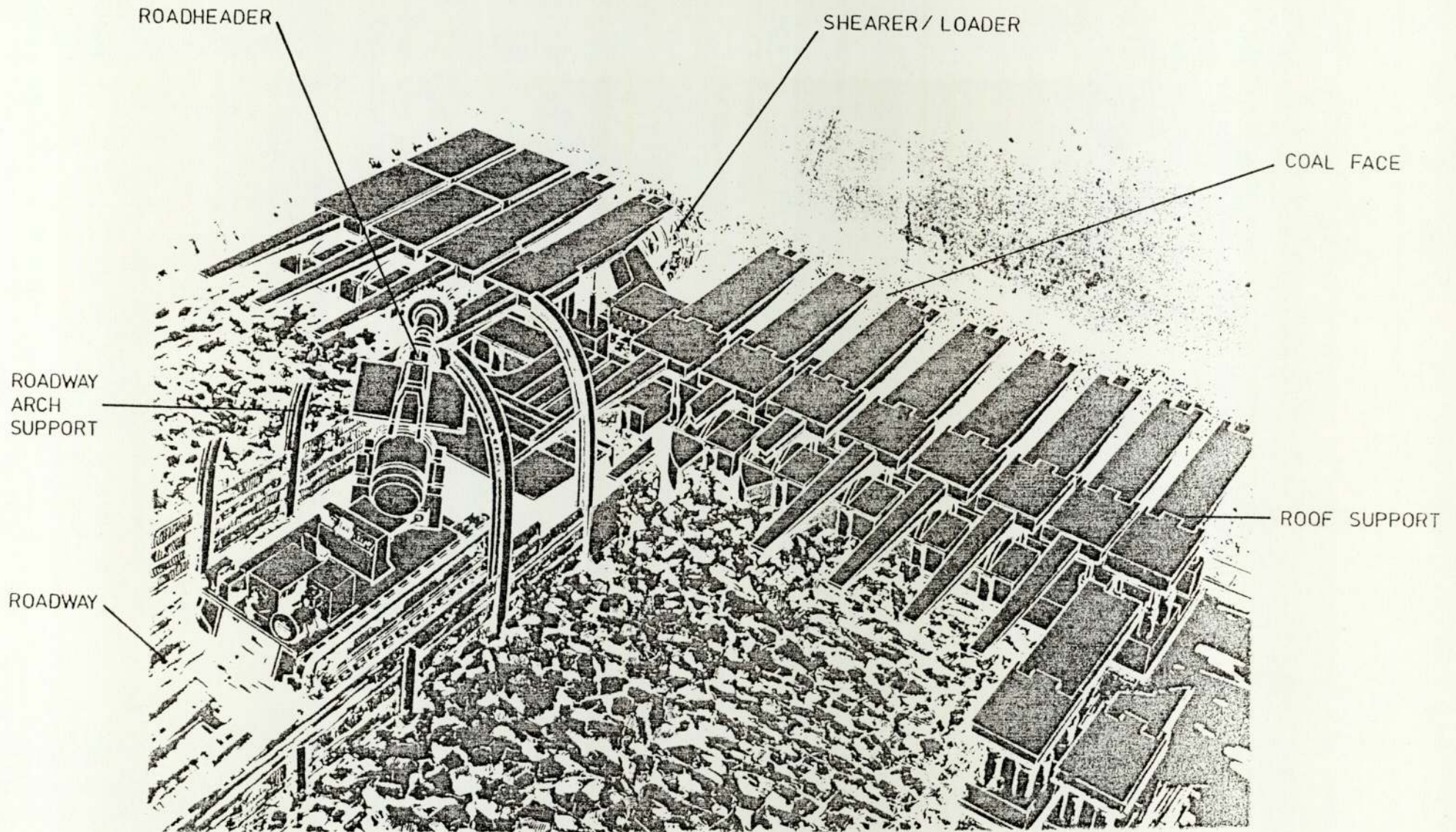


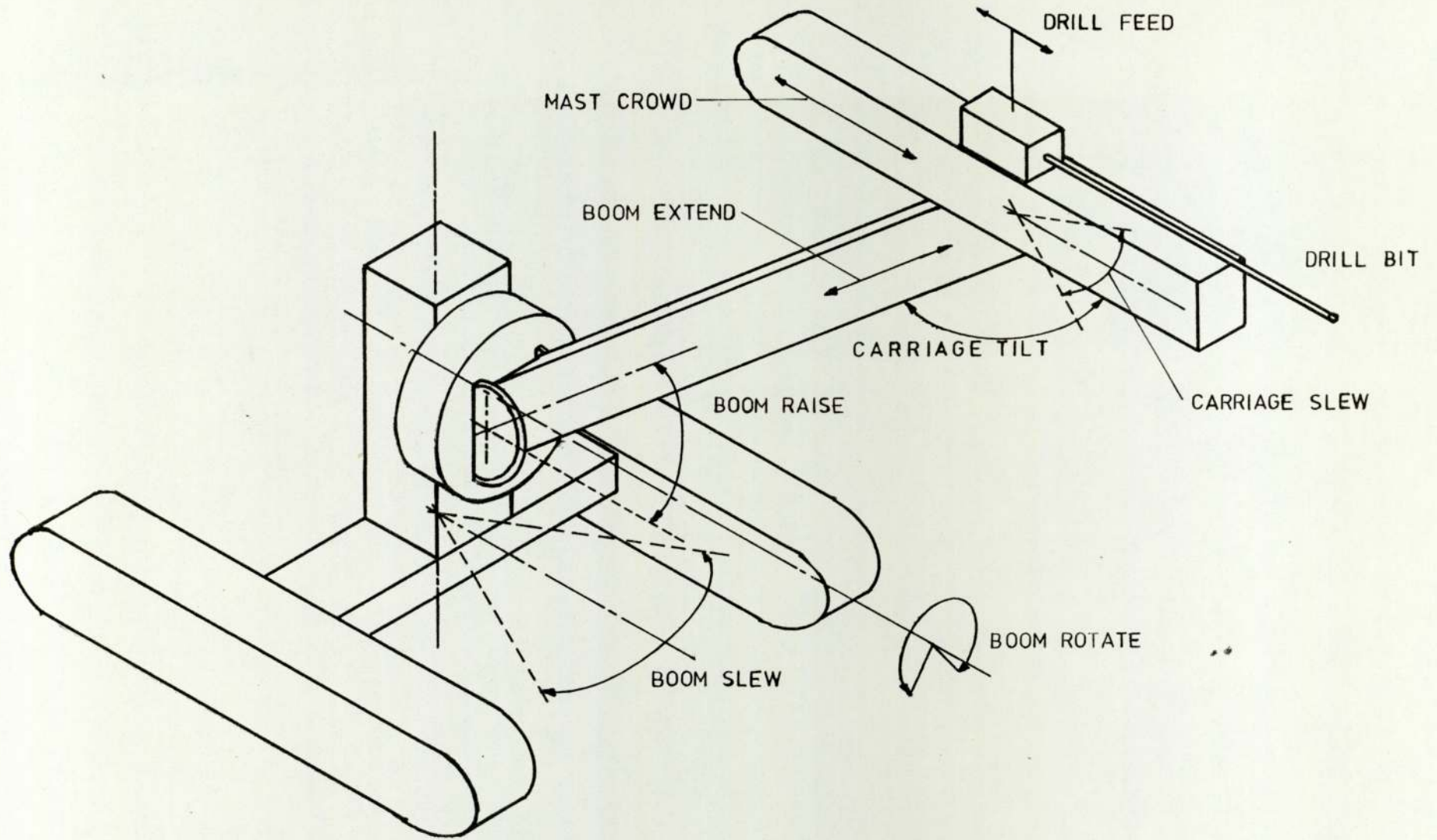
Figure 1.1 Artistic Illustration of the Mining Operations near/at the Coal Face.

## 1.1. COAL MINING OPERATIONS

Coal mining is one of the traditional industries in the United Kingdom. After centuries of development, it has attained a high level of mechanisation<sup>[1]</sup>. Nowadays, electro-hydraulic machineries are heavily used in this industry to increase the output from coalmines and the efficiency<sup>[2]</sup>. Longwall mining is the system used in the UK, some European countries and China; while the room-and-pillar system is generally used in America, Australia and South Africa<sup>[3]</sup>. Figure 1.1. shows an artistic illustration of the typical coal mining operations in a longwall system near and at the coal face.

Essential operations in longwall mining can be briefly described below. Roadways are built at each end of the coal face, which is normally about 200 metres long, to give access to the coal seam, to facilitate the transportation of coal, machineries and personnel and to provide ventilation. Coal is cut along the seam by electrically powered cutters such as shearer-loaders, and is transported through the roadway. The cavity created by coal extraction along the seam is supported by chocks (hydraulic roof supports) which will be advanced after the coal face has been cut.

The advance in technology has given rise to some degree of automation in the coal industry. The National Coal Board (NCB) has developed a minicomputer system known as Mine Operating System (MINOS) to monitor and control some mining operations<sup>[4]</sup>. MINOS is used in some mines to automatically



3

FIG. 1.2 Diagram Showing The Seven Degrees Of Freedom Of A Drilling Boom Used In Coalmining



monitor and control belt conveyors and bunker systems for coal transport, to monitor and record the mine environment for ventilation, to monitor and record the performance of a coal face, to operate fixed plants such as main pumps and fans, and to provide useful management information.

## 1.2. TUNNELLING

Roadways are made by first driving tunnels then by erecting arches as roadway supports. Setting of arches will be easier if a better profile of the tunnel is obtained. In the UK, tunnels are driven either by shotfiring or by direct cutting of rocks. Mechanisation of this process is highly desirable, therefore ripping machines, which are also known as roadheaders, were introduced to the UK coal industry in 1961. The high speed tunnelling techniques in civil engineering have some influences upon the development and design of ripping machines used in the coal industry<sup>[5]</sup>.

The traditional way to excavate tunnels in UK coalmines is the shotfiring method. In this method, a pattern of holes are drilled for placing explosives and then blasted. Conventionally, this drilling of pattern is a two-man operation: one operator stands at the drilling end and indicates the position of the drilling head to the other, who is operating the control valves for the hydraulic actuators of the drilling boom. Electrohydraulic remote control valves are developed to simplify this operation. However, a drilling machine used in coalmining normally has seven degrees of freedom (Figure 1.2 ) and is operated

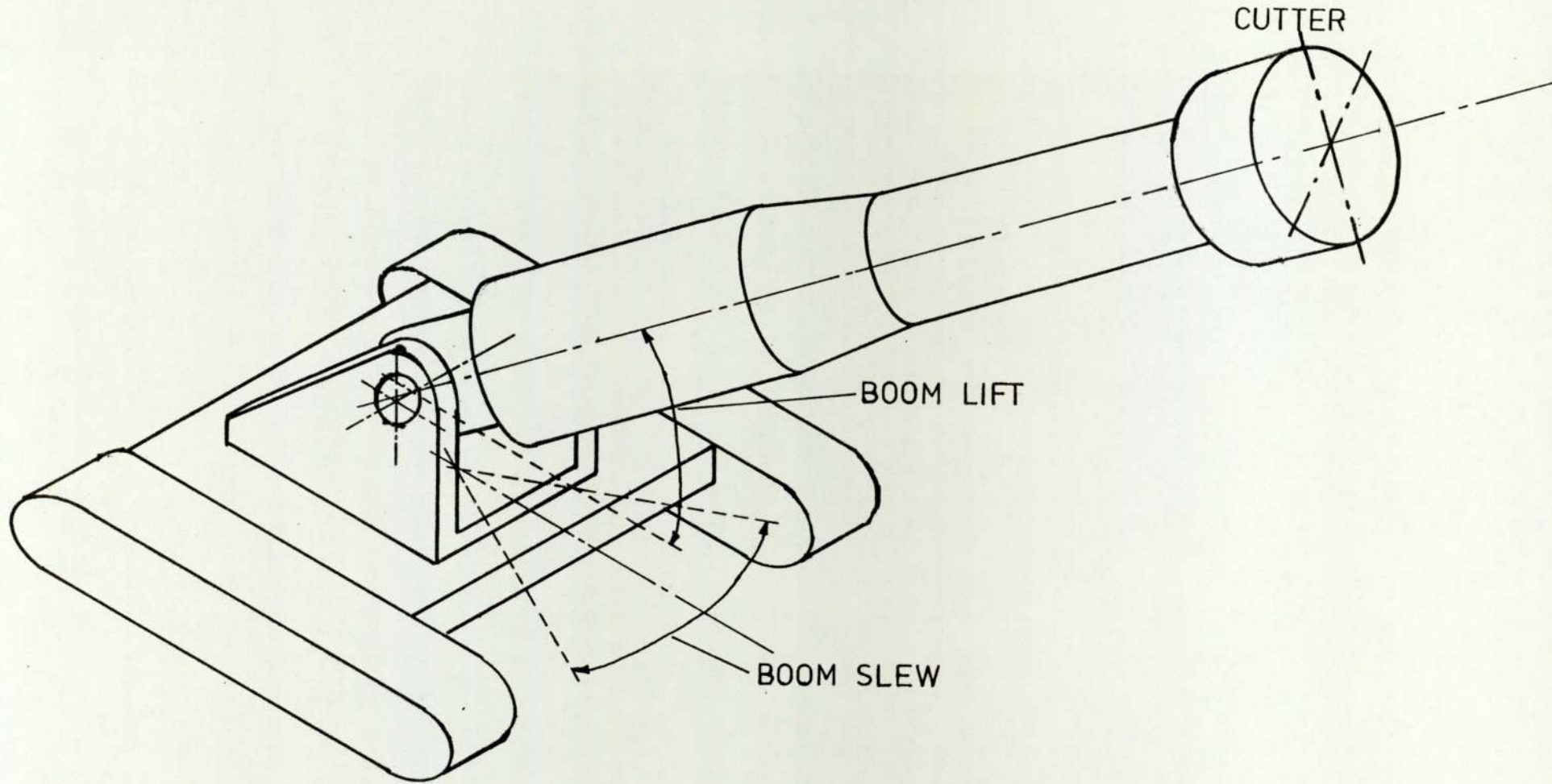


FIG. 1.3 Diagram Showing The Two Degrees Of Freedom Of A Boom Ripper

manually using open-loop control, thus a drilling operation demands high skill and a considerable amount of training for efficient operation. This method is highly labour-intensive and continuous processing is impossible because the machine has to be retracted to permit the blasting of rocks.

Direct cutting of rocks is considered to be an alternative to drilling and blasting. A ripper with a hydraulically powered impact unit mounted at the end of a boom similar to that of a drilling machine has been developed<sup>[6]</sup>. This impact type ripper can be used to drive the tunnel and trim the profile. This machine provides continuous operation in driving the tunnels, but a large amount of time is required to manipulate the impact breaker boom because of the large number of movements of the hydraulic actuators.

There is another type of roadheader used in UK coalmines, the pick type ripper<sup>[7]</sup>. This machine has a cutting head comprising of picks mounted and rotated at the end of a boom, which is installed on a machine chasis. The movement of this boom is reduced to have only lifting and slewing, (Figure 1.3). This arrangement greatly reduces the time required to manipulate the boom. Early pick type machines are restricted to operation on rocks of about  $70 \text{ MN/m}^2$  compressive stress.

Shield tunnelling used in civil engineering leads to the development of shielded machines. These machines are being tested in some mines, Gascoigne Wood being one of them.

Full face tunnel boring machines are also being developed. However, these machines suffer from many disadvantages including high cost, long installation and dismantling times and inflexibility in coping with variable ground conditions. A heavy-duty boom ripper is being developed so that harder rocks can be cut. This in turn increases the size and weight of the machine. Cutting picks assisted by high pressure water jets are used in heavy-duty rippers to improve their cutting ability<sup>[8]</sup>.

Although the main aim in coal mining is to produce coal, a speedy operation in tunnelling will facilitate transporting materials and machineries, and hence increase the coal production. Therefore, in the UK coal industry, the MRDE (Mining Research and Development Establishment) of the NCB and many mining machinery manufacturers are trying to develop high performance tunnelling machines and improve the performance of existing machines.

The average conditions of strata encountered in UK coal-mines make it more economical to use boom rippers in most areas. While shotfiring is still used in areas with hard rocks, e.g. Six Bells Colliery in South Wales. Shielded machines and full face boring machines may be used in areas where high flexibility is not required. Therefore, the aim of this project is to develop a better positional control system for the existing boom rippers. This will result in deskilling the manipulation of a ripping boom and enable further development in automation of the tunnelling process.

It can be seen in the figures (Figures 1.2. and 1.3) that a boom ripper has fewer degrees of freedom than a drilling boom. This is because a boom ripper is used to cut a predetermined profile of simple shape; while a drilling boom is required to drill a prescribed pattern which demands positioning and orientating the drilling bit mounted on the machine. Although the control system described in this thesis has only two degrees of freedom, it can easily be modified to include more degrees of freedom by considering the machine geometry, and hence the same principle can be applied to the positional control of a drilling boom.

### 1.3. STIFFNESS OF A MINING BOOM RIPPER

If the stiffness of a mining boom ripper is defined as the ratio of change in force to the change in displacement of the working end in the direction of the applying force, then better position stability of a boom ripper will be achieved if its stiffness is higher. Although this stiffness depends upon both structural and hydraulic stiffnesses of the ripper, the hydraulic one has a greater effect because the boom is supported by hydraulic actuators. Hence a study has been carried out for the hydraulic stiffness of a boom ripper (Appendix D).

It was found that the stiffness of a system will be higher if the supporting hydraulic actuator is connected to the boom closer to the cutter for a given actuator cross-sectional area. However, the stiffness may be kept within a range if the cross-sectional area of the hydraulic actuator is adjusted accordingly.

Unity positional feedback to the hydraulic actuator valves will have a significant improvement on the stiffness for excitation forces of high frequencies, while the stiffness cannot be much increased if the disturbing force is oscillating at a low frequency.

If the maximum force is the force acting tangentially to the boom<sup>[7]</sup>, and components of the resultant force not acting in the direction of the actuator are resisted by another means, then the boom will have a greater stiffness when the cutter position is higher. This is because the component of the resultant force in the direction of the trunk lifting actuator is less when it is more vertical. Similarly, the situation will be the worst when the boom is in a horizontal position cutting an extremely hard rock and fails to break it. If this happens, the machine will 'climb' over the rock and cause a bounce effect. Vertical supports at far ends will help stabilising the machine's sliding movement on the ground.

Although the performance of a boom will be improved by increasing its stiffness, a different structure and arrangement will be required. Since the aim of this project is to improve the performance of existing boom rippers with minimum modification, alternation of boom arrangement to give a better stiffness is not considered. Alternatively, a better positional control of boom ripper is studied.

As reported in this thesis, the valves of the hydraulic actuator are closed simultaneously based on the switching

function. However, it can be shown that a delay in closing the supply valve from the closure of the exhaust valve will improve the effective stiffness of a boom. This is because that cavitation in the supply chamber, caused by rapid closure of the valve, can be avoided, and hence the actuator will have less oscillation before it rests. This is being studied in a parallel project by J H Knight in the Mechanical Engineering Department at the University of Aston.

#### 1.4. CONTROL PRINCIPLES

At present, most of the working time of a boom ripper is incurred in manipulating the boom. Therefore the overall performance of a roadheader will be improved by shortening the time required to position its cutter. A possible way to achieve this is introducing closed-loop control to the system. Since the movement of a boom is caused by the associated hydraulic actuators, closed-loop control will be developed for these actuators.

Hydraulic actuators are basically integrators<sup>[9]</sup> and the displacement of their rams depend on the fluid flow through the valves, hence one of the possible ways to control these actuators is to control their valve displacements. The conventional system used in this type of control is the hydraulic servomechanism, therefore the basic mathematical model employed in this closed-loop design of the ripping boom is that used for hydraulic servos. Analysis of the model will help developing the

positional control strategy (the switching function) for the mining boom.

Safety in coalmine operations demands the use of fire-resistant fluid in hydraulic machinery. There is a trend to use 5/95 diluted emulsion in UK coalmining machinery because of its fire-resistant property, low cost and non-toxicity. The low viscosity of this fluid increases the leakage of sliding type valves, therefore traditional spool valves are not suitable and poppet valves are being developed. The conventional method in proportional control of the spool valves may not be applied because the displacement of a poppet valve is difficult to control using direct proportional control. But the control can be done by using pulse width modulation (PWM) to give an equivalent opening. PWM demands frequent operation of the valves and may cause fatigue problems.

The operation of poppet valves is basically an on-off mode, and in the current project, PWM is not considered but a 'bang-bang' philosophy is employed so that maximum power is used to accelerate and decelerate the hydraulic ram. To achieve a good accuracy, an optimal control strategy is adopted. Switching functions have been developed so that equilibrium can be attained by one switching from acceleration to deceleration, i.e., to close the valves at a point to give a final position within the neighbourhood of the desired position.



### 1.5. MICROPROCESSOR CONTROL

Optimal control may need a large number of logical decisions, and the conventional way of achieving these operations is the use of expensive computers in which mobility is also restricted. An alternative is to use fluidic elements which are again bulky and expensive. Microprocessors are becoming more commonly used in modern control systems because of their compactness and relatively cheap hardware cost. Development in micro-electronics increases the reliability and speed of microprocessor systems and advance in the large scale integration (LSI) technology further reduces the physical size of a microprocessor system.

Microtechnology has a rapid development in the mining industry for the measurement, information and control of mining operations<sup>[1]</sup>. Therefore it is feasible to develop and use microprocessor control for the current project. The requirements for speed of data handling in capturing and analysing the input and feedback, suggest the use of a multi-microprocessor control system. In this control system, two microprocessors running in parallel are proposed where one is used to capture the input and feedback data so that 'updated' information can be provided for the use of the other microprocessor system, whilst the second is used to analyse the data so that appropriate control signals can be sent to the valves. A special processor, the arithmetic processing unit (APU), is employed to further increase the speed of operation, so

that more sophisticated mathematical analysis can be performed faster.

The microprocessor control system takes care of the appropriate hydraulic actuator movements in positioning the boom cutter, and this will in effect deskill the manipulation of the boom, allowing the operator to concentrate on the quality of the profile. The proposed control strategy can be modified to suit all mining booms including the drilling boom which has seven degrees of freedom. Although the proposed control system is semi-automatic, full automation may be achieved if the necessary sensors have been developed, e.g. a machine position sensor will enable auto-correction of the cutter position.

#### 1.6. ORGANISATION OF THESIS

The current chapter is an introduction to this project and gives a brief description of the coalmining operations incurred in the UK and the importance of positional control in roadheaders. Since the control principles and models employed to develop the boom positional control system are those used for servomechanisms, a brief review of previous researches on this subject is presented in the following chapter. In Chapter 2, a brief review on optimal control is also given, it will be seen that optimal control is normally done by controlling the closure of valves rather than the opening, i.e. control the deceleration rather than the acceleration. Switching functions are derived

for the linear actuators with simple valves in Chapter 3. Since it is desired to employ microprocessor control, arithmetic processing of data is an essential operation in the control system. A decision on using either hardware or software for this process has to be made, a simple comparison between hardware and software arithmetic processing is presented in Chapter 4 to help realising the choice. The main control algorithm and structure of the control programs are given in Chapter 5. The circuits for the microprocessor and associated boards for the main control system are described in Chapter 6. The control algorithm has been tested and the results are presented in Chapter 7. Chapter 8 is the conclusions and recommendations based upon the findings of this project. Details and listings of the programmes may be found in the appendices.

**CHAPTER      2**

**BRIEF   REVIEW   OF   PAST   WORK**

**DONE   IN   APPLYING   THE   CONTROL**

**THEORIES   TO   A   HYDRAULIC   SYSTEM**

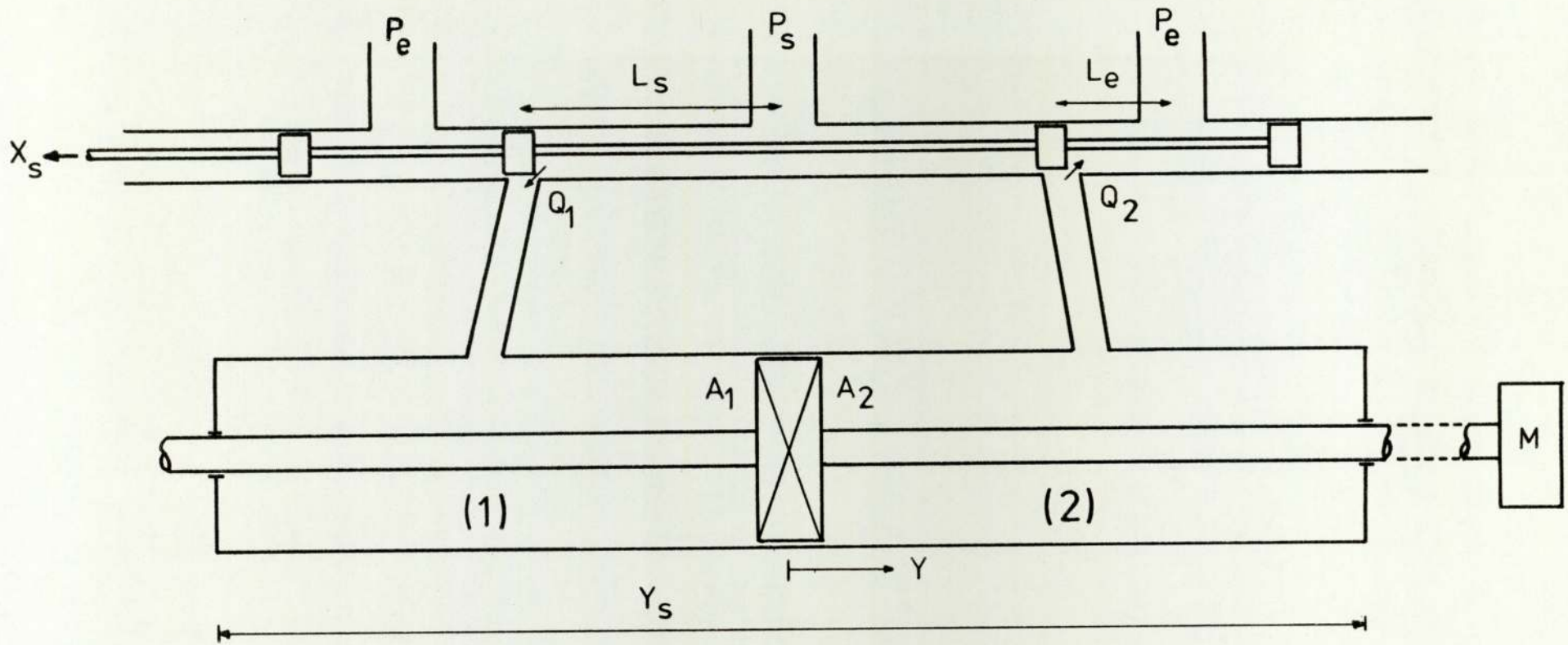


FIG. 2.1 Basic Form Of a Valve Controlled Hydraulic Actuator

## 2.1. CONVENTIONAL CONTROL

The movement of a hydraulic actuator is commonly controlled by the position of valves and the open-loop system is referred to as a hydraulic relay, but called a servo-mechanism when there is a feedback. Traditionally, the valves used in hydraulic servomechanisms are spool valves which are either mechanically or electrically operated. These valves normally have a movement proportional to the input signal and so control both the magnitude and direction of the fluid flow passing through them. The basic form of a hydraulic actuator with spool valves can be represented by the diagram in Figure 2.1 , and Figure 2.2 shows a block diagram for a hydraulic servo.

A hydraulic relay is an integrating device when used in a position control system because the output displacement of the actuator will be directly proportional to the time integral under ideal conditions<sup>[9]</sup>. Mathematical modelling of such a hydraulic system will result in a set of nonlinear differential equations which is difficult to analyse, so analysis based on linearisation is commonly used <sup>[10,11]</sup>.

The flow through the valve ports is turbulent and its rate may be assumed to be proportional to the product of the valve opening area and the square root of the pressure drop across the valve as in the case of flow through an orifice<sup>[10]</sup>. This results in the familiar flow equation (2.1).

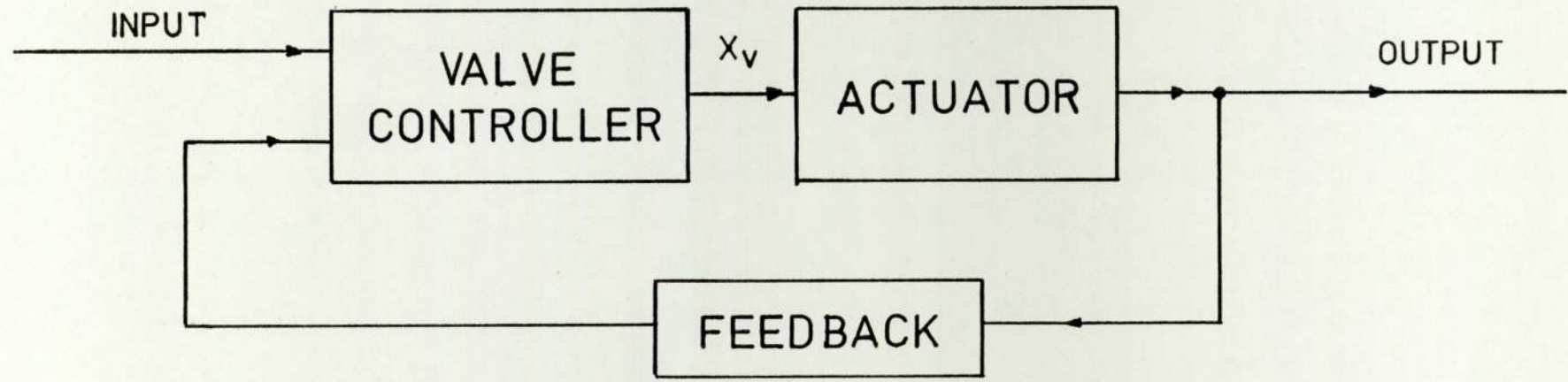


FIG. 2.2 Block Diagram Of A Basic Hydraulic Servo

$$Q = C_d a_v \sqrt{\frac{2\Delta p}{\rho}} \quad (2.1)$$

where  $Q$  = fluid flow rate through the valve.

$C_d$  = flow discharge coefficient of the valve.

$a_v$  = valve opening area.

$\Delta p$  = pressure drop across the valve.

$\rho$  = fluid density.

Analysis of the system is done after combining this flow equation (2.1), the flow equations within the actuator chambers, the equation for the dynamics of the system and the equation for feedback if any.

Since valves are used extensively in hydraulic servo-mechanisms, their performance, behaviour and patterns of stability have been studied in the past. Lee and Blackburn found that there were steady axial and transient flow forces acting on a 4-way spool valve<sup>[12]</sup>. They also demonstrated that the stability of a piston type control valve would depend upon the damping lengths,  $L_s$  and  $L_e$  as defined in Figure 2.1. The length  $L_s$  is normally made greater than  $L_e$  to give a small positive damping on the spool valve to eliminate instability. Taft and Twill had similar findings for a 3-way underlapped spool valve<sup>[13]</sup>. Harpur used a linearised analysis to show that the oil compressibility could cause instability<sup>[11]</sup>.

Notton and Turnbull showed with the aid of describing functions that a spool valve could have continuous



oscillation if the damping in the spool was not sufficient<sup>[14]</sup>. They also used a graphical method to predict the frequency and amplitude of this oscillation and suggested that the stability could be improved by complete reaction-force compensated spool valves. Foster and Kulkarni used a linearisation technique and the approximate method of Kryloff and Bogoliuboff to study the spool oscillations and found that the frequency of self-excited oscillation of a spool might be controlled by varying the supply pressure<sup>[15]</sup>. They also showed that the amplitudes of these oscillations could be controlled by changing the fluid energy losses in the systems and that this type of valve was not suitable for operating as an oscillator because of the oil compressibility and resonant frequency effects at high pressures. Burton, Ukrainetz and Nikiforuk studied an inertially loaded valve with the describing function technique and found that the existence and stability of the limit cycle oscillations could be predicted as a function of the system parameters<sup>[16]</sup>.

Poppet valves attract engineers' attention because of the simplicity in manufacturing, less leakage for fluid with low viscosity (since there is no sliding seal) and less sensitivity to dirt. Poppets are often used as pressure regulators and relief valves in the past. Stone showed that the normal force acting on a poppet face was due to the pressure distribution while the tangential force was viscous in nature. He also reported that the stability could be improved by eliminating the flow forces and that

the downstream configuration had a larger effect on these forces<sup>[17]</sup>. McCloy and Martin found that the flow through a poppet valve was parallel to the poppet face<sup>[18]</sup>. Mansfeld investigated electro-hydraulic actuator control using fast switching on-off valves<sup>[19]</sup>. He found that pulse width modulation (PWM) of these valves could be used to achieve an unsteady fluid flow control, and that PWM might be used for the time-optimal control of electro-hydraulic servomechanisms.

Researches had been carried out for the performance of a hydraulic actuator by numerous people in the past. In the early studies, neglecting the oil compressibility and assuming symmetry were commonly adopted for analysis. Turnbull proposed a graphical method based on the isoclines to find the step response of a symmetrical hydraulic servomechanism<sup>[20]</sup>. He neglected the oil compressibility and pointed out that the load effect was to increase the response time of the servo. McCloy developed a graphical method to study the step response of a closed-loop system<sup>[21]</sup>. He also neglected the oil compressibility but considered the effects of frictional forces, inertial and spring loads. Butler provided analytical and graphical solutions for various kinds of inputs: step, ramp and sinusoidal<sup>[9]</sup>. He neglected the compressibility effects and derived the solutions under certain assumptions for special cases. His assumptions were (i) incompressible fluid and (ii) no leakage and no dynamic friction (viscous damping effects had been neglected). The graphical solutions were presented in the velocity-displacement phase plane.

Cook and Heaps presented a series analysis method to study a closed-loop system with unity feedback<sup>[22]</sup>. Their analysis was based on the assumption that the oil compressibility effect could be ignored. They also investigated systems with velocity feedforward, velocity and error feedbacks. In this method, they expressed the error term in a series and compared the coefficients of like terms.

Royle derived the response equations of a symmetrical actuator for sinusoidal inputs<sup>[23]</sup>. He considered the oil compressibility but neglected the leakage across the ram and found an expression for the ram velocity in terms of the load factor and the input frequency. The load factor is a measure of the inertial load on the system. He presented a curve for the relation between the maximum load factor without cavitation and the driving frequency. As the driving frequency was increased to the actuator's natural frequency, the static maximum load factor was  $1/\sqrt{2}$ . This load factor could be increased by introducing an exhaust valve lap to avoid or delay cavitation.

Lambert and Davies reported that the oil compressibility accounted for damped oscillatory motion<sup>[24,25]</sup>. The damping ratio is a function of leakage if no other forms of damping are present. However, the most important stabilising effect in most physical systems is the damping effect caused by Coulomb friction. They also derived a method to evaluate the equivalent damping ratio for a sinusoidal oscillation. They derived an open-loop transfer

function assuming that the movement of the ram is restricted to a small value so that the (displacement x area) term could be neglected, the equilibrium position was taken to be the position at mid-stroke. Their analysis was done for a closed-loop system with unity feedback.

Wang showed that an electro-hydraulic servomechanism could be represented by reasonably simple and accurate mathematical models with suitable assumptions<sup>[26]</sup>. He demonstrated this with an investigation into the response of a symmetrical actuator to step and pulsed inputs. His method can also be used for time-domain design and near time-optimal design of electro-hydraulic servomechanisms.

Martin had investigated a hydraulic servo with unsymmetrical fluid volume conditions using a linearisation (small perturbation) technique. He found that the marginal stability gain was the least for symmetrical oil volumes, therefore asymmetric oil volumes would give a better stability with faster response and enhanced damping<sup>[27]</sup>.

Montgomery and Lichtarowicz presented a theoretical investigation into the effects of asymmetrical valve lap conditions on the open-loop performance of a valve-controlled hydraulic symmetrical actuator<sup>[28]</sup>. Their studies were carried out in non-dimensional terms and the valves were driven with a sinusoidal excitation. It was reported that valve lap conditions would have a larger effect at low driving frequencies, while these lap effects would decrease when the driving frequency was near and

above the natural frequency of the oil column of the cylinder. Combination of exhaust overlap and supply underlap will give a higher peak pressure and decrease the amplitude of oscillation, and hence reduce the cavitation threshold. Inertial load effect will be significant when the valve-actuator system is used as a vertical oscillator. In these cases, an extra volume term in the compressibility part of the flow equations will help to stabilise the system and stop the ram shifting.

Cavitation reduces the efficiency of a hydraulic servo and may cause undesired damage and instability, therefore work has been done on this subject<sup>[9,23]</sup>. Cavitation may cause high pressure within the chamber and severe attenuation of the ram displacement. Cavitation may occur when the inlet flow of fluid into a chamber cannot fill up the cavity formed, e.g. there may be cavitation caused by the rapid closure of valves. Wang and Ma had derived the approximate cavitation conditions for a valve controlled symmetrical hydraulic actuator under sinusoidal and discontinuous inputs by neglecting the leakage across the ram and the oil compressibility<sup>[29]</sup>.

McCloy reported that cavitation had larger influence on the pressure dynamic response than on the displacement and ram dynamic responses, that damage caused by cavitation in a cylinder was less than that due to orifice cavitation<sup>[30,31]</sup>. Therefore cavitation mode operation of hydraulic servos might be permitted. He also showed that simplified models, which did not take cavitation into account, might adequately

describe the dynamic performance of a system. This technique eases the analytical studies considerably. He showed that an increase in the natural frequency of an actuator was minimised by cavitation while the linear analysis predicted that the frequency would increase as the piston moved towards the end of the cylinder. He also pointed out that high pressures were usually developed in a system with on-off valves. McCloy and Martin investigated the generation of high pressures in an asymmetrical system due to the rapid valve closure and developed a damping using the linearised analysis. Their model assumes no damping, thus an energy conserved system has been modelled. In their paper, they also studied the effects of cavitation and derived an expression for the cavitation conditions<sup>[32]</sup>.

## 2.2. OPTIMAL CONTROL BY CLOSING OF VALVES

Many engineers and researchers had studied optimal control to minimise the response time, overshoot or undershoot within the limits of a system. Oldenburger developed the optimum control transient for the aircraft engine-propellor systems and reported his findings in 1957<sup>[33]</sup>. He also developed some optimal control switching functions for an electrical control system with linear controlling elements<sup>[34]</sup>.

Bang-bang controller was shown to be the optimal if the control variables were linear<sup>[35]</sup>. A bang-bang controller is optimal because the control always operates at either

the upper or the lower limit. This bang-bang strategy is also adopted in designing the optimal electro-hydraulic servomechanisms. However, literature on the optimal control of an electro-hydraulic servo is very limited.

Wang used a simplified model to develop an optimal controller for an electro-hydraulic servo<sup>[36]</sup>. In his system, the valves are opened fully until a predetermined position has been reached, then the valves are closed under control to avoid cavitation. Wang avoided cavitation because he wanted to minimise the damage caused by the actuator cavitation and to use simplified mathematical models.

Davies developed Wang's method further, by not only maintaining the non-cavitating conditions but also introducing a transition stage<sup>[37]</sup>. Davies first fully opened the valves to give the maximum acceleration, and then allowed the system to enter a transition zone by fully closing the valves until a point when cavitation was about to occur. The valves were again opened followed by controlled valve closure to ensure maximum non-cavitating deceleration.

In Wang and Davies models, the leakage across the ram was assumed to be negligible and the oil compressibility effect has been ignored in the regions where the pressure changes were low.

However, many authors studied the optimal controllers operating under cavitation conditions<sup>[38,39,40]</sup>. Since

it had been reported that the mechanical damage caused by the actuator cavitation was less severe than that due to the orifice cavitation, the more complexed sub-controllers, which were required to avoid cavitation, might be simplified. Wilson, Nikiforuk and Lepp developed a graphical method to find the transient response of a time-optimised hydraulic servo operating under cavitation conditions<sup>[38]</sup>.

Armstrong and McCloy had developed an optimal controller with an incompressible, symmetrical and leakage-free model. Then they constructed a sub-optimal controller aiming to give a zero positional error. In designing this controller, they took the oil compressibility into account. They demonstrated that a lightly damped servo would use the sub-optimal controller, while the simple optimal controller was quite adequate for a heavily damped servo. They also suggested that the system performance could be improved by using either a time-optimal controller developed using a more accurate model or a more complex sub-optimal controller<sup>[39]</sup>. They had developed some optimal on-off controllers and dual-mode controllers. They argued that although the cavitating conditions might be permitted, a better performance would be achieved if there was no cavitation<sup>[40]</sup>. They suggested the use of a correction function evaluated from the steady-state errors to improve the performance of the hydraulic servo.



### 2.3. APPLICATION TO THE CURRENT PROJECT

In order to study the system behaviour of the hydraulic actuator, Conway's mathematical model<sup>[10]</sup> of the flow through a valve is adopted, and all the equations are linearised by series expansion. The control valves used will be poppet valves, which can be treated as a single spool valve, if the exhaust and supply valves are operating simultaneously. Therefore, McCloy's finding for the flow pattern through a poppet valve<sup>[18]</sup> is applied to evaluate the equivalent flow area used in the equation (2.1).

The performance of the valves, which are being studied in a parallel project to this one by J H Knight, is not investigated in this thesis. However, the valves are assumed to be simple valves, and the switching functions have been derived for pure on-off valves, on-off valves with time delay and 'linearly' operated valves.

The criterion for the asymmetric actuator cavitation due to the rapid closure of valves<sup>[32]</sup> is used to check the possibility of cavitation in this control system. A simplified system model is adopted in which only inertial, viscous damping and external loads are assumed. The hydraulic actuators used have no experimentally measureable internal leakage, therefore the leakage across the ram has been neglected in deriving the switching functions. The optimal controllers, mentioned in the above literatures, control the valve openings during decelerations<sup>[36,37,38,39,40]</sup>. However, in this project, the proposed control is done by

controlling the moment to initiate the valve switching instead of the way it switches. This is because of the difficulty in controlling the flow area of poppet valves and the requirement for less complex controllers. Since actuator cavitation may be allowed<sup>[31,38]</sup>, this control strategy will ease the analysis and design of the controller.

The use of a correction function to improve the accuracy of the actuator positioning has been considered. However, the steady state errors, predicted by the computer simulations, are so small that the complexity in using the correction function is not justified.

**CHAPTER      3**

**DERIVATION OF THE  
SWITCHING FUNCTIONS**

### 3.1. Introduction

Hydraulic mechanisms are commonly used in positional control with closed-loop arrangements. However, in some applications, conventional feedback techniques may not be effective then alternative methods should be used.

An improved form of control for a ripping boom can be obtained by using a microprocessor based control system. The microprocessor, being a digital device, requires all analogue feedback signals to be converted into digital forms. Output signals from the microprocessor control system will also be in 'ON-OFF' form and hydraulic or mechanical hardware capable of operating directly from these signals would have advantages.

The change of the boom position can be achieved in a minimum time if the maximum available power is used. An 'ON-OFF' mechanism can serve this purpose by driving the ram at the maximum speed, then braking it with the maximum retardation. In fact, the optimal control theory shows that the time used will be a minimum if the valves of a hydraulic actuator are closed fully when the ram has reached its desired position, provided that the fluid is incompressible<sup>[40]</sup>.

Because of the fluid compressibility, the valves cannot be switched off when the ram is at the desired position, they should be closed at some other position so that the ram rests at the required position. This requires the setting up of a switching function to account for the fluid

Pilot Operated Poppet Valves

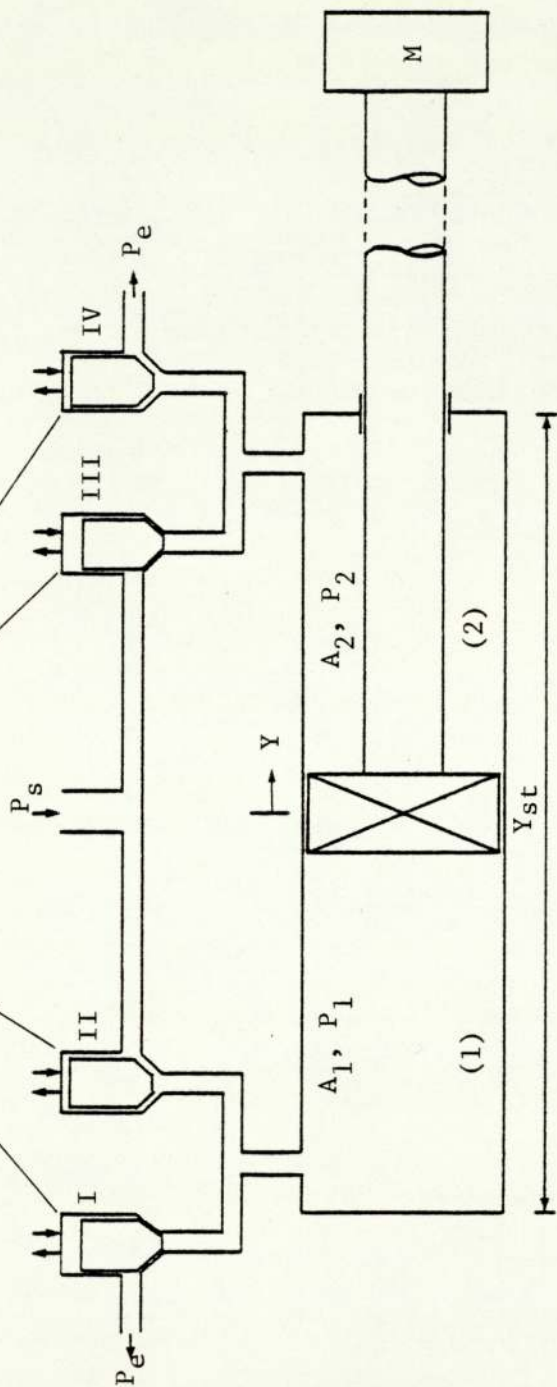


Fig 3.1 A Hydraulic Actuator with Poppet Valves

compressibility. This switching function will depend not only on the desired position but also upon the velocity and acceleration of the ram at switching as well as some other physical properties of the system. However, the coefficients associated with the switching function will remain the same for a particular system, therefore they have to be evaluated only once.

### 3.2 Mathematical Model

The basic form of a hydraulic actuator used in the current project is shown in Figure 3.1. The poppet valves (I) and (IV) are connected to the system exhaust, while the poppets (II) and (III) are connected to the system supply. In this configuration, the poppets are operated in pairs: (I) and (III) or (II) and (IV). A brief description of the hydraulic system for the boom ripper under test is given in Appendix I. In this study, the poppet valve pairs will be operated simultaneously, they can be treated as a spool valve without any lapping as shown in Figure 2.1. The behaviour of this hydraulic actuator can then be described by the set of differential equations (3.1) to (3.5).

Flow

Valves:  $Q_1 = C_{D1} b_1 |X_{1V}| \sqrt{\frac{2}{\rho} |P'_1 - P_1|} \text{Sign} (P'_1 - P_1)$  (3.1)

$$Q_2 = C_{D2} b_2 |X_{2V}| \sqrt{\frac{2}{\rho} |P'_2 - P_2|} \text{Sign} (P'_2 - P_2)$$
 (3.2)

where  $Q_1$  = Fluid flow rate through the valve at chamber 1.

$Q_2$  = Fluid flow rate through the valve at chamber 2.

$C_{D1}$  = Flow discharge coefficient for the valve at chamber 1.

$C_{D2}$  = Flow discharge coefficient for the valve at chamber 2.

$b_1$  = Width of the valve at chamber 1.

$b_2$  = Width of the valve at chamber 2.

$X_{1V}$  = Opening of the valve at chamber 1.

$X_{2V}$  = Opening of the valve at chamber 2.

$P'_1$  = Pressure 'external' to chamber 1.  $\begin{cases} = P_s & \text{when } X_{1V} \geq 0 \\ = P_e & \text{when } X_{1V} < 0 \end{cases}$

$P'_2$  = Pressure 'external' to chamber 2.  $\begin{cases} = P_e & \text{when } X_{2V} \geq 0 \\ = P_s & \text{when } X_{2V} < 0 \end{cases}$

$P_1$  = Pressure in chamber 1.

$P_2$  = Pressure in chamber 2.

$\rho$  = Fluid density.

The signum functions are present to account for any reverse flow. The flow is considered to be positive when it flows into the chamber and the displacement is positive when the volume in chamber 2 is being decreased.

Chambers:  $Q_1 = A_1 \frac{dY}{dT} + C_L (P_1 - P_2) + \frac{V_1}{\beta} \frac{dP_1}{dT}$  (3.3)

$$Q_2 = A_2 \frac{dY}{dT} + C_L (P_2 - P_1) + \frac{V_2}{\beta} \frac{dP_2}{dT}$$
 (3.4)

where

$A_1$  = Sectional area of the piston face in chamber 1.

$A_2$  = Sectional area of the piston face in chamber 2.

$C_L$  = Internal leakage coefficient across the ram.

$V_1$  = Volume of the fluid between the ram face and the valve in chamber 1.

$V_2$  = Volume of the fluid between the ram face and the valve in chamber 2.

$Y$  = Ram displacement.

$T$  = Time.

$\beta$  = Bulk modulus of the fluid.

Dynamics:  $P_1 A_1 - P_2 A_2 = M \frac{d^2 Y}{dt^2} + B \frac{dY}{dT} + KY + F$  (3.5)

where

$M$  = Inertial mass of the moving parts.

$B$  = Effective viscous damping on the system.

$K$  = Effective external spring stiffness of the system.

$F$  = Effective external force applied to the system.



### 3.3. Reference Quantities for Non-dimensionalisation

#### 3.3.1. Reference Position

When the piston is at this position, the total volume of the fluid in the circuit is  $V_T$  with  $V_{1r}$  and  $V_{2r}$  as the volumes of the fluid between the valve and the piston face in chambers 1 and 2 respectively.

$$V_T = V_{1r} + V_{2r} \quad (3.6)$$

The reference position is chosen when:

$$\frac{V_{2r}}{V_{1r}} = \frac{A_2}{A_1} = a \quad (3.7)$$

$$\therefore V_T = V_{1r}(1 + a) \quad (3.8)$$

If this is satisfied when the ram face in chamber 1 is  $Y_r$  from its mid-stroke position then

$$\begin{aligned} V_{1r} &= A_1 (Y_r + \frac{1}{2}Y_{st} + Y_{10}) \\ &= \frac{1}{2} A_1 (Y_{st} + 2Y_r + 2Y_{10}) \end{aligned} \quad (3.9)$$

$$\begin{aligned} V_{2r} &= A_2 (\frac{1}{2}Y_{st} + Y_{20} - Y_r) \\ &= \frac{1}{2} aA_1 (Y_{st} + 2Y_{20} - 2Y_r) \end{aligned} \quad (3.10)$$

$$\therefore Y_{st} + 2Y_{20} - 2Y_r = Y_{st} + 2Y_{10} + 2Y_r$$

$$4Y_r = 2(Y_{20} - Y_{10})$$

$$Y_r = \frac{1}{2}(Y_{20} - Y_{10}) \quad (3.11)$$

where

$$Y_{i0} = \frac{V_{iL} + V_{iC}}{A_i} \quad i=1,2$$

$V_{iL}$  = Volume of the fluid in the  
hose on side  $i \quad i=1,2$

$V_{iC}$  = Clearance volume of the fluid  
in chamber  $i \quad i=1,2$

$Y_{st}$  = Piston stroke measured between  
the two extremes

### 3.3.2. Reference Length

The reference length is chosen such that

$$\begin{aligned} L_r &= \frac{V_{2r}}{A_2} = \frac{V_{1r}}{A_1} \\ &= \frac{1}{2}(Y_{st} + 2Y_{20} - 2Y_r) \\ &= \frac{1}{2}(Y_{st} + 2Y_{10} + 2Y_r) \end{aligned} \quad (3.12)$$

### 3.3.3. Reference Area

The reference area is chosen to be the area of the piston face in the chamber 1.

$$\text{i.e. Area}_{\text{reference}} = A_1$$

### 3.3.4. Reference Time

When the piston is at its reference position, the stiffness of the fluid column in chamber 1 will be given by:

$$S_{1r} = \frac{\beta A_1^2}{V_{1r}} = \frac{\beta A_1^2}{A_1 L_r} = \frac{\beta A_1}{L_r}$$

The stiffness of the fluid column in chamber 2 is:

$$S_{2r} = \frac{\beta A_2^2}{V_{2r}} = \frac{\beta A_2^2}{A_2 L_r} = \frac{\beta A_2}{L_r} = a \frac{\beta A_1}{L_r} = a S_{1r}$$

∴ The total stiffness of the fluid columns at this position

$$S_r = S_{1r} + S_{2r} = (1 + a) \frac{\beta A_1}{L_r} \quad (3.13)$$

The natural hydraulic frequency at this position is:

$$\omega_r = \sqrt{\frac{S_r}{M}} = \sqrt{\frac{\beta A_1}{ML_r} (1 + a)} \quad (3.14)$$

Therefore the natural hydraulic frequency will vary with the physical configuration of the ram and cylinder (including the inertial mass, stroke, piston areas, clearance, etc.) as well as the compressibility of the fluid, and this frequency is a characteristic of a particular system. On non-dimensionalising the quantities, the reference time is chosen to be the inverse of this natural hydraulic frequency:

$$T_r = \frac{1}{\omega_r} = \sqrt{\frac{ML_r}{\beta A_1 (1 + a)}} \quad (3.15)$$

### 3.3.5. Reference Pressure

The reference pressure is chosen to be the system supply pressure,  $P_s$ .

### 3.3.6. Reference Flow Rate

The reference flow rate is chosen to be the reference volume divided by the reference time:

$$Q_r = \frac{V_{1r}}{T_r} = \frac{A_1 L_r}{T_r}$$

$$= A_1 L_r \sqrt{\frac{\beta A_1}{ML_r}} (1 + a) \quad (3.16)$$

where  $V_{1r}$  = Reference volume =  $A_1 L_r$

### 3.4. Non-dimensionalised Equations

After the equations have been non-dimensionalised with the reference quantities, the set of differential equations will become:

$$q_1 = \frac{Q_1}{Q_r} = \frac{A_1 T_r L_r}{A_1 L_r T_r} \frac{dy}{dt} + \frac{P_s}{Q_r} (p_1 - p_2) C_L + \frac{V_{1r} + A_1 L_r y}{\beta Q_r} \frac{P_s}{T_r} \frac{dp_1}{dt}$$

$$= \frac{dy}{dt} + \frac{P_s C_L}{Q_r} (p_1 - p_2) + \frac{P_s}{\beta} (1 + y) \frac{dp_1}{dt} \quad (3.17)$$

$$q_2 = \frac{Q_2}{Q_r} = -a \frac{dy}{dt} + \frac{P_s C_L}{Q_r} (p_2 - p_1) + \frac{P_s}{\beta} (1 - y) a \frac{dp_2}{dt} \quad (3.18)$$

$$p_1 - a p_2 = \frac{ML_r}{P_s A_1 T_r^2} \frac{d^2 y}{dt^2} + \frac{BL_r}{P_s A_1 T_r} \frac{dy}{dt} + \frac{KL_r}{P_s A_1} y + \frac{F}{P_s A_1} \quad (3.19)$$

Also

$$q_1 = \frac{1}{Q_r} C_{D1} b_1 x_v |x_1| \sqrt{\frac{2P_s}{\rho} |p_1' - p_1|} \text{Sign}(p_1' - p_1) \quad (3.20)$$

$$q_2 = \frac{1}{Q_r} C_{D2} b_2 x_v |x_2| \sqrt{\frac{2P_s}{\rho} |p_2' - p_2|} \text{Sign}(p_2' - p_2) \quad (3.21)$$

### 3.5. Possibility for Cavitation to occur

Fluid cannot sustain tension, therefore when the pressure falls below the vapour pressure, cavities form behind the moving ram. A necessary condition for cavitation to occur is that the pressure in either chamber takes on the bound, vapour pressure, for a finite time of duration. Both Wang and McCloy pointed out that rapid braking of hydraulic actuators may induce undesirable cavitation in the working fluid of the actuator [32,36]. Wang also stated that non-cavitation condition is always satisfied during acceleration [36].

McCloy and Martin [32] found that the conditions for cavitation to occur during rapid braking of a hydraulic actuator might be:

$$\frac{dy}{dt}_{\max} \geq \sqrt{1 - y_d} \frac{P_s}{\beta} p_{1e} \quad (3.22)$$

where  $y_d$  is the ram displacement from its reference position divided by the stroke;

$p_{1e}$  is the non-dimensional equilibrium pressure in chamber 1.

If there is no load other than the inertial load in the system, then the equilibrium pressure will be equal to half the supply pressure.

In many applications, the ram has achieved its terminal velocity before the rapid braking of the hydraulic actuator occurs, then this velocity will become the maximum.

For a symmetrical system with the following parameters:

Inertial mass = 2000 kg

System supply pressure = 137 bar gauge.  
(2000 psig approx.)

System exhaust pressure = 0 bar gauge.

Fluid density = 1000 Kg/m<sup>3</sup> (5/95 diluted emulsion  
may be treated as water).

Compressibility = 2.05 x 10<sup>9</sup> N/m<sup>2</sup>.

Dimensions of the rectangular valve = 6 mm x 6 mm.

Working stroke = 1 m.

Effective piston diameter = 15 cm.

Ratio of total volume in circuit at the reference  
position to the swept volume = 1.1.

Flow discharge coefficient for the valve ports = 0.65.

Maximum permissible velocity can be obtained from (3.22):

$$\frac{dy}{dt}_{\max} = 1.30 \times 10^{-3}$$

Terminal velocity can be obtained from (3.17):

$$\frac{dy}{dt}_{\text{terminal}} = 1.098 \times 10^{-3}$$

Therefore with these physical parameters, cavitation is unlikely to occur.

### 3.6. Derivation of the Switching Function for a Symmetric System

#### 3.6.1. Analysis

Subtracting (3.18) from (3.17) gives,

$$q_1 - q_2 = 2 \frac{dy}{dt} + \frac{2P_s C_L}{Q_r} (p_1 - p_2) + \frac{P_s}{\beta} \frac{d(p_1 - p_2)}{dt} + \frac{P_s}{\beta} \frac{d(p_1 + p_2)}{dt} y$$

since  $a = 1$  for a symmetric system.

Furthermore, for a symmetric system,

$$\frac{d(p_1 + p_2)}{dt} = 0$$

$$\therefore q_1 - q_2 = 2 \frac{dy}{dt} + \frac{2P_s C_L}{Q_r} (p_1 - p_2) + \frac{P_s}{\beta} \frac{d(p_1 - p_2)}{dt} \quad (3.23)$$

If the internal leakage is very small so that it can be neglected, equation (3.23) can further be reduced to

$$q_1 - q_2 = 2 \frac{dy}{dt} + \frac{P_s}{\beta} \frac{d(p_1 - p_2)}{dt} \quad (3.24)$$

If there is no external force and no external spring, then equation (3.19) gives

$$p_1 - p_2 = \frac{ML_r}{P_s A_1 T_r^2} \frac{d^2 y}{dt^2} + \frac{BL_r}{P_s A_1 T_r} \frac{dy}{dt} \quad (3.25)$$

It can easily be shown that if there is no cavitation then, by neglecting the compressibility effect, an approximation can be obtained [36]:

$$q_1 = -q_2 = K_1 |x| \sqrt{|p_1' - p_1|} \text{Sign}(p_1' - p_1)$$

$$\text{where } K_1 = \frac{1}{Q_r} C_{D1} b_1 X_V \sqrt{\frac{2P_s}{\rho}}$$

and the valves do not have any lapping.

With the system exhaust pressure approximately equals to zero gauge, the following relationship can be derived:

$$P_1 \begin{cases} = 1 & \text{for } x \geq 0 \\ = 0 & \text{for } x < 0 \end{cases} ;$$

$$P_2 \begin{cases} = 0 & \text{for } x \geq 0 \\ = 1 & \text{for } x < 0 \end{cases}$$

Therefore,

$$p_1 = \frac{1}{2}(1 + p_1 - p_2)$$

$$p_2 = \frac{1}{2}(1 - p_1 + p_2)$$

With these substitutions, equation (3.20) is expanded using Taylor's series for  $x > 0$ , the expression becomes,

$$q_1 = K_1 x \sqrt{\frac{1}{2}} \left[ 1 - \frac{1}{2}(p_1 - p_2) \right] \quad (3.26)$$

when the terms of order higher than one are being neglected.

Combining (3.24), (3.25) and (3.26) gives,

$$\begin{aligned} & 2K_1 x \sqrt{\frac{1}{2}} \left[ 1 - \frac{1}{2} \left( \frac{ML_r}{P_s A_1 T_r^2} \frac{d^2 y}{dt^2} + \frac{BL_r}{P_s A_1 T_r} \frac{dy}{dt} \right) \right] \\ & = 2 \frac{dy}{dt} + \frac{P_s}{\beta} \left[ \frac{ML_r}{P_s A_1 T_r^2} \frac{d^3 y}{dt^3} + \frac{BL_r}{P_s A_1 T_r} \frac{d^2 y}{dt^2} \right] \end{aligned}$$

Rearranging becomes,

$$\begin{aligned} \frac{ML_r}{\beta A_1 T_r^2} \frac{d^3 y}{dt^3} + \left[ \frac{BL_r}{\beta A_1 T_r} + \frac{ML_r}{P_s A_1 T_r^2} K_1 x \sqrt{\frac{1}{2}} \right] \frac{d^2 y}{dt^2} + \left[ 2 + \frac{BL_r}{P_s A_1 T_r} K_1 x \sqrt{\frac{1}{2}} \right] \frac{dy}{dt} \\ = 2 K_1 x \sqrt{\frac{1}{2}} \end{aligned} \quad (3.27)$$



Let  $\zeta = \frac{BT}{2M}r$ ,  $\nu = \frac{\beta}{P_s}$ , and  $\xi = K_1 \sqrt{\frac{1}{2}}$ , then the above equation can be re-written as:

$$\frac{d^3y}{dt^3} + (2\zeta + \nu\xi x) \frac{d^2y}{dt^2} + (1 + 2\nu\zeta\xi x) \frac{dy}{dt} = \xi x \quad (3.28)$$

### 3.6.2. Ideal On-Off Valves

If the valves used can be approximated to ideal ON-OFF valves, then the non-dimensional valve displacement can be one of the following: -1, 0 and 1. The following discussion will be on the cases when the ram is required to move in the positive direction, similar analysis can be used for movement in the other direction.

When the valves are opened, (3.28) can be written as:

$$\frac{d^3y}{dt^3} + (2\zeta + \nu\xi) \frac{d^2y}{dt^2} + (1 + 2\nu\zeta\xi) \frac{dy}{dt} = \xi$$

When the valves are closed, this becomes,

$$\frac{d^3y}{dt^3} + 2\zeta \frac{d^2y}{dt^2} + \frac{dy}{dt} = 0 \quad (3.29)$$

After the valves have been closed, the ram displacement can be obtained by solving the differential equation (3.29).

Integrating this gives,

$$\frac{d^2y}{dt^2} + 2\zeta \frac{dy}{dt} + y = R_0 \quad (3.30)$$

$$\text{where } R_0 = \ddot{y}_0 + 2\zeta \dot{y}_0 + y_0$$

$y_0$  = Initial displacement of the ram on closing the valves.

$\dot{y}_0$  = Initial ram velocity on closing the valves.

$\ddot{y}_0$  = Initial ram acceleration on closing the valves.

Solution to (3.30) is found to be:

$$y(t) = \frac{1}{s_1 - s_2} (\dot{y}_0 + 2\zeta y_0 - s_1 y_0 - \frac{R_0}{s_1}) e^{-s_1 t} - (\dot{y}_0 + 2\zeta y_0 - s_2 y_0 - \frac{R_0}{s_2}) e^{-s_2 t} + \frac{R_0}{s_1 s_2}$$

where  $-s_1$  and  $-s_2$  are the roots of the characteristic equation:

$$s^2 + 2\zeta s + 1 = 0$$

$$\text{As } t \rightarrow \infty, y \rightarrow \frac{R_0}{s_1 s_2} = R_0 = \ddot{y}_0 + 2\zeta \dot{y}_0 + y_0$$

∴ The ram will rest at the desired position,  $y_d$ , if the valves are switched off when  $R_0$  is equal to  $y_d$ . Hence,

$$\ddot{y}_0 + 2\zeta \dot{y}_0 + y_0 = y_d$$

is the switching function for this type of valve.

### 3.6.3. On-Off Valves with Time Delay

The actual operation of electromagnetic valves can be very fast, but there may be a time delay due to the residual electromagnetism. This type of valve can then be approximated to on-off valves with time delay,  $t_d$ . For this type of valve, over the period between the application of 'close' signal and the actual closure of the valves, equation (3.28) can be written as:

$$\frac{d^3 y}{dt^3} + (2\zeta + v\xi) \frac{d^2 y}{dt^2} + (1 + 2v\zeta\xi) \frac{dy}{dt} = \xi \quad (3.31)$$

Integrating this with respect to  $t$  gives,

$$\frac{d^2y}{dt^2} + (2\zeta + \nu\xi) \frac{dy}{dt} + (1 + 2\nu\zeta\xi)y = \xi t + R_1$$

$$\text{where } R_1 = \ddot{y}_0 + (2\zeta + \nu\xi)\dot{y}_0 + (1 + 2\nu\zeta\xi)y_0$$

Solution to this equation is:

$$\begin{aligned} y(t) = & \frac{\xi t}{\phi_1\phi_2} + \frac{\xi}{\phi_1 - \phi_2} \left( \frac{e^{\phi_1 t}}{\phi_1^2} - \frac{e^{\phi_2 t}}{\phi_2^2} \right) + \frac{\phi_1 + \phi_2}{\phi_1^2\phi_2^2} \xi + y_0 \\ & + \frac{\dot{y}_0}{\phi_1 - \phi_2} \left[ \frac{\phi_1}{\phi_2} e^{\phi_2 t} - \frac{\phi_2}{\phi_1} e^{\phi_1 t} - \frac{(\phi_1 + \phi_2)(\phi_1 - \phi_2)}{\phi_1\phi_2} \right] \\ & + \frac{\ddot{y}_0}{\phi_1 - \phi_2} \left[ \frac{e^{\phi_1 t}}{\phi_1} - \frac{e^{\phi_2 t}}{\phi_2} + \frac{\phi_1 - \phi_2}{\phi_1\phi_2} \right] \end{aligned}$$

$$\text{where } (s - \phi_1)(s - \phi_2) = s^2 + (2\zeta + \nu\xi)s + (1 + 2\nu\zeta\xi)$$

At  $t_d$ , the valves are completely closed and the following is required at this instant (Section 3.6.2):

$$y_d = \ddot{y}_{t_d} + 2\zeta \dot{y}_{t_d} + y_{t_d}$$

Therefore, the switching function is:

$$\begin{aligned} y_d = & \frac{\ddot{y}_0}{\phi_1 - \phi_2} \left[ (\phi_1 + 2\zeta + \frac{1}{\phi_1}) e^{\phi_1 t_d} - (\phi_2 + 2\zeta + \frac{1}{\phi_2}) e^{\phi_2 t_d} + \frac{\phi_1 - \phi_2}{\phi_1\phi_2} \right] \\ & + \frac{\dot{y}_0}{\phi_1 - \phi_2} \left[ \phi_1 (\phi_2 + 2\zeta + \frac{1}{\phi_2}) e^{\phi_2 t_d} - \phi_2 (\phi_1 + 2\zeta + \frac{1}{\phi_1}) e^{\phi_1 t_d} - \frac{\phi_1^2 - \phi_2^2}{\phi_1\phi_2} \right] \\ & + \frac{\xi}{\phi_1\phi_2} \left[ (2\zeta + \tau_d + \frac{\phi_1 - \phi_2}{\phi_1\phi_2}) \right] + y_0 \\ & + \frac{\xi}{\phi_1 - \phi_2} \left[ \frac{1}{\phi_1} (\phi_1 + 2\zeta + \frac{1}{\phi_1}) e^{\phi_1 t_d} - \frac{1}{\phi_2} (\phi_2 + 2\zeta + \frac{1}{\phi_2}) e^{\phi_2 t_d} \right] \end{aligned}$$

#### 3.6.4. Linearly Operated Valves

In practice, most of the spool valves dynamics can often be approximated to linear operations [29]. If the time taken to operate the valve is  $t_c$ , then on closing the valve, displacement can be described as: valve displacement (non-dimensional) =  $x = 1 - \frac{t}{t_c}$  where  $t$  is the time after the application of the closing signal but before the complete closure of the valve. Over the period between the application of the closing signal and the complete closure of valves, equation (3.28) can be written as:

$$\frac{d^3y}{dt^3} + \left[ 2\zeta + \left(1 - \frac{t}{t_c}\right) v\xi \right] \frac{d^2y}{dt^2} + \left[ 1 + 2v\zeta\xi \left(1 - \frac{t}{t_c}\right) \right] \frac{dy}{dt} = \xi \left(1 - \frac{t}{t_c}\right) \quad (3.32)$$

The solution to this equation can be found by applying Laplace transform and then inverting the Laplace transform.

With the following substitution:

$$(s - \theta_1)(s - \theta_2)(s - \theta_3) = s^3 + (2\zeta + v\xi)s^2 + \left(1 + 2v\zeta\xi + \frac{2v\xi}{t_c}\right)s + \frac{2v\zeta\xi}{t_c}$$

the expression for the non-dimensional ram displacement can be simplified.

The solution is:

$$\begin{aligned}
 Y(t) = & \frac{\ddot{y}_0}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} \left[ (\theta_3 - \theta_2) e^{\theta_1 t} + (\theta_1 - \theta_3) e^{\theta_2 t} + (\theta_2 - \theta_1) e^{\theta_3 t} \right] \\
 & + \frac{\dot{y}_0}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} \left[ \begin{aligned} & (2\zeta + \nu\xi + \theta_1)(\theta_3 - \theta_2) e^{\theta_1 t} \\ & + (\theta_2 + \nu\xi + 2\zeta)(\theta_1 - \theta_3) e^{\theta_2 t} \\ & + (2\zeta + \nu\xi + \theta_3)(\theta_2 - \theta_1) e^{\theta_3 t} \end{aligned} \right] \\
 & + \frac{y_0}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} \left[ \begin{aligned} & \left\{ 1 + 2\nu\zeta\xi + \frac{\nu\xi}{t_c} + (2\zeta + \nu\xi)\theta_1 + \theta_1^2 \right\} e^{\theta_1 t} (\theta_3 - \theta_2) \\ & + \left\{ 1 + 2\nu\zeta\xi + \frac{\nu\xi}{t_c} + (2\zeta + \nu\xi)\theta_2 + \theta_2^2 \right\} e^{\theta_2 t} (\theta_1 - \theta_3) \\ & + \left\{ 1 + 2\nu\zeta\xi + \frac{\nu\xi}{t_c} + (2\zeta + \nu\xi)\theta_3 + \theta_3^2 \right\} e^{\theta_3 t} (\theta_2 - \theta_1) \end{aligned} \right] \\
 & + \frac{\xi}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} \left[ \begin{aligned} & (\theta_3 - \theta_2) \left( \frac{1}{\theta_1} - \frac{1}{t_c \theta_1^2} \right) e^{\theta_1 t} \\ & + (\theta_1 - \theta_3) \left( \frac{1}{\theta_2} - \frac{1}{t_c \theta_2^2} \right) e^{\theta_2 t} \\ & + (\theta_2 - \theta_1) \left( \frac{1}{\theta_3} - \frac{1}{t_c \theta_3^2} \right) e^{\theta_3 t} \\ & - \left( 1 - \frac{t}{t_c} \right) \left( \frac{\theta_3 - \theta_2}{\theta_1} + \frac{\theta_1 - \theta_3}{\theta_2} + \frac{\theta_2 - \theta_1}{\theta_3} \right) \\ & + \frac{1}{t_c} \left( \frac{\theta_3 - \theta_2}{\theta_1^2} + \frac{\theta_1 - \theta_3}{\theta_2^2} + \frac{\theta_2 - \theta_1}{\theta_3^2} \right) \end{aligned} \right]
 \end{aligned}$$

As in Section 3.6.2, the requirement for the ram to rest at the desired position,  $y_d$ , is that at complete closure of valves the following is satisfied:

$$\ddot{y}_{t_c} + 2\zeta\dot{y}_{t_c} + y_{t_c} = y_d$$

This gives the switching function for this type of valve.

### 3.6.5. Effects of Externally Applied Load

If there is an externally applied load, then equation (3.19) will hold and the equation (3.26) becomes,

$$q_1 = K_1 \times \sqrt{\frac{1}{2}} \left[ 1 - \frac{1}{2} \left( \frac{ML_r}{P_s A_1 T_r^2} \frac{d^2 y}{dt^2} + \frac{BL_r}{P_s A_1 T_r} \frac{dy}{dt} + \frac{F}{P_s A_1} \right) \right] \quad (3.33)$$

Combining (3.19), (3.24) and (3.33) gives,

$$\begin{aligned} & 2K_1 x \sqrt{\frac{1}{2}} \left[ 1 - \frac{1}{2} \left( \frac{ML_r}{P_s A_1 T_r^2} \frac{d^2 y}{dt^2} + \frac{BL_r}{P_s A_1 T_r} \frac{dy}{dt} \right) - \frac{F}{P_s A_1} \right] \\ & = 2 \frac{dy}{dt} + \frac{P_s}{\beta} \left[ \frac{ML_r}{P_s A_1 T_r} \frac{d^3 y}{dt^3} + \frac{BL_r}{P_s A_1 T_r} \frac{d^2 y}{dt^2} \right] \end{aligned}$$

Rearranging gives,

$$\begin{aligned} & \frac{ML_r}{\beta A_1 T_r^2} \frac{d^3 y}{dt^3} + \left[ \frac{BL_r}{\beta A_1 T_r} + \frac{ML_r}{P_s A_1 T_r^2} K_1 x \sqrt{\frac{1}{2}} \right] \frac{d^2 y}{dt^2} + \left[ 2 + \frac{BL_r}{P_s A_1 T_r} K_1 x \sqrt{\frac{1}{2}} \right] \frac{dy}{dt} \\ & = 2K_1 x \sqrt{\frac{1}{2}} \left( 1 - \frac{1}{2} \frac{F}{P_s A_1} \right) \end{aligned}$$

$$\therefore \frac{d^3 y}{dt^3} + (2\zeta + v\xi x) \frac{d^2 y}{dt^2} + (1 + 2v\zeta\xi x) \frac{dy}{dt} = \xi \left( 1 - \frac{1}{2} \frac{F}{P_s A_1} \right) x \quad (3.34)$$

By comparing this equation with equation (3.28), it can be seen that the externally applied load only affects the coefficient of the constant term in the switching function. However, it does not have any effect on the switching function for ideal ON-OFF valves since the valves will be closed immediately on the application of the switching signal.

### 3.7. Derivation of the Switching Function for an Asymmetric System

If the ram is at its reference position, i.e.  $y = 0$ , and is non-cavitative, then with a small change in the ram position due to the fluid flow will be described by:-

$$q_1 = K_1 x \sqrt{1 - p_1} ; \quad q_2 = -K_1 x \sqrt{p_2}$$

and

$$q_1 = \dot{y} + \frac{P_S}{\beta} \dot{p}_1 \quad (y \approx 0 \text{ for small movement})$$

$$q_2 = -a\dot{y} + \frac{P_S}{\beta} a\dot{p}_2$$

the dot (·) represents the operator  $\frac{d}{dt}$

therefore,

$$q_1 - q_2 = (1 + a)\dot{y} + \frac{P_S}{\beta} (\dot{p}_1 - a\dot{p}_2)$$

Equation (3.19) can be written as the following by neglecting the externally applied force and external springs:

$$p_1 - a p_2 = \alpha_m \ddot{y} + \alpha_b \dot{y}$$

$$\text{where } \alpha_m = \frac{ML_r}{P_S A_1 T_r^2}$$

$$\alpha_b = \frac{BL_r}{P_S A_1 T_r}$$

$$\therefore q_1 - q_2 = (1 + a) \dot{y} + \frac{P_S}{\beta} (\alpha_m \ddot{y} + \alpha_b \dot{y}) \quad (3.35)$$

Because of the small movement of the ram, an approximation can be obtained by neglecting the compressibility effect<sup>[32]</sup>, and the following relationship is obtained:-

$$aq_1 \approx -q_2$$

$$\therefore aK_1 x \sqrt{1 - p_1} \approx K_1 x \sqrt{p_2}$$

$$\therefore p_1 + \frac{p_2}{a^2} = 1$$

With these substitutions for  $p_1$  and  $p_2$ , from (3.35) gives,

$$\begin{aligned} q_1 - q_2 &\approx K_1 x \sqrt{1 - p_1} (1 + a) \\ &= \frac{1 + a}{\sqrt{1 + a^3}} K_1 x \left[ 1 - (p_1 - ap_2) \right] \end{aligned} \quad (3.36)$$

By Taylor's series expansion gives,

$$(1 + a)\dot{y} + \frac{P_s}{\beta}(\alpha_m \ddot{y} + \alpha_b \ddot{y}) = \frac{1 + a}{\sqrt{1 + a^3}} K_1 x (1 - \frac{1}{2} \alpha_m \ddot{y} - \frac{1}{2} \alpha_b \ddot{y})$$

Rearranging to obtain:

$$\begin{aligned} \frac{P_s}{\beta} \alpha_m \ddot{y} + \left[ \frac{P_s}{\beta} \alpha_b + \frac{1 + a}{\sqrt{1 + a^3}} K_1 x \frac{1}{2} \alpha_m \right] \ddot{y} + \left[ \frac{1}{\sqrt{1 + a^3}} K_1 x \alpha_b \frac{1}{2} + 1 \right] (1 + a) \dot{y} \\ = \frac{1 + a}{\sqrt{1 + a^3}} K_1 x \end{aligned} \quad (3.37)$$

For the symmetric case,  $a = 1$ , (3.37) can be reduced to:

$$\frac{P_s}{\beta} \alpha_m \ddot{y} + \left[ \frac{P_s}{\beta} \alpha_b + \frac{1}{\sqrt{2}} K_1 x \alpha_m \right] \ddot{y} + \left[ \frac{1}{\sqrt{2}} K_1 x \alpha_b + 2 \right] \dot{y} = \frac{2}{\sqrt{2}} K_1 x$$

this is the same as (3.27).

Hence the switching functions for various types of valves can be obtained by the similar procedures as in the symmetric case but with the coefficients:

$$\xi = \frac{1 + a}{2 \sqrt{1 + a^3}} K_1$$

$$\nu = \frac{\beta}{P_s}$$

$$\zeta = \frac{BT_r}{2M}$$



### 3.8. Simulation Studies

The system behaviours have been simulated on a desk-top computer, HP 9845B, with the switching functions for the valve types. A modified Euler method is used to solve the differential equations [41], the computer program for this simulation study can be found in Appendix C.

Figures 3.2 and 3.3 predict the non-dimensional ram displacement\* responses of the symmetric systems with ideal ON-OFF valves and the inertial masses varying from 500 kg to 2000 kg in the non-dimensional time domain\*\* and the physical time domain respectively. The ram will take approximately the same amount of physical time to reach the same desired position. This can be explained by the fact that the same amount of fluid has been displaced to give the same ram displacement, although the response for a lighter system is faster than the one with a higher inertial mass. The dimensional ram velocity and acceleration are slightly smaller in a heavier system and the switching of the valves has to occur earlier with reference to the desired position. However, the non-dimensional times taken are significantly different because the reference time varies with the actuator's natural hydraulic frequency, which depends upon the inertial mass (the heavier the mass, the lower is the frequency).

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\* Recall that the ram displacement is non-dimensionalised by the reference length,  $L_r$  which will be half-stroke in the case of a symmetrical system. See Section 3.3.2.

\*\* The time domain is non-dimensionalised by the reference time,  $T_r$ , which is the inverse of the natural hydraulic frequency of the ram at its reference position. See Section 3.3.4.

The switching function for an ideal ON-OFF valve system is:

$$\ddot{y} + 2\zeta\dot{y} + y = y_d,$$

therefore, the damping factor,  $\zeta$ , plays an important role in determining the switching of the valves. If one keeps the damping factor constant (in the non-dimensional unit), the switching will be determined by the ram velocity and its acceleration for the same desired position. However, these two quantities will vary according to the inertial mass: the higher the loading, the lower will be these quantities (in the physical units) for the same given system supply and exhaust pressures. This is because the pressure differences across the valves will be reduced to give a higher driving force in moving a heavier ram, hence a smaller fluid flow rate will result. It should be recalled that the non-dimensional time is also inertial load dependant (the reference time will increase with the inertial mass), therefore the non-dimensional velocity and acceleration will be higher for a system with a heavier inertial load, this is one of the reasons why the switching for the system with a heavier inertial load has to occur earlier. It can also be seen that the overshoot is higher for a larger inertial mass system since the kinetic energy is higher at the switching point for a heavier system, hence the switching occurs earlier for a heavier system. Figures 3.4 and 3.5 review the similar behaviours for the asymmetrical systems.

The effects of damping on the system behaviour can be seen in Figures 3.6 and 3.7: the smaller is the damping, the more will be the overshoot and the oscillations before the

ram reaches its equilibrium state. According to the switching functions, the higher the damping, the earlier the valves should be closed for the same desired position with the same initial conditions, if the other quantities remain unchanged. The distance of travel is normally long enough for the ram to achieve its terminal velocity before closing the valves, therefore the determination of the switching point depends highly upon the velocity term,  $2 \zeta \dot{y}$ . Although the terminal velocity will decrease with an increase in the damping factor, the effect of damping has larger influence on the switching point if the system is underdamped. As the damping increases, the effects of lower velocity will become more significant. This can be reviewed from the graphs (Figures 3.6 and 3.7) that apparently an exponential curve might be constructed through the switching points.

If one examines the definition of the damping factor,  $\zeta$ , he will find that the damping factor can be increased by decreasing the inertial mass, hence an increase in the inertial loading is expected to have a similar effect as decreasing the damping factor. However, this damping effect is outweighed by the effects of the fluid flow rate, though a system with a heavier mass will take less non-dimensional time to reach the switching point, since the reference time is different.

The effects of the time delay on the ON-OFF valves are shown in Figures 3.8 and 3.9 for the symmetric and asymmetric systems respectively. It will take a longer time

to reach the desired position for a system with valves of a longer time delay, since these valves will take longer to respond to the switching signals. Consequently, the switching signal has to be applied much earlier for the valves with a longer time delay.

Figures 3.10 and 3.11 show the similar effects of 'linear' valves with various operating times. However, the intermediate oscillations are fewer because the valves are closed gradually and the pressures within the chambers do not change rapidly. The graphs indicate that the final errors for these systems are higher, it may be explained by the fact that the switching functions are derived based on the assumption that the higher order terms in the Taylor's series expansion can be neglected. Therefore, in deriving the switching functions, the estimated fluid flow is less than that obtained in the simulation. In other words, on deriving the switching functions, the flow through the valve is approximately linear, while the simulated flow is non-linear. This can be seen from Figure 3.12 that the non-linearity is more significant on the side with the piston rod (i.e. the side with a smaller sectional area). Figures 3.13 and 3.14 show that the deviation of the simulated flow from the linearity becomes larger as the operating time increases. This is why the final error increases as the valves take longer time to operate. However, the magnitudes of these errors are in the order of  $10^{-2}$  of the reference length, and are insignificant for this application: mining boom positional control. Therefore, correction functions are not introduced so as to keep the switching functions simple.

Figures 3.15 and 3.16 show the effects of the externally applied force: the initial overshoot will be higher when the force is lower, and it takes a longer time to reach the desired position if the force is higher. This is because the force is a measure of the resistance to the ram movement: the higher the force, the lower will be the velocity and acceleration. Consequently, the switching should occur earlier (with reference to the desired position) for a system with a lower externally applied force.

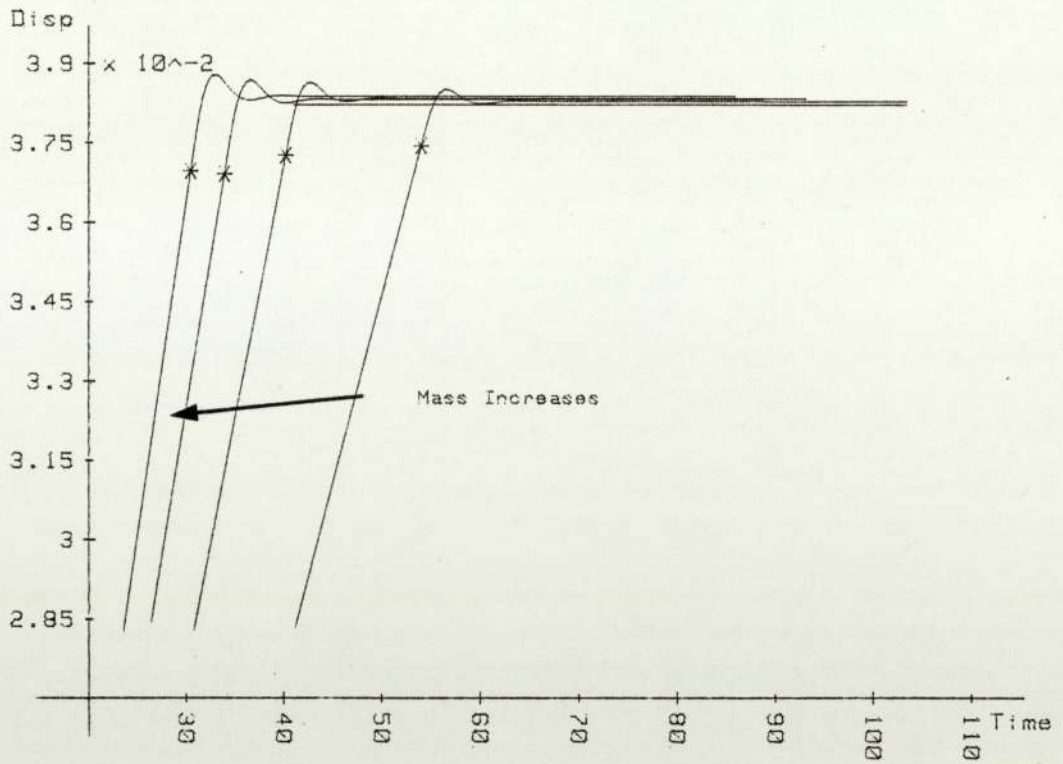


Fig. 3.2 Non-dimensional Ram Displacement - Non-dimensional Time Response Symmetrical Systems With Different Inertial Masses

ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 mSec. )  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor = .50

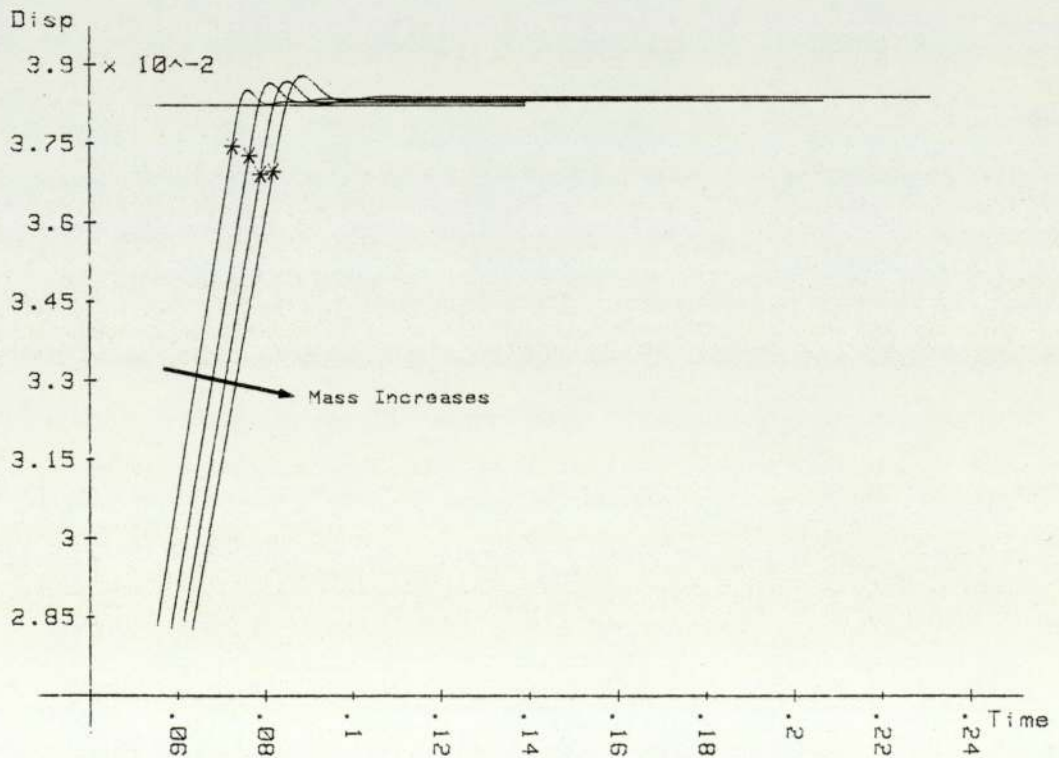


Fig. 3.3 Non-dimensional Ram Displacement - Dimensional Time Response Symmetrical Systems With Different Inertial Masses

ON-OFF Valves With Time-Delay = 0.0 mSec.  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor = .50

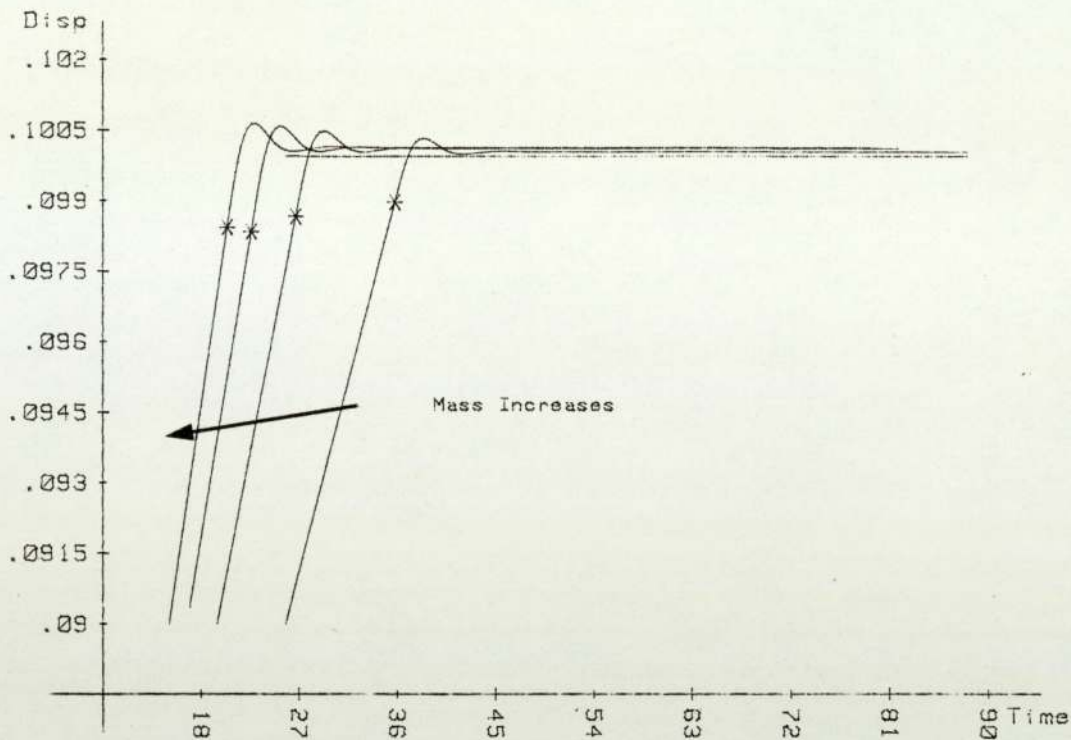


Fig. 3.4 Non-dimensional Ram Displacement - Non-dimensional Time Response  
 Asymmetrical Systems With Different Inertial Masses  
 Area Ratio =  $6.400E-01$   
 ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 mSec. )  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor = .50

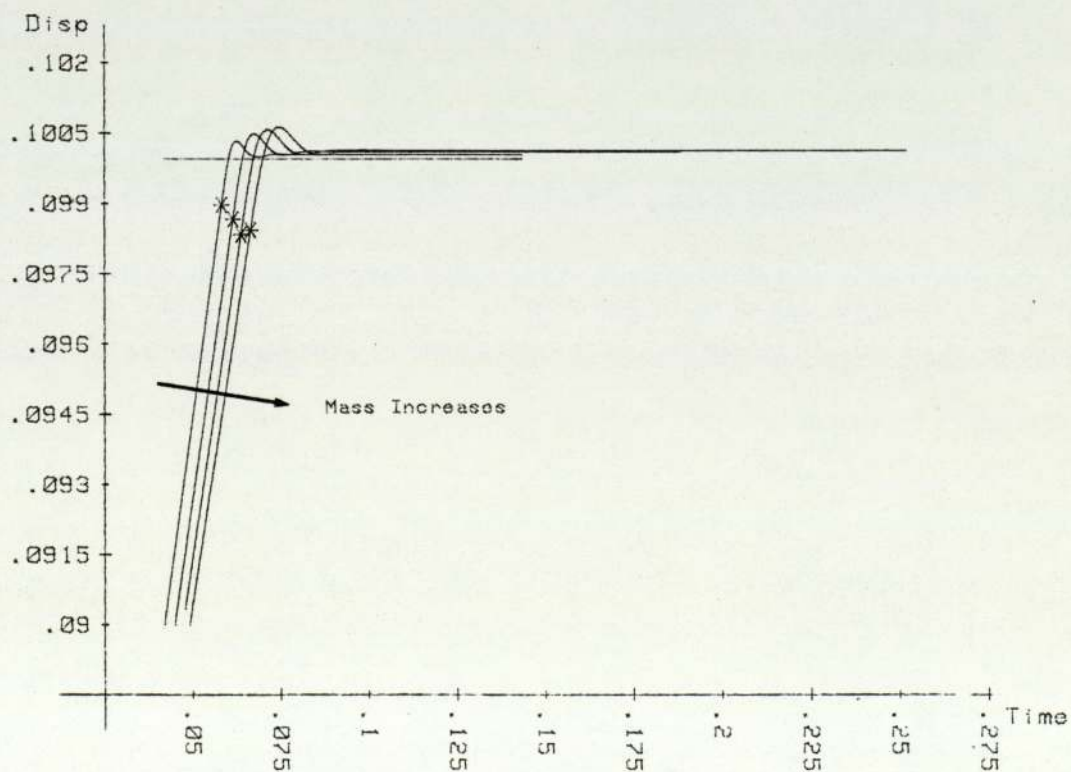


Fig. 3.5 Non-dimensional Ram Displacement - Dimensional Time Response  
 Asymmetrical Systems With Different Inertial Masses  
 Area Ratio =  $6.400E-01$   
 ON-OFF Valves With Time-Delay = 0.0 mSec.  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor = .50

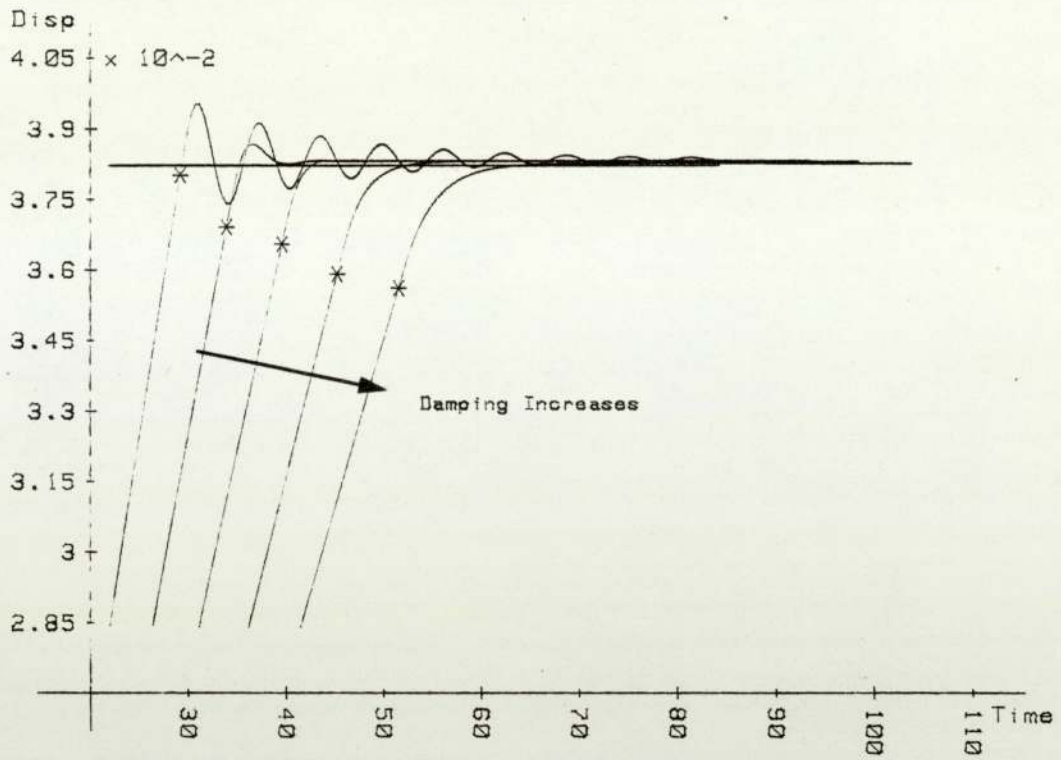


Fig. 3.6 Non-dimensional Ram Displacement - Non-dimensional Time Response Symmetrical Systems With Inertial Masses = 1500 Kg.

ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 mSec. )  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor Varies

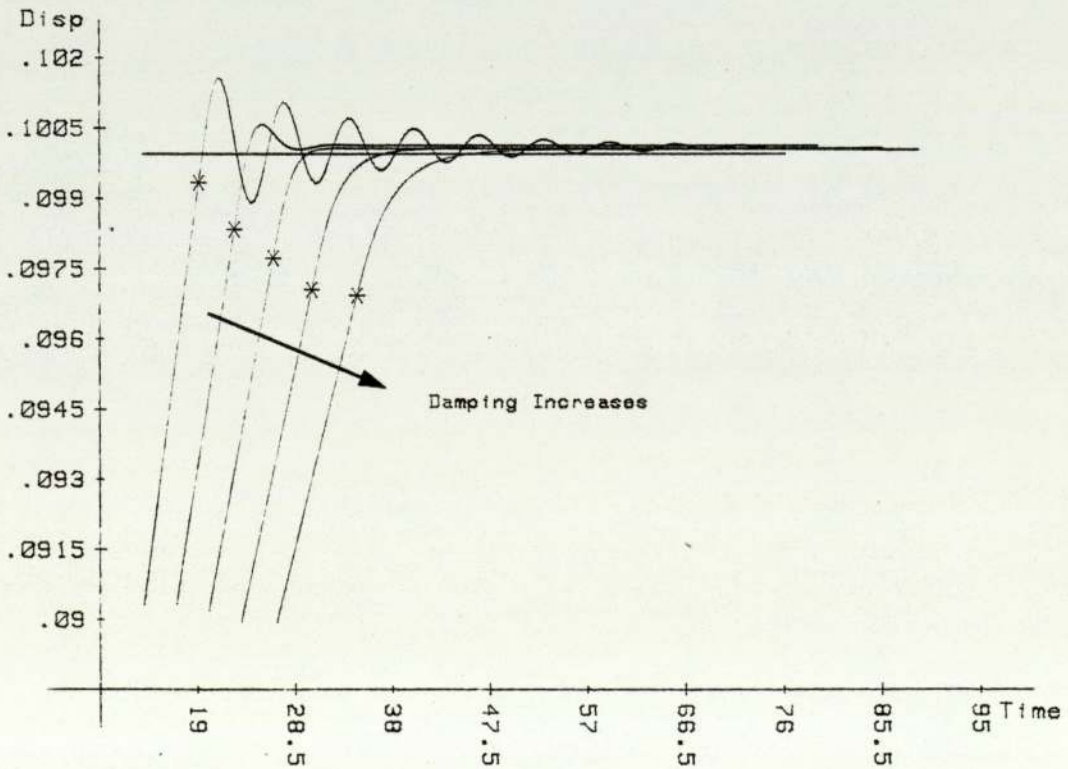


Fig. 3.7 Non-dimensional Ram Displacement - Non-dimensional Time Response Asymmetrical Systems With Inertial Masses = 1500 Kg.

Area Ratio =  $6.400E-01$   
 ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 mSec. )  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor Varies



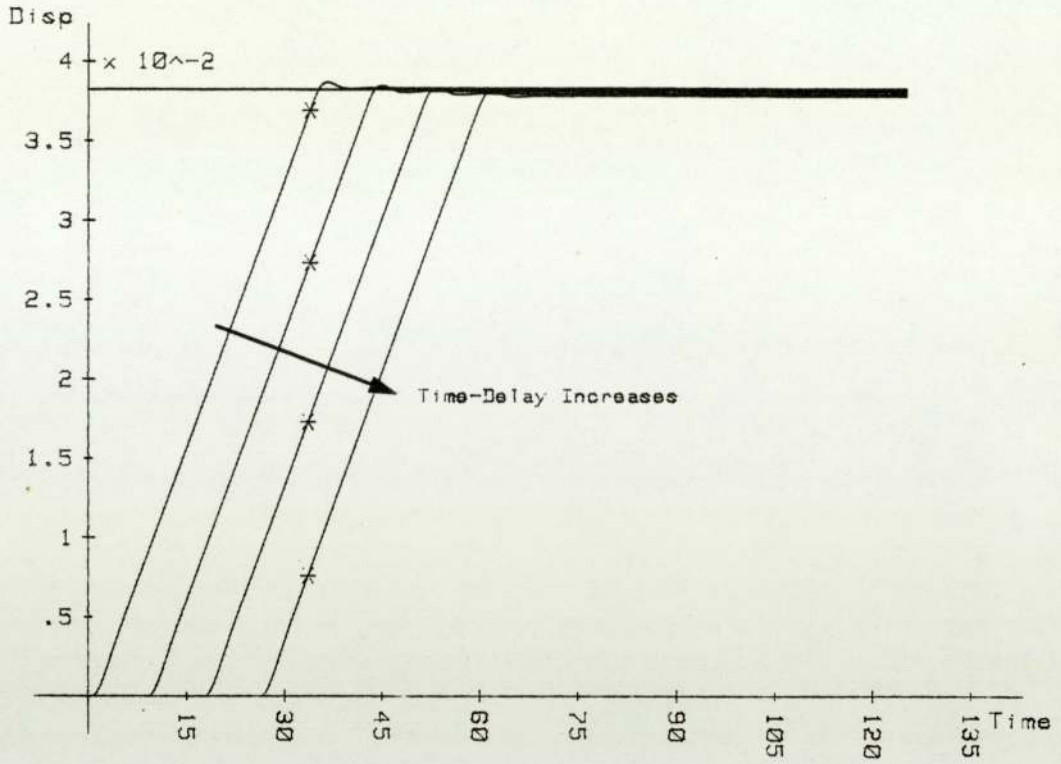


Fig. 3.8 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Symmetrical Systems With Inertial Mass = 1500 Kg.

ON-OFF Valves With Different Time-Delay  
External Force Applied = 0.0000E+00 Effective Damping Factor = .50

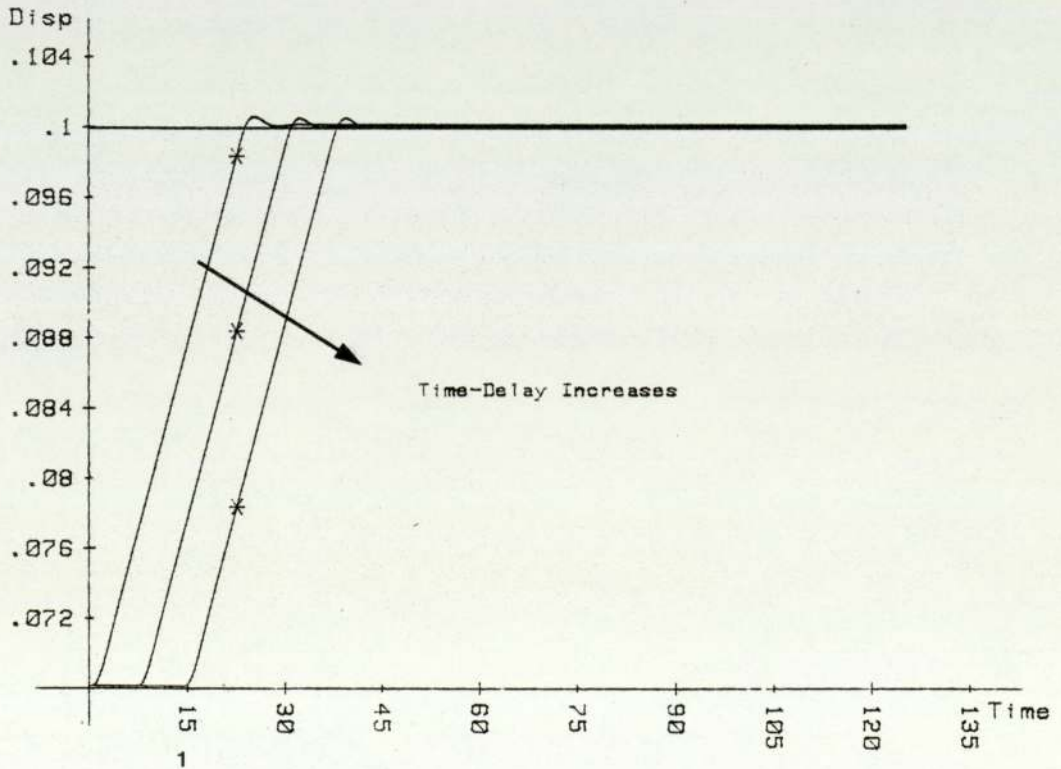


Fig. 3.9 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Asymmetrical Systems With Inertial Mass = 1500 Kg.

Area Ratio = 6.400E-01  
ON-OFF Valves With Different Time-Delay  
External Force Applied = 0.0000E+00 Effective Damping Factor = .50

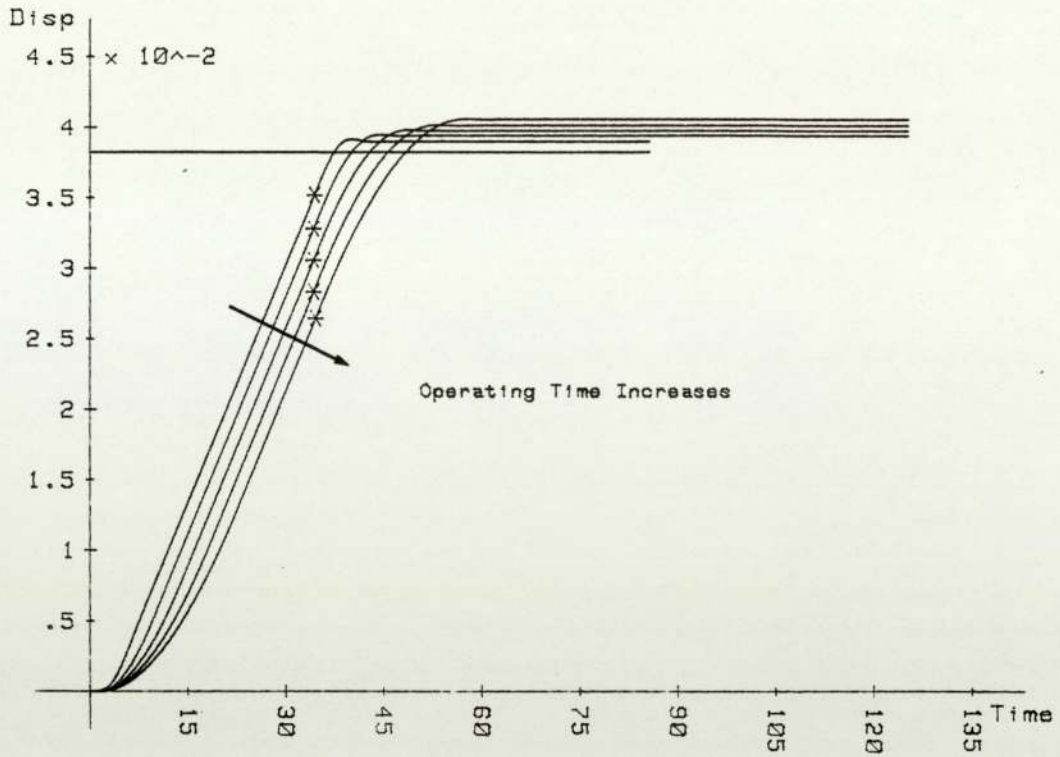


Fig. 3.10 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Symmetrical Systems With Inertial Mass = 1500 Kg.

Linear Valves With Different Operating Times  
External Force Applied = 0.0000E+00 Effective Damping Factor = .50

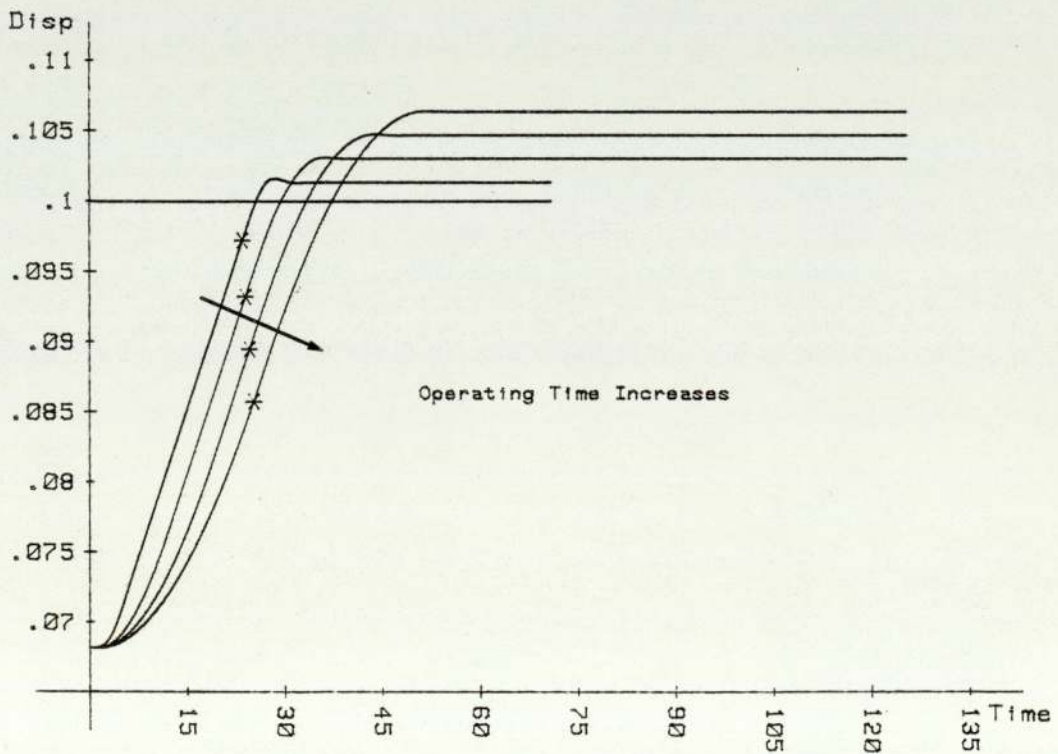


Fig. 3.11 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Asymmetrical Systems With Inertial Mass = 1500 Kg.

Area Ratio = 6.400E-01  
Linear Valves With Different Operating Times  
External Force Applied = 0.0000E+00 Effective Damping Factor = .50

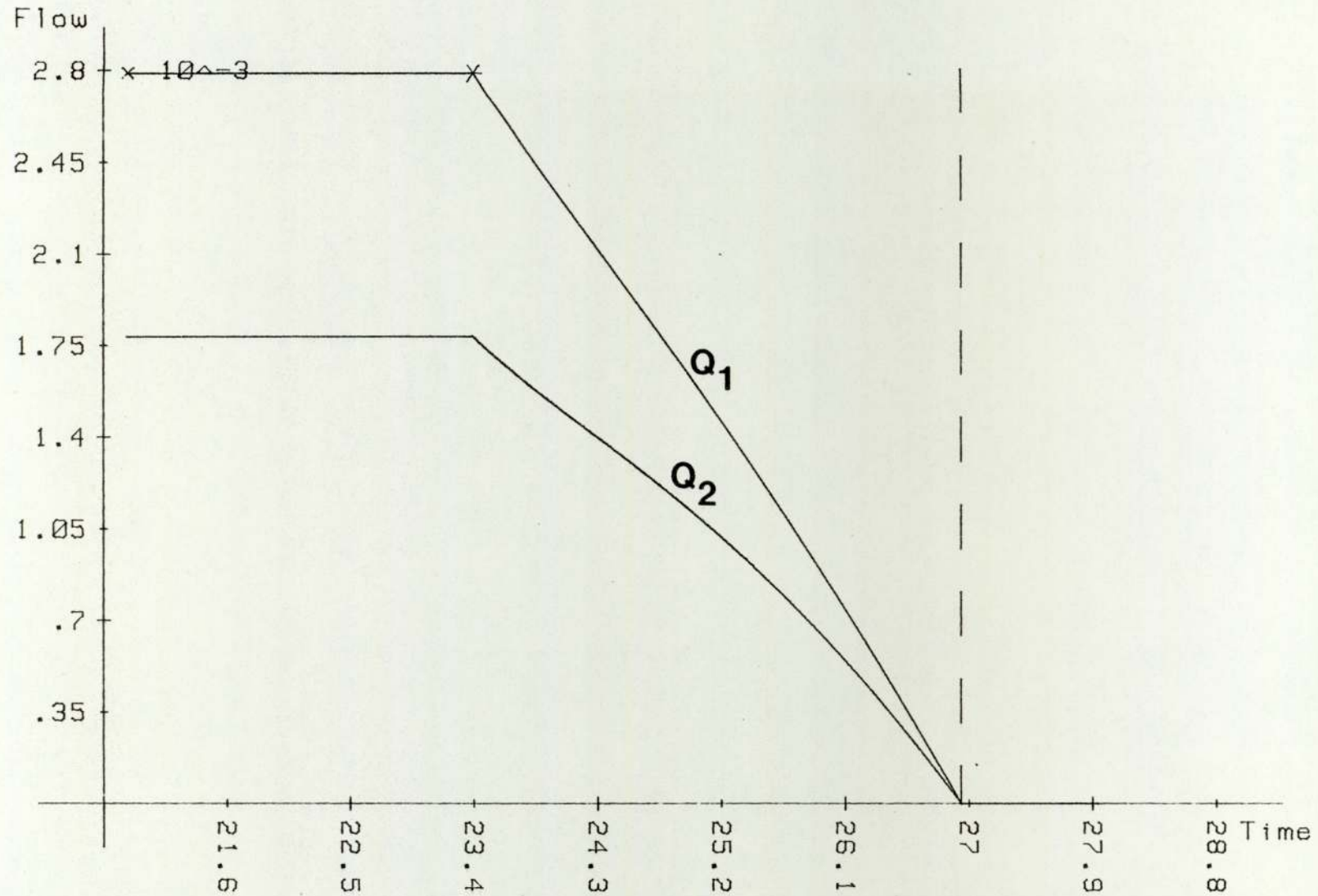


Fig. 3.12 Fluid Flow Rate - Time Response (Non-dimensional) (Mass = 1500 Kg)  
 Asymmetrical System With Area Ratio =  $6.400E-01$   
 Linearly Operated Valves With Time =  $3.5497E+00$  ( 10.0 mSec. )  
 External Force Applied =  $0.0000E+00$  Effective Damping Factor = .50

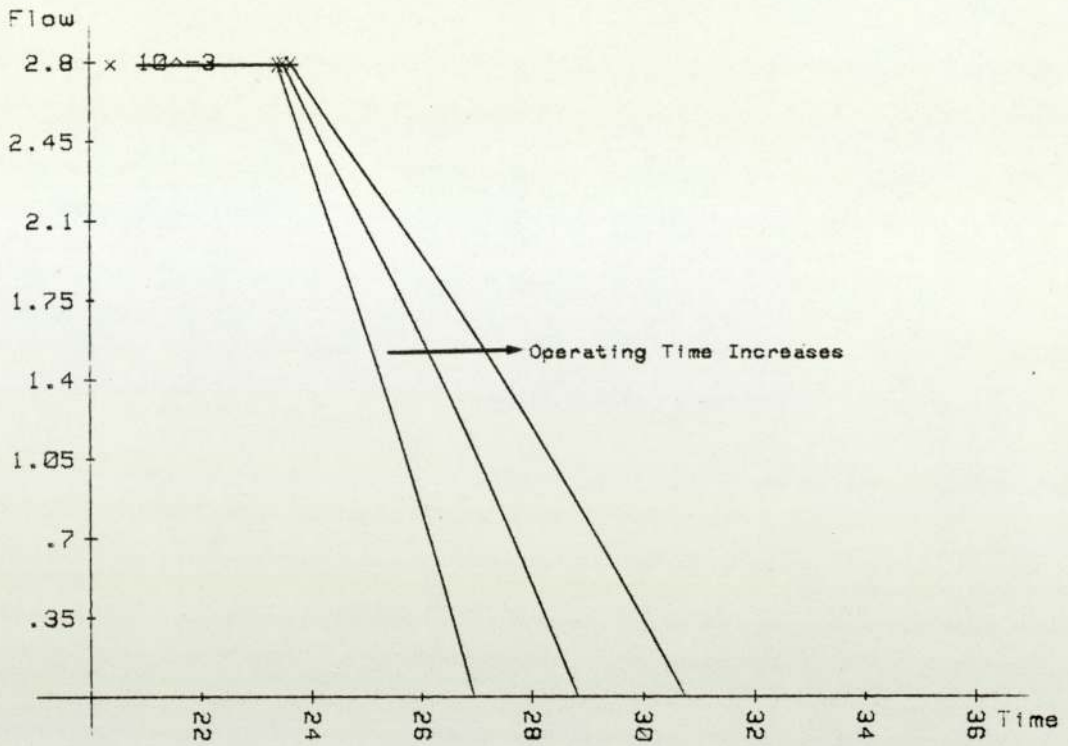


Fig. 3.13 Fluid Flow Rate Through Valve 1 - Time Response (Non-dimensional)  
 Asymmetrical Systems With Inertial Mass = 1500 Kg.  
 Area Ratio = 6.400E-01  
 Linear Valves With Different Operating Times  
 External Force Applied = 0.0000E+00 Effective Damping Factor = .50

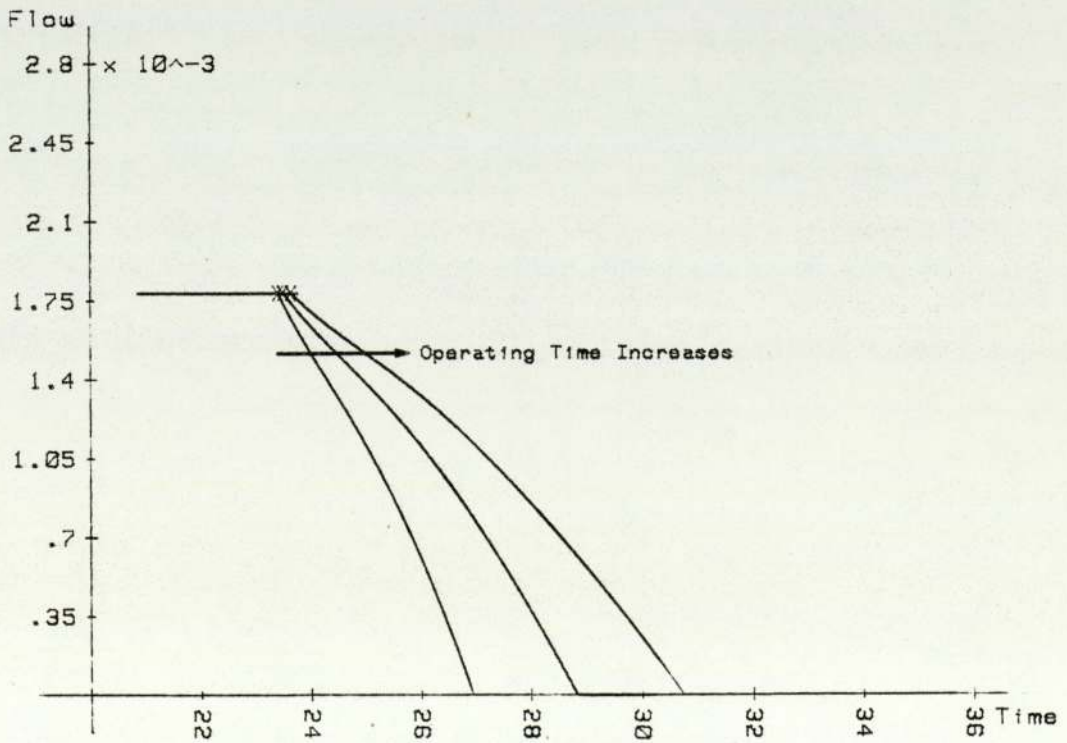


Fig. 3.14 Fluid Flow Rate Through Valve 2 - Time Response (Non-dimensional)  
 Asymmetrical Systems With Inertial Mass = 1500 Kg.  
 Area Ratio = 6.400E-01  
 Linear Valves With Different Operating Times  
 External Force Applied = 0.0000E+00 Effective Damping Factor = .50

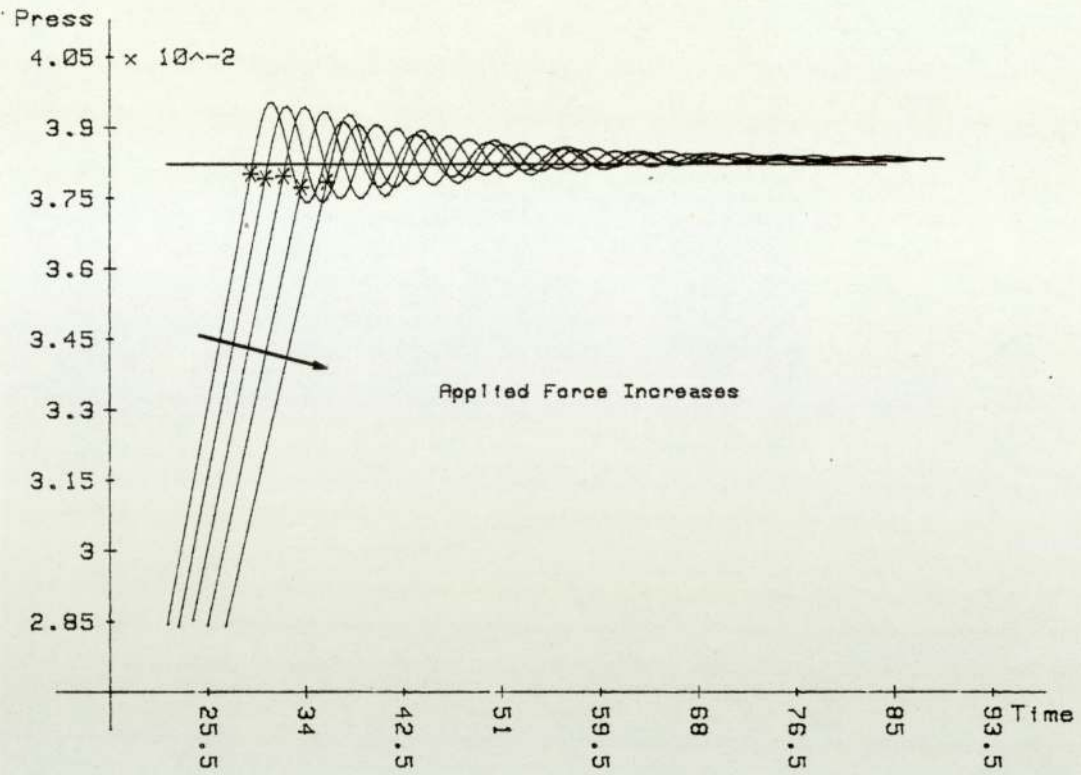


Fig. 3.15 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Symmetrical Systems With Inertial Masses = 1500 Kg.

ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 msec. )  
External Force Applied Varies Effective Damping Factor = .10

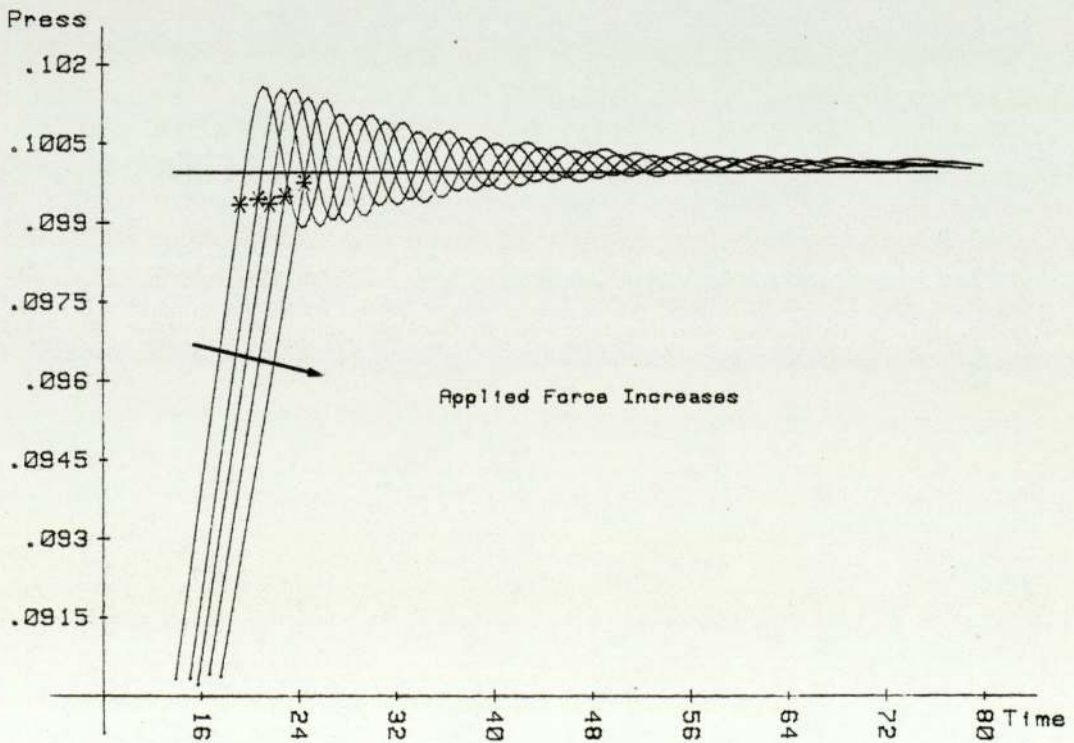


Fig. 3.16 Non-dimensional Ram Displacement - Non-dimensional Time Response  
Asymmetrical Systems With Inertial Masses = 1500 Kg.

Area Ratio =  $6.400E-01$   
ON-OFF Valves With Time-Delay =  $0.0000E+00$  ( 0.0 msec. )  
External Force Applied Varies Effective Damping Factor = .10

**CHAPTER      4**

**ARITHMETIC PROCESSING**

#### 4.1. Introduction

Arithmetic processing can be performed in either hardware or software. Hardware operation has definite advantage over software in speed. On the other hand, if the software for the mathematical operations is available and is ready for programming into read only memories (ROMs), then a special arithmetic processing unit (APU) is more expensive than the ROMs. For example, in 1982, an APU (Intel-8231A) costs £120 while a 2K erasable programmable read only memory (EPROM), 2716, can be bought for £3.

Using the Intel system design kit, SDK-85, a comparison between the hardware arithmetic processing and the software mathematical operation has been made to see whether the software arithmetic processing is suitable for this boom positional control.

#### 4.2. Function to be Evaluated

In order to account for the fluid compressibility effects, a switching function has been set up to give the positional control. This function is simple if there are no significantly variable forces acting on the system (please refer to Chapter 3). This function is used for each pair of the hydraulic rams: the trunk lifting rams and the arcing rams.

For any pair of rams, closing (switching) of the valves should be taken place when:

$$fn(y_d, y, \dot{y}, \ddot{y}) = \lambda_1 \ddot{y} + \lambda_2 \dot{y} + \lambda_3 y + \lambda_4 y_d + \lambda_5 = 0 \quad (4.1)$$

where  $y_d$  = required ram position for the desired boom position.

$y$  = current ram position.

$\dot{y}$  = current ram velocity.

$\ddot{y}$  = current ram acceleration.

$\lambda_i$  = coefficients determined by the system parameters;  $i = 1, 2, 3, 4, 5$ .

All the coefficients in (4.1) are dependant upon the system properties (e.g. ram diameters, inertial load, damping, valve characteristics, etc.) and their expressions are complicated. However, they are constants for a particular system and can be evaluated and converted into the binary numbers when the boom is installed. Therefore, in calculating the switching function, there is no need to re-evaluate these coefficients.

#### 4.3. Timing Test Program

The time taken by an APU to evaluate the above function (4.1) is compared with that used by the software evaluation. The actual switching function encountered will be more complicated than the one given above because of the geometry of the boom (since the aim is to control the boom cutter position). This simplified switching function is used to give an insight into the differences in the performance between the two techniques. The block structure of this test program



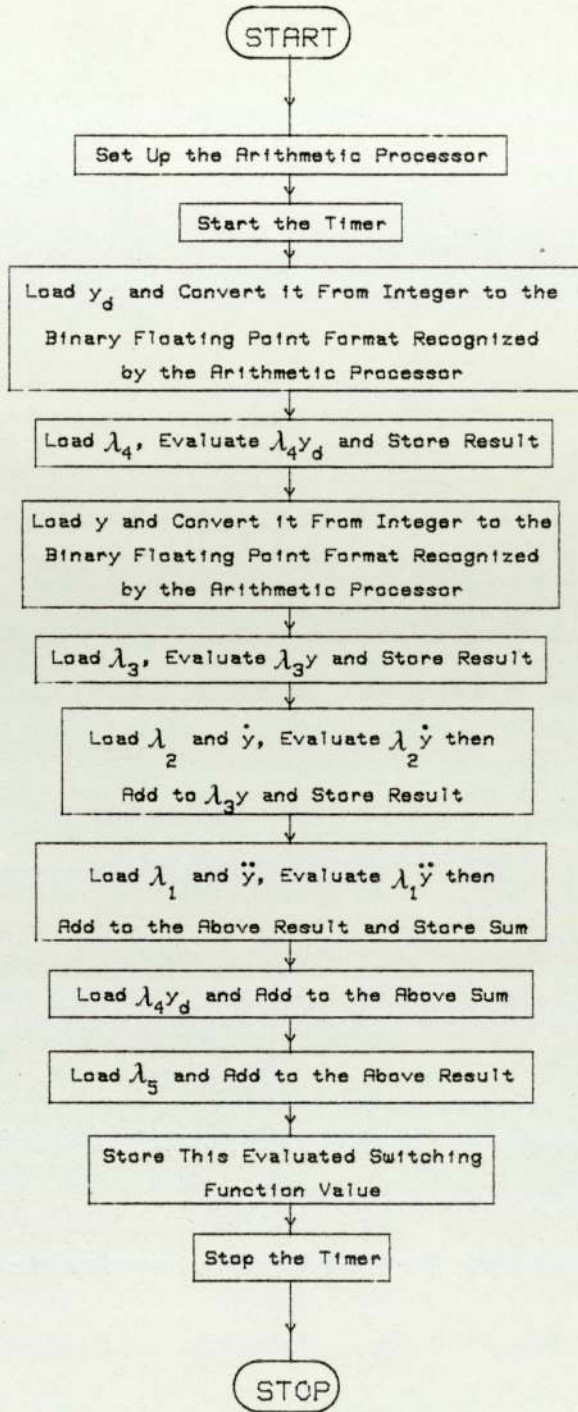


FIG 4.1 Block Structure of the Timing Test Program

is given in Fig. 4.1. It is similar for both arithmetic processors\*, hence only one diagram is presented. The actual assembly language programs are given in Appendix E.

#### 4.4. Explanation of the Test Program

##### 4.4.1. Set up the 'Arithmetic Processor'

For the hardware arithmetic processor, the APU, the set up procedure is simply applying a reset signal to the RESET pin of this APU [42]. This operation can be synchronised when the system is reset, since the microprocessor will issue a RESET OUT signal on recognising an active RESET IN signal. On the other hand, in case of the software arithmetic processor, the FPAL, setting is done by calling a set up procedure to initialise the floating point record (FPR) [43].

##### 4.4.2. Timing

It would be unfair to start the timing at the beginning of the program, because of the difference in the set up procedures, therefore the following timing scheme is used. When the actual procedure for evaluating the switching function is called, the timing will be started and it will be stopped on exit from that procedure. Hence only the time taken for evaluating the simplified switching is measured. This timing is done using the hardware timer resided in the 8155 on the SDK-85.

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\* A term arithmetic processor is introduced to represent either the special arithmetic processing unit (APU) for hardware operations or the Intel 8080/85 floating point library (FPAL) routines for software operations.

#### 4.4.3. Format Conversion

In the data capturing system, all the positional information is provided as single byte integers (OOH to FFH, or OD to 255D), therefore conversion from the integer format into the binary floating point format recognised by the arithmetic processor must be performed before any floating point arithmetic processing can be applied. However, all the other information is provided in the binary floating point format.

#### 4.4.3. Evaluation of the Switching Function

The function evaluation is done as in the expression (4.1) for the function  $fn(y_d, y, \dot{y}, \ddot{y})$  and the intermediate results are stored in temporary memory locations. This temporary storing of the results is required by the FPAL because it does not have any stack for keeping the intermediate results. On the other hand, the APU has a 16-byte data stack, therefore this storing operation may be omitted to increase the speed provided that the arithmetic operation performed does not destroy the current data stack contents [42].

#### 4.5. Binary Floating Point Format

Floating point representation (format) is more commonly known as the scientific notation in the decimal number system. The reason for using the floating point notation is to increase the number range for the calculations so that very large or small numbers can be handled. The floating point format consists of three fields, namely, the sign, mantissa and exponent. The sign field indicates



the sign of the number represented, while the mantissa and the exponent fields give the magnitude of this number. For example, in the decimal number system, the number, 123400, can be represented as +1.234E 5 in the scientific notation. In this format, the sign field is '+' indicating a positive number with a mantissa of '1.234' giving the four significant figures and the exponent field '5', meaning that the decimal point should be shifted five places to the right yielding the actual number's magnitude, i.e. the actual magnitude of this number is obtained by multiplying  $10^5$  to the mantissa.

Since the microprocessor is a binary digital device, the binary number system must be used. In a binary number system, all the fields are in a binary format with the exponent field indicating the power of two to be multiplied in order to obtain the actual magnitude. Both the APU and the FPAL recognise a 32-bit (4-byte) floating point format consisting of these three fields. These fields appear in this sequence:

Sign	exponent	mantissa
------	----------	----------

A floating point zero is represented by thirty two 0-bits in both the APU and FPAL floating point formats. However, non-zero numbers are represented differently in these two formats.

For non-zero numbers, in both formats, the sign field is the most significant bit (i.e. bit-31). A zero in this bit position indicates a positive or non-negative number while a one means that the number is negative.

Decimal Number	Decimal Floating Point Format	APU's Binary Floating Point Format			FPAL's Binary Floating Point Format		
		Sign 1 Bit	Exponent 7 Bits	Mantissa 24 Bits	Sign 1 Bit	Exponent 8 Bits	Mantissa 23 Bits
256	2.56 E 2	0	0001001	10000000 00000000 00000000	0	10000111	00000000 00000000 00000000
-256	-2.56 E 2	1	0001001	10000000 00000000 00000000	1	10000111	00000000 00000000 00000000
0.015625	1.5625 E -2	0	1111011	10000000 00000000 00000000	0	01111001	00000000 00000000 00000000
-0.015625	-1.5625 E -2	1	1111011	10000000 00000000 00000000	1	01111001	00000000 00000000 00000000
11	1.1 E 1	0	0000100	10110000 00000000 00000000	0	10000010	01100000 00000000 00000000

Table 4.1 Examples of the Number Formats

There are seven bits in the exponent field of the APU's floating point format representing the power of two to be multiplied to give the magnitude of this number. These seven bits are given in two's complement for the negative power of 2. The exponent field of the FPAL recognised format consists of 8 bits and is offset by a bias of  $2^7 - 1$  (i.e. the indicated exponent is 127D larger than the actual value).

Both formats have 24 significant bits. In the APU format, the mantissa field consists of the remaining 24 bits and represents the fraction field beginning with a 1-bit, and a binary point is assumed to precede the most significant fraction bit (bit-23 of the 4-byte format), i.e., the mantissa in APU's format is a 24-bit binary fraction number in the range [0.1B, 1.0B) or [0.5D, 1.0D). On the other hand, in the FPAL format, an implicit one bit is assumed at the left of the binary point of the fraction field, therefore only 23 bits are needed to be specified for this 24-bit mantissa. The number range of this mantissa field is therefore [1.0B, 10.0B) or [1.0D, 2.0D). A few examples have been given in Table 4.1.

These floating point formats can better be illustrated by the following examples:-

- (a) The decimal number 55D can be represented as +5.5E 1 in the decimal floating point format. In the binary number system, it is 110111 B and can be represented in binary floating point format either as  $1.10111 \text{ B} \cdot 10^{101} \text{ B}$

or  $0.110111B \cdot 10^{110} B$ . Hence in the APU format, 55D is represented as 0 0000110 11011100 00000000 00000000; while in the FPAL format, the exponent field is offset by 127D to the value, 132D, therefore this number (55D) is represented as

0 10000100 1011100 00000000 00000000.

- (b) The decimal number 0.3125D can be represented as +3.125E -1 in the decimal floating point format. As a binary number, it is represented as 0.0101B and can be put in the binary floating point format as either  $0.101B \cdot 10^{-1} B$  or  $1.01B \cdot 10^{-10} B$ . Hence in the APU format, 0.3125D is represented as 0 1111111 10100000 00000000 00000000; while in the FPAL format, it is 0 01111101 0100000 00000000 00000000 (the exponent field is offset by 127D).

Since the magnitude of a floating point number is obtained by scaling the mantissa with the exponent, the size of the exponent determines the number range which can be represented in the format. The range of the exponent field of the APU format is from -1000000B to 111111B (-64D to 63D), therefore the number range represented in the APU format is  $(-1.0 \cdot 2^{63} D, -0.5 \cdot 2^{-64} D], 0, [0.5 \cdot 2^{-64} D, 1.0 \cdot 2^{63} D)$ .

While the range of the numbers represented in the FPAL format is

$(-2.0 \cdot 2^{128} D, -1.0 \cdot 2^{-127} D], 0, [1.0 \cdot 2^{-127} D, 2.0 \cdot 2^{128} D)$ ,

since the range of the exponent field is from -127D to 128D.

## 4.6 Operation Required

### 4.6.1 FPAL

The evaluation of the switching function is done with the floating point arithmetic routines provided by the FPAL, therefore the mathematical operations are performed by calling the appropriate subroutines. The routines called upon include: two conversions, four multiplications, four additions, four storing operations and two loadings.

### 4.6.2 APU

All the mathematical operations are performed in hardware within the APU. Every mathematical operation is initiated by issuing (writing) a command code to the APU's command port. Since the APU has a 16-byte data stack, storing of intermediate results may be omitted if these mathematical operations do not destroy the current stack contents. However, every operand should be loaded onto the data stack before an operation can be initiated, so there are at least nine loadings to be performed. The numbers of conversions, additions and multiplications are the same as those required in using the FPAL. It has to be pointed out that the time taken for transferring the data from memory to the APU data stack depends upon the hardware configuration of the system. Instructions in the data transfer group are used for a memory mapped APU, while input and output instructions are used for an I/O mapped APU. Normally, the time taken by an instruction in the data transfer group is less than that used by an I/O instruction. For example, an 8085A microprocessor requires ten states to execute an



Operating Condition	Number of Clock Cycles Used	Time Used in mSecs for a 3.072 MHz System Clock	Ratio = $\frac{\text{Time Used in Software Evaluation}}{\text{Time Used Under Given Operating Condition}}$
FPAL	13619 D	4.433	1.000
I/O Mapped APU With Storing of Intermediate Results	4706 D	1.532	2.894
I/O Mapped APU Without Storing of Intermediate Results	3276 D	1.066	4.157
Memory Mapped APU With Storing of Intermediate Results	4529 D	1.474	3.007
Memory Mapped APU Without Storing of Intermediate Results	3162 D	1.029	4.307

Table 4.2 Time Taken to Evaluate the Simplified Switching Function Under Various Operating Conditions

IN instruction while it takes only seven states to perform the instruction, MOV A,M.

#### 4.7. Results

Since the actual time used in an arithmetic operation also depends on the values of the operands for both arithmetic processors, the same numerical values were used for this timing test. Timing has been done for both the memory mapped and the I/O mapped APU and the results are presented in Table 4.2. These timings confirm that the hardware operations are considerably faster than the software operations.

#### 4.8. Boom Moving Rate

The time taken for the boom lifting has been measured and the normalised result is shown in Figure 4.2. The boom took 5.5 seconds to move down from the upper level to the bottom one. The ram displacement was measured using a linear displacement transducer and then this analogue signal is converted into a digital signal by an analogue-to-digital converter (A/D converter). If an 8-bit A/D converter is used, the resolution will be  $1/(2^8 - 1)$  or  $1/255$  of the full range. If the range is adjusted to cover only the ram stroke, then for every bit change of the A/D reading, the analogue signal should have changed by  $1/255$  of the whole range. For this particular boom arrangement, the time taken to give this change after the ram has reached its terminal velocity is 21.76 mSec which is comparatively much longer than the time taken to evaluate the simplified switching function.

Time Intervals: 0.015 Sec

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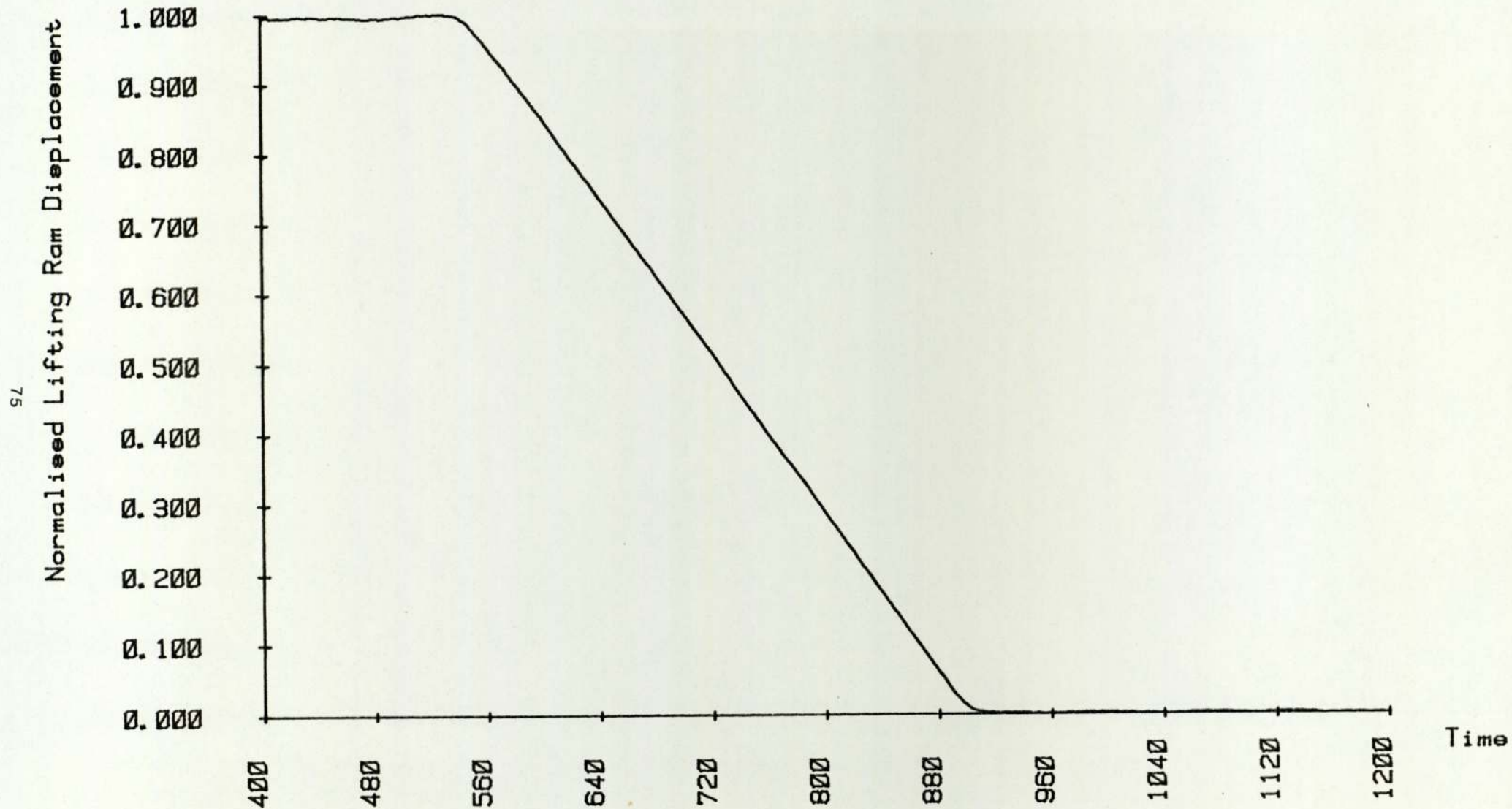


FIG. 4.2 NORMALISED LIFTING RAM DISPLACEMENT - TIME RESPONSE

#### 4.9. Comments

Should the switching function be that simple, the function evaluation can either be done in software or in hardware. Since the hardware operation has no 'apparent' advantage over the software in speed and is more expensive, the software treatment is recommended.

However, in the actual boom system, the desired position will be given for the boom cutter and the required ram position are to be evaluated according to its geometry. This will greatly increase the time taken to calculate the switching function. If more complicated mathematical operations are required, a considerable amount of time will be saved by the hardware APU. A timing test has been done for evaluating the sine function of an angle, the time has been reduced from 45.9 mSec to 1.36 mSec by using the M/M APU (i.e. the APU operates about 34 times faster than the software routines).

Considerable amounts of memories have also been saved when using the APU, even though it is memory mapped at the suggested addresses which amount to 0.5K (512D bytes). For example, the program length of these simple timing tests are 1945D for the software evaluation, 230D bytes for the I/O mapped APU hardware evaluation and 214D bytes for the memory mapped APU hardware operation (in addition to the 0.5K bytes).

Furthermore, with larger sized valves or a higher system supply pressure the boom moving speed can easily be increased. Therefore high speed mathematical operations provided by an APU are strongly recommended for the real system.

**CHAPTER 5**

**CONTROL ALGORITHMS**

**AND**

**PROGRAM STRUCTURE**

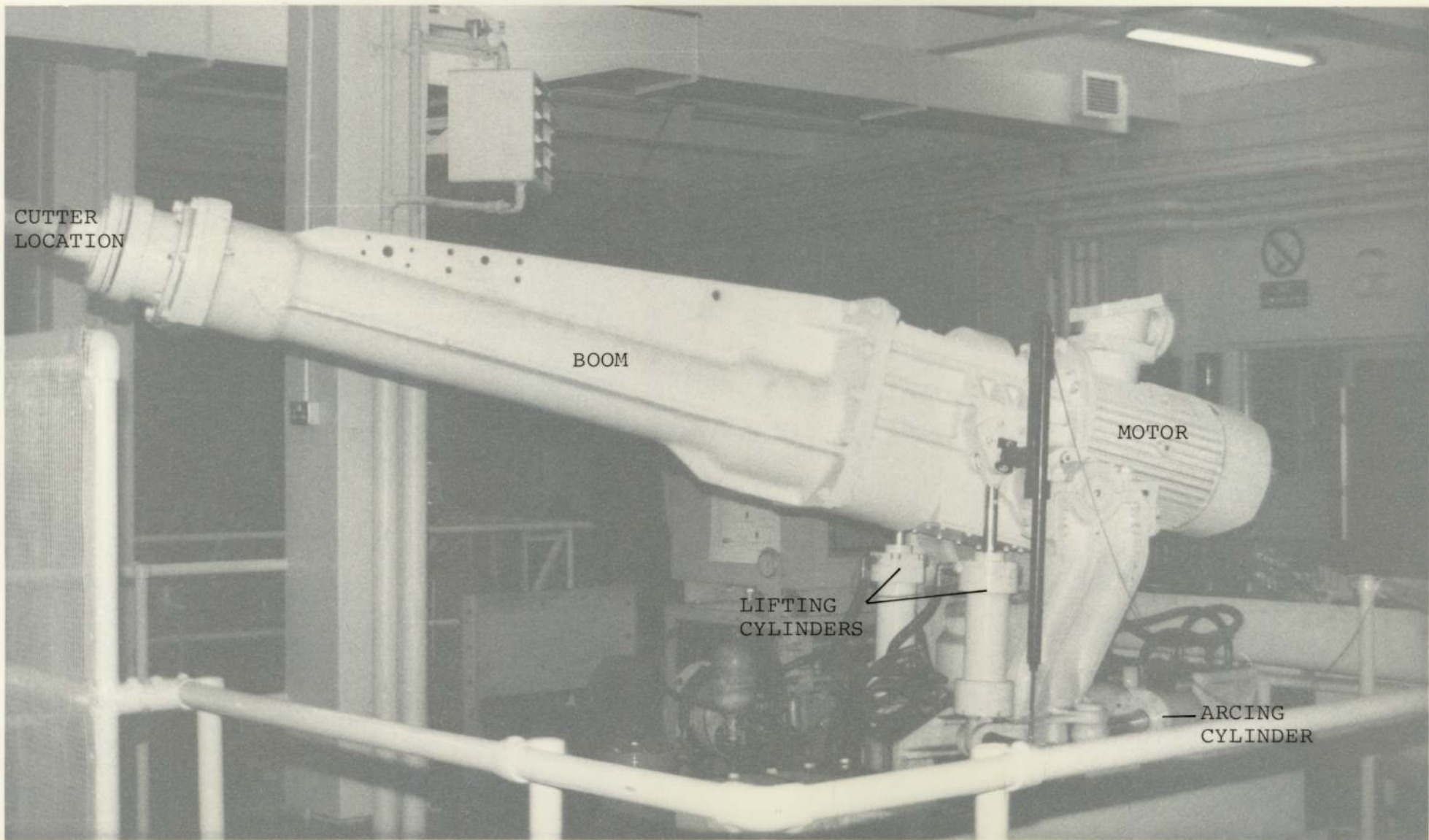


Fig 5.1 The Boom Ripper Assembly under Test

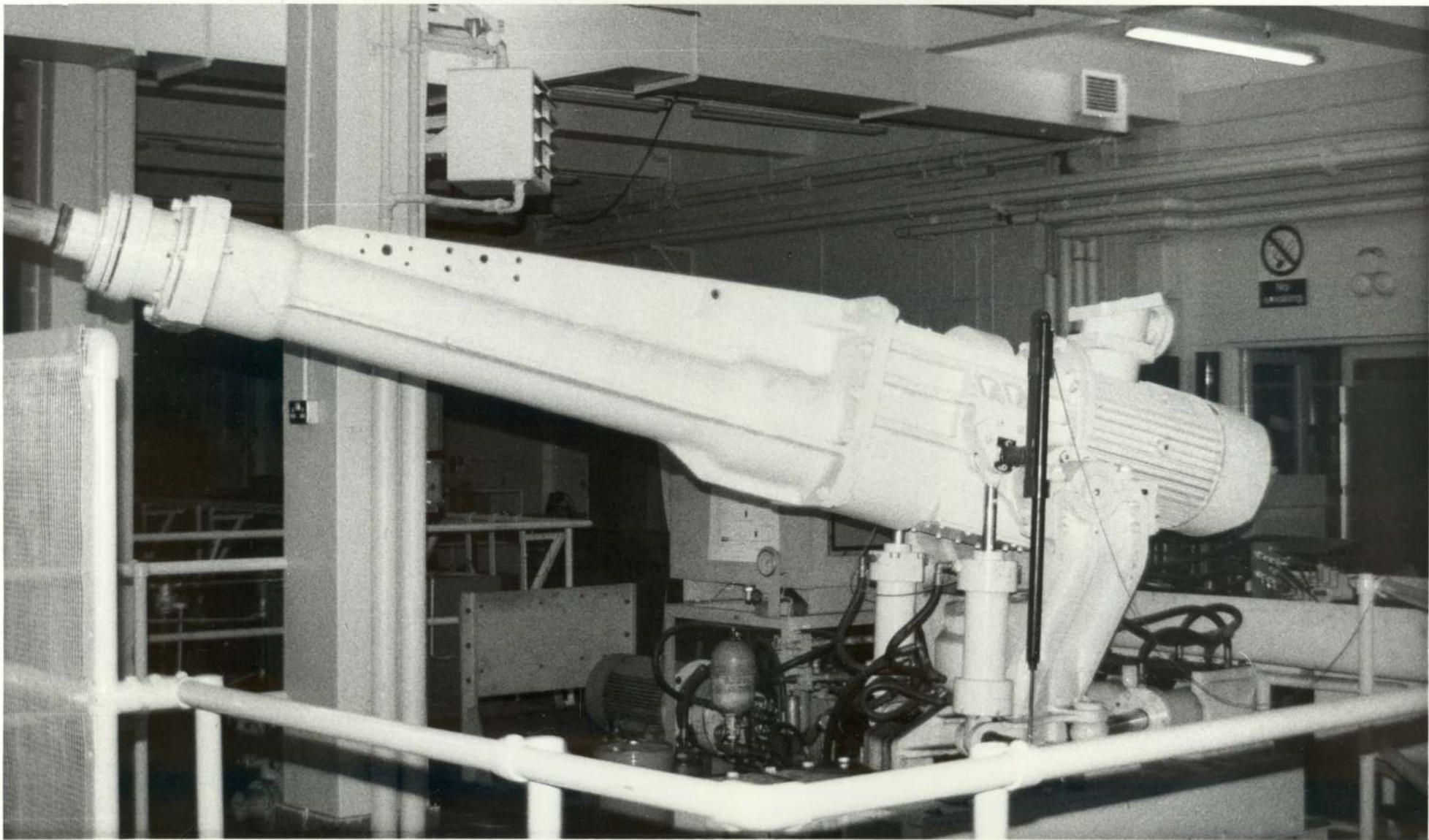


Fig 5.1 The Boom Ripper Assembly under Test



### 5.1. INTRODUCTION

It should be recalled that the aim of this project is to improve the control of the cutter position of a boom ripper, thus to deskill the tunnelling operation and to increase the overall performance of the roadheader. It would be helpful in deriving the control algorithm, if one has a rough idea of how the tunnel is driven using a boom ripper. In general, the cavity of the roadway is first cut layer by layer from the lowest one, the tunnel profile is then trimmed and followed by the setting up of the support arches. Hence the proposed system should be able to provide the cutting and trimming facilities in addition to moving the boom cutter from one position to another.

It can be seen from Figure 5.1 that the boom ripper has two degrees of freedom, namely slewing and lifting, provided by the paired hydraulic rams. When the machine is operating underground, it advances using its caterpillar tracks (see Figure 1.1) after one face has been cut, but this advancing of the roadheader is not considered in the current study. However, this can easily be included in the control system if the necessary sensors and mechanisms are available.

Because there are two degrees of freedom associated with the boom movement, it will be more convenient if the tunnel profile is defined in a coordinate system that resembles these degrees of freedom. Therefore, in this positional control system, the profile data are captured in pairs which describe the altitude and slew position of a point on the profile boundary, these data pairs will later be referred

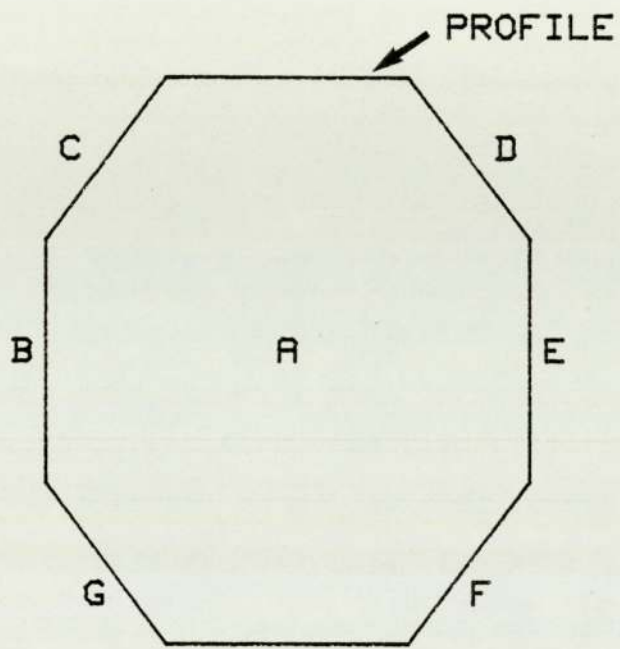


FIG 5.2 A Hypothetical Profile

to as altitude-slew pairs. Similarly, the desired position and feedback data will also be supplied in altitude-slew pairs.

It is undesirable for the boom to move outside the defined profile boundary, an algorithm has been developed to restrict the boom movement. Considering the hypothetical profile shown in Figure 5.2, the following actions will be required. When the current position of the cutter is at (A), it is free from the profile boundary, therefore its movement in all directions is allowed. When the cutter is currently at (B), it will only be allowed to move vertically and/or towards the right; if it is currently at (E), the boom cutter movement will be restricted to vertical and/or left only. In addition to the downward movement permitted, the cutter is allowed to move towards the right while it is currently at (C), but towards the left if its current position is at (D). Similarly, the cutter is allowed to move up and towards the left at (F); while it is permitted to move up and towards the right at (G). In other words, the cutter is only permitted to move to a position without crossing the profile boundary.

The proposed positional control system is composed of the mechanical, hydraulic, electrical and micro-electronic sections. The control program resides in the micro-electronic section of the system. In this chapter, the structure of the control algorithm is presented and the listing of the assembly language programs can be found in Appendix G.

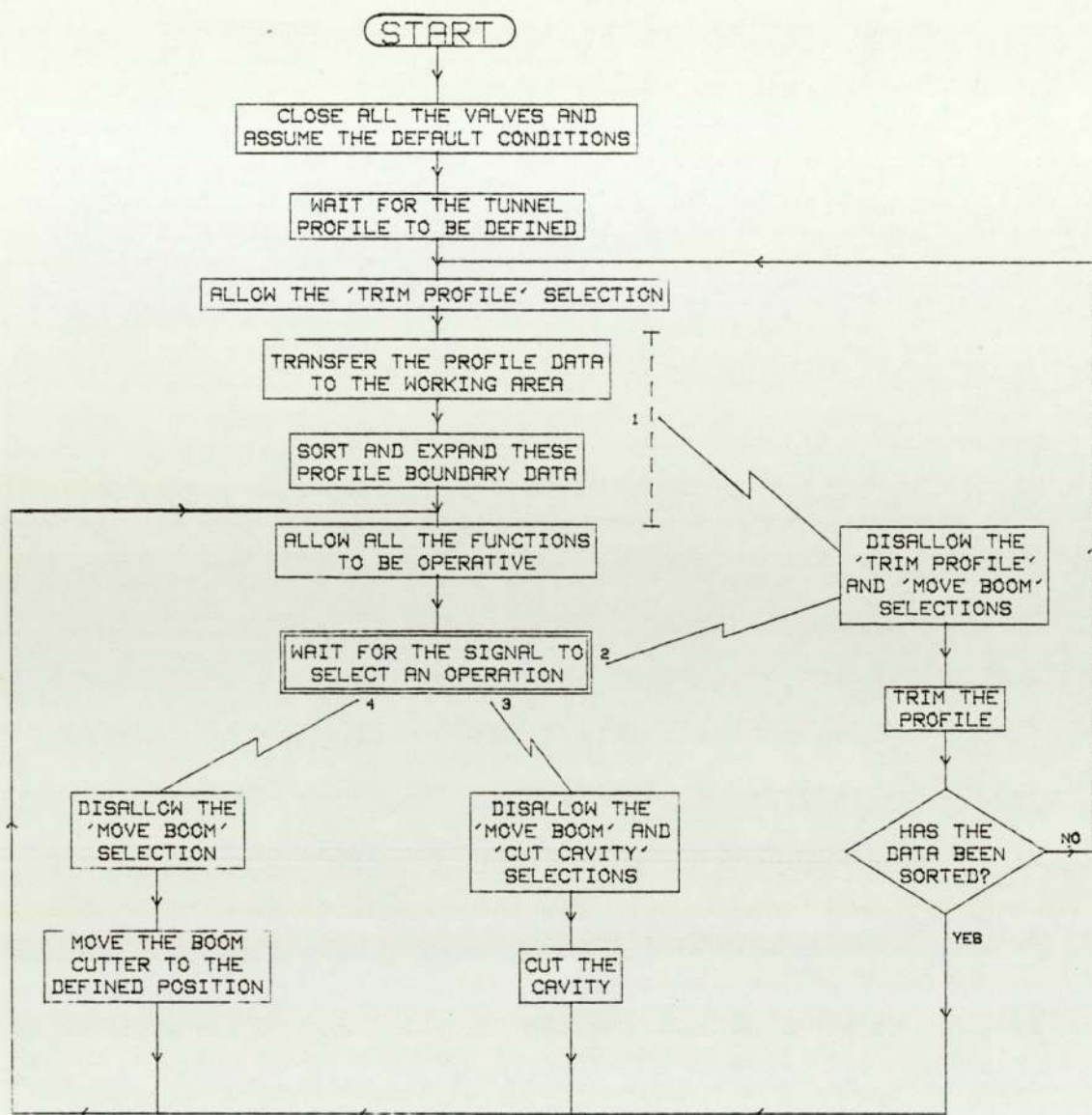


FIG 5.3 Flow Diagram of the General Structure of the Control Algorithm

## 5.2. GENERAL STRUCTURE OF THE CONTROL ALGORITHM

There is normally an initialisation procedure which will be performed on a fresh (sometimes known as a cold) start; and it is often to begin the operation with all the hydraulic valves closed. Therefore the control program will first ensure that all the valves are closed, assume the default conditions and then wait for the tunnel profile to be defined. There are three functions available in the system: moving the boom cutter from the current position to that defined by the operator (Move Boom), cutting the roadway cavity (Cut Cavity) and trimming the tunnel profile (Trim Profile) defined by the operator. However, for the reasons mentioned later, there will be four branching possibilities: the first and the second for trimming the profile, the third for cutting the cavity and the fourth for moving the boom cutter. A flow diagram for the general structure of this control algorithm is shown in Figure 5.3.

'Trim Profile' is the first operation that can be selected as soon as the tunnel profile has been defined. If this operation is not selected, the profile data will then be transferred to the data manipulating area (so that the original data can be preserved), where they will be sorted and expanded into a table in order to establish the profile boundary. This table consists of data sets, which contain the minimum and maximum slew bounds for every altitude from the lowest level to the highest one. Hence the table enables the cutting of the cavity layers and the checking to see if the boom cutter is moving within the specified profile boundary.

After the data have been sorted and expanded, all the function selections will be enabled. However, re-selection of a function will not be permitted once it has commenced its operation, since there is no need to select the same function twice. Hence the function selection is just an initiating procedure for that activity. After the completion of an operation, its function selection will again be enabled to permit a second trial if desired.

The 'Move Boom' control has two functions: selecting the 'Move Boom' operation and controlling the boom moving speed (there are two speeds available, fast and slow). It is undesirable to move the boom cutter to another position (away from its path) when it is cutting the cavity or is trimming the profile. Hence the 'Move Boom' selection control will be disabled if any one of these operations is selected unless that operation has been stopped or completed. During these two operations, the boom cutter movement is controlled by the program.

### 5.3. INITIALISATION

This is the first segment to be executed on starting the operation or resetting the system. All the input/output (I/O) ports are first declared so that the appropriate signals can be sent to or received from these ports. All the directional valves will then be closed. Initially, all the function selections are disabled, therefore no operation can be performed while the system is waiting for the tunnel profile to be defined by the operator. The system will also assume all the default conditions: stop

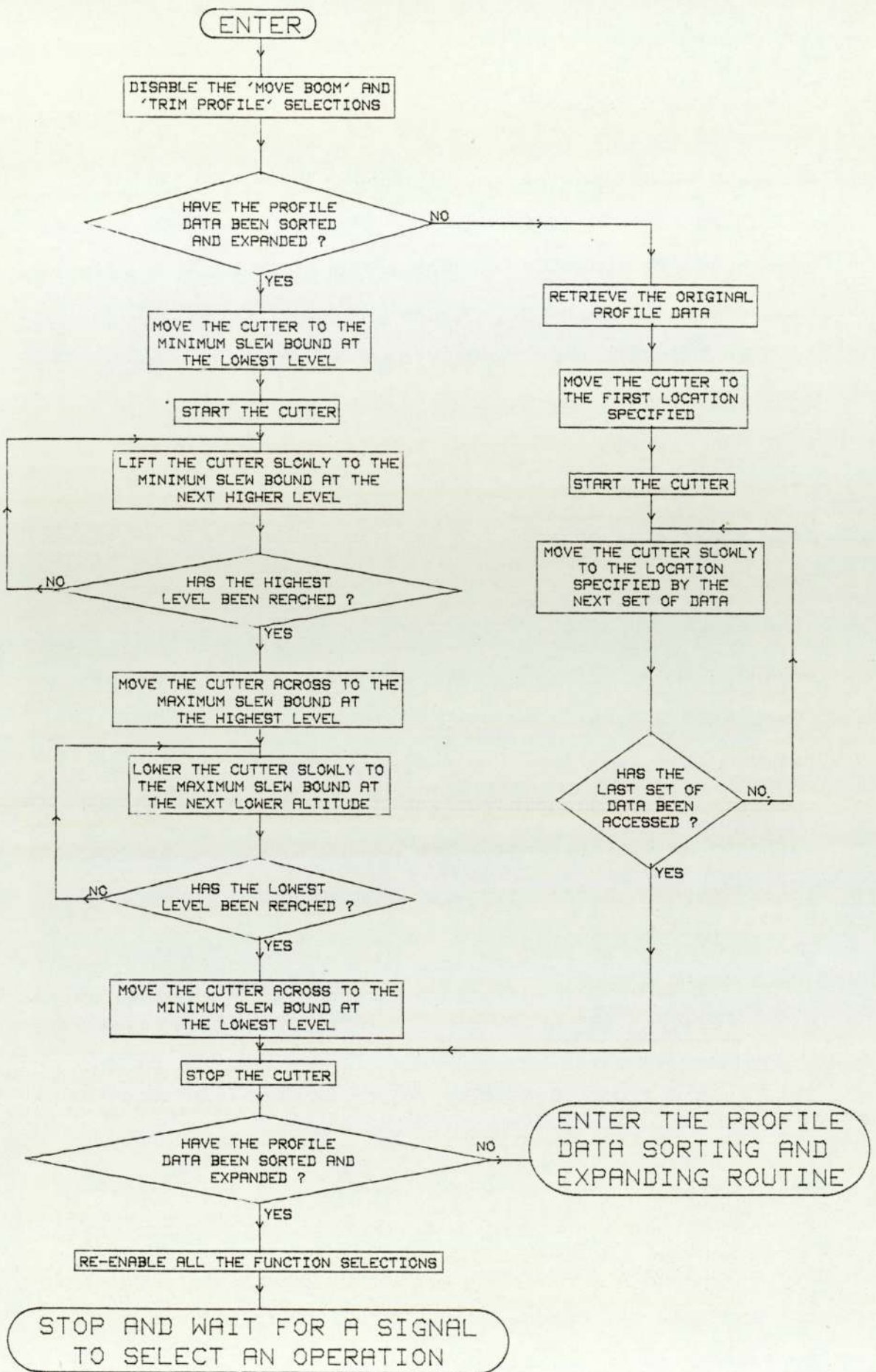


FIG 5.4 Flow Diagram for the 'Trim Profile' Routine

the boom cutter, assume slow speed for the 'Move Boom' mode and set up the appropriate entry locations for the various routines when the selection signals for different functions are recognised.

#### 5.4. PROFILE TRIMMING

After the profile has been defined by the operator, the 'Trim Profile' operation may be selected. Although in normal practice, this operation will not be performed until the cavity has been excavated. However, since the profile has been defined, it can be trimmed if desired. It has been mentioned in Section 5.2 that, as soon as the profile has been defined, the program will call up the data sorting and expanding procedures which take a finite amount of time to complete. The 'Trim Profile' operation can therefore be selected before or after these sorting and expanding procedures have been completed (indicated as the branching at 1 or 2 in Figure 5.3). Different actions will therefore be taken to trim the profile depending on when the branching is originated. A flow diagram for this program segment is shown in Figure 5.4.

If the 'Trim Profile' operation is selected after the profile data have been sorted and expanded, then the boom will be moved to the minimum slew bound at the lowest level and the cutter will be started. The profile trimming process then commences with the cutter being moved slowly along the defined profile. On the other hand, if it is selected before the data sorting and expanding procedures have been completed, the original profile data



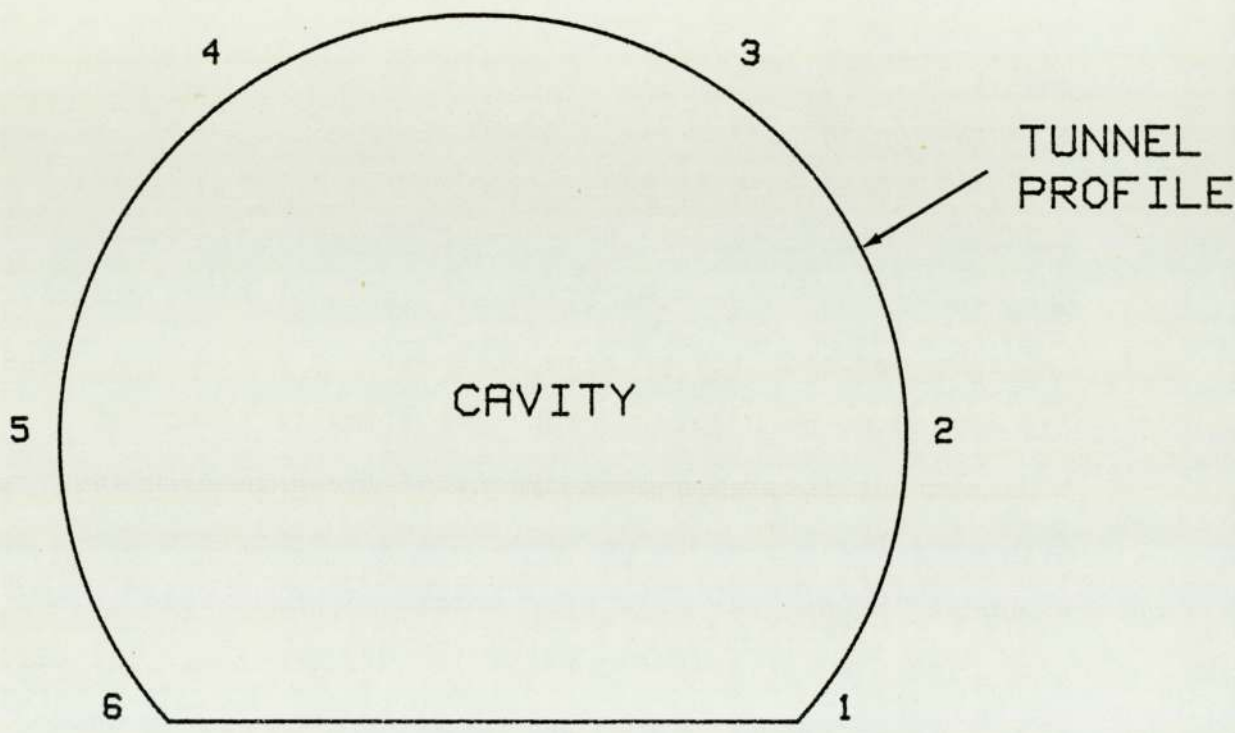


FIG 5.5 A Sample Tunnel Profile

are retrieved from the data capturing system, the boom cutter will then be moved to the first position defined by the operator in specifying the tunnel profile. The cutter will also be started and moved slowly along the profile. The boom cutter movement will follow the way in which the profile is defined by the operator.

If the sorting and expanding procedures have not been completed, the profile data will be sorted and expanded after the completion of this profile trimming process. Otherwise, the system will then wait for the signal to select an operation.

It has been mentioned that there are two ways in moving the boom cutter along the profile: one for the original data and the other for the sorted and expanded data. Figure 5.5 shows a sample tunnel profile which has been defined as the path: 1-2-3-4-5-6-1.

The algorithm for moving the boom cutter along the profile using the unsorted data is relatively simple. The profile data are first transferred to the working area in the form as they are captured. Then the cutter will be moved to the location specified by the first data pair (the profile data are captured as altitude-slew pairs). It will be started and moved slowly to the locations specified by the data pairs in the order as they are captured. After the boom cutter has been moved to the location defined by the last pair of the profile data, it will then be stopped. In other words, the boom cutter will follow the path defined by the operator (the path: 1-2-3-4-5-6-1 in Fig. 5.5).

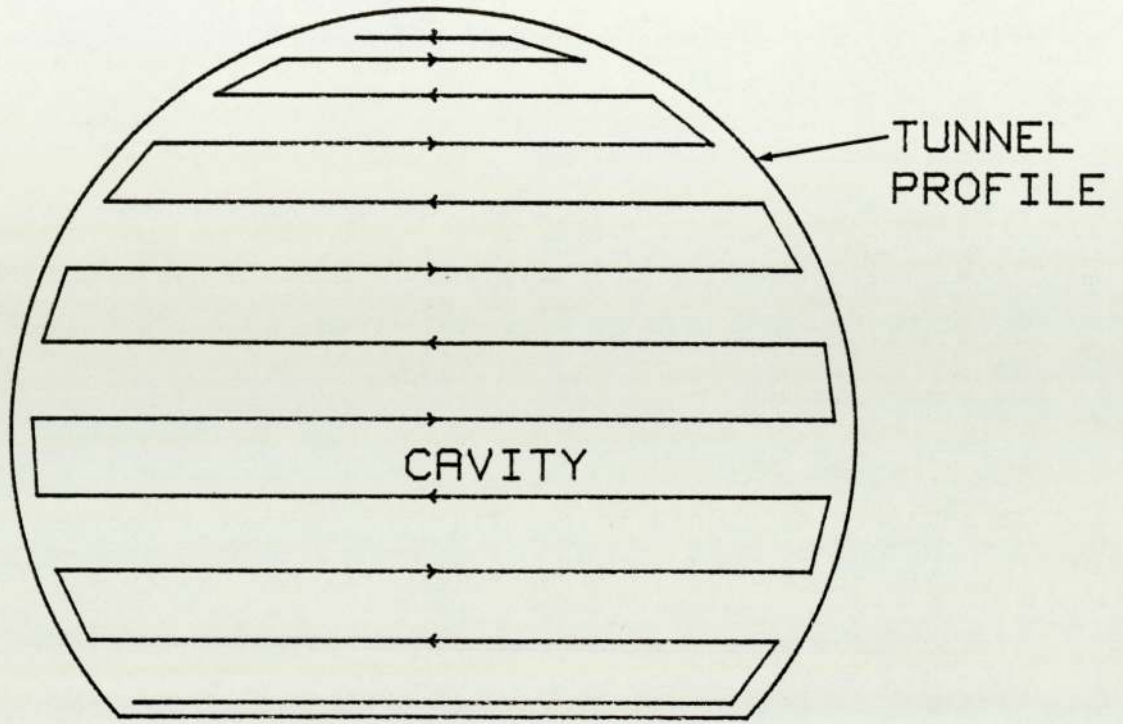


FIG 5.6 A Sample Cavity Cutting

The algorithm for moving the cutter along the profile using the sorted and expanded data is different. The profile data have been sorted and expanded into a table that contains the minimum and maximum slew bounds for every altitude from the lowest level to the highest one. Therefore the boom cutter can be moved to the location where is the minimum slew bound at the lowest altitude. After the cutter has been positioned at this location, it will then be started and raised slowly to the minimum slew bound at the next higher level, until the cutter reaches the highest altitude. Then it will be moved across from the minimum slew bound to the maximum one at this altitude. Afterwards it will be lowered to the maximum slew bound at the next lower level, until the lowest one has been accessed. The cutter will finally be moved across this lowest boundary from the maximum slew location to the minimum one, before it is stopped. Effectively, the cutter has been moved along the defined profile (along the path: 6-5-4-3-2-1-6 as shown in Figure 5.5).

#### 5.5. CAVITY CUTTING

The 'Cut Cavity' operation cannot be selected until the tunnel profile has been defined and the data have been sorted and expanded. After this operation has been selected, the boom cutter movement is controlled by the program. A sample path of this movement is shown in Figure 5.6. Similar to the 'Trim Profile' operation, the 'Move Boom' and the 'Cut Cavity' selections are disabled once this routine has been entered. A flow diagram for

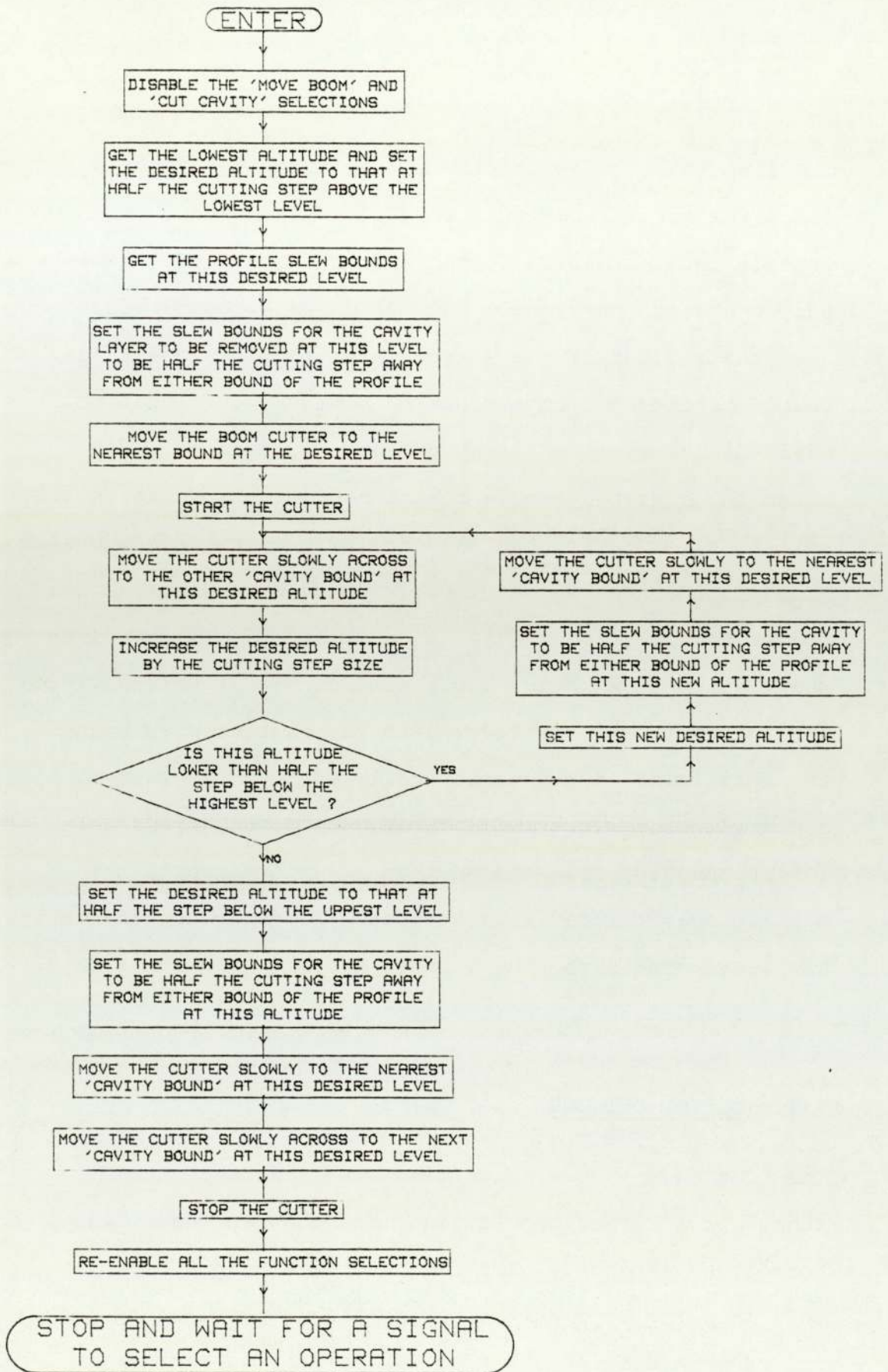


FIG 5.7 Flow Diagram for the 'Cut Cavity' Routine

this routine is shown in Figure 5.7.

After this operation has been selected, the highest and the lowest levels of the cavity layers to be excavated by this operation are first evaluated. These cavity levels\* are chosen to be half the desired vertical depth of cut away from but contained in the actual profile boundary. The slew bounds at this lowest cavity altitude are similarly reduced. The boom cutter will be located to the nearest cavity slew bound at the lowest cavity altitude. After the cutter has been moved to this position, it is started and the 'Cut Cavity' process commences. The cutter is now moved slowly across to the other reduced slew bound so that this cavity layer will be removed. After this layer has been cut, the slew bounds at the level, which is one desired vertical depth of cut higher, are similarly reduced (half the depth of cut smaller on both sides than the actual profile slew bounds). The cutter will be advanced to the reduced slew bound on the same side at that higher level. The movement of the cutter is restricted to within the prescribed profile boundary. Then this cavity layer will be cut. The process is repeated until the highest cavity level has been removed.

After the completion of this operation, a cavity, which is smaller than the actual profile by half the vertical

---

\* In this section, when a term is preceded by the word cavity, it refers to the reduced value. Therefore cavity slew bound is the same as reduced slew bound. When a term follows the word profile, it means the actual value. Hence the profile boundary is another way of referring to the actual boundary.

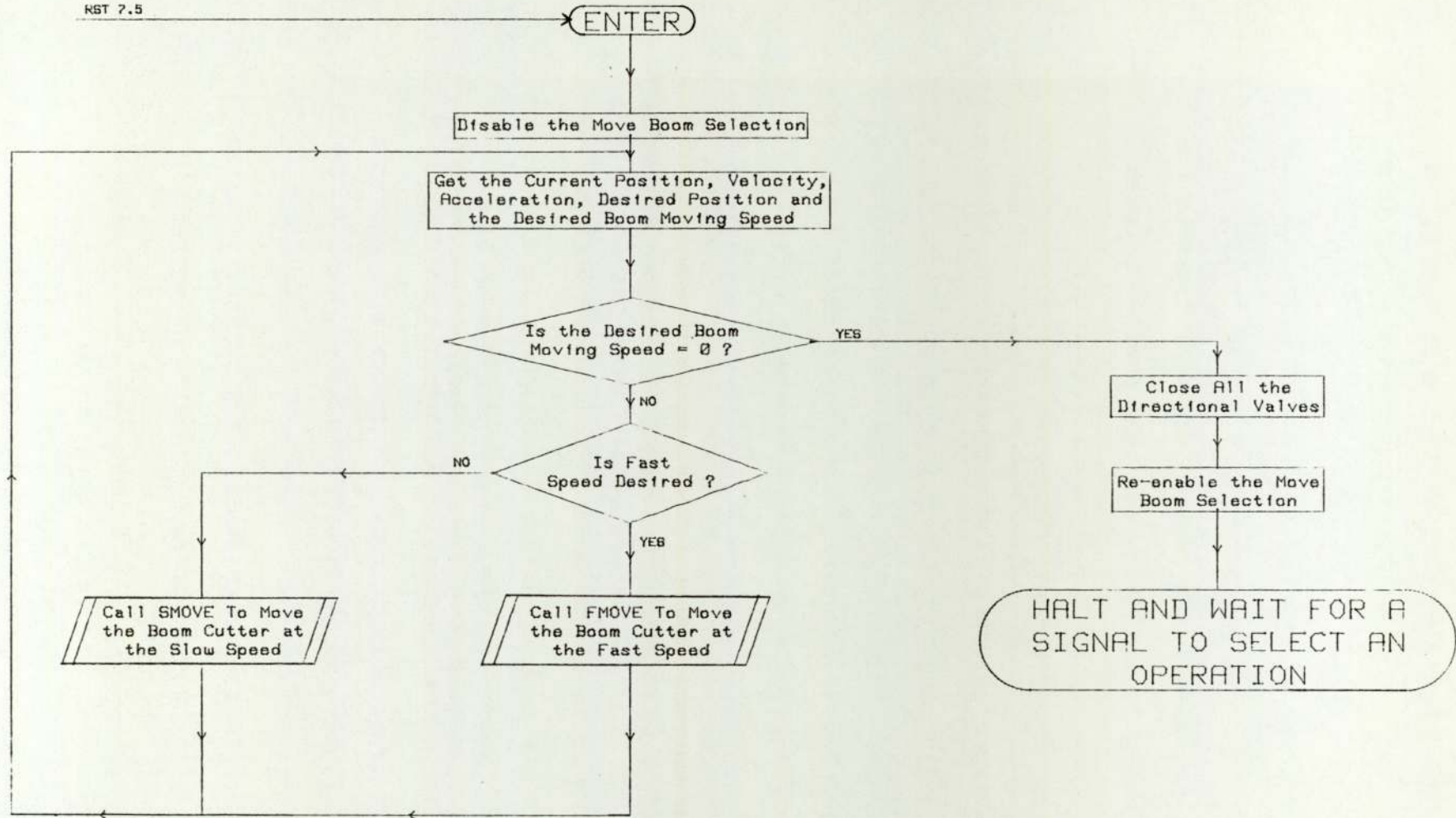


FIG 5.8 A Flow Diagram for the 'Move Boom' Routine

cutting step, will be formed. All the three function selections will then be enabled. The final profile boundary can then be removed by the 'Trim Profile' operation.

#### 5.6. BOOM MOVING

After the tunnel profile has been defined and the completion of the profile data sorting and expanding procedures, the 'Move Boom' operation can be selected while the cutter is neither cutting the cavity nor trimming the tunnel profile. It is desirable to have the 'Move Boom' and the 'Boom Moving Speed' selectors combined into a single control. When this control is operated, it first sends a selection signal to put the system into the 'Move Boom' mode and continuously informs the system the chosen boom moving speed. A flow diagram of this 'Move Boom' control algorithm is given in Fig. 5.8.

After the 'Move Boom' selection signal has been recognised, this function will only act as the 'boom moving speed' selector. It selects one of these three possibilities: fast, slow and no-move. On entering this routine, the main control system will collect the information on the current and desired positions as well as the chosen speed for the boom cutter (from the 'boom moving speed' selector). The appropriate subroutine will be called for moving the boom cutter at the desired speed. The subroutine, SMOVE, for moving the boom cutter slowly, will be described in a later section. The subroutine, FMOVE, for the fast moving of the cutter will be discussed in Chapter 7. This process will



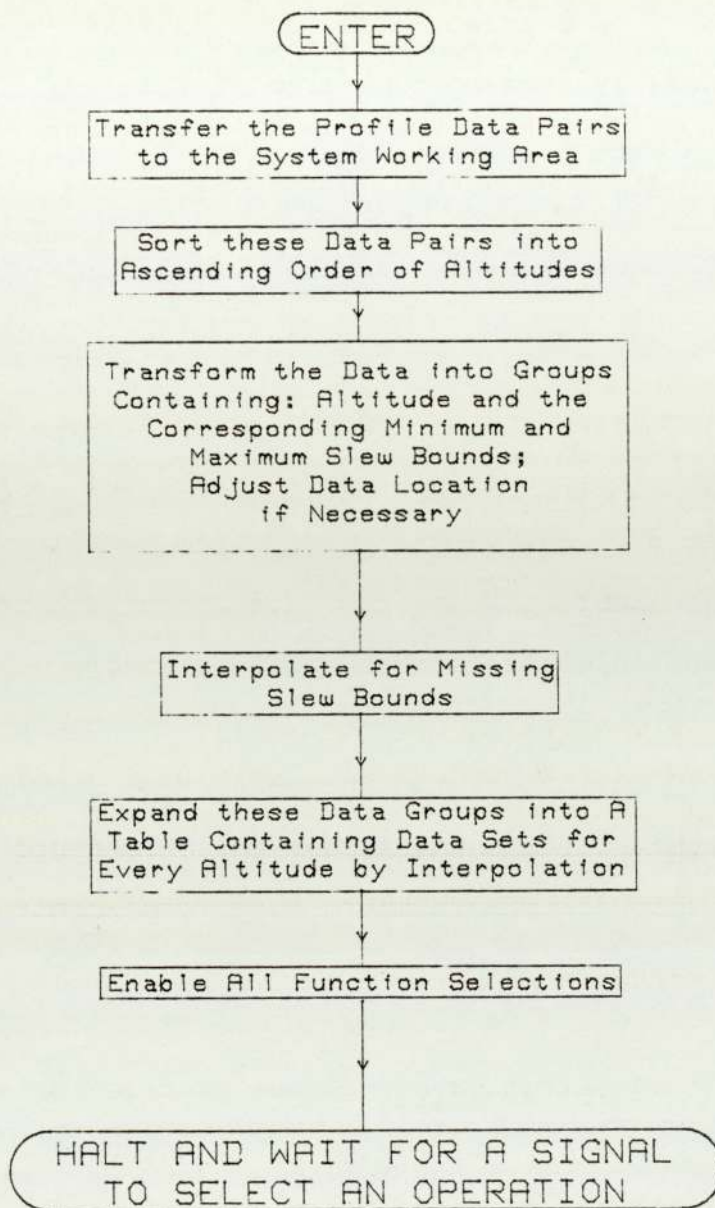


FIG 5.9 A Flow Diagram for the Data Sorting and Expanding Procedures

be repeated until the 'boom moving speed' selector selects a 'no-move' mode. When this 'no-move' selection is recognised, all the directional valves will be closed and all the function selections will also be re-enabled. This allows the system to wait for a signal to select an operation.

#### 5.7. SORTING AND EXPANDING THE PROFILE DATA

These procedures are performed immediately after the tunnel profile has been defined. When the data are being sorted and expanded, only the 'Trim Profile' operation can be selected. Fig. 5.9 shows the flow diagram for these routines. The captured data are first transferred to the working area so that the original data can be preserved. These data will then be sorted in the ascending order of the profile altitudes. The sorted data will be transformed into groups containing the altitudes with their two slew bounds for the defined profile. Therefore, the regrouped data will be in sets of three and will appear as:

lowest altitude, min. slew bound, max. slew bound, ...,  
highest altitude, min. slew bound, max. slew bound.

The original data are captured in altitude-slew pairs when the operator defines the tunnel profile. This data capturing is done using the system developed by J H Knight in a parallel project at the University of Aston. He built into this data capturing system an algorithm to avoid capturing incorrectly specified profile data. These data can be either the duplicating data pairs (because the data sampling rate is faster than the speed that the operator

can define the profile), or the data 'mistakenly' generated by the analogue-to-digital converter due to its accuracy. A tolerance has been introduced in this algorithm, it is therefore possible to miss out one of the slew bounds during the data capturing process.

In the transformed data groups, there are three data per group instead of four to define the two slew bounds at any altitude. If only one profile slew bound has been captured, the system will generate the 'missing' slew bound by assuming the same value as the captured data. An algorithm has been derived to prevent the system from loss of data in the data re-grouping process: all the remaining data are copied to their next higher locations if there will be a 'crash' resulted from any data generating process.

After the data have been transformed, any data group with the same left and right slew bounds (i.e. only one slew bound has been captured for that altitude), except the groups for the lowest and the highest altitudes, can be recognised as deficient in one data. The missing information will be interpolated from the adjacent data sets (data groups with the next lower and higher altitudes), to give the appropriate slew bound.

The system can only detect single data point, but cannot determine whether the maximum slew bound or the minimum value is missing without additional checks, therefore interpolation will be performed for both bounds whenever the data group contains only one slew bound. Fig. 5.10

*		*		*		*
*				*		
*		*		*		*

(A)

(B)

*		*		*		*
			*			*
*		*		*		*

(C)

(D)

\* -- CAPTURED PROFILE DATA POINT  
 (Captured Minimum slew is on the Left  
 Captured Maximum slew is on the Right  
 Only One Slew Bound at the Mid-level  
 has been Captured in Each Case)

FIG 5.10 Sample Discrete Profile Data Captured

gives the four possibilities of single data point being captured. In (A) and (B), the maximum slew bound can be found by interpolating the maximum slew bounds of the two adjacent data groups. In (C) and (D), the minimum slew value can be found by interpolating the minimum slew bounds of the two adjacent data sets. However, in (B), the minimum value obtained by interpolation will be greater than the captured data; while in (D) the interpolated maximum slew bound will be less than the captured value. Therefore, a simple algorithm for interpolating the data can be derived: the minimum slew bound can be found by taking the smaller of the captured minimum and the interpolated minimum, while the maximum slew bound can be found by taking the larger of the captured maximum and the interpolated maximum.

The table will finally be constructed to contain data group at every altitude. However, because of the way that the profile data are captured, there may be missing data sets at some altitudes. These missing data sets, if any, will be generated by interpolation for their altitudes and the slew bounds. After this table has been formed, all the function selections will be allowed, and the system will wait for a signal to select one of the three operations available.

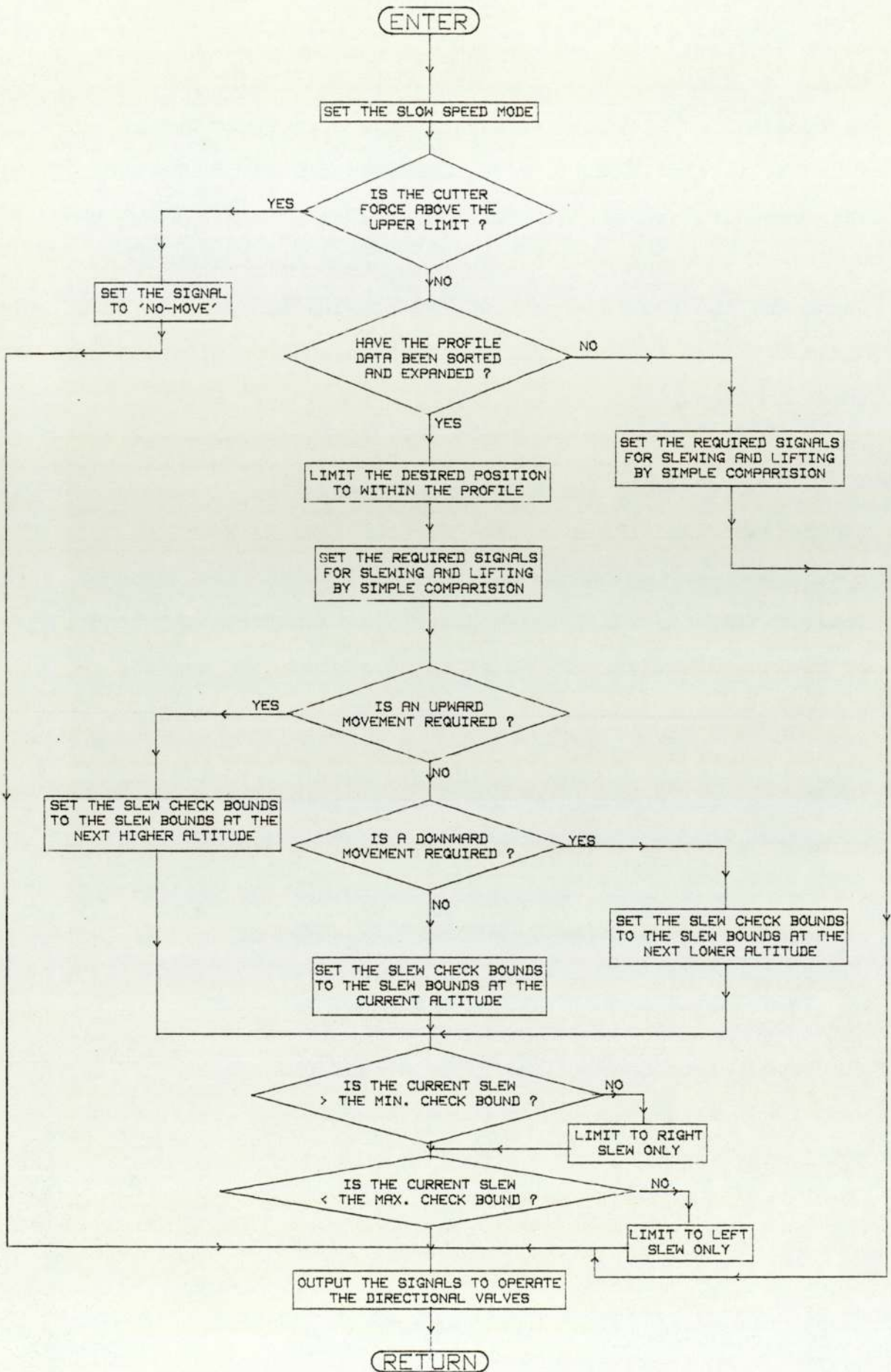


FIG 5.11 A Flow Diagram for the 'SMOVE' Subroutine

5.8. ROUTINE FOR MOVING THE  
BOOM CUTTER SLOWLY - 'SMOVE'

When the boom cutter is being moved slowly, the signals for the directional valves are determined by purely comparing the desired and the current positions of the cutter. A flow diagram for this routine is shown in Fig. 5.11.

This routine is employed whenever the boom cutter is required to move slowly. The checking for the cutter movement to stay within the profile can only be possible after the data table has been established. Hence this checking will not be exercised if the profile data have not been sorted and expanded. There is therefore a possibility for the cutter to move outside the prescribed profile boundary. However, this 'no-check' movement is only performed when the 'Trim Profile' operation is selected before the completion of the data sorting and expanding procedures; during this operation, the cutter itself is moving along the profile and the overshoot, if any, is not significant after the "smoothing" action of the cutter.

If the desired cutter movement is being checked to ensure that the boom cutter will move within the profile boundary, the following algorithm will be used. The desired altitude is first checked to ensure that it is within the profile. The desired slew at the required altitude will then be checked to ensure that it is not outside the defined profile boundary. The desired altitude is compared with the current altitude to determine the required movement: upward,

downward or horizontal. The comparison between the desired and the current slew values will give the required arcing movement: left, right and no slewing. After these signals have been determined, the effect of the desired movement will then be checked against the profile data table using the algorithm described in Section 5.1. This additional checking will prevent the boom cutter being moved outside the defined profile boundary.

In addition to the positional checks, the cutter force (indicated by the cutter motor current) is also examined: if the force is above the permissible upper limit, then the boom cutter movement will be completely stopped. The finalised signals will then be sent to operate the directional valves for the required boom cutter movement.

#### 5.9. MODE CHANGING

Since the 'Move Boom' selection will be disabled when either the 'Trim Profile' or the 'Cut Cavity' operation is selected, a facility, which stops the current operation and enables the operator to select the 'Move Boom' operation, may be desirable. (The operator may want to move the boom cutter away from a hard stratum which causes the cutter to stop). On selecting this 'Change Mode' operation, all the directional valves will be closed. The data sorting and expanding procedures will be executed if the profile data table has not yet been established. Otherwise, the system will wait for the operator to select the desired operation.



**CHAPTER      6**

**CONTROL CIRCUIT DESIGN**

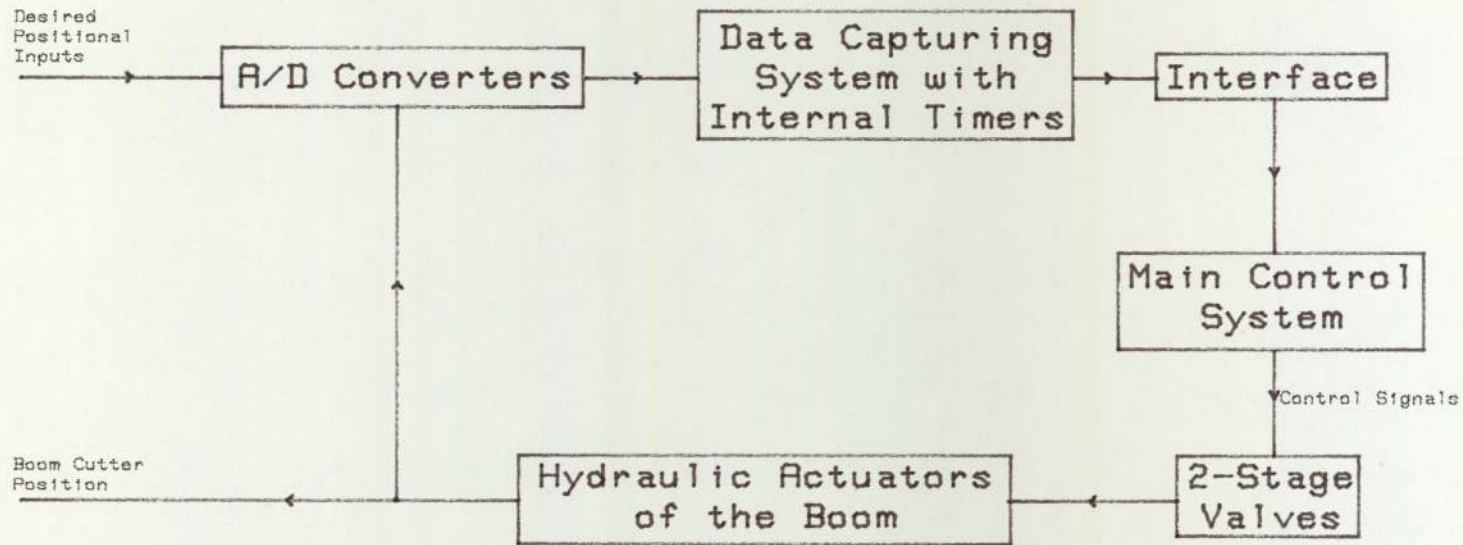


FIG 6.1 Block Structure of the Proposed Dual Microprocessor Control System for the Mining Boom Positions

## 6.1. INTRODUCTION

It has been decided to control the position of a mining boom cutter with a microprocessor based system. There are many microprocessors available on the market. However, the one commonly used in the Department of Mechanical Engineering at the University of Aston is Intel-8085, hence this microprocessor is chosen for the proposed control system.

A microprocessor by itself can do nothing, it needs other supporting elements, such as address decoders, random access memories (RAMS), read only memories (ROMs), buffers, logic gates, input/output (I/O) ports and other accessories to perform the appropriate operations.

The proposed system is a dual microprocessor system with a main system for the control algorithms and a second one for capturing the system performance and input data ready for use by the main system. Since the two microprocessors are running in parallel, this arrangement will enable a high speed operation. The block diagrams of this control system are shown in Figures 6.1 and 6.2. The 'hand-shaking' facility is provided by an interface to enable the main control system to get data from the data capturing system. Figure 6.3 shows the procedures taken by the two systems in communicating with each other.

In this chapter, a brief account is given on the design of the electronic circuit for the main microprocessor system and the interface to the data capturing system. The data

SYSTEM INPUTS

OPERATIVE CONTROLS

TUNNEL PROFILE DISPLACEMENTS

BOOM MOVE/VELOCITY

DISPLACEMENT FEEDBACK

BOOM RAISE

BOOM SLEW

MACHINE DEGREES OF FREEDOM SIMULATION INPUTS

LINEAR DISPLACEMENTS

X

Y

ANGULAR DISPLACEMENTS

$\alpha_x$

$\alpha_y$

$\alpha_z$

SYSTEM OUTPUTS

MULTI-DIRECTIONAL AND VOLUME FLOW HYDRAULIC VALVES

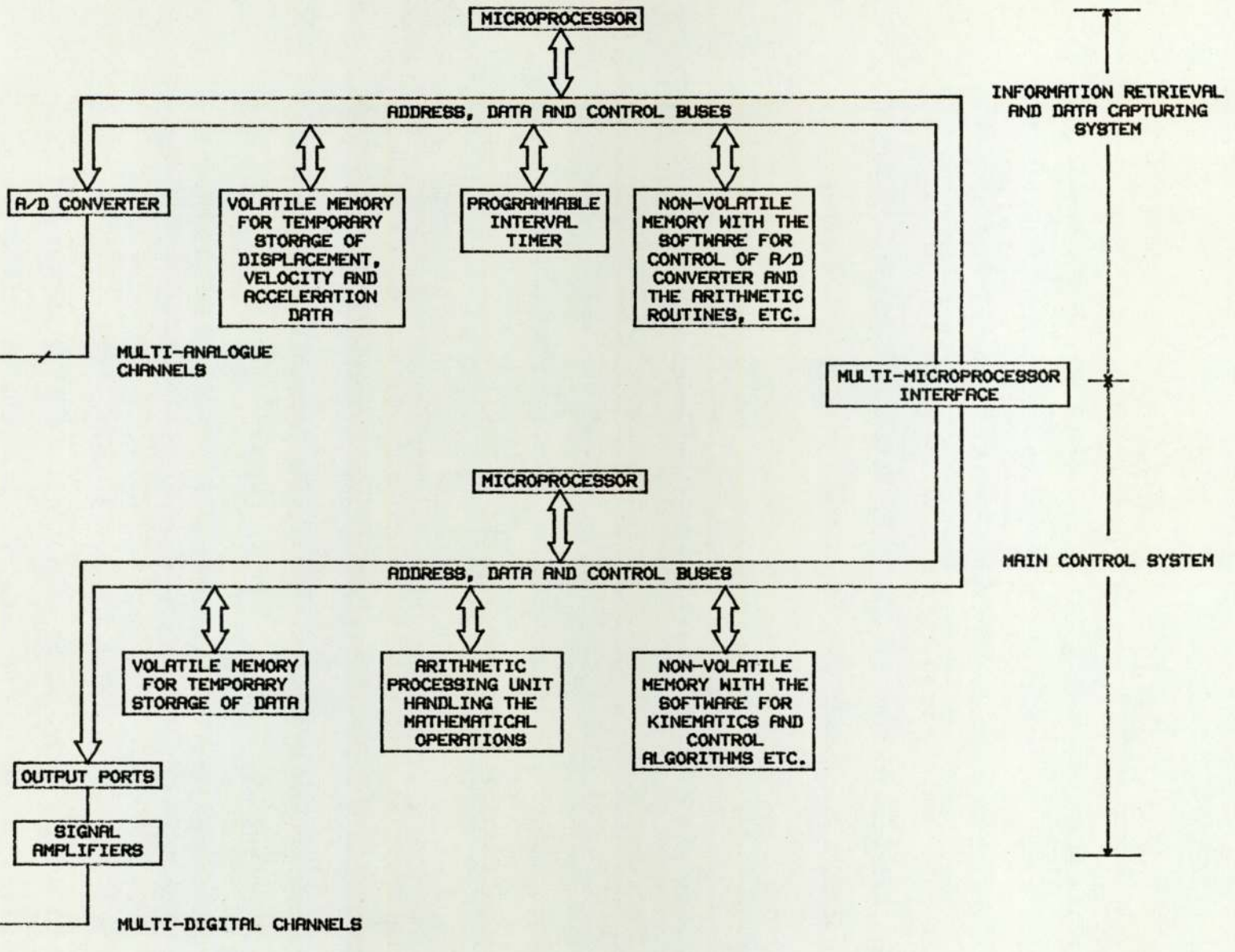


FIG 6.2 Proposed Dual-Microprocessor Control System for Boom Cutter Position

MAIN CONTROL SYSTEM  
MICROPROCESSOR

DATA CAPTURING SYSTEM  
MICROPROCESSOR

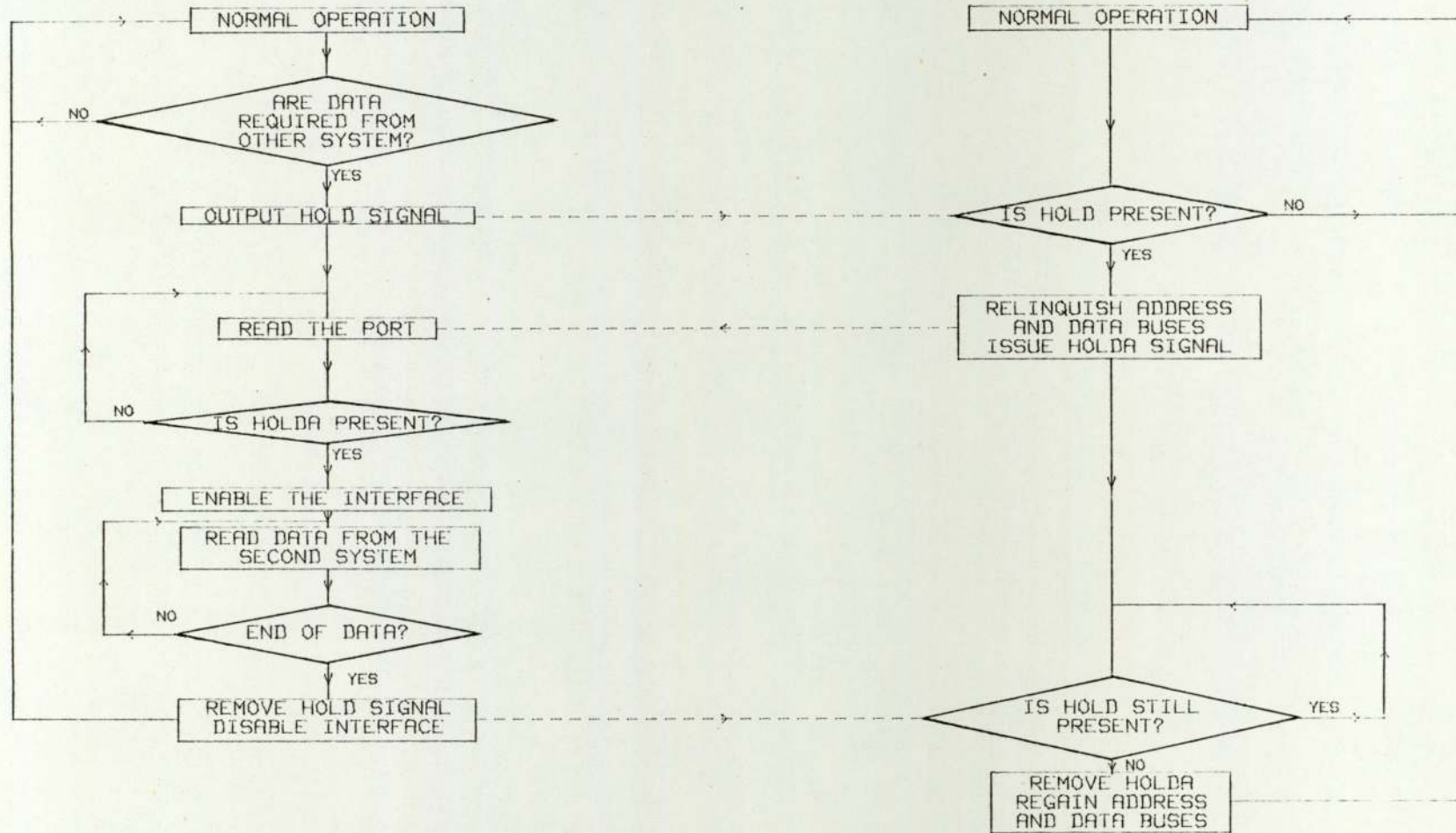


FIG 6.3 Flow Diagram for the Actions Taken by the Two Microprocessors in Communicating With Each Other

capturing microprocessor system and the interface to the pilot valves of the hydraulic actuators are developed by J H Knight in a parallel project at Aston University.

## 6.2. MAIN MICROPROCESSOR AND MONITOR BOARD

The first board in the main microprocessor control system consists of the microprocessor, an address decoder, the first monitor ROM, two 8-bit I/O ports, buffers, the crystal and logic gates. A schematic diagram of this board is given in Figure 6.4.

The Intel-8085 is an 8-bit parallel central processing unit (CPU). It is designed with N-channel depletion loads, and requires a single +5v supply. Multiplexed lower order address and data bus reduces the number of lines required for transferring the data. Detailed functional description of this CPU can be found in the Intel publication [42].

### 6.2.1. Clock Generation

A CPU is normally dynamic and requires a clock signal to operate. The Intel-8085 CPU has an internal clock generator which derives the timing inputs for the microprocessor from the 6.144 MHz crystal. It generates an internal clock frequency of 3.072 MHz with a clock cycle period of about 326 nSec. It also makes this clock (3.072 MHz) available on the clock output pin of the CPU so that this signal can be used as the system clock.

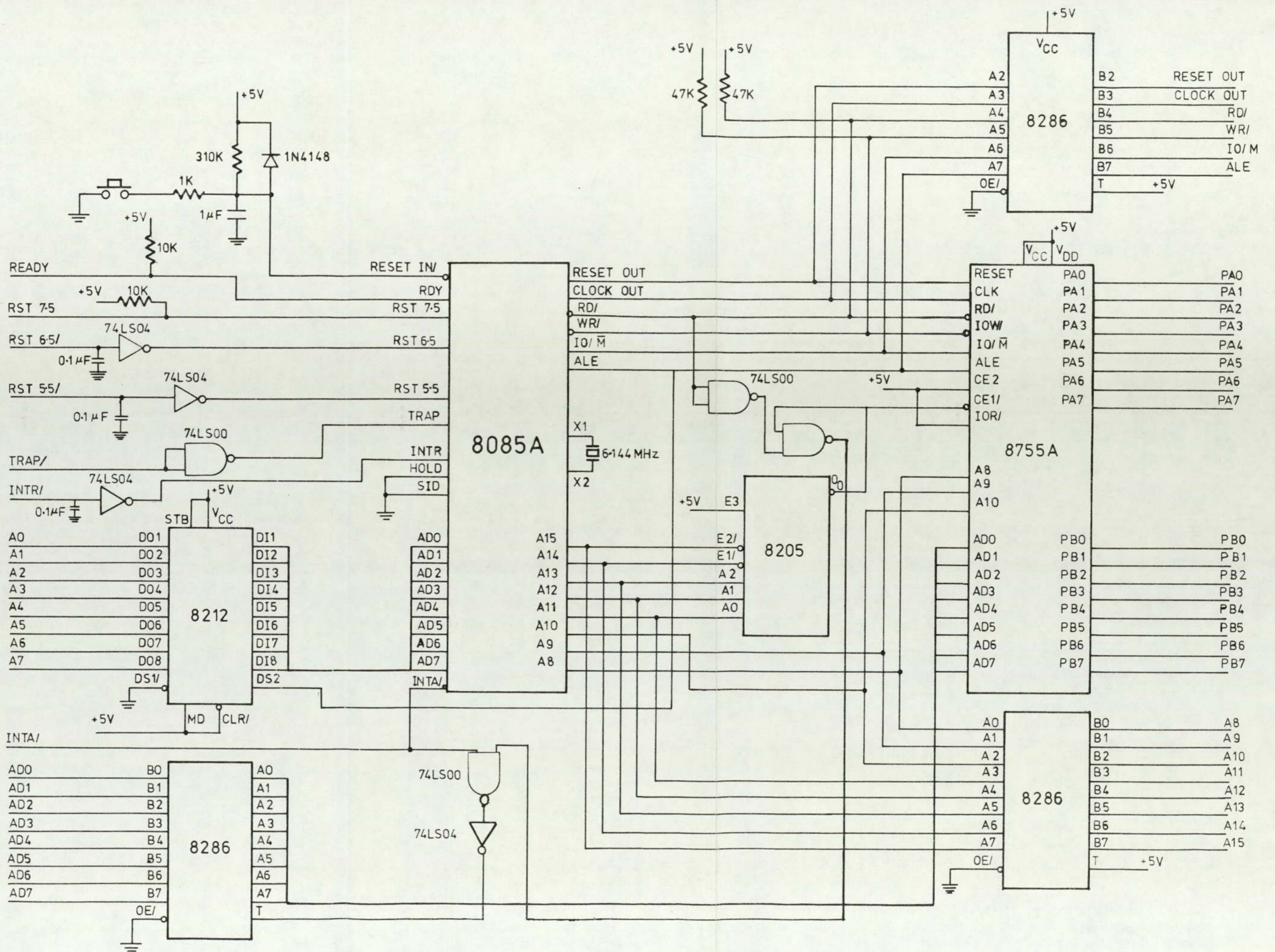


FIG. 6.4

SCHEMATIC DIAGRAM FOR THE MAIN MICROPROCESSOR BOARD

### 6.2.2. RESET IN and Power On

This system can be reset by closing the reset switch to low activate the RESET IN input (pin 36) of the CPU. This CPU will also be automatically reset when the power is switched on. The Intel-8085 CPU employs a special internal circuit, a substrate bias generator, to increase its speed and needs time for this circuit to stabilise, therefore it is not guaranteed to work until 10 mSec after the supply voltage ( $V_{CC}$ ) has reached +4.75v. A RC (resistor-capacitor) circuit is introduced to keep the RESET IN signal low during this period [44] so that the CPU can be reset at power on.

On recognising the RESET IN low signal, the CPU will generate the high RESET OUT signal which can be used to reset the whole system. Since various signal lines and buses will be floating during the reset process, the low active control signal outputs (e.g.  $\overline{RD}$ ,  $\overline{WR}$ ) are connected to pull up resistors to ensure the inactive state during this resetting period.

### 6.2.3. Address Decoding

The address is decoded using a 3-to-8 line decoder, Intel-8205 or 74LS138 (a low power Schottky transistor-transistor-logic integrated circuit). The decoding is performed in 2K (2048D bytes) blocks by examining the address lines  $A_{11}$  to  $A_{15}$ . The decoder will drive the selected output low, therefore this output can be used for any device that requires a low active chip select input.



#### 6.2.4. First Monitor and I/O Ports

The first monitor program resides in the erasable programmable read only memory (EPROM), Intel-8755A, which has a memory of 2048D x 8 bits. Following the reset process, the CPU will start executing instructions from the location with address equal to 0, therefore this monitor EPROM is configured to have its addresses from 0 to 7FF H (i.e. the first 2K block). The Intel-8755A is compatible with the Intel-8085 CPU and has two general purpose 8-bit I/O ports, it facilitates the construction of the control system.

The boom is moved by operating two pairs of hydraulic actuators: one for lifting the trunk and the other for arcing (slewing) it. Each pair of these actuators use four valves to control the direction of the ram movement, hence require four bits of the output port. This means that a total of eight bits should be made available for output. These are provided by assigning one of the two I/O ports of the Intel-8755A to be an 8-bit output port. Three bits of the other port are used for interfacing to the data capturing system. This arrangement leaves 5 bits of that I/O port free for other purposes if required. In fact, these five bits are used as outputs for resetting the APU, setting the boom moving speed (one bit for slewing and one bit for lifting), starting and stopping the boom cutter revolving.

#### 6.2.5. Low Order Address Latching

Since the CPU uses a multiplexed low order address and data bus, the low order address has to be latched, if the low

order address is required, throughout the machine cycle of the CPU. This address latching is done by employing an 8-bit latch, 8212. The low order address is put onto the bus during the first part of a machine cycle and is latched to the output of the 8212 by the address latch enable (ALE) signal when this ALE signal goes from high to low.

#### 6.2.6. Buffers

Buffers are installed to provide the expansion facilities. The latched low order address has already been buffered through the latch, 8212. The control signals, the high order address bus and the multiplexed low order address/data bus are buffered by the octal bus transceivers, Intel-8236.

The buffer for the control signals is configured to transmit signals from side A to side B of the transceiver. These buffered signals are:  $IO/\overline{M}$ ,  $\overline{WR}$ ,  $\overline{RD}$ , ALE, CLK OUT and RESET OUT. Similar arrangement is made for the high order address bus. For these transceivers, the  $\overline{OE}$  inputs are tied low to enable the data transmission through the buffer, while the T inputs are connected to +5v (high) to configure the transceivers  $B_0 - B_7$  as outputs with its  $A_0 - A_7$  as inputs.

The buffer for the multiplexed low order address and data bus is connected with  $\overline{OE}$  low to enable data transfer through the buffer. The buffer's transmit control signal is configured so that this signal is normally high for

PIN	MNEMONIC	DESCRIPTION
1a	GND	SIGNAL GROUND
2a	+5V	+5 VOLTS
3a	A0	DEMULTIPEXED LOW ORDER ADDRESS BIT 0
4a	A1	DEMULTIPEXED LOW ORDER ADDRESS BIT 1
5a	A2	DEMULTIPEXED LOW ORDER ADDRESS BIT 2
6a	A3	DEMULTIPEXED LOW ORDER ADDRESS BIT 3
7a	A4	DEMULTIPEXED LOW ORDER ADDRESS BIT 4
8a	A5	DEMULTIPEXED LOW ORDER ADDRESS BIT 5
9a	A6	DEMULTIPEXED LOW ORDER ADDRESS BIT 6
10a	A7	DEMULTIPEXED LOW ORDER ADDRESS BIT 7
11a	PA0	} CONTROL SIGNALS FOR THE DIRECTIONAL VALVES FOR THE BOOM ELEVATION
12a	PA1	
13a	PA2	
14a	PA3	
15a	PA4	} CONTROL SIGNALS FOR THE DIRECTIONAL VALVES FOR THE BOOM SLEW
16a	PA5	
17a	PA6	
18a	PA7	
19a	READY	READY SIGNAL
20a	RESET OUT	SYSTEM RESET SIGNAL FROM THE MICROPROCESSOR
21a	CLOCK OUT	SYSTEM CLOCK OUTPUT [3.072 MHz]
22a	HOLD B	HOLD THE DATA CAPTURING SYSTEM [HIGH ACTIVE]
23a	PB7	HOLD ACKNOWLEDGE FROM THE DATA CAPTURING SYSTEM
24a	PB6	ENABLE THE INTERFACE BETWEEN THE TWO SYSTEMS [LOW ACTIVE]
25a	PB5	INVERTED HOLD B
26a	PB4	APU RESET SIGNAL
27a	PB3	STOP THE CUTTER [LOW ACTIVE]
28a	PB2	START THE CUTTER [LOW ACTIVE]
29a	PB1	} FAST/SLOW SPEED CONTROL SIGNALS [0 FOR FAST]
30a	PB0	
31a	+5V	+5 VOLTS
32a	GND	SIGNAL GROUND
1c	GND	SIGNAL GROUND
2c	+5V	+5 VOLTS
3c	AD0	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 0
4c	AD1	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 1
5c	AD2	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 2
6c	AD3	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 3
7c	AD4	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 4
8c	AD5	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 5
9c	AD6	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 6
10c	AD7	MULTIPEXED LOW ORDER ADDRESS/DATA BIT 7
11c	A8	HIGH ORDER ADDRESS BIT 8
12c	A9	HIGH ORDER ADDRESS BIT 9
13c	A10	HIGH ORDER ADDRESS BIT 10
14c	A11	HIGH ORDER ADDRESS BIT 11
15c	A12	HIGH ORDER ADDRESS BIT 12
16c	A13	HIGH ORDER ADDRESS BIT 13
17c	A14	HIGH ORDER ADDRESS BIT 14
18c	A15	HIGH ORDER ADDRESS BIT 15
19c	IO/M	I/O PORT OR MEMORY CONTROL SIGNAL
20c	RD/	READ CONTROL [LOW ACTIVE]
21c	ALE	ADDRESS LATCH ENABLE
22c	TRAP/	INVERTED TRAP INTERRUPT
23c	RST 7.5	RESTART 7.5 INTERRUPT (FOR MOVING THE BOOM)
24c	RST 6.5/	INVERTED RESTART 6.5 INTERRUPT (FOR CUTTING THE CAVITY)
25c	RST 5.5/	INVERTED RESTART 5.5 INTERRUPT (FOR TRIMMING THE PROFILE)
26c	WR/	WRITE CONTROL [LOW ACTIVE]
27c	INTR/	INVERTED INTERRUPT FOR STOPPING THE BOOM [LOW ACTIVE]
28c	INTA/	INTERRUPT ACKNOWLEDGE [LOW ACTIVE]
29c	RESET IN/	SYSTEM RESET INPUT [LOW ACTIVE]
30c	+12V	+12 VOLTS
31c	+5V	+5 VOLTS
32c	GND	SIGNAL GROUND

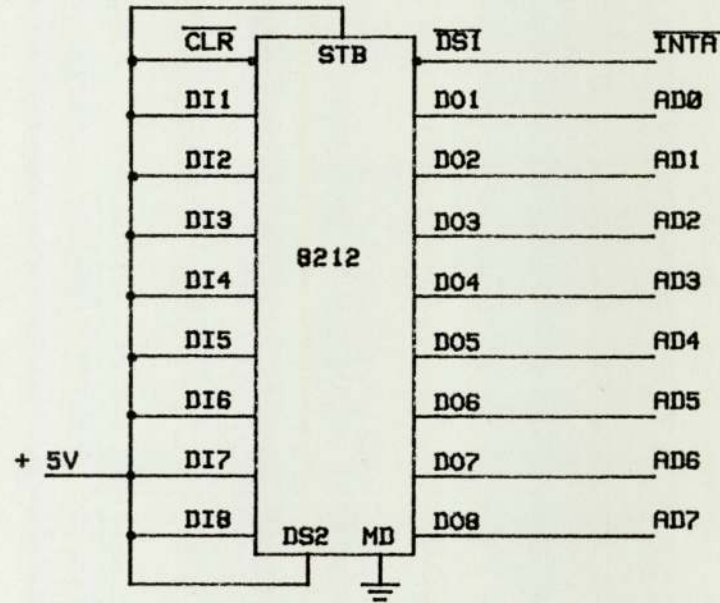
TABLE 6.1 PIN ASSIGNMENT ON THE BACK PLATE OF THE MAIN CONTROL SYSTEM

transmitting information from side A to side B of the transceiver, while the data flow direction will be reversed if the CPU is requiring (reading) data from a device not residing on the CPU board. In order to avoid bus contention, the controlling of this T signal must be done by some logic. Since the only on-board device is the monitor EPROM, the chip select signal for the monitor can be used with other control signals to generate the T signal for this buffer. The CPU will read in data if it performs one of the following operations: reading data from memory, inputting from an input port, or getting an instruction on recognising an interrupt. This T signal generation logic will be done by the NAND gates (74LS00) and the inverter (74LS04). T will be high whenever the CPU requires information from the monitor EPROM and its ports, or writes information onto the data bus. T will become low if the CPU sends out a low  $\overline{\text{INTA}}$  signal, or a low  $\overline{\text{RD}}$  signal with a high monitor EPROM chip select signal.

#### 6.2.7. Others

The CPU ready line input is connected to the back plate of the system and is normally pulled high through a pull up resistor unless a low is sent from a device indicating a busy status. Table 6.1 shows the pin assignment on this back plate for the main control system. The built-in hardware interrupts of the Intel-8085 are constructed in the following way:

FIG 6.5 INTERRUPT INSTRUCTION PORT SUPPLYING THE RESTART 7 INSTRUCTION



- TRAP - This interrupt has got the highest priority and is used to continue the system operations after the tunnel profile has been defined.
- RST 7.5 - This interrupt is used to execute the routine for moving the boom cutter to the position defined by the operator with the desired speed (fast or slow).
- RST 6.5 - This interrupt is used to execute the routine for cutting the tunnel cavity layer by layer.
- RST 5.5 - This interrupt is used to execute the routine for trimming the tunnel profile.
- INTR - This interrupt is used to stop the execution of the current program segment so that other operations can be selected. This therefore provides a 'change mode' facility to the operator. Fig. 6.5 shows the schematic diagram for the RST 7 instruction port.

All other unused high active input control signals (e.g. HOLD, SID) are grounded to enable normal operation.

### 6.3. EXPANSION RAM AND EPROM BOARD

This board contains the address decoder, EPROMs and RAMs. There are three 2K EPROMs with addresses from 8000 H to 97FF H and two 1K RAMs with addresses from B800 H to BFFF H. Similar to the first board, the address decoding is done in blocks of 2K from 8000 H to BFFF H. Since the

microprocessor uses the high order addresses for both memories and I/O ports, the address decoder is enabled only when the  $\text{IO}/\overline{\text{M}}$  signal is low to avoid accidental selection of the RAM or the EPROM on this board. A schematic diagram of this board is shown in Fig. 6.6.

The EPROMs and RAMs used are 2716 and Intel-8185 respectively. Since they have standby facilities, they may be disabled when the chips are not selected to reduce the power consumption. These chips are forced to enter a standby state by applying high signals to their  $\overline{\text{CS}}$  inputs, therefore the use of Intel-8205 (or 74LS138) provides this facility.

The Intel-8185 is an 1K 8-bit static RAM with a multiplexed address and data bus, therefore the buffered  $\text{AD}_0 - \text{AD}_7$  and  $\text{ALE}$  of the microprocessor are connected to the corresponding pins of this RAM. These RAMs are selected when the address is within the 2K block from B800 H. However, only one chip is enabled by the buffered  $\text{A}_{10}$  which is connected to  $\overline{\text{CE}}_1$  of one chip and to  $\text{CE}_2$  of the other. On the contrary, the 2716 EPROMs are selected purely on the chip-select ( $\overline{\text{CS}}$ ) signals, since they are 2K EPROMs, and the  $\overline{\text{OE}}$  pins are enabled by the buffered  $\overline{\text{RD}}$  signal of the CPU.

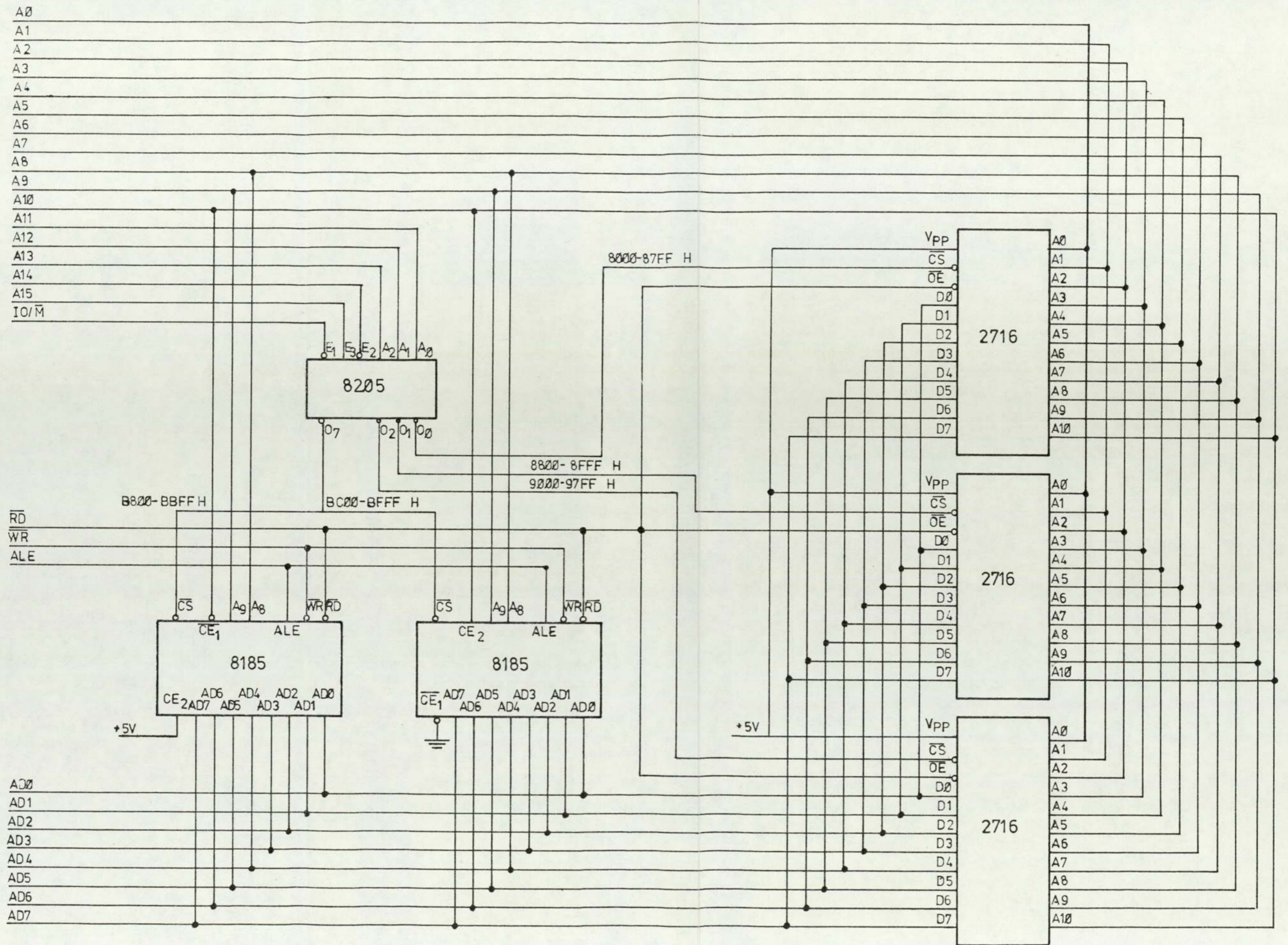


FIG. 6.6 Schematic Diagram For Expansion RAM and EPROM Board



#### 6.4. ARITHMETIC PROCESSING UNIT BOARD

Mathematical operations can be carried out either by software (program) or by hardware (arithmetic processing unit, APU). Hardware operation will speed up the performance and enable the valve control signals to be given at a high frequency. A schematic diagram for this APU board is given in Fig. 6.7.

The APU chosen is the Intel-8231A for its high operation speed and ease of interfacing to the CPU. This APU is selected with the high order address FD H and FC H for its command port and its on-chip data stack respectively. It can therefore be selected as an I/O port or a memory.

Memory mapping technique can increase the flexibility by enabling more instructions to be used for communication with the APU at the expense of the memory addresses FCOO H to FDOO H ( $\frac{1}{2}$  K) which cannot be used for ordinary memories.

The APU's ready signal output is connected to the CPU so that the CPU can be halted by the 8231A when this APU is busy and data transfer is requested by the CPU. The ready line is pulled low when the APU is busy and will be pulled high again when it is ready for data transfer.

Since the APU can operate on a maximum frequency of 4 MHz, it is driven by the 3.072 MHz clock output of the CPU employed. Similar to the CPU, all the unused high active control inputs are grounded, while those low active control inputs are tied to high (+5v).

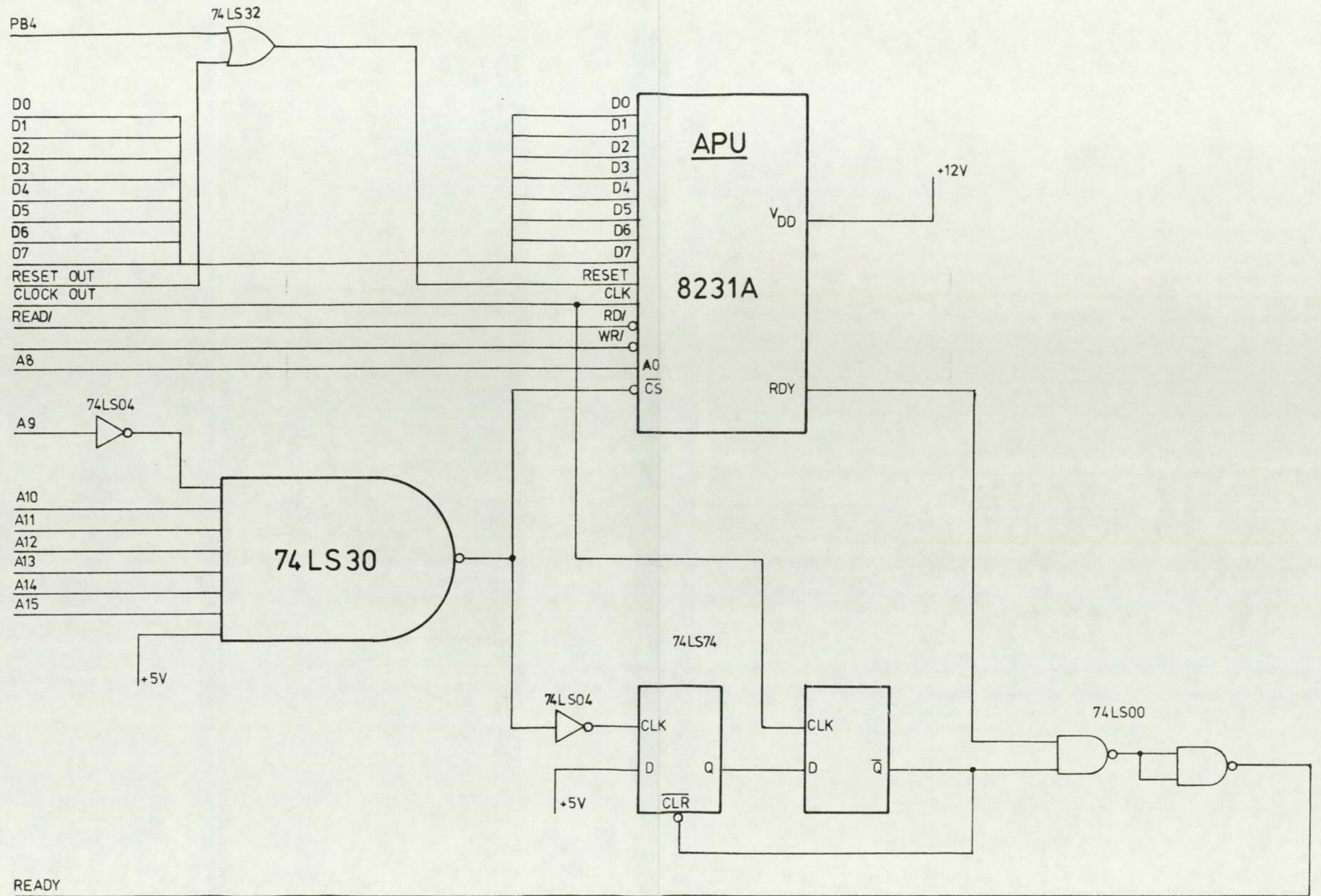


FIG. 6.7 SCHEMATIC DIAGRAM FOR THE APU EXPANSION BOARD

## 6.5. INTERFACE TO THE DATA CAPTURING SYSTEM

The proposed system is a dual microprocessor one, therefore an interface must be provided to enable the communication between these two parts. A schematic diagram for this interface is presented in Fig. 6.8.

Both the microprocessors will be running in parallel unless data transfer from one to another is performed. When the main control microprocessor wants data from the data capturing system, it latches a low signal onto bit 5 of port 1 and this signal is inverted and then transmitted to the HOLD input of the data capturing system's CPU. The inverter is installed to ensure normal operation at power on. On recognising this signal, the second CPU will issue a HOLDA (hold acknowledge) signal onto its HOLDA output and relinquish the address and data buses as soon as the completion of the current bus transfer. The  $\overline{RD}$ ,  $\overline{WR}$  and  $IO/\overline{M}$  control lines of this CPU will also be tristated. On receipt of the HOLDA signal, the main microprocessor can start to read data from the data capturing system.

The buffers for address, data and control lines in this interface card normally have tristated outputs unless data transfer is requested. This tristate arrangement enables the two systems to operate normally without any undesirable interference. The data and address buffers (8286s) are enabled by pulling the output enable ( $\overline{OE}$ ) lines low; when the buffer (8216) for the control lines is enabled by pulling the  $\overline{CS}$  signal low. The  $\overline{RD}$  signal is connected to the T

MAIN CONTROL SYSTEM

DATA CAPTURING SYSTEM

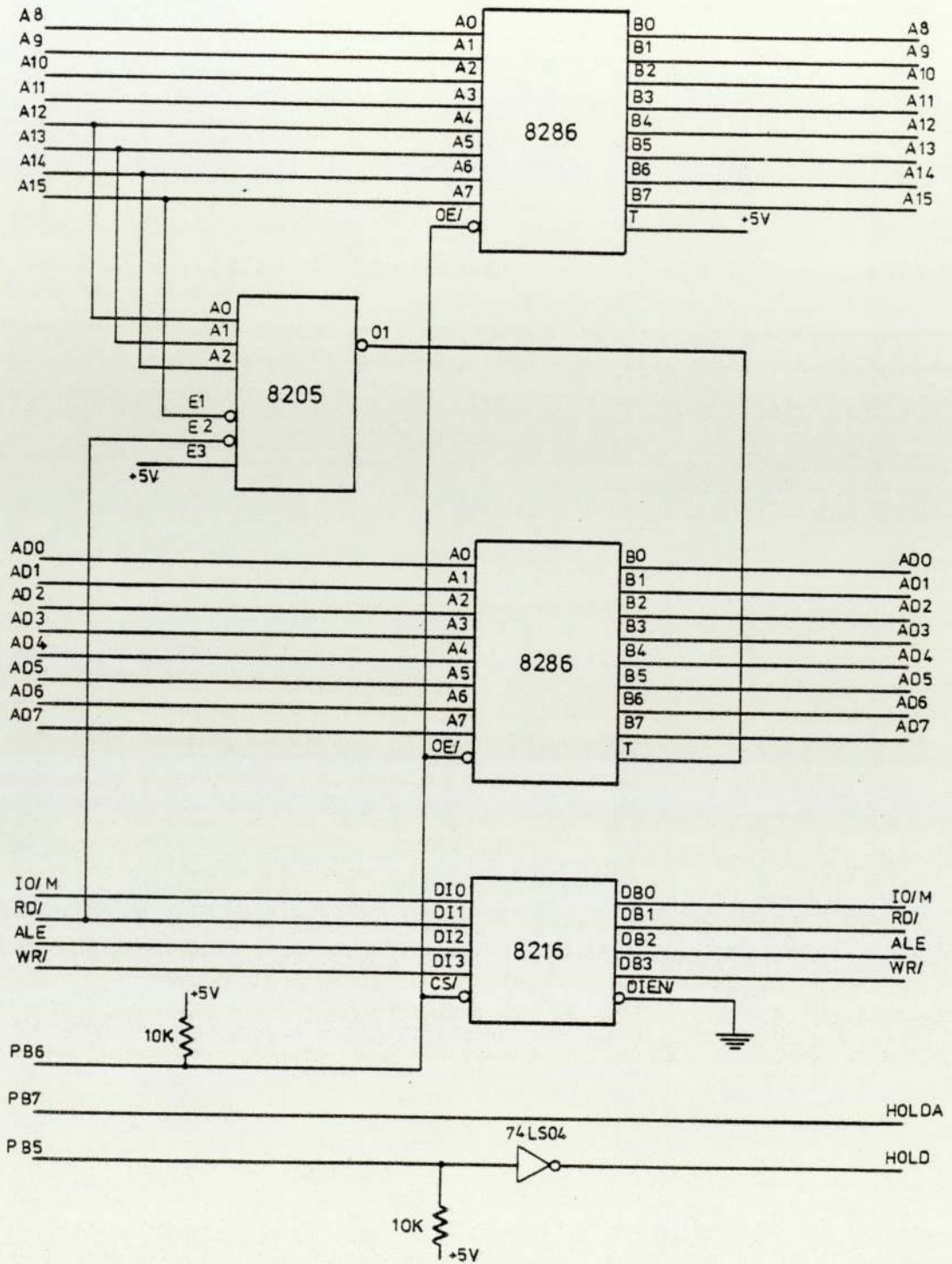


FIG.6-8 SCHEMATIC DIAGRAM FOR INTERFACING THE MAIN CONTROL SYSTEM WITH THE DATA CAPTURING ONE

PIN	MNEMONIC	DESCRIPTION
1c	GND	SIGNAL GROUND
2c	+5V	+5 VOLTS
3c	AD0	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 0 (Main System)
4c	AD1	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 1 (Main System)
5c	AD2	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 2 (Main System)
6c	AD3	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 3 (Main System)
7c	AD4	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 4 (Main System)
8c	AD5	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 5 (Main System)
9c	AD6	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 6 (Main System)
10c	AD7	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 7 (Main System)
11c	A8	HIGH ORDER ADDRESS BIT 8 (Main System)
12c	A9	HIGH ORDER ADDRESS BIT 9 (Main System)
13c	A10	HIGH ORDER ADDRESS BIT 10 (Main System)
14c	A11	HIGH ORDER ADDRESS BIT 11 (Main System)
15c	A12	HIGH ORDER ADDRESS BIT 12 (Main System)
16c	A13	HIGH ORDER ADDRESS BIT 13 (Main System)
17c	A14	HIGH ORDER ADDRESS BIT 14 (Main System)
18c	A15	HIGH ORDER ADDRESS BIT 15 (Main System)
19c	IO/ $\bar{M}$	I/O PORT OR MEMORY CONTROL SIGNAL (Main System)
20c	RD/	READ CONTROL SIGNAL [LOW ACTIVE] (Main System)
21c	ALE	ADDRESS LATCH ENABLE (Main System)
22c	TRAP/	INVERTED TRAP INTERRUPT (Main System)
23c	RST 7.5	RESTART 7.5 INTERRUPT FOR MOVING THE BOOM (Main System)
24c	-	-
25c	-	-
26c	WR/	WRITE CONTROL SIGNAL [LOW ACTIVE] (Main System)
27c	-	-
28c	-	-
29c	-	-
30c	-	-
31c	+5V	+5 VOLTS
32c	GND	SIGNAL GROUND
1a	GND	SIGNAL GROUND
2a	+5V	+5 VOLTS
3a	AD0	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 0 (Data System)
4a	AD1	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 1 (Data System)
5a	AD2	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 2 (Data System)
6a	AD3	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 3 (Data System)
7a	AD4	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 4 (Data System)
8a	AD5	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 5 (Data System)
9a	AD6	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 6 (Data System)
10a	AD7	MULTIPLEXED LOW ORDER ADDRESS/DATA BIT 7 (Data System)
11a	A8	HIGH ORDER ADDRESS BIT 8 (Data System)
12a	A9	HIGH ORDER ADDRESS BIT 9 (Data System)
13a	A10	HIGH ORDER ADDRESS BIT 10 (Data System)
14a	A11	HIGH ORDER ADDRESS BIT 11 (Data System)
15a	A12	HIGH ORDER ADDRESS BIT 12 (Data System)
16a	A13	HIGH ORDER ADDRESS BIT 13 (Data System)
17a	A14	HIGH ORDER ADDRESS BIT 14 (Data System)
18a	A15	HIGH ORDER ADDRESS BIT 15 (Data System)
19a	IO/ $\bar{M}$	I/O PORT OR MEMORY CONTROL SIGNAL (Data System)
20a	RD/	READ CONTROL [LOW ACTIVE] (Data System)
21a	ALE	ADDRESS LATCH ENABLE (Data System)
22a	HOLD	HOLD THE DATA CAPTURING MICRO [HIGH ACTIVE]
23a	HOLDA	HOLD ACKNOWLEDGE FROM THE DATA CAPTURING MICRO
24a	PB6	ENABLE THE INTERFACE BETWEEN THE 2 SYSTEMS [LOW ACTIVE]
25a	PB5	INVERTED HOLD SIGNAL (FROM MAIN SYSTEM)
26a	WR/	WRITE CONTROL [LOW ACTIVE] (Data System)
27a	-	-
28a	-	-
29a	-	-
30a	-	-
31a	+5V	+5 VOLTS
32a	GND	SIGNAL GROUND

TABLE 6.2 PIN ASSIGNMENT ON THE BACK PLATE FOR THE INTERFACE CARD

input of the multiplexed low order address and data buffer to enable the read data operation. When T is high, signal flow is from side A to side B, hence passes the  $AD_0 - AD_7$  onto the respective bus of the second system and addresses the appropriate memory. When  $\overline{RD}$  becomes low, the signal flow direction will be reversed from side B to side A of the buffer, then the host microprocessor can read in data through this buffer. The other two buffers are configured to have information flowing from the main microprocessor system to the data capturing one when they are enabled. The flow diagram for this interfacing procedure has been given in Fig. 6,3, and the pin assignment on the back plate for this interface card is shown in Table 6.2.

**CHAPTER      7**

**TESTS   AND   RESULTS**

## 7.1. Introduction

The control algorithms and the control circuits have been derived and built. It has been mentioned in Chapter 3 that a switching function can be derived to determine when the valves should be closed. The use of this switching function will be expected to reduce the time taken in positioning the boom ripper. It has also been shown in Chapter 4 that the use of an APU will facilitate the mathematical operations. However, this will mean a more expensive system. Tests have been done using various control strategies: (A) control based on simply comparing the feedback position with the desired position, and (B) the more complicated control employing the switching function evaluations.

## 7.2. Control Strategies

### 7.2.1. Simple Comparison With Slow Speed

The simplest control strategy in this boom positional control is to include a unity feedback. In a parallel project, J H Knight of the Mechanical Engineering Department at Aston University has developed a data capturing and information retrieval system, which provides the positional information on the desired input and the actual output. These data are supplied in the digital forms and are also arranged to give a unity feedback.

In this simple comparison method, the input and the output data are compared in determining the required signals sent to the electro-hydraulic valves to give the desired boom



movement. It has been mentioned in Chapter 3 that the control by simply comparing the input and the output data would not be good enough because the physical properties (e.g. the inertial load, the fluid compressibility, the valve characteristics etc.) of the system might cause excessive overshoot. However, the inherent limitation in the accuracy of the 8-bit analogue-to-digital converter (ADC),  $\pm 1$  least significant bit (LSB), suggests that this simple comparison method will be adequate when the boom is moving at the slow speed.

#### 7.2.2. Switching Function Evaluation

It has been shown in Chapter 3 that the employment of the switching function, which determines the switching signals depending on the physical properties and the current conditions of the system, would reduce the overshoot and hence would improve the positional control of the boom ripper. This will be particularly helpful if the boom is moving at its fast speed, therefore tests have been done using the switching function for the boom fast move mode.

The information on the system performance is provided by the data capturing system developed by J H Knight; while the mathematical operations are handled by the APU to increase the speed of operation and to reduce the complexity in the software of the control system. The valve performance has been tested by J H Knight and his results suggest that the valves can be approximated to 'On-Off' valves with time delays. Therefore the switching functions tested are those derived for this valve type (i.e. 'On-Off' valve with time delay).

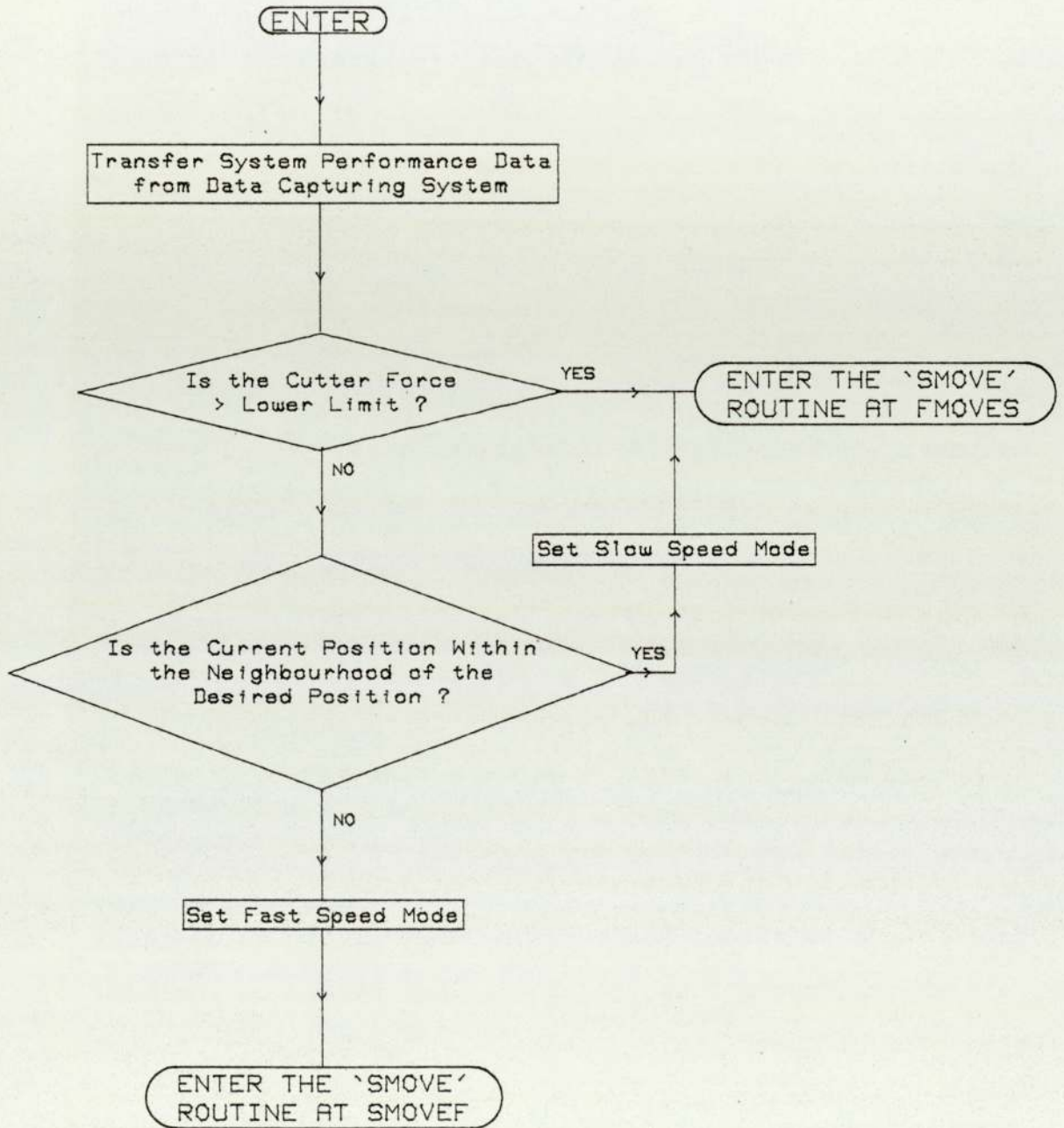


FIG 7.1 Flow Diagram for the 'FMOVE' Routine Using Simple Comparison and the 'SMOVE' Routine

### 7.3. Boom Moving at High Speed - 'FMOVE' Routine

It has been mentioned that two control strategies have been tested for the positional control of the boom ripper. The simple comparison method may cause an undesirable overshoot if the boom is moving at a high speed. Therefore, in the simple comparison method, the boom movement will be restricted to the slow move mode, if the cutter is within a specified neighbourhood of the desired position, to avoid any undesirable overshoot. Figure 7.1 shows the flow diagram for this control method, and the listing of the assembly language program for this routine is included in Appendix G.

When this routine is entered, the necessary information will be transferred from the data capturing system. The current boom slew position is checked against the desired slew value: if it is within the specified neighbourhood of the desired slew position, then the slew movement will be set to the slow move mode. Similarly, the current boom altitude is compared with the desired elevation and the slow move mode will be set if it is within the neighbourhood of the desired altitude. After this checking for the permitted speed mode has been done, the 'SMOVE' routine will be entered (please refer to the listing in Appendix G). The determination of the appropriate switching signals and the other checkings will be done in the 'SMOVE' routine by simple comparison. A detailed description of the 'SMOVE' routine has been given in Chapter 5.

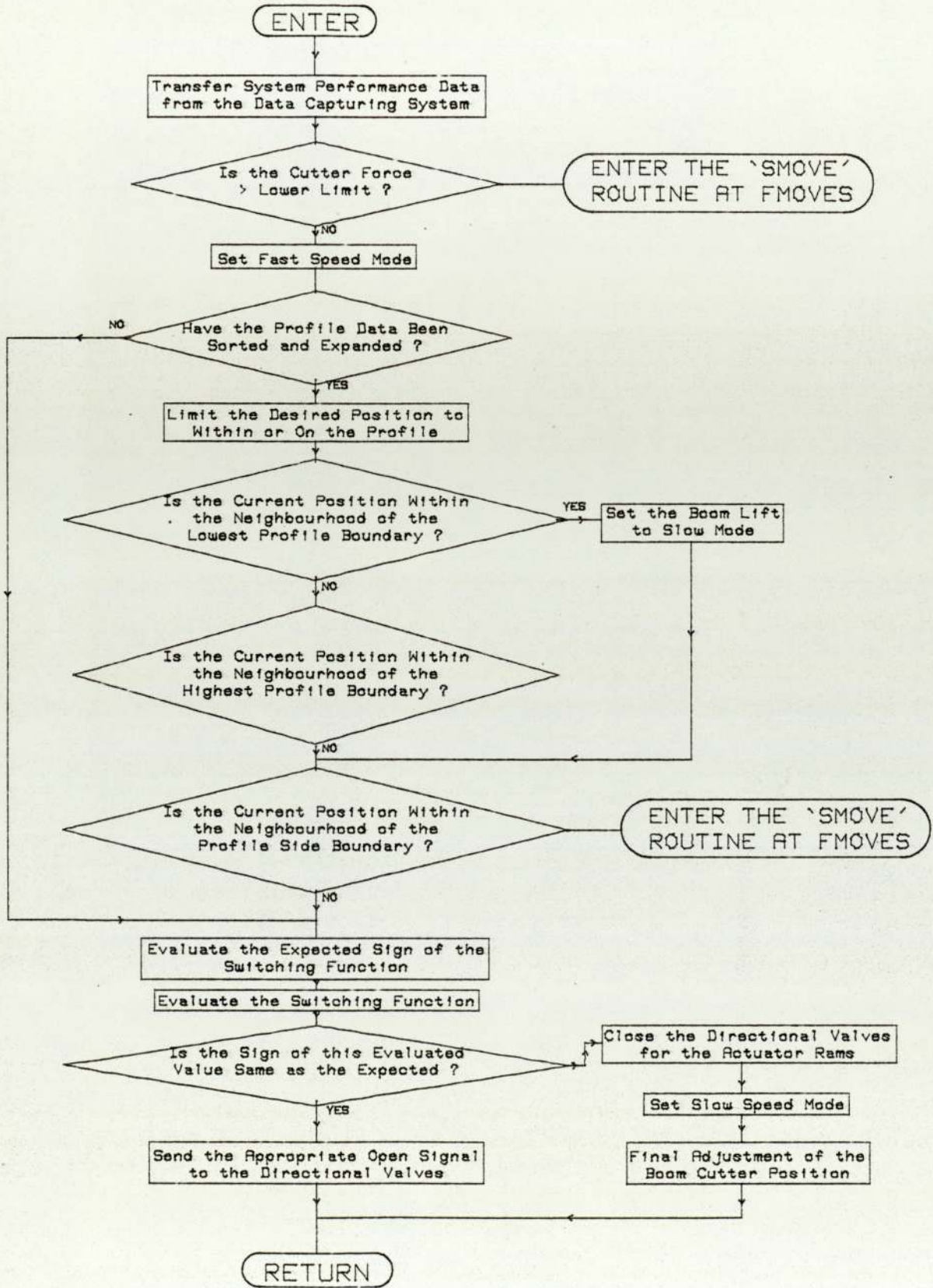


FIG 7.2 Flow Diagram for the 'FMOVE' Routine Using the Switching Function and the 'SMOVE' Routine

Another control strategy can be used for this boom movement at its high speed is the employment of the switching functions. Figure 7.2 shows the flow diagram for this algorithm and the listing of the assembly language program for this routine is given in Appendix H. Similar to the other routine, the system performance data are transferred from the data capturing system on entering this routine. If the cutter force is greater than the upper limit for the high speed movement, then the 'SMOVE' routine will be entered at FMOVES so that the boom can be moved only at the slow speed. Otherwise, the fast speed will be set. If the profile data have not been sorted and expanded, then the program will go to continue its execution from determining the switching signals.

However, if the profile data table has been properly established as described in Chapter 5, the following actions will be taken before it evaluates the switching function. Similar to the 'SMOVE' routine, the desired position is limited to within or on the profile boundary defined by the operator. The current position is examined: if it is close to either the lowest profile boundary or the highest one, then the slow move mode will be set for the vertical movement of the boom; if it is close to either side of the profile boundary, then the 'SMOVE' routine will be entered at FMOVES. The reason for restricting the boom movement to its slow mode, when it is currently near the profile boundary, is to make the checking for permissible directions of the boom movement easier (please refer to Chapter 5). In this system, the criterion for being close to the boundary is 16D (10H) bits.

If either the current boom position is free from the profile side boundary or the profile data have not been sorted and expanded, then the 'FMOVE' routine will continue its operation. It will determine the switching signals for the two movements (i.e. the lifting and the slewing) separately using similar procedures. The expected sign of the switching function will be determined by simply using the current and the desired positions. Then the captured data will be converted into the APU recognised binary floating point (BFP) format. These BFP data are then transferred to the APU stack and the appropriate switching function will be evaluated. The result will be used to determine the signal: if the evaluated function value has the same sign as the expected, then a signal will be sent to operate the pilots of the directional valves (the direction of movement will depend on the sign of the function value); otherwise, it indicates a change in the sign of the switching function (i.e. the criterion  $fn=0$ , for switching off the valves has been satisfied), then the directional valves will be closed. After all the directional valves (both valve sets for the lifting and the slewing) have been closed, the final position will be adjusted in the slow speed mode (this is similar to the 'inching' process used by the operator with an open-loop control system).

Because of the hardware configuration of the hydraulic circuit for the boom ripper under consideration, the heavy mass of the boom causes (i) the lowering movement having preference over the boom slewing and (ii) the slewing

movement having preference over the boom raising. It has been observed in the experiment that the boom would lower down whenever the fast mode is set and both sets of valves (i.e. the valves for both the vertical and the slewing actions) are operating. The boom will lower down even though the raise up signal has been sent to the pilots of the directional valves. Therefore an algorithm has been built into the system for the vertical movement to have preference over the slewing whenever the fast speed mode is set.

Another consideration has been the time delay in the valve operation. Because of the time delay, the boom will neither move nor stop immediately after the signal has been sent to the directional valves. If the boom is near the desired position (say 2 bits from it) and the correction signal has been sent to the valves, then the boom will not move until the valves are opened and its movement cannot be detected unless a two-bit change in the boom position has been observed (the data capturing system regards a change within 1 bit as 'no change' because the ADC has an accuracy of  $\pm 1$  LSB). As soon as this boom movement has been detected, the switching off signal will be sent to close the directional valves. However, because of the time delay in the valve operation, the boom will not stop immediately, therefore an overshoot results and another correction signal will then be sent to the valves. Consequently, the boom will be hunting about this desired position. Therefore, the boom movement is restricted to its slow move mode if it is near the desired position (in this system, the criterion is  $\pm 4$  bits from the desired position) to prevent hunting.

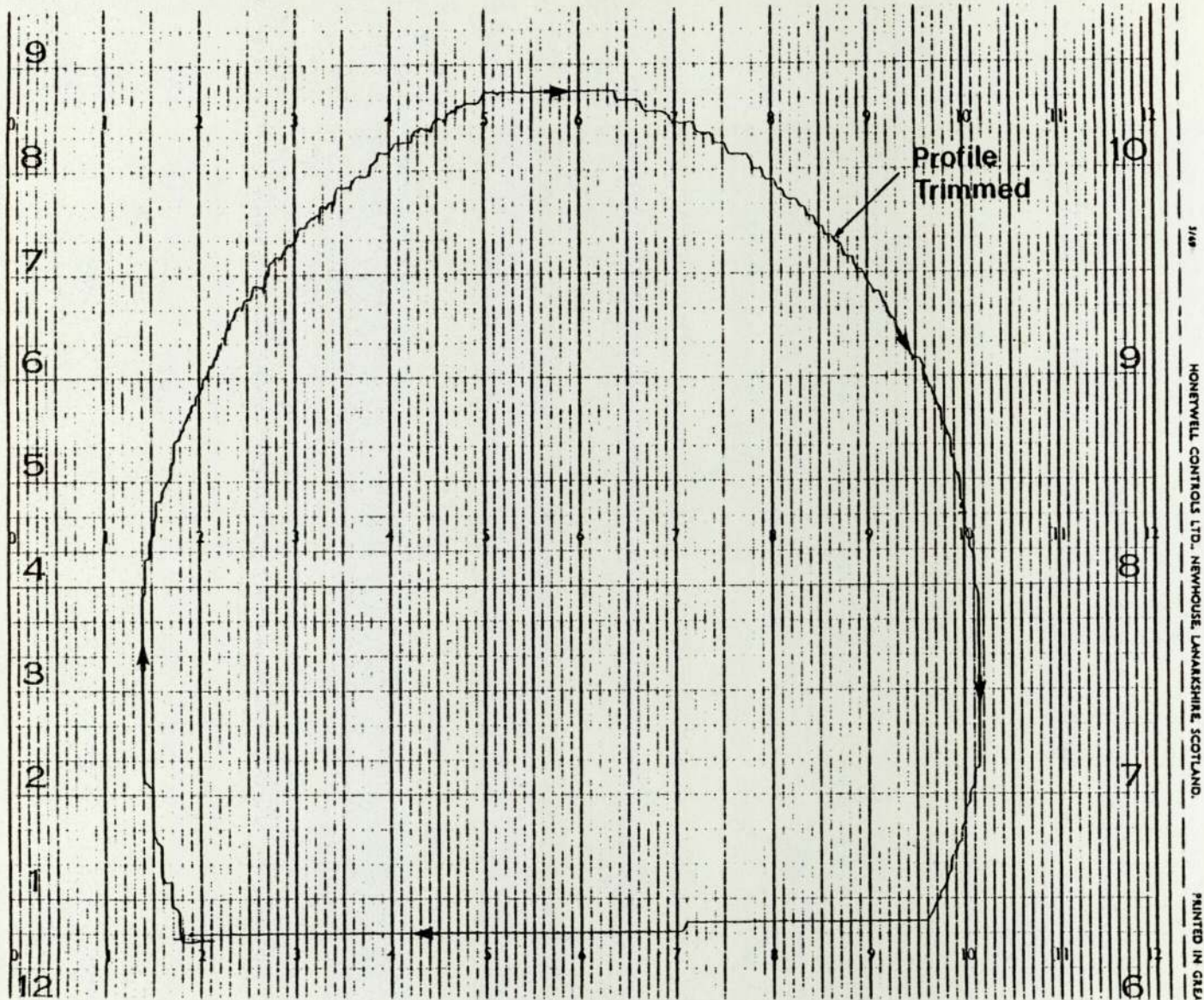


Fig. 7.3 Trace of the Boom Cutter Movement in Trimming the Profile.



## 7.4. Test Results

### 7.4.1. Boom Control by Simple Comparison With Speed Limitation

The control strategy, using simple comparison between the current and the desired boom positions, has been coded (programmed) and tested. In this method, the slow move mode will be set when the comparison reveals that the two positions are near to each other. The results are shown in Figures 7.3 and 7.4 for the profile trimming and the cavity cutting processes respectively. It can be seen from these traces that this control strategy is satisfactory: the accuracy is within the ADC tolerance ( $\pm 1$  LSB). It can be noted from Figure 7.3 that the downward movement of the boom in trimming the profile boundary has got a larger step than the upward movement, this is because the dynamics of the boom ripper are different in both directions. The heavy mass of the boom causes it to lower more before the valve can be closed completely, while this heavy mass will help in stopping the boom from its upward movement.

It can also be seen from these figures that the boom movement is within the defined profile boundary. This can be easily observed when the cutter is ripping the upper cavity layers. As described in Chapter 5, the starting and ending points of the cavity layer to be cut are half the vertical cutting step size away from the profile boundary. In order to simplify the control program, the cutter movement is restricted to within or on the defined profile boundary for all movements. Therefore the cutter will advance to the next layer along the profile boundary instead of a distance

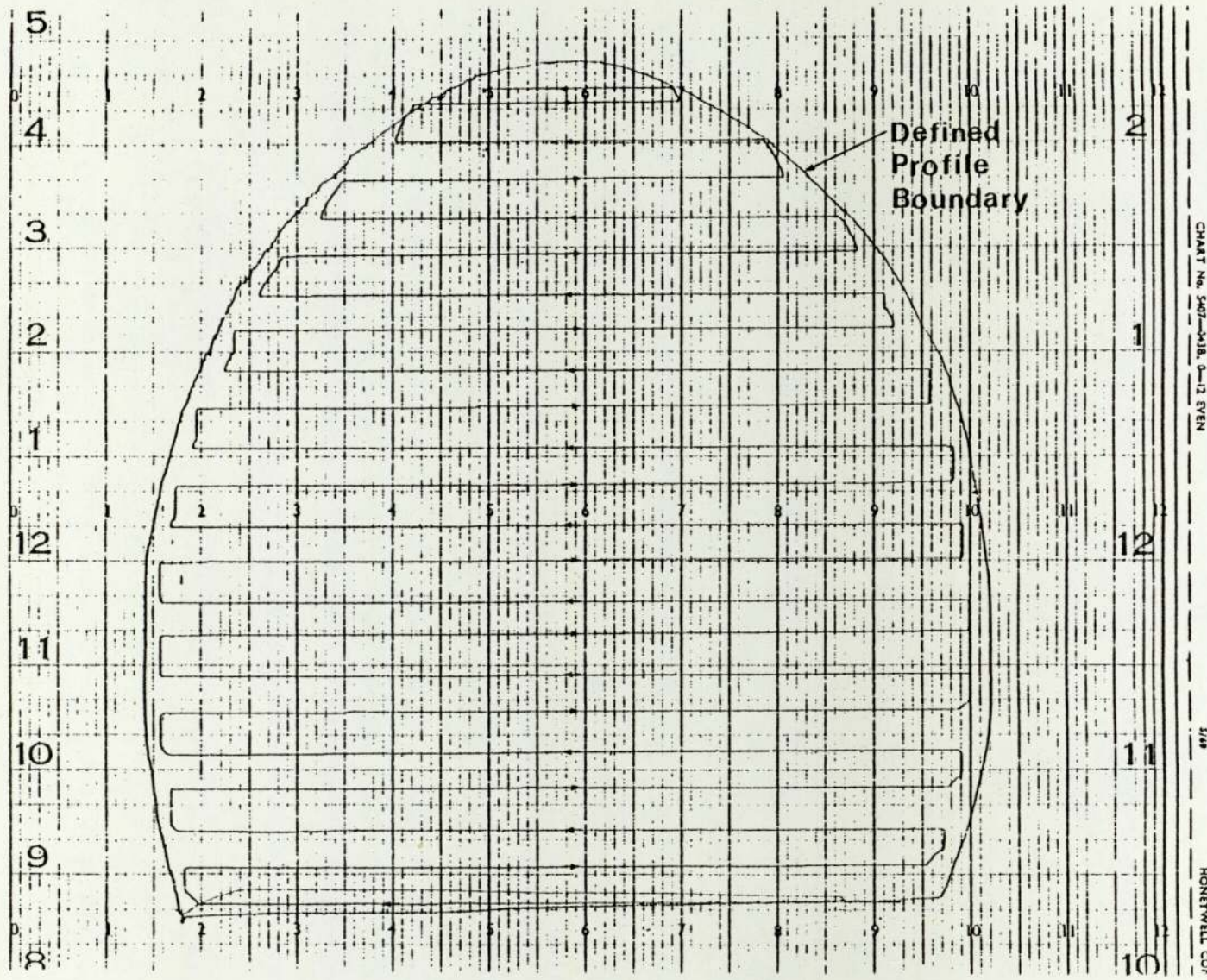


Fig. 7.4 Trace of the Boom Cutter Movement in Cutting the Cavity

away from it when the cutter is ripping the uppermost cavity layers and hits the profile boundary.

It is difficult to demonstrate the 'Move Boom' facility in a still figure (or recording), therefore only the observations are reported. When the boom is required to move fast to a position, it will move at the high speed to a region near the desired position and then it is restricted to moving slowly towards that and stop when it reaches this position. If the boom starts from a position near the profile boundary, it will move slowly away from the boundary until it enters the 'free' space where it can move at high speed if the cutter load permits. The boom will not move outside the defined profile boundary.

#### 7.4.2. Boom Control with Switching Function Evaluation

It can be seen from the simulation studies in Chapter 3 that the time delay in the valve operation, the external force and the ram velocity are the dominant factors in determining the instant to send the switching off (closing) signals to the valves.

The time delay in the valve operation has been measured by Knight to be approximately 390 mSec with a system supply pressure of 138 bars. However, this time delay will depend on the system supply pressure and the pressure difference across the valve. In the system under test, a metering out technique is used to provide the different boom moving speeds, therefore the time delay in the valve operation will vary if the speed changes. This time variation



and the complexity in the system dynamics make it difficult to theoretically determine the coefficients in the switching functions for this system, and experimental determination of the coefficients has been used.

The external force is mainly due to the heavy weight of the boom. It has been mentioned in Chapter 3 that the effect of the external force is mainly on the constant term in the switching function, while the time delay in valve operation also affects the coefficient of the velocity term in this function, and the acceleration is so small that its contribution to the switching function can be neglected (because the steady state can normally be achieved before the switching signals are sent).

Therefore a plot of the amount of overshoot (the boom travel since the application of the closing signal before it comes to a rest) against the velocity at which the boom is moving when the switching signal is sent, will give the required coefficients. Figures 7.5 and 7.6 are obtained for the horizontal and vertical (lifting) movements in the units used by the data capturing system. The horizontal slew and the vertical elevation ranges are both scaled from 0 to 255D bits in the data capturing system. In physical units, the horizontal slew is from  $-52^{\circ}57'$  to  $52^{\circ}57'$ , while the vertical elevation is from  $-34^{\circ}12'$  to  $44^{\circ}46'$ .

These graphs show that the overshoots and the initial boom velocities (when the closing valve signals are sent) have got approximately linear relationships. Therefore

the slopes of these graphs will give the coefficients for the velocity terms in the switching functions and the y-intercepts will yield the constant terms.

Although it is difficult to theoretically determine the coefficients for the switching functions, an attempt has been made to estimate the coefficients. The following parameters are used for the lifting rams:

Mass of Boom = 3070 kg (Calculated from the measured boom dimensions and assumed densities).

Mass of Ram = 7 kg.

Effective External Force = 40 kN (due to the weight of boom, this ranges from 37.65 to 43.67 kN depending on the boom elevation).

Valve Dimensions = 6 mm x 3 mm.

On-Off Valve with Time Delay = 390 mSec.

From the switching function in Section 3.6.3 with the parameter adjustment (for an asymmetric system with external force) mentioned in Sections 3.6.5 and 3.7, the switching function for this lifting ram has been reduced to the form as Equation (4.1):-

$$\lambda_1 \ddot{Y} + \lambda_2 \dot{Y} + \lambda_3 Y + \lambda_4 Y_d + \lambda_5 = 0$$

where  $\lambda_1 = 0.998$

$\lambda_2 = 2472$  (mainly affected by the valve time delay)

$$\lambda_3 = 1$$

$$\lambda_4 = -1$$

$\lambda_5 = -0.04568$  (both the applied force and the valve time delay will contribute to this).

The magnitude of the acceleration term is very small compared with the others in this function, therefore the switching function can be reduced to:

$$\lambda_2 \dot{y} + (y - y_d) + \lambda_5 = 0$$

After these coefficients have been converted into the units used by the control system. The coefficients for the velocity ( $\lambda_2$ ) is 27113, and the constant term ( $\lambda_5$ ) is -0.188. The experimentally determined values (from Figure 7.6) are 29179 and -0.5 respectively. This suggests that the time delay in the system is longer than that used in the theoretical prediction, while the actual load acting on the hydraulic rams is higher. The reasons for a longer time delay in the system are thought to be an increase in the chamber pressures due to the metering out technique used and the actual working system pressure has been set at 110 Bars. The theoretically determined coefficient for the velocity term differs from the experimental value by 0.8%. Because of the complexity in the dynamics of the system, estimation of the external force cannot be accurate. If the quoted weight of the boom (3.25 tons) by the manufacturer has been assumed, the external force will range from 42.2 to 47.1 kN. If the average value (45 kN) of this force range is used, then the theoretical prediction will give 27310 and -0.161 for  $\lambda_2$

and  $\lambda_5$  respectively. When the experimental accuracy of the data capturing system is considered: the 8-bit ADC used has an error of  $\pm 1$  LSB and the boom velocity will be calculated whenever a positional change of three bits or more has been detected. Therefore an absolute positional error of 1 bit and an absolute error of 2 bits in the velocity calculated are expected for the worst cases. Since the accuracy in the 16-bit timer is  $1/(2^{16}-1)$ , the error in the velocity calculation is mainly due to the ADC, this is expected to be  $\pm 2/3$  ( $\pm 66.67\%$ ). If these have been considered, then the slopes ( $\lambda_2$ ) will lie between 29200 and 27090, while the y-intercepts ( $\lambda_5$ ) will have values between 6.16 and -6.9. The theoretical values based on the 390 mSec time-delay on-off valve and a 45 kN external force also fall within this region.

Similarly the theoretically predicted values of  $\lambda_2$  and  $\lambda_5$  for the slewing rams are 23443 and 0.01 based on on-off valves with 390 mSec time-delay. However, Figure 7.5 shows that the relationships between the overshoots and the velocities can be described by two distinct lines depending on the direction of the boom slew. Since the boom slewing is controlled by two different sets of poppet valves, the time-delay for valve operation may be different. The experimentally obtained coefficients (from Figure 7.5) are:  $\lambda_2$  equals to 24894 and  $\lambda_5$  is -0.59 when the boom slews to the right, but  $\lambda_2$  is 30303 and  $\lambda_5$  equals to 0.98 if the boom slews to the left. The slopes of these lines will give the approximate time-delays in the valves: 414 mSec for the valves giving the right



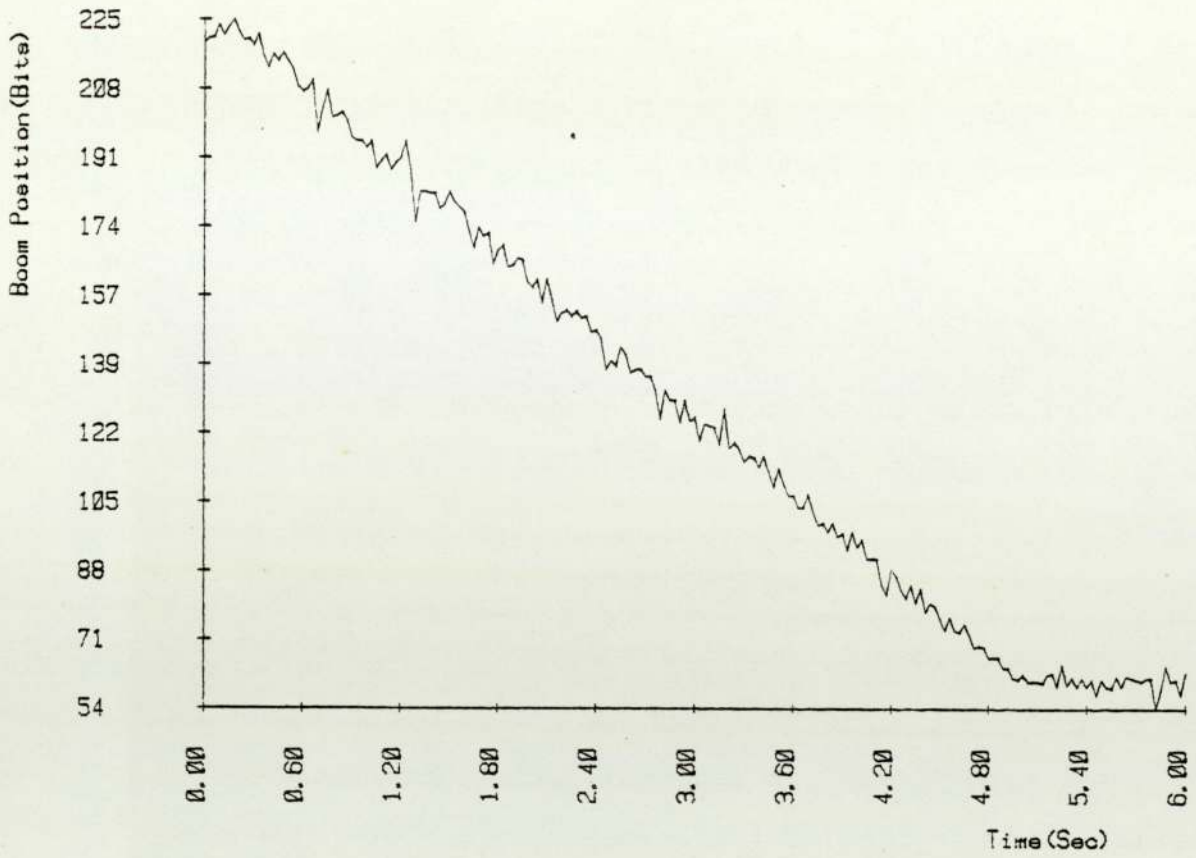


FIG 7.7 Boom Lowering - Time Response

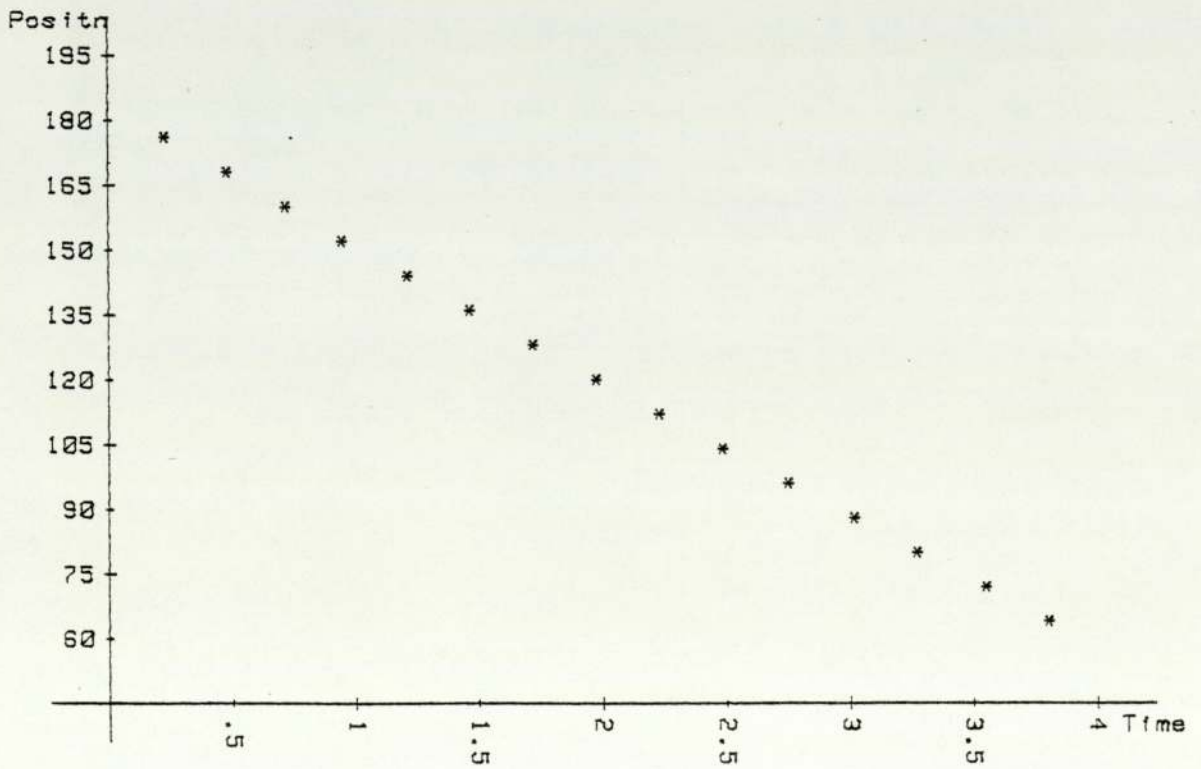


Fig 7.8 Boom Lowering Position (Bits) - Time (Sec) Response.

slew, and 505 mSec for the valves giving the left slew (these valves are slower than the measured value for the lifting rams --- 390 mSec). If these values are used in re-calculating the switching coefficients. The following values are obtained: 24049 for  $\lambda_2$  and -.05 for  $\lambda_5$  if the boom slews to the right, 30065 for  $\lambda_2$  and .06 for  $\lambda_5$  if the boom slews to the left. If the experimental errors in the data capturing system have been taken into account, the values for  $\lambda_2$  can lie between 18182 and 90910 for the right slew valves, while they are within the range 14936 and 74682 for the left slew valves. Similarly, the ranges for the coefficients,  $\lambda_5$ , are: -1.59 to 0.41 for the valves slewing the boom to the right, -0.02 to 1.98 for those causing the boom to slew towards the left.

Figure 7.7 shows an analogue recording of the time response for the boom lowering while Figure 7.8 shows the equivalent recording by the data capturing system. It can be seen from the analogue trace that the signal is very noisy. However, the magnitude of this noise is so small that it can hardly be detected by the data capturing process. Therefore, the values of the velocities and the positions captured can be used in evaluating the switching function without any correction (adjustment) to compensate for the noise.

Figures 7.9 and 7.10 show the results of the tests on the boom slewing and lifting using the simplified switching functions as shown on P.139. The initial and final errors together with the velocities, at which the boom was moving when the 'close' signals

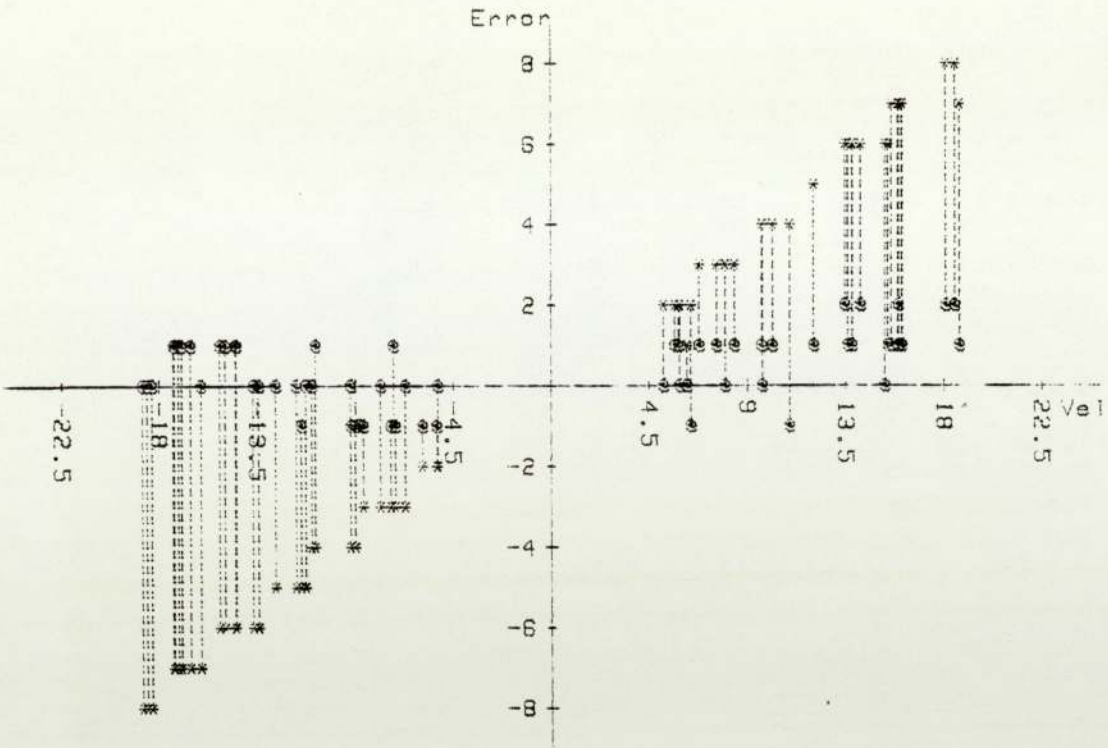


Fig 7.9 Error (Bits) - Boom Slowing Velocity (Bits/Sec)

\* --- Initial Error When the Switching Signal is Sent  
 @ --- Final Error When the Boom Comes to a Rest

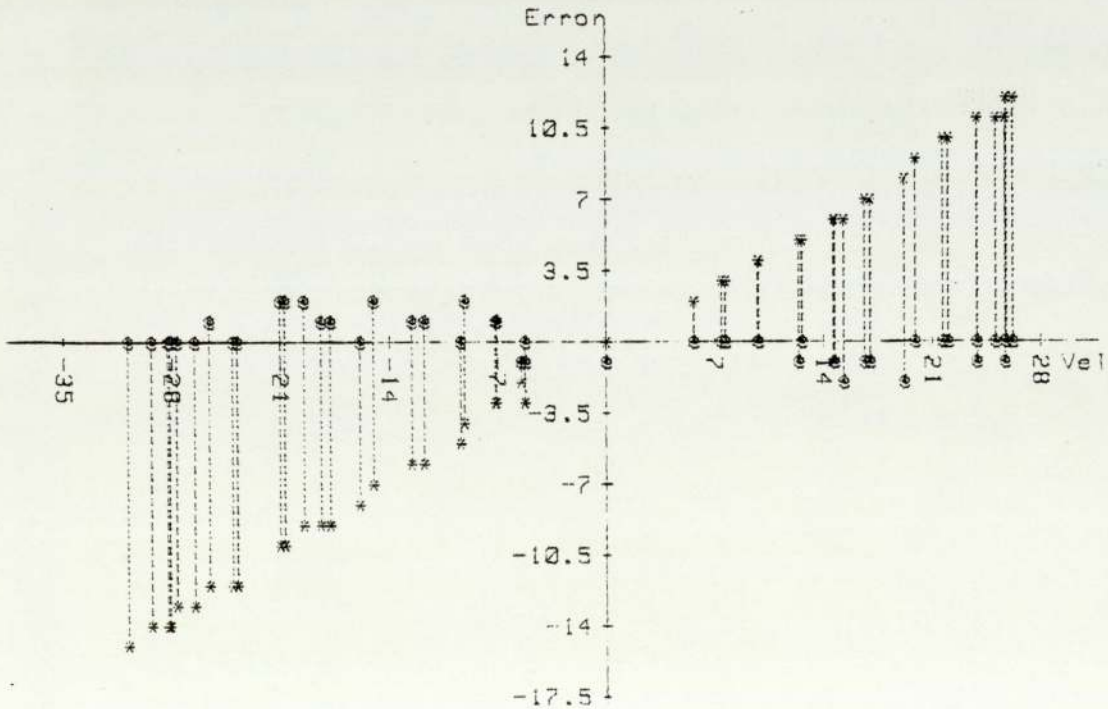


Fig 7.10 Error (Bits) - Boom Lifting Velocity (Bits/Sec)

\* --- Initial Error When the Switching Signal is Sent  
 @ --- Final Error When the Boom Comes to a Rest

were sent to the valves, are presented in these graphs. It can be observed that the absolute final errors are never more than 2 bits. Since the ADC has an accuracy of  $\pm 1$  LSB, the worst error due to this ADC limitation is expected to be  $\pm 2$  LSB ( $\pm 1$  LSB when the switching signal is determined coupled with  $\pm 1$  LSB when the final boom positional reading is taken). This confirms that the switching function can determine the switching point giving a final position within the expected accuracy of the system.

The switching functions have been programmed and tested for the boom positional control in the three activities (Move Boom, Trim Profile and Cut Cavity). Similar results to those obtained using the comparison method with slow speed limitation (Figures 7.3 and 7.4) have been obtained, therefore the recordings of these tests are not included. The only difference between these two methods is that the overall time taken for the boom to move by the comparison method (with slow speed limitation) is longer than the other one. This is because the band, in which only the slow speed mode is permitted, is larger when the comparison method is used ( $\pm 18$  bits compared with  $\pm 4$  bits). This band is included in the switching function method to prevent the boom from hunting when it is initially very near the desired position.

**CHAPTER      8**

**CONCLUSIONS AND RECOMMENDATIONS**

### 8.1. Conclusions

The test results in Chapter 7 show that the positional control of a boom ripper can be obtained. This involves the use of a dual microprocessor system, which provides a high operation speed. This is because the two microprocessors are running in parallel: one microprocessor system can concentrate on capturing and updating the system performance and the required data, while the other system can analyse these data and determine the control signals. Furthermore, the use of an APU will simplify the software and increase the speed in handling the mathematical operations.

Poppet valves have advantages over the conventional spool valves for this control application. Firstly, poppet valves are less sensitive to dirt and therefore are ideal for the mining environment. Secondly, poppet valves have two states: they can either be fully opened or closed. Therefore poppet valves can be operated directly on the 'On-Off' (or digital) signals sent by the microprocessor which is a digital device itself. Thirdly, poppet valves are easier and less expensive to manufacture. They also have less leakage for a fluid with a low viscosity compared with the conventional spool valves which have sliding seals. All these features make the poppet valves suitable for use in this mining boom ripper positional control system.

The optimal control theory shows that the response time can be minimised by maximising the acceleration and carefully

controlling the retardation. However, a 'bang-bang' control philosophy is adopted in this project instead of the normal way of controlling the valve closure. In this 'bang-bang' control strategy, simple On-Off signals are sent to the electro-hydraulically operated poppet valves. Two methods can be used in determining the control signals. They are both satisfactory in their performance in this application.

The simple comparison method is the simpler of the two. In this method, the control signals are determined by comparing the desired with the current boom positions: the valves will be opened if the two positions are not the same, but they will be closed when the two data match. In order to prevent any undesirable overshoot when the boom is moving at its high speed, it is necessary to restrict the speed of the boom to its slow mode when it is near the desired position. This region should be large enough to prevent overshooting when the boom is travelling at its highest speed. Because of this band, the response time is not a minimum when the boom changes its position at high speed.

The second method is to use the switching function evaluation in determining the signals sent to the control valves. Various switching functions have been derived in Chapter 3. J H Knight has studied the performance of the valves used in this control system, his results suggest that the valves can be approximated to On-Off valves with time delays. Because of the complexity in the system geometry and the

dynamics, it is extremely difficult to theoretically determine the coefficients in the switching function. So experimental evaluations have been used, which correlate favourably with the theory. These switching functions are then tested individually for each axis of movement (i.e. lifting and slewing). The results in Chapter 7 show that the use of this switching function can shorten the overall response time and hence an optimal control will be obtained when the boom is travelling at its high speed. It is worth noting that the data capturing system cannot detect a velocity when the boom moves slowly because of the limitation in the accuracy of the ADC and the capacity of the timer installed. Therefore, the evaluation of the switching function at the slow speed can effectively be reduced to a simple comparison.

## 8.2. Recommendations

The results in Chapter 7 show that the simple comparison method with speed limitation near the desired position can provide the necessary activities and adequate control. The simple comparison method yields a simpler and less expensive control system, which consumes less energy compared with a system using the switching function evaluation method. In this particular application, minimising the time, taken by the boom ripper to move from one position to another, is not very important. On the contrary, the intrinsic safety requirement demands a control circuit with less power consumption. Therefore, the simple comparison method with slow speed limitation near the desired position is recommended for this system.



A dual microprocessor system can provide a fast and continuous monitoring of the electro-hydraulic valves. The control signals sent to the valves can be obtained in a much shorter time, when the simple comparison method is used instead of the switching function evaluation method. This is because the function evaluation method requires more complicated mathematical operations, which need more time to handle, although the data manipulation time can be shortened by using an APU. However, the requirement of a wide region, in which only the slow speed mode is allowed for the boom to travel, makes the total time taken to fast move the cutter from one position to another longer in the simple comparison method. Therefore, if a fast response in moving the boom at its high speed is desired, the switching function evaluation method is recommended.

### 8.3. Suggestions for Further Work

A microprocessor based system can simplify and increase the facilities in the automatic operations for a boom ripper. Since the microprocessor is a digital device, it must operate on digital signals. This demands that all the analogue signals should be converted into the digital forms before they can be processed by the microprocessors. The accuracy of the system performance therefore depends highly upon the resolution power of the ADC installed. For the boom ripper under consideration, the accuracy in positioning the boom ripper at its cutter end, is approximately 1 cm. If a finer positional control is desired for the boom ripper, then an ADC with a higher

resolution power should be used, and the associated hardware and software have to be developed. For example, a 12-bit ADC with 1-bit accuracy can increase the resolution power from 1/255 (for an 8-bit ADC) to 1/4095.

A basic or the simplest control system has been developed, and extra facilities can be built around this system. One example is to construct a selector that can choose, from a variety of patterns, the method used to cut the cavity.

In the control system under consideration, no provisions have been made for the machine chassis movement. However, it will be easy to include the necessary correction, if the current machine position related to the tunnel frame is known, by considering the kinematics of the boom ripper. This can be done by converting all the positional data (desired and the current positions) into a common reference co-ordinate system and then determining the control signals with the transformed data. However, this will require a high amount of arithmetic operations, and the use of an APU will definitely facilitate these. Since the APU and its associated circuit board have been included in this study, the software development will not present a great difficulty. The main problem is to provide the information on the machine position and its orientation, therefore further work in developing a system to give these data is suggested.

The control system studied in this project has been designed for a mining boom ripper. This can be extended to provide control to the more sophisticated drilling boom machine by considering the machine geometry. Because the drilling machine has seven degrees of freedom, more mathematical operations will be required. Hardware and software development based on the APU is strongly recommended to provide a faster and simpler control system.

It has been observed that the hydraulic configuration of the boom ripper under consideration makes the fluid flow through the path with less resistance. In this project, an algorithm has been built into the system to give preference to the boom's vertical movement at high speed. Further work is suggested to develop a fluid flow divider so that the maximum available fluid power can be utilised.

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Abbreviations used:

AIEE	American Institution of Electrical Engineers.
ASME	American Society of Mechanical Engineers.
Appl.	Applied.
BHRA	British Hydrodynamics Research Association.
Conf.	Conference on.
Engng.	Engineering.
Hyd.	Hydraulic.
IChemE	Institution of Chemical Engineers.
IEE	Institution of Electrical Engineers.
IEEE	Institution of Electrical and Electronic Engineers.
IMechE	Institution of Mechanical Engineers.
IMinE	Institution of Mining Engineers.
Int.	International.
J.	Journal of.
JDSMC	Journal of Dynamic Systems, Measurement and Control.
Mech.	Mechanical or Mechanics.
MRDE	Mining Research and Development Establishment.
NCB	National Coal Board.
Proc.	Proceedings of.
Pt.	Part.
Sci.	Science.
Symp.	Symposium.
Trans.	Transactions of.

Year of publication is enclosed in parenthesis ( ).

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## APPENDICES

APPENDIX A

Derivation of Switching Function for  
ON-OFF Valves with Time Delay

After the application of switching signal but before the actual operation of valve, the governing equation is given by

$$\frac{d^3y}{dt^3} + (2\zeta + v\xi)\frac{d^2y}{dt^2} + (1 + 2v\zeta\xi)\frac{dy}{dt} = \xi \quad (3.31)$$

Integrating with respect to t gives,

$$\frac{d^2y}{dt^2} + (2\zeta + v\xi)\frac{dy}{dt} + (1 + 2v\zeta\xi)y = \xi t + R_1$$

$$\text{where } R_1 = \ddot{y}_0 + (2\zeta + v\xi)\dot{y}_0 + (1 + 2v\zeta\xi)y_0$$

Applying Laplace transform to both sides of this equation gives,

$$s^2 \mathcal{L}y - s\dot{y}_0 - y_0 + (2\zeta + v\xi)(s \mathcal{L}y - y_0) + (1 + 2v\zeta\xi)\mathcal{L}y = \frac{\xi}{s^2} + \frac{R_1}{s}$$

Rearranging give,

$$\begin{aligned} Y &= \frac{1}{s^2 + (2\zeta + v\xi)s + (1 + 2v\zeta\xi)} \left[ \frac{\xi}{s^2} + \frac{R_1}{s} + s y_0 + \dot{y}_0 + (2\zeta + v\xi)y_0 \right] \\ &= \frac{1}{s^2 + (2\zeta + v\xi)s + (1 + 2v\zeta\xi)} \left[ \frac{\xi}{s^2} + \frac{\ddot{y}_0}{s} + \frac{2\zeta + v\xi}{s} \dot{y}_0 + \dot{y}_0 \right] + \frac{y_0}{s} \end{aligned}$$

$$\begin{aligned} \text{Let } (s - \phi_1)(s - \phi_2) &= s^2 + (2\zeta + v\xi)s + (1 + 2v\zeta\xi) \\ &= s^2 - (\phi_1 + \phi_2)s + \phi_1\phi_2 \end{aligned}$$

$$\therefore Y = \frac{1}{\phi_1 - \phi_2} \left[ \frac{1}{s - \phi_1} - \frac{1}{s - \phi_2} \right] \left[ \frac{\xi}{s^2} + \frac{\ddot{y}_0}{s} - \frac{\phi_1 + \phi_2}{s} \dot{y}_0 + \dot{y}_0 \right] + \frac{y_0}{s}$$

Applying inverse Laplace transform to this equation gives,

$$\begin{aligned}
 y(t) = & \frac{\xi t}{\phi_1 \phi_2} + \frac{\xi}{\phi_1 - \phi_2} \left( \frac{e^{\phi_1 t}}{\phi_1^2} - \frac{e^{\phi_2 t}}{\phi_2^2} \right) + \frac{\phi_1 + \phi_2}{\phi_1^2 \phi_2^2} \xi + y_0 \\
 & + \frac{\dot{y}_0}{\phi_1 - \phi_2} \left[ \frac{\phi_1}{\phi_2} e^{\phi_2 t} - \frac{\phi_2}{\phi_1} e^{\phi_1 t} - \frac{(\phi_1^2 - \phi_2^2)}{\phi_1 \phi_2} \right] \\
 & + \frac{\ddot{y}_0}{\phi_1 - \phi_2} \left[ \frac{e^{\phi_1 t}}{\phi_1} - \frac{e^{\phi_2 t}}{\phi_2} + \frac{\phi_1 - \phi_2}{\phi_1 \phi_2} \right]
 \end{aligned}$$

Expression of  $\dot{y}(t)$  can be obtained by differentiating  $y(t)$  with respect to  $t$ , the non-dimensional time. Hence,  $\dot{y}(t) =$

$$\begin{aligned}
 \frac{dy}{dt} = & \frac{\xi}{\phi_1 \phi_2} + \frac{\xi}{\phi_1 - \phi_2} \left( \frac{e^{\phi_1 t}}{\phi_1} - \frac{e^{\phi_2 t}}{\phi_2} \right) + \frac{\dot{y}_0}{\phi_1 - \phi_2} \left[ \phi_1 e^{\phi_2 t} - \phi_2 e^{\phi_1 t} \right] \\
 & + \frac{\ddot{y}_0}{\phi_1 - \phi_2} \left[ e^{\phi_1 t} - e^{\phi_2 t} \right]
 \end{aligned}$$

Differentiating  $\frac{dy}{dt}$  with respect to  $t$  gives the expression for  $\ddot{y}(t)$ ,

$$\begin{aligned}
 \ddot{y}(t) = \frac{d^2y}{dt^2} = & \frac{\xi}{\phi_1 - \phi_2} \left( e^{\phi_1 t} - e^{\phi_2 t} \right) + \frac{\phi_1 \phi_2 \dot{y}_0}{\phi_1 - \phi_2} \left( e^{\phi_2 t} - e^{\phi_1 t} \right) \\
 & + \frac{\ddot{y}_0}{\phi_1 - \phi_2} \left( \phi_1 e^{\phi_1 t} - \phi_2 e^{\phi_2 t} \right)
 \end{aligned}$$

After the time delay,  $t_d$ , the system behaviour will be described by the same set of differential equations as the ideal ON-OFF case. Therefore, for the ram to rest at the desired position,  $y_d$ , the valves have to be switched so that at  $t_d$  after the application of switching (closing) signal the following will be satisfied:

$$y_d = \ddot{y}_{t_d} + 2\zeta\dot{y}_{t_d} + y_{t_d}$$

Therefore the switching function is:

$$\begin{aligned}
 y_d = & \frac{\ddot{y}_o}{\phi_1 - \phi_2} \left( \phi_1 + 2\zeta + \frac{1}{\phi_1} \right) e^{\phi_1 t_d} - \left( \phi_2 + 2\zeta + \frac{1}{\phi_2} \right) e^{\phi_2 t_d} + \frac{\phi_1 - \phi_2}{\phi_1 \phi_2} \\
 & + \frac{\dot{y}_o}{\phi_1 - \phi_2} \left[ \phi_1 \left( \phi_2 + 2\zeta + \frac{1}{\phi_2} \right) e^{\phi_2 t_d} - \phi_2 \left( \phi_1 + 2\zeta + \frac{1}{\phi_1} \right) e^{\phi_1 t_d} - \frac{\phi_1^2 - \phi_2^2}{\phi_1 \phi_2} \right] \\
 & + \frac{\xi}{\phi_1 \phi_2} \left( 2\zeta + t_d + \frac{\phi_1 + \phi_2}{\phi_1 \phi_2} \right) + y_o \\
 & + \frac{\xi}{\phi_1 - \phi_2} \left[ \frac{1}{\phi_1} \left( \phi_1 + 2\zeta + \frac{1}{\phi_1} \right) e^{\phi_1 t_d} - \frac{1}{\phi_2} \left( \phi_2 + 2\zeta + \frac{1}{\phi_2} \right) e^{\phi_2 t_d} \right]
 \end{aligned}$$



APPENDIX B

Derivation of Switching Function for  
Linearly Operated Valves with Operating Time  $t_c$

After the application of switching signal (closing signal) but before the complete closure of valves, the governing equation is given by:

$$\frac{d^3y}{dt^3} + [2\zeta + v\xi(1 - \frac{t}{t_c})] \frac{d^2y}{dt^2} + [1 + 2v\zeta\xi(1 - \frac{t}{t_c})] \frac{dy}{dt} = \xi(1 - \frac{t}{t_c}) \dots \quad (3.32)$$

Applying Laplace transform to both sides of this equation gives,

$$s^3 \mathcal{L}y - s^2 y_0 - s \dot{y}_0 - \ddot{y}_0 + (2\zeta + v\xi)(s^2 \mathcal{L}y - s y_0 - \dot{y}_0) + \frac{v\xi}{t_c} (2s \mathcal{L}y - y_0) + (1 + 2v\zeta\xi)(s \mathcal{L}y - y_0) + \frac{2v\zeta\xi}{t_c} \mathcal{L}y = \xi \left( \frac{1}{s} - \frac{1}{t_c s^2} \right)$$

Rearranging gives,

$$y = \frac{1}{s^3 + (2\zeta + v\xi)s^2 + (1 + 2v\zeta\xi + \frac{2v\xi}{t_c})s + \frac{2\zeta\xi v}{t_c}} *$$

$$\left\{ \begin{array}{l} \ddot{y}_0 + (s + 2\zeta + v\xi)\dot{y}_0 + [s^2 + (2\zeta + v\xi)s + 1 + 2v\zeta\xi + \frac{v\xi}{t_c}]y_0 \\ + \xi \left( \frac{1}{s} - \frac{1}{t_c s^2} \right) \end{array} \right\}$$

Taking the inverse Laplace transform gives the expression for  $y(t)$ ,

$$\begin{aligned}
 y(t) &= \frac{1}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} * \\
 & \left[ y_0 \left[ (\theta_3 - \theta_2) e^{\theta_1 t} + (\theta_1 - \theta_3) e^{\theta_2 t} + (\theta_2 - \theta_1) e^{\theta_3 t} \right] \right. \\
 & + \dot{y}_0 \left[ \begin{aligned} & (2\zeta + v\xi + \theta_1)(\theta_3 - \theta_2) e^{\theta_1 t} + (2\zeta + v\xi + \theta_2)(\theta_1 - \theta_3) e^{\theta_2 t} \\ & \left. + (2\zeta + v\xi + \theta_3)(\theta_2 - \theta_1) e^{\theta_3 t} \right] \right. \\
 & + y_0 \left[ \begin{aligned} & \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_1 + \theta_1^2 \right\} (\theta_3 - \theta_2) e^{\theta_1 t} \\ & + \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_2 + \theta_2^2 \right\} (\theta_1 - \theta_3) e^{\theta_2 t} \\ & + \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_3 + \theta_3^2 \right\} (\theta_2 - \theta_1) e^{\theta_3 t} \end{aligned} \right] \\
 & + \xi \left[ \begin{aligned} & \left( 1 - \frac{1}{t_c \theta_1} \right) \frac{\theta_3 - \theta_2}{\theta_1} e^{\theta_1 t} + \left( 1 - \frac{1}{t_c \theta_2} \right) \frac{\theta_1 - \theta_3}{\theta_2} e^{\theta_2 t} \\ & + \left( 1 - \frac{1}{t_c \theta_3} \right) \frac{\theta_2 - \theta_1}{\theta_3} e^{\theta_3 t} - \left( 1 - \frac{t}{t_c} \right) \left( \frac{\theta_3 - \theta_2}{\theta_1} + \frac{\theta_1 - \theta_3}{\theta_2} + \frac{\theta_2 - \theta_1}{\theta_3} \right) \\ & + \frac{1}{t_c} \left( \frac{\theta_3 - \theta_2}{\theta_1^2} + \frac{\theta_1 - \theta_3}{\theta_2^2} + \frac{\theta_2 - \theta_1}{\theta_3^2} \right) \end{aligned} \right]
 \end{aligned}$$

$$\therefore \dot{y}(t) = \frac{dy}{dt} = \frac{1}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} *$$

$$\begin{aligned} & \left[ \ddot{y}_0 \left[ \theta_1(\theta_3 - \theta_2)e^{\theta_1 t} + \theta_2(\theta_1 - \theta_3)e^{\theta_2 t} + \theta_3(\theta_2 - \theta_1)e^{\theta_3 t} \right] \right. \\ & + \dot{y}_0 \left[ \begin{array}{l} \theta_1(2\zeta + v\xi + \theta_1)(\theta_3 - \theta_2)e^{\theta_1 t} + \theta_2(2\zeta + v\xi + \theta_2)(\theta_1 - \theta_3)e^{\theta_2 t} \\ + \theta_3(2\zeta + v\xi + \theta_3)(\theta_2 - \theta_1)e^{\theta_3 t} \end{array} \right] \\ & + y_0 \left[ \begin{array}{l} \theta_1 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_1 + \theta_1^2 \right\} (\theta_3 - \theta_2)e^{\theta_1 t} \\ + \theta_2 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_2 + \theta_2^2 \right\} (\theta_1 - \theta_3)e^{\theta_2 t} \\ + \theta_3 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_3 + \theta_3^2 \right\} (\theta_2 - \theta_1)e^{\theta_3 t} \end{array} \right] \\ & + \xi \left[ \begin{array}{l} \left( 1 - \frac{1}{t_c \theta_1} \right) (\theta_3 - \theta_2)e^{\theta_1 t} + \left( 1 - \frac{1}{t_c \theta_2} \right) (\theta_1 - \theta_3)e^{\theta_2 t} \\ + \left( 1 - \frac{1}{t_c \theta_3} \right) (\theta_2 - \theta_1)e^{\theta_3 t} + \frac{1}{t_c} \left( \frac{\theta_3 - \theta_2}{\theta_1} + \frac{\theta_1 - \theta_3}{\theta_2} + \frac{\theta_2 - \theta_1}{\theta_3} \right) \end{array} \right] \end{aligned}$$

$$\text{and } \ddot{y}(t) = \frac{d^2y}{dt^2} = \frac{1}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} *$$

$$\begin{aligned} & \left[ \ddot{y}_0 \left[ \theta_1^2(\theta_3 - \theta_2)e^{\theta_1 t} + \theta_2^2(\theta_1 - \theta_3)e^{\theta_2 t} + \theta_3^2(\theta_2 - \theta_1)e^{\theta_3 t} \right] \right. \\ & + \dot{y}_0 \left[ \begin{array}{l} \theta_1^2(2\zeta + v\xi + \theta_1)(\theta_3 - \theta_2)e^{\theta_1 t} + \theta_2^2(2\zeta + v\xi + \theta_2)(\theta_1 - \theta_3)e^{\theta_2 t} \\ + \theta_3^2(2\zeta + v\xi + \theta_3)(\theta_2 - \theta_1)e^{\theta_3 t} \end{array} \right] \\ & + y_0 \left[ \begin{array}{l} \theta_1^2 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_1 + \theta_1^2 \right\} (\theta_3 - \theta_2)e^{\theta_1 t} \\ + \theta_2^2 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_2 + \theta_2^2 \right\} (\theta_1 - \theta_3)e^{\theta_2 t} \\ + \theta_3^2 \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_3 + \theta_3^2 \right\} (\theta_2 - \theta_1)e^{\theta_3 t} \end{array} \right] \\ & + \xi \left[ \begin{array}{l} \theta_1 \left( 1 - \frac{1}{t_c \theta_1} \right) (\theta_3 - \theta_2)e^{\theta_1 t} + \theta_2 \left( 1 - \frac{1}{t_c \theta_2} \right) (\theta_1 - \theta_3)e^{\theta_2 t} \\ + \theta_3 \left( 1 - \frac{1}{t_c \theta_3} \right) (\theta_2 - \theta_1)e^{\theta_3 t} \end{array} \right] \end{aligned}$$

Similarly the switching function can be obtained by satisfying the following requirement when the valves are completely closed, i.e. at time,  $t_c$ , after the application of switching signal:

$$y_d = \ddot{y}_{t_c} + 2\zeta\dot{y}_{t_c} + y_{t_c}$$

Therefore, the switching function is:

$$y_d = \frac{1}{(\theta_1 - \theta_2)(\theta_2 - \theta_3)(\theta_3 - \theta_1)} * \left[ \begin{aligned} & \ddot{y}_0 \left[ \begin{aligned} & (1 + 2\zeta\theta_1 + \theta_1^2)(\theta_3 - \theta_2)e^{\theta_1 t_c} + (1 + 2\zeta\theta_2 + \theta_2^2)(\theta_1 - \theta_3)e^{\theta_2 t_c} \\ & + (1 + 2\zeta\theta_3 + \theta_3^2)(\theta_2 - \theta_1)e^{\theta_3 t_c} \end{aligned} \right] \\ & + \dot{y}_0 \left[ \begin{aligned} & (1 + 2\zeta\theta_1 + \theta_1^2)(2\zeta + v\xi + \theta_1)(\theta_3 - \theta_2)e^{\theta_1 t_c} \\ & + (1 + 2\zeta\theta_2 + \theta_2^2)(2\zeta + v\xi + \theta_2)(\theta_1 - \theta_3)e^{\theta_2 t_c} \\ & + (1 + 2\zeta\theta_3 + \theta_3^2)(2\zeta + v\xi + \theta_3)(\theta_2 - \theta_1)e^{\theta_3 t_c} \end{aligned} \right] \\ & + y_0 \left[ \begin{aligned} & (1 + 2\zeta\theta_1 + \theta_1^2) \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_1 + \theta_1^2 \right\} (\theta_3 - \theta_2)e^{\theta_1 t_c} \\ & + (1 + 2\zeta\theta_2 + \theta_2^2) \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_2 + \theta_2^2 \right\} (\theta_1 - \theta_3)e^{\theta_2 t_c} \\ & + (1 + 2\zeta\theta_3 + \theta_3^2) \left\{ 1 + 2v\zeta\xi + \frac{v\xi}{t_c} + (2\zeta + v\xi)\theta_3 + \theta_3^2 \right\} (\theta_2 - \theta_1)e^{\theta_3 t_c} \end{aligned} \right] \\ & + \xi \left[ \begin{aligned} & (1 + 2\zeta\theta_1 + \theta_1^2) \left( 1 - \frac{1}{t_c\theta_1} \right) \frac{\theta_3 - \theta_2}{\theta_1} e^{\theta_1 t_c} \\ & + (1 + 2\zeta\theta_2 + \theta_2^2) \left( 1 - \frac{1}{t_c\theta_2} \right) \frac{\theta_1 - \theta_3}{\theta_2} e^{\theta_2 t_c} \\ & + (1 + 2\zeta\theta_3 + \theta_3^2) \left( 1 - \frac{1}{t_c\theta_3} \right) \frac{\theta_2 - \theta_1}{\theta_3} e^{\theta_3 t_c} \end{aligned} \right] \\ & + \frac{1}{t_c} \left( \frac{\theta_3 - \theta_2}{\theta_1^2} + \frac{\theta_1 - \theta_3}{\theta_2^2} + \frac{\theta_2 - \theta_1}{\theta_3^2} \right) - \left( 1 - \frac{t}{t_c} - \frac{2\zeta}{t_c} \right) \left( \frac{\theta_3 - \theta_2}{\theta_1} + \frac{\theta_1 - \theta_3}{\theta_2} + \frac{\theta_2 - \theta_1}{\theta_3} \right) \end{aligned} \right]$$

APPENDIX C

Computer Programs for the Simulation Studies

The simulation studies were carried out using the modified Euler method [41] to solve the differential equations. This method has a long term stability which is desirable for continuous simulations over an extended period. It evaluates the forward states using the derivatives which are currently updated. The block structure of this program is shown in Figure C.1. The system data have been written onto a file before this program is executed. When this program is executed, the system data are first read and modified if necessary. Then the coefficients of the function will be evaluated according to the appropriate equations (Sections 3.6 and 3.7). The roots of the polynomial are found by the subprogram "Roots" which was described by Chuen [45]. The initial states and the desired position are then entered into the computer. The program will then evaluate the switching function and determine the switching signal. The system behaviours are then evaluated by the modified Euler method and stored if necessary for a small step increment of time. If the simulation is to be continued, then it will repeat from the switching function evaluation; otherwise it will modify the system parameters and repeat the whole simulation process if required.

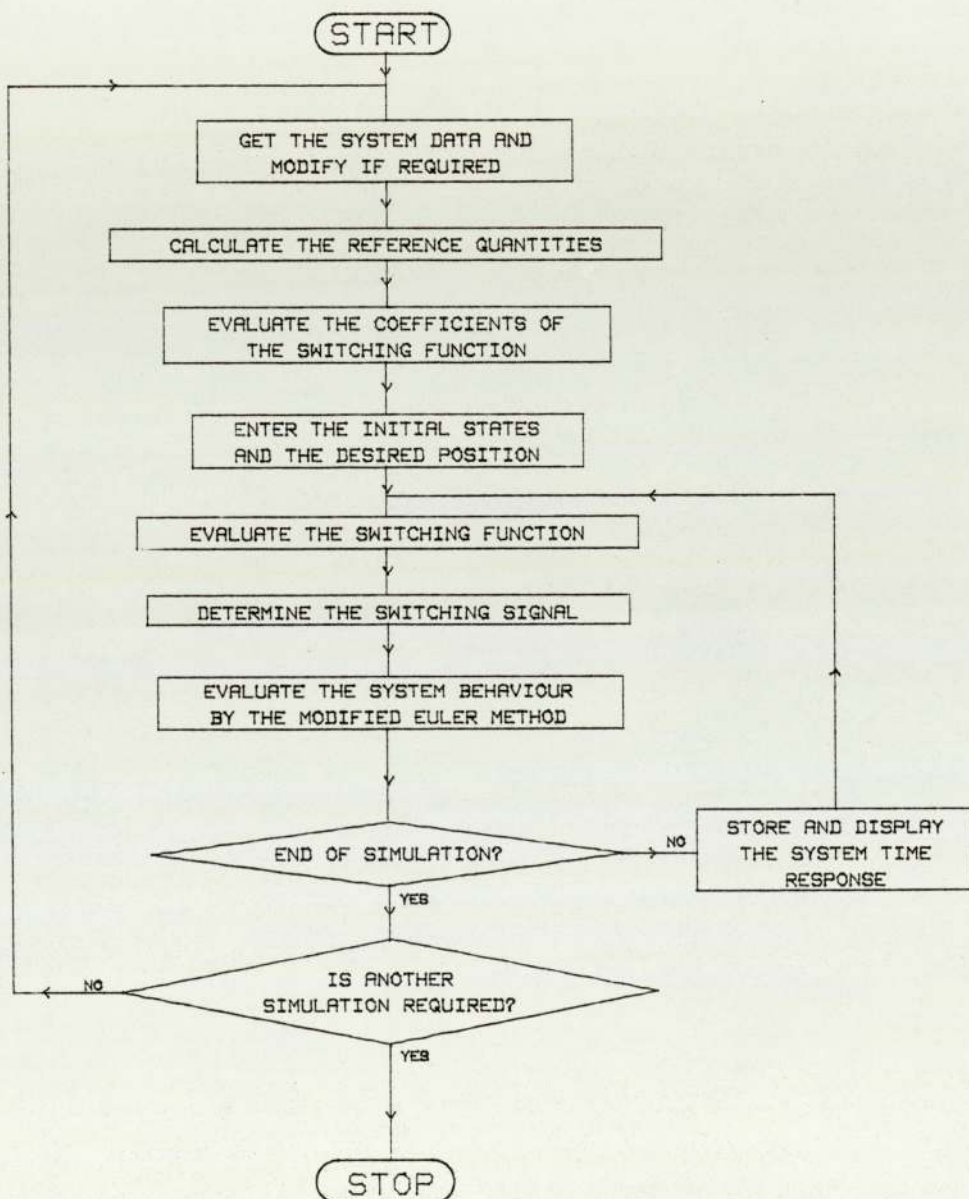


FIG C.1 BLOCK STRUCTURE OF THE SIMULATION PROGRAM



```

20  OPTION BASE 1
30  COM Thetaf,Thetai,Tlimit,Veli,Zmin,Zmax,Ep(500),Z(500)
40  COM Tauc(500),Fo,Zeta,Damping,P1i,P2i,Wtval,Change#[3]
50  COM Deltai,Deltaf,Sety,Tdummyc,M,Tauclose,Type,Sym,Aratio
60  COM INTEGER Rerun,Npoints,Again,Plotter,Dump,No,Printer,
    Pswitch(20),Psws
70  DIM Coefficient(4),Solution(3,2),Result(2),Answer(2)
80  DIM Operand(2),Operand1(2),Operand2(2),Tempres1(2)
90  DIM Tempres2(2),Coeffdum(4),Ramp1(500),Ramp2(500)
100  INTEGER Divide,Multiply,Subtract,Add,Rtorder,Rtorder1
110  SHORT Shortdummy1,Shortdummy2,Shortdummy3
120  PRINTER IS 16
130  PRINT PAGE,"SELECT MASS STORAGE DEVICE WITH THESE CODES:"
140  PRINT
150  PRINT SPA(20),"1 - for Flexible Disk"
160  PRINT
170  PRINT SPA(20),"2 - for Right Cartridge marked T15"
180  PRINT
190  PRINT SPA(20),"3 - for Left Cartridge marked T14"
200  INPUT Selectcode
210  IF (Selectcode>0) AND (Selectcode<4) THEN 240
220  BEEP
230  GOTO 200
240  IF Selectcode=1 THEN Msus$=":F8"
250  IF Selectcode=2 THEN Msus$=":T15"
260  IF Selectcode=3 THEN Msus$=":T14"
270  MASS STORAGE IS Msus$
280  PRINT PAGE
290  READ Divide,Multiply,Subtract,Add
300  DATA 4,3,2,1
310  ASSIGN #5 TO "SOLN"&Msus$,Check
320  IF Check=0 THEN 390
330  IF Check=1 THEN 360
340  DISP "FILE SOLN ERROR:";
350  GOTO 630
360  DISP "CREATE FILE: SOLN"
370  CREATE "SOLN",5
380  GOTO 310
390  BUFFER #5
400  Deltam=Deltas=0
410  Again=No=Deltafo=0
420  ! ***** RE-ENTER AT THIS POINT FOR RE-RUN ***** !
430  Npoints=Pmin=Pmax=Psws=0
440  Zmin=Wmin=Amin=9E99
450  Zmax=Wmax=Amax=-9E99
460  ! ***** INITIALIZATION OF SYSTEM PARAMETERS ***** !
470  Printer=16
480  IF Again THEN 560
490  INPUT "PRINTER DEVICE CODE: 0 for Thermal Printer 16
    for CRT",Printer
500  INPUT "DO YOU WANT TO SPECIFY DURATION OF SIMULATION ",A$
510  Tlimit=9E99
520  Rerun=1
530  IF POS(UPC$(A$),"N") THEN 560
540  INPUT "NON-DIMENSIONAL TIME DURATION",Tlimit
550  Tdummyc=Tlimit
560  Sym=Onoff=Epsilo=Tau=Signal=Print=Sw=0
570  Dratio=Aratio=1
580  Va=Vb=.5
590  ASSIGN #1 TO "SPDATA"&Msus$,Check
600  IF Check=0 THEN 680
610  DISP "FILE SPDATA ERROR: ";
620  IF Check=1 THEN 660
630  DISP "WRONG FILE TYPE, OR FILE IS PROTECTED"
640  BEEP

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650 STOP
660 DISP "FILE IS MISSSING"
670 GOTO 640
680 BUFFER #1
690 Readfile: ! **** READING DATA FROM FILE "SPDATA" **** !
700 READ #1;Sym,Onoff,Time,Cd1,B1,Xs,Ps,Pe,Rho,Beta,Ys
710 READ #1;D1,A1,Ratio,Vr,C1,M,Bd,Ko,Fo
720 Cd2=Cd1
730 D2=D1
740 A2=A1
750 B2=B1
760 IF Sym THEN 780
770 READ #1;Cd2,B2,Dratio,Aratio,Va
780 READ #1;Zeta
790 READ #1,1 ! MOVE THE FILE POINT TO THE FIRST RECORD
800 Thetas=Thetas+Deltas ! ADJUST Theta values on Re-run
810 Thetaf=Thetas
820 IF NOT Again THEN 850
830 A#=Change$
840 GOTO 950
850 PRINT PAGE,"Sym = ";Sym;"Onoff = ";Onoff;"Time = ";Time;"B1
= ";B1;"Xs = ";Xs
860 PRINT "Cd1 = ";Cd1;"Ps = ";Ps;"Pe = ";Pe;"Rho = ";Rho;"Beta
= ";Beta
870 PRINT "Ys = ";Ys;"D1 = ";D1;"Ratio = ";Ratio;"Vr = ";Vr
880 PRINT "C1 = ";C1;"M = ";M;"Bd = ";Bd;"Ko = ";Ko;"Fo = ";Fo
890 PRINT "Zeta = ";Zeta;"Cd2 = ";Cd2;"B2 = ";B2
900 PRINT "Dratio = ";Dratio;"Va = ";Va
910 PRINT LIN(5),"Note that if Zeta was specified then Bd would
be calculated automatically"
920 IF Again THEN 970
930 Change$="NO"
940 INPUT "ANY CHANGE IN THE ABOVE QUANTITIES? (YES/NO)",A#
950 IF POS(UPC$(A#),"N") THEN 1090
960 IF Again THEN 850
970 DISP "MAKE THE CHANGES THEN PRESS <CONT>"
980 PAUSE
990 Aratio=Dratio*Dratio
1000 A1=PI*D1*D1/4E4 ! D1 IS IN cm
1010 ! PRINT NEW DATA INTO FILE
1020 PRINT #1;Sym,Onoff,Time,Cd1,B1,Xs,Ps,Pe,Rho,Beta,Ys
1030 PRINT #1;D1,A1,Ratio,Vr,C1,M,Bd,Ko,Fo
1040 IF Sym THEN 1060
1050 PRINT #1;Cd2,B2,Dratio,Aratio,Va
1060 PRINT #1;Zeta,END
1070 PRINT LIN(2),"Will there be ANY MORE CHANGES when rerun
with different ",LIN(1),"initial & final conditions?"
1080 INPUT Change$
1090 D2=Dratio*D1
1100 Type=Onoff
1110 Aratio=Dratio*Dratio
1120 A2=Aratio*A1
1130 ON KEY #4 GOTO Endcal
1140 ON KEY #2 GOSUB Changeprinter
1150 ASSIGN * TO #1
1160 PRINTER IS Printer
1170 IF Sym THEN 1240
1180 PRINT "SYSTEM PARAMETERS",LIN(2),"SPOOL:"
1190 PRINT " DISCHARGE COEFFICIENT FOR SPOOL VALVE 1: ";Cd1
1200 PRINT " DISCHARGE COEFFICIENT FOR SPOOL VALVE 2: ";Cd2
1210 PRINT " WIDTH OF SPOOL VALVE 1: ";B1;"mm."
1220 PRINT " WIDTH OF SPOOL VALVE 2: ";B2;"mm."
1230 GOTO 1270
1240 PRINT "SYSTEM PARAMETERS (SYMMETRIC)",LIN(2),"SPOOL:"
1250 PRINT " DISCHARGE COEFFICIENT FOR SPOOL VALVE: ";Cd1
1260 PRINT SPA(22);"WIDTH OF SPOOL VALVE: ";B1;"mm."

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1270 PRINT SPA(7);"MAX. SPOOL DISPLACEMENT FROM CENTRE:";Xs;"mm."
1280 IF Onoff THEN PRINT SPA(10);"TIME DELAY FOR THIS ON-OFF
SPOOL:";Time;"mSec."
1290 IF NOT Onoff THEN PRINT SPA(9);"TIME REQUIRED FOR SPOOL
OPERATION:";Time;"mSec."
1300 PRINT "ACTUATOR:"
1310 Vt=Vr*Ratio
1320 PRINT " AT MID POSITION, TOTAL VOLUME OF FLUID IN CIRCUI
T:";Vt;"cu.m."
1330 Var=Vbr=Ratio*Va
1340 IF Sym THEN PRINT "AT MID POSITION, VOLUME OF FLUID IN EACH
CHAMBER:";Va*Vt;"cu.m."
1350 IF Sym THEN 1450
1360 PRINT " AT MID POSITION, VOLUME OF FLUID IN CHAMBER
1:";Va*Vt;"cu.m."
1370 Vb=1-Va
1380 Vbr=Ratio*Vb
1390 PRINT " AT MID POSITION, VOLUME OF FLUID IN CHAMBER
2:";Vb*Vt;"cu.m."
1400 PRINT SPA(21);"PISTON DIAMETER (CHAMBER 1):";D1;"cm."
1410 PRINT SPA(21);"PISTON DIAMETER (CHAMBER 2):";D2;"cm."
1420 PRINT SPA(25);"PISTON AREA (CHAMBER 1):";A1;"sq.m."
1430 PRINT SPA(25);"PISTON AREA (CHAMBER 2):";A2;"sq.m."
1440 GOTO 1470
1450 PRINT SPA(33);"PISTON DIAMETER:";D1;"cm."
1460 PRINT SPA(37);"PISTON AREA:";A1;"sq.m."
1470 PRINT SPA(34);"WORKING STROKE:";Ys;"m."
1480 PRINT SPA(20);"INTERNAL LEAKAGE COEFFICIENT:";C1;"(Litre
/sec)/Bar"
1490 PRINT SPA(13);"EFFECTIVE EXTERNAL SPRING STIFFNESS:";Ko;
"kN/m"
1500 PRINT "OTHERS:",LIN(1)," SYSTEM SUPPLY PRESSURE:";Ps;
"Bar"
1510 PRINT " SYSTEM EXHAUST PRESSURE:";Pe;"Bar"
1520 PRINT " EFFECTIVE EXTERNAL FORCE:";Fo;"kN (ASSUMED CONS
TANT)"
1530 PRINT SPA(8);"FLUID BULK MODULUS:";Beta;"GN/m^2"
1540 PRINT SPA(13);"FLUID DENSITY:";Rho;"kg/m^3"
1550 ! ***** CONVERSION TO S.I. ***** !
1560 Time=Time/1000 ! Time from mSec
1570 B1=B1/1000 ! B1 from mm
1580 B2=B2/1000 ! B2 from mm
1590 Xs=Xs/1000 ! Xs from mm
1600 Ps=Ps*1E5 ! Ps from Bar
1610 Pe=Pe*1E5 ! Pe from Bar
1620 Beta=Beta*1E9 ! Beta from GN/sq.m.
1630 D1=D1/100 ! D1 from cm
1640 D2=D2/100 ! D2 from cm
1650 Ko=Ko*1000 ! Ko from kN/m
1660 Fo=Fo*1000 ! Fo from kN
1670 ! ***** END OF CONVERSIONS ***** !
1680 ! ***** CALCULATION OF CONSTANTS ***** !
1690 Refposn=-Vt*((1+Aratio)*Va-1)/(2*A1*Aratio) ! Ref position
1700 Lref=Va*Vt/A1+Refposn ! Ref length
1710 S1=Beta*A1/Lref ! Hydraulic stiffness column 1
1720 S2=S1
1730 IF Sym THEN 1750
1740 S2=Beta*A2/Lref ! Hydraulic stiffness column 2
1750 S=S1+S2 !Effective hydraulic stiffness at ref position
1760 Omega=SQR(S/M) ! Natural hydraulic frequency
1770 PRINT "AT REF POSITION, HYDRAULIC FREQUENCY OF FLUID COLU
MN:";Omega;"Rad/sec"
1780 PRINT SPA(49);"=";Omega/(2*PI);"Hz"
1790 Tr=1/Omega ! Ref time =time for 1 radian
1800 Qr=A1*Lref/Tr ! Ref flow rate
1810 IF Zeta=0 THEN 1890

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1820 Damping=Zeta
1830 Bd=2*M*Zeta*Omega      ! External Damping in physical units
1840 PRINT SPA(11);"EFFECTIVE EXTERNAL DAMPING COEFFICIENT:";
      Bd;"N/(m/sec)"
1850 PRINT SPA(44);"Zeta =";Damping
1860 Tauclose=Time/Tr      ! Non-dimensional valve operation time
1870 PRINT LIN(2),SPA(8);" NON-DIMENSIONAL TIME FOR CLOSING
      VALVES: ";Tauclose
1880 GOTO 1920
1890 Zeta=Bd/(2*M*Omega)
1900 Damping=Zeta
1910 GOTO 1840
1920 Bdtm=Bd*Tr/M
1930 Pe=Pe/Ps      ! Non-dimensionalize exhaust pressure
1940 K1=Cd1*B1*Xs*SQR(2*Ps/Rho)/Qr
1950 Zeta3=Cd1*B1*Xs*(1+Aratio)/2*SQR(2*Ps/Rho/(1+Aratio*Arat
      io*Aratio))/Qr
1960 Alpha=Beta/Ps
1970 Rtorder=3
1980 IF Onoff THEN Rtorder=2
1990 Rtorder1=Rtorder+1
2000 REDIM Coefficient(Rtorder1),Solution(Rtorder,2),Coeffdum
      (Rtorder1)
2010 IF Onoff THEN Funonoff
2020 Coefficient(1)=2*Alpha*Zeta*Zeta3/Tauclose
2030 Coefficient(2)=1+2*Alpha*Zeta*Zeta3+2*Alpha*Zeta3/Tauclose
2040 Coefficient(3)=2*Zeta+Alpha*Zeta3
2050 Coefficient(4)=1
2060 GOTO Cont1
2070 Funonoff: ! EVALUATE ROOTS(ZEROES) FOR CHARACTERISTIC EQU
      ATION FOR ON-OFF
2080 Coefficient(1)=1+2*Zeta*Alpha*Zeta3
2090 Coefficient(2)=2*Zeta+Alpha*Zeta3
2100 Coefficient(3)=1
2110 Cont1: ! Re-enter at this point
2120 MAT Coeffdum=Coefficient
2130 FOR Idummy=1 TO Rtorder+1
2140 PRINT "Coefficient for y^";Idummy-1;" term = ";Coefficient
      (Idummy)
2150 NEXT Idummy
2160 CALL Roots(Coefficient(*),Rtorder,#5)
2170 PRINTER IS 16
2180 READ #5,1
2190 FOR Idummy=1 TO Rtorder
2200 READ #5;Solution(Idummy,1),Solution(Idummy,2)
2210 NEXT Idummy
2220 READ #5,1
2230 MAT Coefficient=Coeffdum
2240 IF Onoff THEN Function1
2250 REM To evaluate the coefficients for the switching function
2260 REM applied to linearly-operated valves
2270 RAD      ! Declare angular units in radians
2280 Operand1(1)=Solution(1,1)-Solution(2,1)
2290 Operand1(2)=Solution(1,2)-Solution(2,2)
2300 Operand2(1)=Solution(2,1)-Solution(3,1)
2310 Operand2(2)=Solution(2,2)-Solution(3,2)
2320 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
2330 Operand1(1)=Solution(3,1)-Solution(1,1)
2340 Operand1(2)=Solution(3,2)-Solution(1,2)
2350 CALL Complexfn(Operand1(*),Answer(*),Result(*),Multiply)
2360 MAT Tempres1=ZER
2370 Idummy=1
2380 Lb1: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
2390      Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
2400 Operand1(1)=Solution(Idummy,1)
2410 Operand1(2)=Solution(Idummy,2)

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2420 CALL Fun1(Operand1(*),Answer(*),Zeta,Tauclose)
2430 Operand1(1)=Solution(Idummy2,1)-Solution(Idummy1,1)
2440 Operand1(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
2450 CALL Complexfn(Operand1(*),Answer(*),Operand2(*),Multiply)
2460 Tempres1(1)=Tempres1(1)+Operand2(1)
2470 Tempres1(2)=Tempres1(2)+Operand2(2)
2480 Idummy=Idummy+1
2490 IF Idummy<=3 THEN Lb1
2500 CALL Complexfn(Tempres1(*),Result(*),Answer(*),Divide)
2510 Coeff3=Answer(1)
2520 MAT Tempres1=ZER
2530 Idummy=1
2540 Lb2: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
2550     Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
2560 Operand1(1)=Solution(Idummy,1)
2570 Operand1(2)=Solution(Idummy,2)
2580 CALL Fun1(Operand1(*),Answer(*),Zeta,Tauclose)
2590 Operand1(1)=Operand1(1)+2*Zeta+Zeta3*Alpha
2600 CALL Complexfn(Operand1(*),Answer(*),Operand2(*),Multiply)
2610 Operand1(1)=Solution(Idummy2,1)-Solution(Idummy1,1)
2620 Operand1(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
2630 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
2640 Tempres1(1)=Tempres1(1)+Answer(1)
2650 Tempres1(2)=Tempres1(2)+Answer(2)
2660 Idummy=Idummy+1
2670 IF Idummy<=3 THEN Lb2
2680 CALL Complexfn(Tempres1(*),Result(*),Answer(*),Divide)
2690 Coeff2=Answer(1)
2700 MAT Tempres1=ZER
2710 MAT Tempres2=ZER
2720 Idummy=1
2730 Lb3: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
2740     Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
2750 Operand1(1)=Solution(Idummy,1)
2760 Operand1(2)=Solution(Idummy,2)
2770 Operand2(1)=Solution(Idummy2,1)-Solution(Idummy1,1)
2780 Operand2(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
2790 CALL Complexfn(Operand2(*),Operand1(*),Answer(*),Divide)
2800 Tempres1(1)=Tempres1(1)+Answer(1)
2810 Tempres1(2)=Tempres1(2)+Answer(2)
2820 CALL Fun1(Operand1(*),Answer(*),Zeta,Tauclose)
2830 CALL Complexfn(Operand2(*),Answer(*),Operand1(*),Multiply)
2840 Sqdummy=2*Zeta/(Solution(Idummy,1)*Solution(Idummy,1)+So
lution(Idummy,2)*Solution(Idummy,2))
2850 Operand2(1)=1+Sqdummy*Solution(Idummy,1)
2860 Operand2(2)=-1*Sqdummy*Solution(Idummy,2)
2870 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
2880 Tempres2(1)=Tempres2(1)+Answer(1)
2890 Tempres2(2)=Tempres2(2)+Answer(2)
2900 Idummy=Idummy+1
2910 IF Idummy<=3 THEN Lb3
2920 Answer(1)=1-Alpha*Zeta3/Tauclose*(Tempres2(1)-Tempres1(1))
2930 Answer(2)=-1*(Alpha*Zeta3/Tauclose)*(Tempres2(2)-Tempres
1(2))
2940 Coeff1=Answer(1)
2950 MAT Tempres1=ZER
2960 MAT Tempres2=ZER
2970 Idummy=1
2980 Lb4: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
2990     Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
3000 Operand1(1)=Solution(Idummy,1)
3010 Operand1(2)=Solution(Idummy,2)
3020 CALL Fun2(Operand1(*),Operand2(*),Tauclose)
3030 CALL Fun1(Operand1(*),Answer(*),Zeta,Tauclose)
3040 CALL Complexfn(Operand2(*),Answer(*),Operand1(*),Multiply)
3050 Operand2(1)=Solution(Idummy2,1)-Solution(Idummy1,1)

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3060 Operand2(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
3070 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
3080 Tempres1(1)=Tempres1(1)+Answer(1)
3090 Tempres1(2)=Tempres1(2)+Answer(2)
3100 Idummy=Idummy+1
3110 IF Idummy<=3 THEN Lb4
3120 MAT Tempres2=ZER
3130 Idummy=1
3140 Lb5: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
3150 Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
3160 Operand2(1)=Solution(Idummy,1)
3170 Operand2(2)=Solution(Idummy,2)
3180 Operand1(1)=Solution(Idummy2,1)-Solution(Idummy1,1)
3190 Operand1(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
3200 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Divide)
3210 Tempres2(1)=Tempres2(1)+Answer(1)
3220 Tempres2(2)=Tempres2(2)+Answer(2)
3230 Idummy=Idummy+1
3240 IF Idummy<=3 THEN Lb5
3250 Tempres1(1)=Tempres1(1)+2*Zeta/Tauclose*Tempres2(1)
3260 Tempres1(2)=Tempres1(2)+2*Zeta/Tauclose*Tempres2(2)
3270 MAT Tempres2=ZER
3280 Idummy=1
3290 Lb6: Idummy1=(Idummy+1) MOD 4+((Idummy+1) DIV 4>0)
3300 Idummy2=(Idummy+2) MOD 4+((Idummy+2) DIV 4>0)
3310 Operand1(1)=Solution(Idummy2,1)-Solution(Idummy1,1)
3320 Operand1(2)=Solution(Idummy2,2)-Solution(Idummy1,2)
3330 Operand2(1)=Solution(Idummy,1)^2-Solution(Idummy,2)^2
3340 Operand2(2)=2*Solution(Idummy,1)*Solution(Idummy,2)
3350 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Divide)
3360 Tempres2(1)=Tempres2(1)+Answer(1)
3370 Tempres2(2)=Tempres2(2)+Answer(2)
3380 Idummy=Idummy+1
3390 IF Idummy<=3 THEN Lb6
3400 Tempres1(1)=Tempres1(1)+Tempres2(1)/Tauclose
3410 Tempres1(2)=Tempres1(2)+Tempres2(2)/Tauclose
3420 CALL Complexfn(Tempres1(*),Result(*),Answer(*),Divide)
3430 Answer(1)=Answer(1)*Zeta3*(1-Fo/Ps/R1/2)
3440 Answer(2)=Answer(2)*Zeta3
3450 Coeff0=Answer(1)
3460 GOTO Funend
3470 Pause: DISP "PAUSE At Pause"
3480 PAUSE
3490 Function1: REM To evaluate the coefficients for switching
function for ON-OFF valves with/without time delay
3500 RAD
3510 Coeff0=0
3520 Coeff1=1
3530 Coeff2=2*Zeta
3540 Coeff3=1
3550 GOTO Funend
3560 IF (Tauclose=0) AND (Fo=0) THEN Funend
3570 Idummy=1
3580 La1: ! *****
3590 Operand1(1)=1
3600 Operand1(2)=0
3610 Operand2(1)=Solution(Idummy,1)
3620 Operand2(2)=Solution(Idummy,2)
3630 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Divide)
3640 Answer(1)=Solution(Idummy,1)+2*Zeta+Answer(1)
3650 Answer(2)=Solution(Idummy,2)+Answer(2)
3660 IF Idummy>=2 THEN La2
3670 Tempres1(1)=Answer(1)
3680 Tempres1(2)=Answer(2)
3690 Idummy=Idummy+1
3700 GOTO La1

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3710 La2:          ! *****
3720 Tempres2(1)=Answer(1)
3730 Tempres2(2)=Answer(2)
3740 Operand1(1)=EXP(Tauclose*Solution(1,1))*COS(Tauclose*Sol
    ution(1,2))
3750 Operand1(2)=EXP(Tauclose*Solution(1,1))*SIN(Tauclose*Sol
    ution(1,2))
3760 Operand2(1)=Solution(2,1)
3770 Operand2(2)=Solution(2,2)
3780 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
3790 CALL Complexfn(Answer(*),Tempres1(*),Result(*),Multiply)
3800 Operand1(1)=EXP(Tauclose*Solution(2,1))*COS(Tauclose*Sol
    ution(2,2))
3810 Operand1(2)=EXP(Tauclose*Solution(2,1))*SIN(Tauclose*Sol
    ution(2,2))
3820 Operand2(1)=Solution(1,1)
3830 Operand2(2)=Solution(1,2)
3840 CALL Complexfn(Operand1(*),Operand2(*),Answer(*),Multiply)
3850 CALL Complexfn(Answer(*),Tempres2(*),Operand1(*),Multiply)
3860 CALL Complexfn(Operand1(*),Result(*),Answer(*),Subtract)
3870 Operand1(1)=Solution(1,1)
3880 Operand1(2)=Solution(1,2)
3890 Operand2(1)=Solution(2,1)
3900 Operand2(2)=Solution(2,2)
3910 CALL Complexfn(Operand1(*),Operand2(*),Result(*),Subtract)
3920 CALL Complexfn(Answer(*),Result(*),Operand1(*),Divide)
3930 Coeff2=Operand1(1)+Coefficient(2)/Coefficient(1)
3940 Imaginary=Operand1(2)
3950 PRINT "Coefficient for velocity term [Real] = ";Coeff2
3960 PRINT SPA(25),"[Imaginary] = ";Imaginary
3970 Operand2(1)=EXP(Tauclose*Solution(1,1))*COS(Tauclose*Sol
    ution(1,2))
3980 Operand2(2)=EXP(Tauclose*Solution(1,1))*SIN(Tauclose*Sol
    ution(1,2))
3990 CALL Complexfn(Operand2(*),Tempres1(*),Answer(*),Multiply)
4000 Operand2(1)=EXP(Tauclose*Solution(2,1))*COS(Tauclose*Sol
    ution(2,2))
4010 Operand2(2)=EXP(Tauclose*Solution(2,1))*SIN(Tauclose*Sol
    ution(2,2))
4020 CALL Complexfn(Operand2(*),Tempres2(*),Operand1(*),Multiply)
4030 CALL Complexfn(Answer(*),Operand1(*),Operand2(*),Subtract)
4040 CALL Complexfn(Operand2(*),Result(*),Operand1(*),Divide)
4050 Coeff3=Operand1(1)+1/Coefficient(1)
4060 Imaginary=Operand1(2)
4070 PRINT "Coefficient for acceleration term [Real]: ";Coeff3
4080 PRINT SPA(29),"[Imaginary]: ";Imaginary
4090 Operand2(1)=EXP(Tauclose*Solution(1,1))*COS(Tauclose*Sol
    ution(1,2))
4100 Operand2(2)=EXP(Tauclose*Solution(1,1))*SIN(Tauclose*Sol
    ution(1,2))
4110 CALL Complexfn(Operand2(*),Tempres1(*),Operand1(*),Multiply)
4120 Operand2(1)=Solution(1,1)
4130 Operand2(2)=Solution(1,2)
4140 CALL Complexfn(Operand1(*),Operand2(*),Tempres1(*),Divide)
4150 Operand2(1)=EXP(Tauclose*Solution(2,1))*COS(Tauclose*Sol
    ution(2,2))
4160 Operand2(2)=EXP(Tauclose*Solution(2,1))*SIN(Tauclose*Sol
    ution(2,2))
4170 CALL Complexfn(Operand2(*),Tempres2(*),Operand1(*),Multiply)
4180 Operand2(1)=Solution(2,1)
4190 Operand2(2)=Solution(2,2)
4200 CALL Complexfn(Operand1(*),Operand2(*),Tempres2(*),Divide)
4210 CALL Complexfn(Tempres2(*),Tempres1(*),Answer(*),Subtract)
4220 CALL Complexfn(Answer(*),Result(*),Tempres1(*),Divide)
4230 Tempres2(1)=Zeta3*Tempres1(1)
4240 Tempres2(2)=Zeta3*Tempres1(2)

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4250 Coeff0=(1-Fo/(2*Ps*A1))*Zeta3/Coefficient(1)*(2*Zeta+Tau
      close-Coefficient(2)/Coefficient(1))+Tempres2(1)
4260 Imaginary=Tempres2(2)
4270 PRINT "Coefficient for constant term [Real]: ";Coeff0
4280 PRINT SPA(25),"[Imaginary]: ";Imaginary
4290 Funend:      REM Coefficients have been evaluated
4300 PRINTER IS 0 ! Print these onto paper
4310 PRINT "Coeff. for Accel term = ";Coeff3
4320 PRINT "Coeff. for Vel term = ";Coeff2
4330 PRINT "Coeff. for Disp term = ";Coeff1
4340 PRINT "Coeff. for Const term = ";Coeff0
4350 PRINTER IS Printer
4360 K2=K1
4370 IF Sym THEN 4390
4380 K2=K1*(B2/B1)*(Cd2/Cd1)
4390 K3=Tr*Tr*Ps*A1/(M*Lref) ! Evaluate non-dimensional quantities
4400 Bd=Bd*Lref/(Ps*A1*Tr)
4410 Ko=Ko*Lref/(Ps*A1)
4420 Fo=Fo/(Ps*A1)
4430 Velr=Lref/Tr
4440 Accelr=Velr/Tr
4450 K4=C1*Ps/QR
4460 K5=Beta/Ps
4470 Thic=K5*K3*2/Var
4480 Epsilopi=0
4490 IF Onoff THEN 4510
4500 Epsilopi=Tr/Time ! Rate of change of valve opening
4510 G=K1*Beta*A1*Tr*Tr/(M*Vr*Va*Lref*2)
4520 IF Onoff THEN G=2*G
4530 ! ***** INPUT INITIAL CONDITIONS ***** !
4540 IF Again THEN 4640
4550 BEEP
4560 INPUT "INITIAL DISPLACEMENT OF PISTON FROM CENTRE IN m",Y
4570 Y=Y-Refposn
4580 IF ABS(Y)<=Ys/2 THEN 4610
4590 BEEP
4600 GOTO 4550
4610 INPUT "INCREMENT FOR INITIAL DISPLACEMENT IN m",Deltai
4620 Deltai=Deltai/Lref
4630 GOTO 4650
4640 Y=(Deltai+Thetai)*Lref
4650 PRINT "INITIAL CONDITIONS:"
4660 PRINT " PISTON DISPLACEMENT FROM CENTRE :";Y+Refposn;"m."
4670 Theta=Y/Lref !Non-dimensional Displacement
4680 Thetai=Theta !Initial Displacement
4690 IF NOT Again THEN 4720
4700 Vel=Veli
4710 GOTO 4740
4720 BEEP
4730 INPUT "INITIAL RAM VELOCITY IN m/s",Vel
4740 Veli=Vel
4750 PRINT SPA(22);"RAM VELOCITY:";Vel;"m/sec"
4760 Thetapi=Vel/Velr !Non-dimensionalize velocity
4770 IF Veli<>0 THEN 4820
4780 Aratio3=Aratio*Aratio*Aratio
4790 P1=Ps*(Aratio3+Fo)/(1+Aratio3)*1E-5
4800 P2=Aratio*Aratio*(Ps-P1*1E5)*1E-5
4810 GOTO 4890
4820 IF NOT Again THEN 4860
4830 P1=P1i
4840 P2=P2i
4850 GOTO 4890
4860 BEEP
4870 INPUT "INITIAL PRESSURE IN CHAMBER 1 IN Bar",P1
4880 P1i=P1

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4890 PRINT SPA(13);"PRESSURE IN CHAMBER 1:";P1;"Bar"
4900 P1=P1*1E5/Ps !Non-dimensionalize P1
4910 IF Again OR (Vel=0) THEN 4950
4920 BEEP
4930 INPUT "INITIAL PRESSURE IN CHAMBER 2 IN Bar",P2
4940 P2i=P2
4950 PRINT SPA(13);"PRESSURE IN CHAMBER 2:";P2;"Bar"
4960 P2=P2*1E5/Ps
4970 PRINT LIN(1),"REFERENCE QUANTITIES:"
4980 PRINT " TIME:";Tr*1000;"mSec."
4990 PRINT " PRESSURE:";Ps/1E5;"Bar"
5000 PRINT " FLOW RATE:";Qr*1000;"Litre/sec"
5010 PRINT " LENGTH:";Lref;"m."
5020 PRINT " POSITION:";Refposn;"m. from Mid-stroke position"
5030 PRINT " AREA:";A1;"sq.m."
5040 PRINT " VOLUME:";A1*Lref;"cu.m."
5050 PRINT " VELOCITY:";Velr;"m/s"
5060 PRINT "ACCELERATION:";Accelr;"m/s^2"
5070 IF NOT Again THEN 5180
5080 Thetaf=Thetaf+Deltaf
5090 Yf=Lref*Thetaf
5100 IF ABS(Thetaf*Lref+Refposn)<=.5*Ys THEN 5270
5110 DISP "Thetaf is now greater than .5 on either side"
5120 PRINTER IS 16
5130 PRINT PAGE,LIN(5),SPA(20),"Please remove the disc from
the drive"
5140 PRINT LIN(1),SPA(23),"and place it in the pocket marke
d:",LIN(2)
5150 PRINT SPA(40),"Flexible Disc"
5160 BEEP
5170 STOP
5180 ! ***** INPUT FINAL DESIRED POSITION ***** !
5190 BEEP
5200 INPUT "FINAL DESIRED POSITION FROM CENTRE IN m.",Yf
5210 IF ABS(Yf)<=Ys/2 THEN 5230 ! Check if within the stroke
5220 GOTO 5180
5230 Yf=Yf-Refposn ! Position relative to the Ref position
5240 INPUT "INCREMENT OF FINAL DESIRED POSITION IN m",Deltas
5250 Deltas=Deltas/Lref ! Non-dimensionalize this quantity
5260 Deltaf=Deltas
5270 PRINT LIN(1),"FINAL DESIRED POSITION FROM REF POSN.:";Yf
;"m.",LIN(1)
5280 Thetas=Yf/Lref
5290 Thetaf=Thetas
5300 Thi2=Bdtm*Bdtm*Thetaf*(4*Thic-1)/4
5310 Thetaf2=Thetaf
5320 Thetar=Thetas-Theta
5330 REM Alpha1=Alpha2*Thetaf-1
5340 Alpha2=2*Ps*A1*Tr*Tr/((Ratio*Va-Thetaf)*M*Ys)
5350 Qrootsq=(Alpha2-Alpha1*Alpha1)/4
5360 Alpha3=Thetaf*(Qrootsq-Alpha2)
5370 Wtval=1
5380 ! ***** CALCULATE TIME INCREMENT ***** !
5390 Deltat=.02*PI
5400 PRINT "THE TIME INCREMENT IS CHOSEN TO BE:";Deltat*1E6;"
microSec.",LIN(6)
5410 IMAGE " ERROR S",2X,5(10A,4X)
5420 PRINTER IS 16
5430 ! Print messages onto the CRT
5440 PRINT PAGE,SPA(25),"Please DO NOT Interrupt",LIN(2)
5450 PRINT SPA(25),"Please USE other machine if available",
LIN(2)
5460 PRINT SPA(10),"PERMISSION HAS BEEN GIVEN TO LEAVE COMPUTER
RUNNING OVERNIGHT",LIN(5)
5470 PRINT USING 5410;"THETA","THETA-PI","THETA-2-PI","Signal
F","EPSILO"

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5480 ! End of print message segment
5490 ! Print information on paper
5500 PRINTER IS 0
5510 PRINT LIN(2),"Mass of moving parts = ";M;" Kg","Damping
    Coeff = ";Damping
5520 PRINT "Final desired position: ";Thetas;" ( ";Thetas#Lre
    f+Refposn;" m. from centre)"
5530 PRINT "External Force Applied: ";Fo;" ( ";Fo#Ps#A1/1000;"
    kN)"
5540 PRINT USING 5550;Tauclose,Tauclose#Tr*1000
5550 IMAGE #,"Tau close : ",MD.3DE,2X,"( ",3D.D,2X,"mSec."
5560 IF Onoff THEN PRINT "ON-OFF Values)",LIN(2)
5570 IF NOT Onoff THEN PRINT "Linearly Operated Values)",LIN(2)
5580 PRINT USING 5590;"Tau","Theta","Thetapi","Theta2pi","Fun
    ction"
5590 IMAGE 3X,6(10A,4X)
5600 PRINTER IS 16
5610 PLOTTER IS 13,"GRAPHICS"
5620 Tausc=80
5630 IF Tlimit>1E99 THEN 5650
5640 Tausc=1.1*Tlimit
5650 Dummymin=MIN(Theta,Thetaf)
5660 Dummymax=MAX(Theta,Thetaf)
5670 Dummyrange=Dummymax-Dummymin
5680 Dummymin=Dummymin-.1*Dummyrange
5690 Dummymax=Dummymax+.1*Dummyrange
5700 Tausc=Tausc*MAX(INT(ABS(Dummyrange/.05)),1)
5710 IF Dummyrange<>0 THEN 5740
5720 SCALE -Tausc/10,Tausc,Thetaf-.01,.1+Thetaf
5730 GOTO 5750
5740 SCALE -Tausc/10,Tausc,Dummymin,Dummymax
5750 AXES
5760 MOVE 0,Thetaf
5770 LINE TYPE 7
5780 DRAW Tausc,Thetas
5790 LINE TYPE 1
5800 MOVE Tausc/2,Thetaf+(Thetaf-Theta)/4
5810 LORG 6
5820 LABEL "FINAL DESIRED POSITION: "&VAL$(Thetas)
5830 LORG 5
5840 MOVE 0,0
5850 ON KEY #3 GOSUB Graph
5860 ON KEY #4 GOTO Endcal
5870 ON ERROR GOTO Error5
5880 ON KEY #5 GOSUB Chsignal
5890 ON KEY #2 GOSUB Changeprinter
5900 GOSUB Deriv
5910 GOSUB Signchk
5920 IF Signal<>Presign THEN Switch_value
5930 GOSUB Intrgl
5940 IF (Tau)=Tlimit) OR (Npoints)=400) THEN Endcal
5950 IF Sw<>2 THEN 5900
5960 IF Pscor THEN 5900
5970 Sw=Sw+2
5980 IF Scorer THEN Sw=10
5990 Sw=Sw+2
6000 GOTO 5900
6010 Switch_value: REM SWITCHING TAKES PLACE NOW
6020 PRINTER IS 0 ! Print information at the point of switching
6030 PRINT USING 6040;Tau,Theta,Thetapi,Theta2pi,Signalfn
6040 IMAGE 2X,5(MD.5DE,2X),7X
6050 PRINTER IS 16
6060 IF Thetaf<>Thetaf1 THEN Sw=0
6070 IF Sw THEN 6110
6080 Scorer=0
6090 IF Signal=SGN(Thetae) THEN Scorer=1

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6100  Song=Signal
6110  Sw=Sw+1      ! COUNT THE NUMBER OF SWITCHINGS
6120  IF Sw=1 THEN 6150
6130  Pscor=0
6140  IF Presign1=Signal THEN Pscor=1
6150  IF Sw<>5 THEN 6170
6160  IF Signal=Song THEN Sw=Sw+2
6170  IF Onoff THEN Switch=1
6180  IF Signal<0 THEN 6210
6190  LABEL "*"
6200  GOTO 6220
6210  LABEL "+"
6220  Psws=Psws+1
6230  Pswitch(Psws)=Npoints
6240  MOVE Tau,Theta
6250  IF Time=0 THEN 6430
6260  Deltatd=Deltat
6270  Deltatr=Time/Tr/100
6280  Deltat=MIN(Deltatr,Deltatd)
6290  Tau=Tau
6300  Tauf=Tau+Time/Tr-Deltat
6310  IF Onoff THEN Pureonoff
6320  GOSUB Intrgl
6330  GOSUB Deriv
6340  IF Tau<Tauf THEN 6320
6350  LABEL "#"
6360  IF ABS(Epsilon)<.5 THEN Signal=0
6370  GOTO 5930
6380  Pureonoff:  Tau=0
6390  GOSUB Intrgl
6400  GOSUB Deriv
6410  IF Tau<Tauf THEN 6380
6420  LABEL "#"
6430  Switch=0
6440  GOSUB Intrgl
6450  IF ABS(Epsilon)<.5 THEN Signal=0
6460  GOTO 5940
6470  Endcal:  PRINTER IS 0
6480  PRINT USING 6040;Tau,Theta,Thetapi,Theta2pi,Signalf1
6490  PRINT LIN(2),"PRESSURE IN CHAMBER 1 : ",P1*Ps/1E5;" Bar"
6500  PRINT "PRESSURE IN CHAMBER 2 : ",P2*Ps/1E5;" Bar"
6510  PRINT "RESULTS HAVE BEEN WRITTEN ON FILE NUMBERED: ";No,
      LIN(5)
6520  PRINTER IS Printer
6530  ASSIGN #6 TO "SMDA"&VAL$(No)&Msus$,Check
6540  IF Check=0 THEN 6570
6550  CREATE "SMDA"&VAL$(No)&Msus$,20
6560  GOTO 6520
6570  PRINT #6;Fo,Damping,Wtval,Sety,M,Tauclose,Tr,Type,Sym,Ar
      atio,Npoints,Psws
6580  FOR Dummy=1 TO Psws
6590  PRINT #6;Pswitch(Dummy)
6600  NEXT Dummy
6610  FOR Dummy=1 TO Npoints-1
6620  Shortdummy1=Tauc(Dummy)
6630  Shortdummy2=Z(Dummy)
6640  Shortdummy3=Ep(Dummy)
6650  PRINT #6;Shortdummy1,Shortdummy2,Shortdummy3
6660  NEXT Dummy
6670  PRINT #6;END
6680  ASSIGN * TO #6
6690  No=No+1
6700  ASSIGN #6 TO "SMDA"&VAL$(No)&Msus$,Check
6710  IF Check=0 THEN 6740
6720  CREATE "SMDA"&VAL$(No)&Msus$,20
6730  GOTO 6700

```

```

6740 PRINT #6;Fo,Damping,Wtval,Sety,M,Tauclose,Tr,Type,Sym,Ar
      atio,Npoints,Psws
6750 FOR Dummy=1 TO Psws
6760 PRINT #6;Pswitch(Dummy)
6770 NEXT Dummy
6780 FOR Dummy=1 TO Npoints-1
6790 Shortdummy1=Tauc(Dummy)
6800 Shortdummy2=Ramp1(Dummy)
6810 Shortdummy3=Ramp2(Dummy)
6820 PRINT #6;Shortdummy1,Shortdummy2,Shortdummy3
6830 NEXT Dummy
6840 PRINT #6;END
6850 ASSIGN * TO #6
6860 EXIT .GRAPHICS
6870 IF Again THEN 6910
6880 IF ERRN THEN PRINT ERRM$
6890 INPUT "INPUT 1 to run again, OTHERWISE INPUT 0 to stop
      ",Again
6900 IF NOT Again THEN STOP
6910 No=No+1
6920 REM IF INT(No/2)*2-No=0 THEN Tlimit=Tlimit+Tdummyc
6930 Tlimit=Tlimit+Tdummyc
6940 GOTO 420
6950 STOP
6960 Graph: REM ENTERING GRAPHIC MODE
6970 GRAPHICS
6980 RETURN
6990 Changeprinter: REM CHANGE PRINTER DEVICE
7000 Printer=16-Printer
7010 PRINTER IS Printer
7020 RETURN
7030 REM PROGRAMME STORED IN "SW2809" ON 1ST October 1981 ON
      DISC #51
7040 Signchk: REM CHECK SIGNAL
7050 Presign=Signal
7060 IF Sw<>0 THEN 7220
7070 IF ABS(Thetaf-Theta)<1E-6 THEN 7100
7080 IF (Thetapi<>0) OR (Theta2pi<>0) THEN 7220
7090 IF ABS(Thetaf-Theta)>1E-6 THEN 7120
7100 Signal=Presign
7110 RETURN
7120 Signalfn=Coeff3*Theta2pi+Coeff2*Thetapi+Coeff1*(Theta-Th
      etaf)+SGN(Thetapi)*Coeff0
7130 IF Epsilon<0 THEN 7160
7140 Signalfn=Signalfn+(1+Onoff)*Time/Tr/2*Thetapi
7150 GOTO 7170
7160 Signalfn=Coeff3*Theta2pi+Coeff2*Thetapi+Coeff1*Theta-Th
      etaf+Coeff0
7170 Signal=SGN(-Signalfn)
7180 RETURN
7190 Signsw: Signal=-Signal
7200 Sw=100
7210 RETURN
7220 Signalf1=Coeff3*Theta2pi+Coeff2*Thetapi+Coeff1*(Theta-Th
      etaf)+SGN(Thetapi)*Coeff0
7230 IF Epsilon<0 THEN 7260
7240 Signalf1=Signalf1+(1+Onoff)*Time/Tr/2*Thetapi
7250 GOTO 7270
7260 Signalf1=Coeff3*Theta2pi+Coeff2*Thetapi+Coeff1*Theta-Th
      etaf+Coeff0
7270 IF Sw>7 THEN Set0s
7280 Signal=SGN(-Signalf1)
7290 Signalfn=Signalf1
7300 RETURN
7310 Chsignal: Signal=-Signal
7320 Signalfn=Signalf1

```

```

7330 RETURN
7340 Set0s: Epsilo=Presign=Signal=0
7350 RETURN
7360 Deriv: REM CALCULATE DERIVATIVES
7370 Thetae=Thetaf-Theta
7380 Ppi1=1
7390 Ppi2=Pe
7400 IF Epsilo>=0 THEN Skippi
7410 Ppi1=Pe
7420 Ppi2=1
7430 Skippi: REM CALCULATE FLOW RATES
7440 Q1=K1*ABS(Epsilo)*SQR(ABS(Ppi1-P1))*SGN(Ppi1-P1)
7450 Q2=K2*ABS(Epsilo)*SQR(ABS(P2-Ppi2))*SGN(P2-Ppi2)
7460 P1pi=K5*(Q1-Thetapi-K4*(P1-P2))/(1+Theta)
7470 P2pi=K5*(K4*(P1-P2)+Aratio*Thetapi-Q2)/((1-Theta)*Aratio)
7480 Theta2pi=K3*(P1-Aratio*P2-Bd*Thetapi-Ko*Theta-Fo)
7490 IF Print THEN Noprint
7500 PRINT USING Prtim;Thetae,Signal,Theta,Thetapi,Theta2pi,Signalf1,Epsilo
7510 IF Npoints>1 THEN 7530
7520 IF ABS(Theta-Thetas)>.1 THEN 7590
7530 Npoints=Npoints+1
7540 Tauc(Npoints)=Tau
7550 Z(Npoints)=Theta
7560 Ep(Npoints)=Thetaf
7570 Ramp1(Npoints)=P1
7580 Ramp2(Npoints)=P2
7590 DISP Tau;Npoints;Thetaf,P1*Ps/1E5;P2*Ps/1E5
7600 Noprint: Print=Print+1
7610 PLOT Tau,Theta
7620 IF Print>=5 THEN Print=0
7630 Prtim: IMAGE +,MD.DE ,X,MD,X,4(MD.5DE,2X),MD.5DE
7640 RETURN
7650 Intrgl: REM CALCULATE INTEGRALS
7660 Presign1=-SGN(Signalf1)
7670 Signalf1=Coeff3*Theta2pi+Coeff2*Thetapi+Coeff1*(Theta-Thetaf)+Coeff0
7680 IF Epsilo<0 THEN 7710
7690 Signalf1=Signalf1+(1+Onoff)*Time/Tr/2*Thetapi
7700 GOTO 7720
7710 Signalf1=Signalf1+(1+Onoff)*Time/Tr/2*Thetapi*Aratio
7720 Thetaf1=Thetaf
7730 Thetaf2=Thetaf2+Deltaf
7740 IF ABS(Thetaf2)>=ABS(Thetas) THEN Thetaf2=Thetas
7750 Tau=Tau+Deltat
7760 Thetaf=Thetaf1
7770 Theta=Theta+Deltat*Thetapi
7780 Thetapi=Thetapi+Deltat*Theta2pi
7790 Epsilo=Epsilo+Epsilopi*Deltat*Signal+Signal*Onoff*(Switch=0)
7800 IF ABS(Epsilo)>1 THEN Epsilo=SGN(Epsilo)
7810 P1=P1+Deltat*P1pi
7820 P2=P2+Deltat*P2pi
7830 IF P1<=0 THEN P1=0
7840 IF P2<=0 THEN P2=0
7850 RETURN
7860 Error5: REM ERROR DURING EXECUTION
7870 DISP ERRM$
7880 BEEP
7890 OFF ERROR
7900 GOTO Endcal
7910 END
7920 SUB Roots(K(*),INTEGER N,#5)
7930 REM *****
7940 REM * PROGRAM STORED IN "ROOTS" *
7950 REM *****
7960 OPTION BASE 1

```

```

7970 REM *****
7980 REM * FINDS THE ROOTS OF POLYNOMIAL *
7990 REM * VERSION 3/DECEMBER 1981 *
8000 REM * N --- ORDER OF POLYNOMIAL (LESS THAN 3) *
8010 REM * K --- COEFFICIENTS OF POLYNOMIAL *
8020 REM * ROOTS ARE STORED IN THE FILE ASSIGNED TO #5 *
8030 REM * THIS FILE IS CALLED "SOLN" *
8040 REM *****
8050 H1=N !Conserve N
8060 IF N<=0 THEN 9370
8070 DIM A(4),B(4),X(4)
8080 FOR I=1 TO N+1
8090 A(I)=K(N+2-I)
8100 B(I)=A(I)
8110 NEXT I
8120 PRINT
8130 C1=0
8140 D1=4
8150 O=0
8160 PRINT
8170 PRINT
8180 PRINT
8190 PRINT " The roots for the characteristic equation are:"
8200 PRINT
8210 PRINT TAB(14);"Real Part";TAB(32);"Imag Part"
8220 PRINT
8230 IF N<=2 THEN 8940
8240 IF A(N+1)=0 THEN 9030
8250 IF N/2-INT(N/2)=0 THEN 8280
8260 GOSUB 9120
8270 GOTO 8230
8280 IF ABS(A(N-1))<1E-60 THEN 8320
8290 P=A(N)/A(N-1)
8300 Q=A(N+1)/A(N-1)
8310 GOTO 8340
8320 P=A(N)
8330 Q=A(N+1)
8340 FOR I=1 TO N+1
8350 X(I)=A(I)
8360 NEXT I
8370 GOSUB 9070
8380 FOR I=1 TO N-1
8390 B(I)=X(I)
8400 NEXT I
8410 R=X(N)
8420 S=A(N+1)-P*X(N)-Q*X(N-1)
8430 GOSUB 9070
8440 X(N)=P*X(N-1)-Q*X(N-2)
8450 D=X(N-1)^2-X(N)*X(N-2)
8460 IF ABS(D)>1E-60 THEN 8490
8470 PRINT "Solution unobtainable with this Program"
8480 GOTO 9370
8490 P1=P+(R*X(N-1)-S*X(N-2))/D
8500 Q1=Q+(S*X(N-1)-R*X(N))/D
8510 IF ABS(P)>1E-60 THEN 8550
8520 IF ABS(P1)>1E-60 THEN 8550
8530 IF ABS(Q)>1E-60 THEN 8560
8540 GOTO 8570
8550 IF ABS(P1/P-1)>.000001 THEN 8570 !Relative error allowed
      1E-6
8560 IF ABS(Q1/Q-1)<.000001 THEN 8710
8570 P=P1
8580 Q=Q1
8590 C1=C1+1
8600 IF C1=D1*25 THEN 8620
8610 GOTO 8340

```

```

8620 PRINT
8630 PRINT "The solution did not converge after";C1;"iterations"
8640 PRINT "To CONTINUE for 25 more iterations TYPE 1 otherwise"
8650 PRINT "TYPE 0"
8660 INPUT K1
8670 IF K1=1 THEN 8690
8680 GOTO 9370
8690 D1=D1+1
8700 GOTO 8340
8710 FOR I=2 TO N-1
8720 A(I)=B(I)
8730 NEXT I
8740 N=N-2
8750 D=P*P-4*Q
8760 IF D<0 THEN 8860
8770 D=SQR(D)
8780 PRINT USING 8810;(-P+D)/2,0
8790 PRINT USING 8810;(-P-D)/2,0
8800 PRINT #5;(-P+D)/2,0,(-P-D)/2,0
8810 IMAGE 4X,2(6X,MD.5DXE),6X,MD.5D
8820 C1=0
8830 D1=4
8840 IF N-2>0 THEN 8240
8850 GOTO 8940
8860 D=SQR(-D)
8870 Z1=P/2/SQR((P/2)^2+(D/2)^2)
8880 PRINT USING 8810;-P/2,D/2
8890 PRINT USING 8810;-P/2,-D/2
8900 PRINT #5;-P/2,D/2,-P/2,-D/2
8910 C1=0
8920 D1=4
8930 IF N-2>0 THEN 8240
8940 IF N=1 THEN 9000
8950 IF N=0 THEN 9370
8960 P=B(2)/B(1)
8970 Q=B(3)/B(1)
8980 N=0
8990 GOTO 8750
9000 PRINT USING 8810;-B(2)/B(1),0
9010 PRINT #5;-B(2)/B(1)
9020 GOTO 9370
9030 PRINT USING 8810;0,0
9040 PRINT #5;0,0
9050 N=N-1
9060 GOTO 8230
9070 X(2)=X(2)-P*X(1)
9080 FOR I=3 TO N
9090 X(I)=X(I)-P*X(I-1)-Q*X(I-2)
9100 NEXT I
9110 RETURN
9120 IF B(2)=0 THEN 9150
9130 X=-B(2)/B(1)
9140 GOTO 9160
9150 X=-B(N+1)/B(1)
9160 F=0
9170 F1=0
9180 FOR I=1 TO N+1
9190 J=N-I+2
9200 IF B(J)=0 THEN 9240
9210 F=B(J)*X^(I-1)+F
9220 IF I-1=0 THEN 9240
9230 F1=(I-1)*B(J)*X^(I-2)+F1
9240 NEXT I
9250 X1=X-F/F1
9260 IF ABS(X/X1-1)<.000001 THEN 9290
9270 X=X1

```

```

9280 GOTO 9160
9290 PRINT USING 8810;X1,0
9300 PRINT #5;X1,0
9310 N=N-1
9320 FOR I=2 TO N+1
9330 A(I)=B(I)+X1*A(I-1)
9340 B(I)=A(I)
9350 NEXT I
9360 RETURN
9370 N=H1
9380 SUBEND
9390 Lcomp: SUB Complexfn(Operand1(*),Operand2(*),Answer(*),
INTEGER Operator)
9400 REM *****
9410 REM * SUB-PROGRAMS TO OPERATE ON COMPLEX NUMBERS *
9420 REM *****
9430 OPTION BASE 1
9440 DIM A(2),B(2),C(2)
9450 FOR I=1 TO 2
9460 A(I)=Operand1(I)
9470 B(I)=Operand2(I)
9480 NEXT I
9490 ! *****
9500 ! * Operator = 1 for Addition *
9510 ! * Operator = 2 for Subtraction *
9520 ! * Operator = 3 for Multiplication *
9530 ! * Operator = 4 for Division *
9540 ! *****
9550 ON Operator GOTO Add,Subtract,Multiply,Divide
9560 DISP "OPERATOR INCORRECTLY SPECIFIED"
9570 BEEP
9580 STOP
9590 Add: REM Add two complex numbers together
9600 C(1)=A(1)+B(1)
9610 C(2)=A(2)+B(2)
9620 GOTO Complete
9630 Multiply: REM Multiply two complex numbers together
9640 C(1)=A(1)*B(1)-A(2)*B(2)
9650 C(2)=A(1)*B(2)+A(2)*B(1)
9660 GOTO Complete
9670 Subtract: REM Subtract two complex numbers: A - B
9680 C(1)=A(1)-B(1)
9690 C(2)=A(2)-B(2)
9700 GOTO Complete
9710 Divide: REM Divide two complex numbers: A / B
9720 Dummy=B(1)*B(1)+B(2)*B(2)
9730 C(1)=(A(1)*B(1)+A(2)*B(2))/Dummy
9740 C(2)=(A(2)*B(1)-A(1)*B(2))/Dummy
9750 Complete: ! Operation completed
9760 Answer(1)=C(1)
9770 Answer(2)=C(2)
9780 SUBEXIT
9790 SUBEND
9800 Lfun1: SUB Fun1(Operand(*),Answer(*),Zeta,Tauclose)
9810 OPTION BASE 1
9820 ! *****
9830 ! * Programme to evaluate: *
9840 ! * (1+2*Zeta*Operand+Operand^2)*EXP(Operand*Tauclose) *
9850 ! *****
9860 RAD
9870 DIM A(2),B(2)
9880 A(1)=1+2*Zeta*Operand(1)+Operand(1)*Operand(1)-Operand(2)
) *Operand(2)
9890 A(2)=2*Zeta*Operand(2)+2*Operand(1)*Operand(2)
9900 B(1)=EXP(Operand(1)*Tauclose)*COS(Operand(2)*Tauclose)
9910 B(2)=EXP(Operand(1)*Tauclose)*SIN(Operand(2)*Tauclose)

```



```
9920 CALL Complexfn(A(*),B(*),Answer(*),3)
9930 SUBEXIT
9940 SUBEND
9950 Lfun2: SUB Fun2(Operand(*),Answer(*),Tauclose)
9960 ! *****
9970 ! * To evaluate the expression: *
9980 ! * (Operand*Tauclose-1)/(Tauclose*Operand)/Operand *
9990 ! *****
10000 ! 1-1/(Tauclose*Operand)=(Tauclose*Operand-1)/(Tauclose
    *Operand)
10010 OPTION BASE 1
10020 DIM A(2),B(2)
10030 MAT A=(Tauclose)*Operand
10040 B(1)=A(1)-1
10050 B(2)=A(2)
10060 CALL Complexfn(B(*),A(*),Answer(*),4)
10070 MAT A=Operand
10080 MAT B=Answer
10090 CALL Complexfn(B(*),A(*),Answer(*),4)
10100 SUBEXIT
10110 SUBEND
```

APPENDIX D

STIFFNESS ANALYSIS OF A MINING BOOM

## D.0. STIFFNESS ANALYSIS OF A BOOM

One may define the stiffness of a boom of a mining machinery as the ratio of change in force to the change in displacement of the working end in the direction of the applying force. Although this quantity depends on both the structural and hydraulic stiffnesses, only the hydraulic stiffness is investigated because the boom arms are normally supported by the hydraulic rams.

Suppose there is a boom-ripper as shown in Figure D.1.

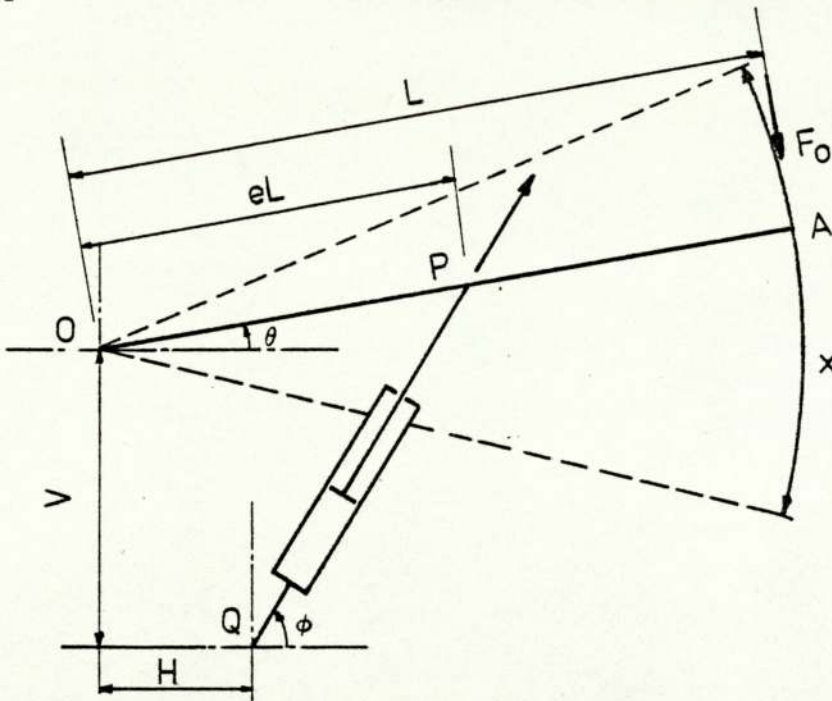


Figure D.1. Diagrammatic Representation  
of a Boom-ripper

The movement of this boom (OA) in the vertical plane is provided by the hydraulic ram (QP). The analysis is simplified by assuming that all movements in other planes are resisted by other means. The worst case of this system will be that when the cutter at boom-end attempts to cut a hard rock, the force resisting will lift up the

boom so that the cutter and boom "climb" over the rock. This force will be a maximum when it is cutting in a perpendicular direction to the boom. By considering the geometry of the system, the hydraulic properties of the oil in the ram, the maximum force,  $F_o$ , and applying the virtual work principle, we have,

$$\text{Stiffness} = \frac{d}{dx} F_o = \frac{e^2 \beta A^2 \sin(\theta - \phi)}{V}$$

where  $\beta$  = Bulk modulus of the fluid in the ram

$V$  = Volume of the fluid column

$A$  = Cross-sectional area of the piston

$e, \theta, \phi$  as defined in Figure D.1

It can be shown that, if the ram's cross-sectional area is adjusted so that the pressure force within the cylinder can just resist the maximum external force when the ram is fully extended, the stiffness will only vary slightly with the location of the ram; while there is a substantial change in stiffness if the area has been kept constant.

#### D.1. STIFFNESS OF A DOSCO BOOM-RIPPER

The DOSCO boom-ripper has a slightly different configuration from that shown in Figure D.1. It can be diagrammatically represented in Figure D.2.

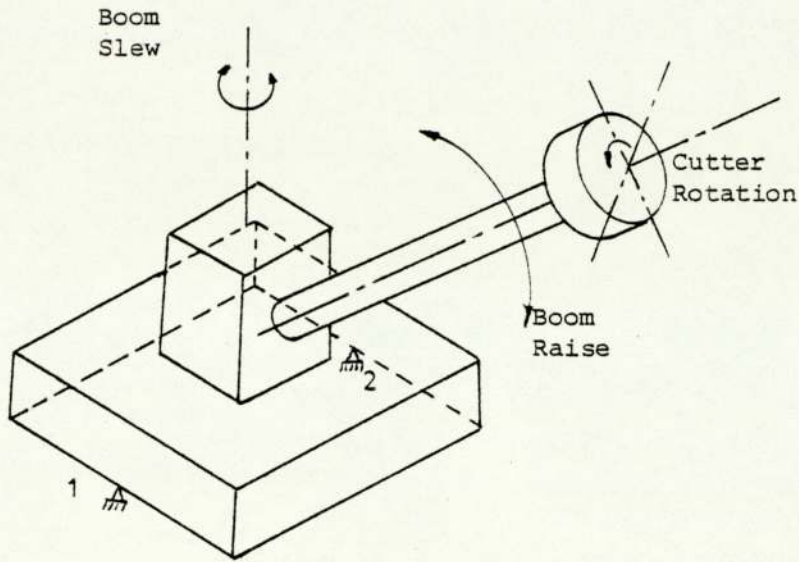


Figure D.2. A Boom-ripper

The vertical movement of the boom can be represented as in Figure D.3.

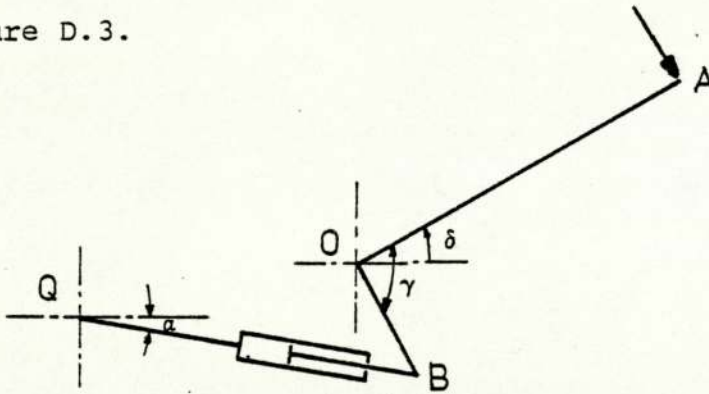


Figure D.3. Diagram to Show the Arrangement to Provide Vertical Movement of the Boom

The boom  $OA$  is rotating about  $O$  in the vertical plane when the link  $AOB$  may be considered as rigid. The hydraulic ram  $PB$  provides the movement in any vertical plane. Since a fixed configuration is considered, the stiffness is investigated with the angle of elevation varying.

If the profile to be cut is considered and has a radius,  $R$ , then the position of the cutter can be represented as a function of  $R$  and the angle of rotation,  $\theta$ , between the line joining the profile centre point (as in Figure D.4), and the horizontal.

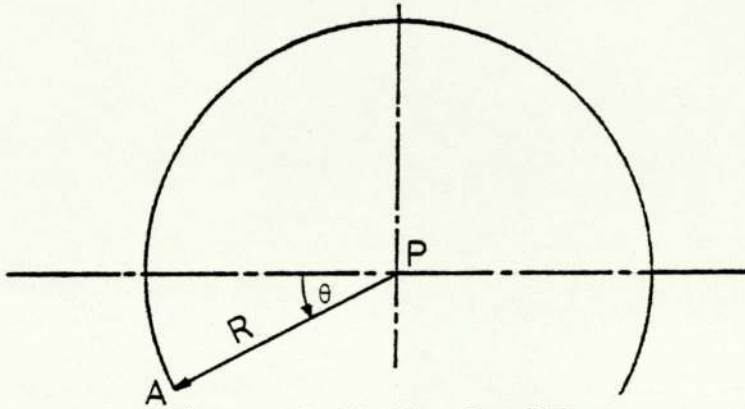


Figure D.4. The Profile

There will be a certain relationship among these quantities and other physical dimensions of the machine. The mechanism of the machine can be represented by Figure D.5.

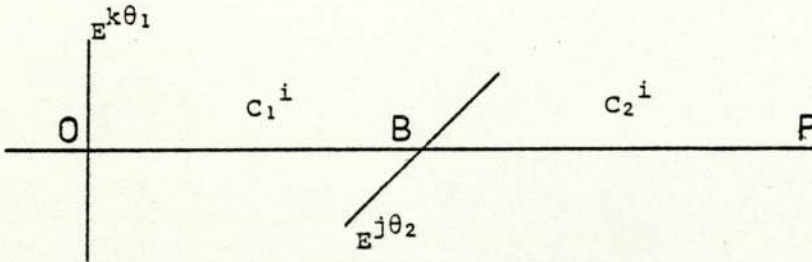


Figure D.5. Diagrammatic Representation of the Mechanism of the Boom-ripper

Figure D.5. indicates that the links  $C_1^i$  and  $C_2^i$  will rotate about  $O$  in a horizontal plane while the link  $C_2^i$  will rotate about  $B$  in a vertical plane that contains  $C_2^i$ . Therefore any position of  $P$  can be found by applying this model.

The stiffness for this type of boom is found to be:-

$$\text{Stiffness} = \left[ \frac{a \sin(\gamma - \delta - \alpha)}{L \cos \delta} \right]^2 S$$

where  $\delta$  = Angle of elevation of the boom

$\alpha$  = Angle of inclination of the ram to  
horizontal

$\gamma$  = Angle AOB

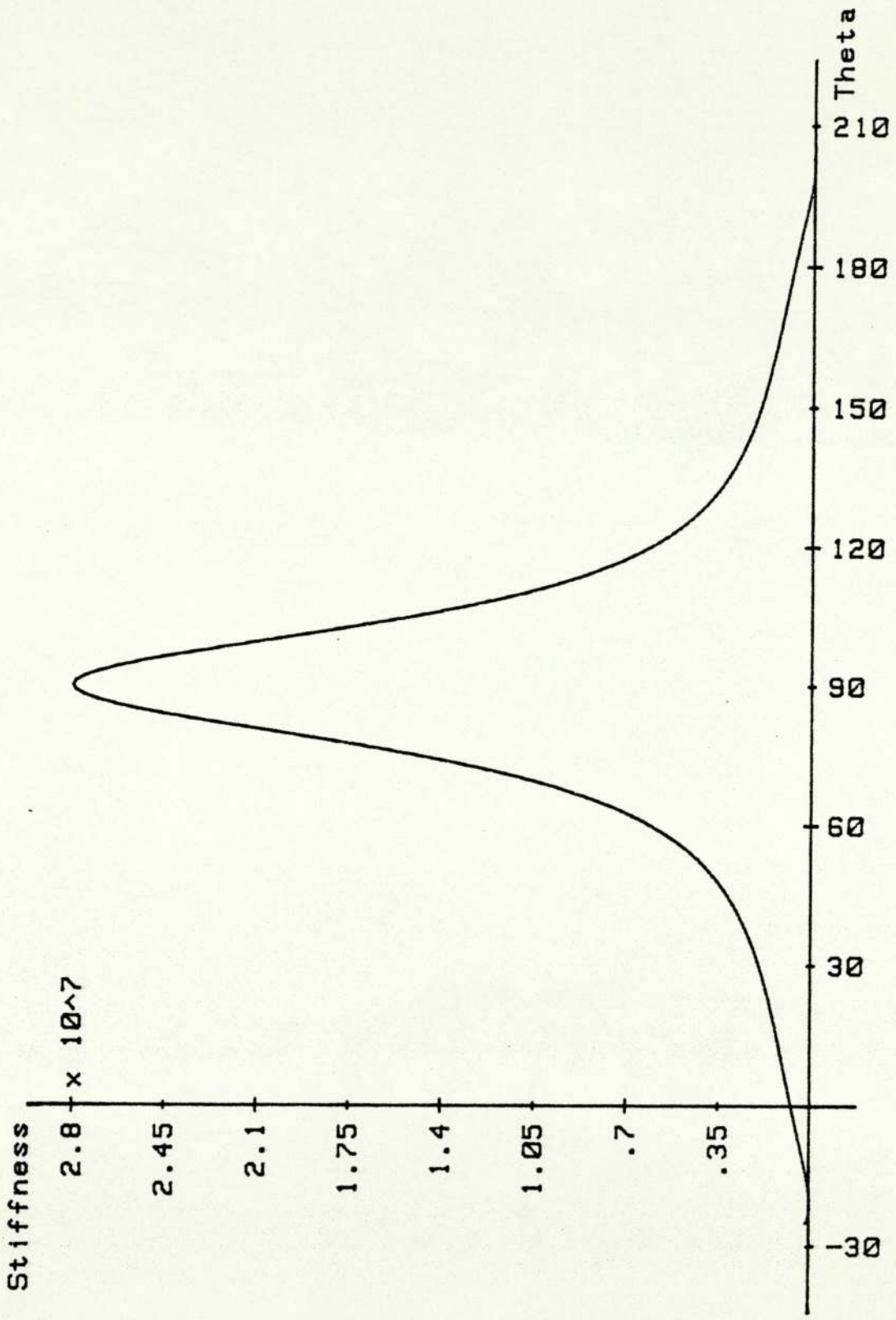
$a$  = Length of the link OB

$L$  = Length of the link OA (the boom)

$S$  = Hydraulic stiffness of the ram

All the above quantities are defined with reference to Figure D.3.

This stiffness is then evaluated and plotted against the angle  $\theta$  as defined in Figure D.4. (Graph D.1). The graph shows that the stiffness is higher when the boom is cutting the top-part of the profile. This is because of the assumption that the reaction force is acting in a tangential direction to the cutter, i.e. when the cutter is cutting the top-part of the profile, this force has a small vertical component and the horizontal force component is resisted by other means which are assumed "rigid" that can take up a large force. In this type of boom ripper, the hydraulic rams, which provide the rotational movement, have fairly constant stiffnesses, therefore the rotational stiffness hardly varies and has less effect on the variation of the boom's stiffness than the lifting trunks have.



Graph D.1 Boom Stiffness - Profile Angle (Theta) Relationship



## D.2. EFFECT OF FEEDBACK ON STIFFNESS

The above two sections show that the effective stiffness of the system will depend on the hydraulic stiffness of the rams. Therefore any improvement in this hydraulic stiffness will improve the system stiffness. Analysis of a simple control system with a proportional position feedback will review this effect. Suppose that the hydraulic ram under consideration is a simple servo-mechanism with a proportional position feedback.

The stiffness of this ram

$$= \frac{1}{\omega} \sqrt{\frac{1}{2} \left( \frac{K_1}{A} \right)^2 + \left( \frac{\omega}{A} \right)^2 \left( A - \frac{\omega^2 MV}{2\beta A} \right)^2} S_o$$

where  $\omega$  = Exciting frequency of disturbing force

$A$  = Area of the ram piston

$M$  = Inertial load

$V$  = Volume of the oil column

$\beta$  = Bulk modulus of the fluid in the ram

$S_o$  = Hydraulic stiffness of the oil column  
in open-loop condition

$$K_1 = C_{DK} \sqrt{\frac{2P_s}{\rho}}$$

$C_D$  = Discharge coefficient of the valve

$W$  = Width of the valve

$K$  = Positional gain

$P_s$  = System supply pressure

$\rho$  = Fluid density

This stiffness will be increased to a high value as the exciting frequency decreases. If the position gain is increased, the stiffness can also be increased.

### D.3. RIGIDITY (STABILITY OF MACHINE ON GROUND)

The machine is supported on ground by two simple supports 1 and 2 which are  $X_1$  and  $X_2$  apart from the vertical plane that contains the axis of slew (this plane faces the profile to be cut, Figure D.2).

The reactions and frictions of these supports can be estimated with reference to the position of the boom (Angle,  $\theta$ , as defined in Figure D.4). The reactions will be the least when the angle is in the range of  $90^\circ$  to  $110^\circ$ , this is because that, in this range, the cutter is cutting horizontally. The force/reaction is maximum in this range, therefore the machine will have minimum resistance to slide movement, so sliding of the machine is likely to occur when the machine is cutting in this position. The force over reaction ratio can be reduced by making the supports as far apart as possible, therefore vertical supports at the far ends can help in stabilising the machine on the ground.

If the rock is too hard for the cutter, then a different story will result. Suppose that the ripper is cutting in a horizontal position and the rock is too hard, then there will be a grinding effect at the boom end and the large reaction at this point will cause the machine to slew.

It can be shown that the chance for the machine to be lifted in this situation is very small. Suppose that the boom in Figure D.2 is in a horizontal position and is pointing forward without any slew, and it cuts a hard rock which causes the rotation of the cutter as in Figure D.6.

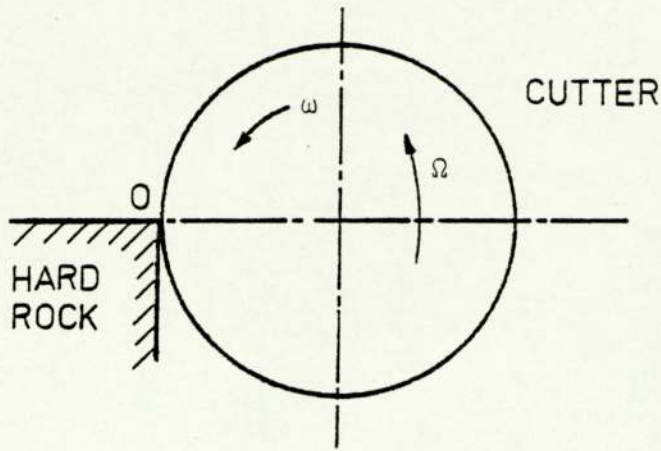


Figure D.6. Cutter cuts a Hard Rock and Stops

By considering the conservation of angular momentum and the impulse force created by the impact, the angular velocity of the machine about O (Fig. D.6) in the vertical plane was found to be in the order of  $10^{-2}$  rad/sec. Therefore it is unlikely for the machine to lift off the ground in this situation.

#### D.4. DRILLING BOOM

The stiffness of the boom can be analysed in the same way as in Section D.0, but with more degrees of freedom, therefore it is not studied here. The stiffness of this machine will affect the machine performance in a different way from that of the boom-ripper. In this machine, the "static" stiffness will be more significant while the "dynamic" stiffness will be more significant in the boom-rippers. Hence for this kind of machine, the stiffness of the machine will contribute to holding the drill in position and is subject to very small disturbance only. The problem associated with this machine is the positioning of the drilling boom rather than keeping the drill in position while it is drilling.

APPENDIX E

Listing of the Assembly Programs for testing  
the time taken to evaluate the simplified  
switching function using different methods.

<u>NAME OF PROGRAM SEGMENT</u>	<u>FUNCTION</u>
PTEST	Driver for testing time used to evaluate the function in software.
MMTEST	Driver for testing the time used to evaluate the function in hardware.
PSWITCH	Routine to time and evaluate function in software with floating point arithmetic library (FPAL).
SWITCH	Routine to time and evaluate function in hardware without storing intermediate results using memory mapped arithmetic processing unit (M/M APU).
MAPSW	Routine to time and evaluate function in hardware with intermediate results stored using M/M APU.
ASWITCH	Routine to time and evaluate function in hardware without storing intermediate results using input or output mapped APU (I/O mapped APU).
APUSW	Routine to time and evaluate function in hardware with intermediate results stored by I/O mapped APU.
RDSND	Routines for data transfer between system memories and I/O mapped APU.
MRDSND	Routines for data transfer between system memories and M/M APU.

<u>METHOD USED TO EVALUATE THE SIMPLIFIED SWITCHING FUNCTION</u>	<u>PROGRAM SEGMENTS TO BE COMBINED</u>	<u>LENGTH OF FINAL PROGRAM IN BYTES</u>
Arithmetic processing in software using FPAL.	PTEST, PSWITCH, FPAL (Supplied by Intel Corporation).	1945 D.
I/O mapped APU with intermediate results stored	MMTEST, APUSW, RDSND.	260 D.
I/O mapped APU without storing of intermediate results.	MMTEST, ASWITCH, RDSND.	230 D.
M/M APU with intermediate results stored.	MMTEST, MAPSW, MRDSND.	244 D.
M/M APU without storing of intermediate results.	MMTEST, SWITCH, MRDSND.	214 D.

IDENTIFIER IN THE ASSEMBLY  
PROGRAM LISTING

NAME IN THE FLOW  
DIAGRAM FIGURE 4.1

COEFF3	$\lambda_1$
COEFF2	$\lambda_2$
COEFF1	$\lambda_4$
COEFF $\phi$	$\lambda_5$
COEFFD	$\lambda_3$
ACCEL	$\ddot{y}$
VELO	$\dot{y}$
POSN	$y$
DESIR	$y_d$
TEMP	Result
RESUL	Evaluated Switching Function Value.

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	A PROGRAMME TO FIND THE TIME TAKEN TO EVALUATE THE
		3	SWITCHING FUNCTION BY FLOATING POINT ARITHMETIC LIBRARY
		4	SOFTWARE REQUIRED: PSWCH.OBJ,FPAL.LIB,PLM80.LIB
		5	HARDWARE REQUIRED: SDK-85 BOARD WITH SECOND 8155
		6	EXPANSION ROMs AND RAMs
		7	WRITTEN BY: C W CHUEN
		8	DATE: 1 JUNE 1982
		9	*****
		10	NAME PTEST
		11	\$INCLUDE(:F1:COEFF.PAL)
		12	*****
		13	SECRET TO CONTAIN THE COEFFICIENTS AND
		14	OTHER RAM AREA DEFINITIONS
		15	TO BE USED WITH THE ROUTINE: PSWCH
		16	*****
		17	PUBLIC COEFF3,COEFF2,COEFF1,COEFF0,ACCEL,VELO,POSN
		18	PUBLIC DESIR,TEMP,RESUL
		19	ASEC
		20	ORG 02000H
2000 00		21	COEFF3: DB 00H,00H,80H,3FH ; LOCATE STARTING RAM ADDRESS
2001 00			; COEFFICIENT FOR ACCELERATION TERM
2002 80			
2003 3F		22	COEFF2: DB 00H,00H,80H,3FH ; FOR VELOCITY TERM
2004 00			
2005 00			
2006 80		23	COEFF1: DB 00H,00H,80H,3FH ; FOR DISPLACEMENT TERM
2007 3F			
2008 00			
2009 00			
200A 80		24	COEFF0: DB 00H,00H,80H,3FH ; CONSTANT TERM
200B 3F			
200C 00			
200D 00			
200E 80			
200F 3F		25	COEFFD: DB 00H,00H,80H,3FH ; DESIRED POSITION TERM
2010 00			
2011 00			
2012 80			
2013 3F		26	ACCEL: DB 00H,00H,00H,40H ; ACCELERATION
2014 00			
2015 00			
2016 00			
2017 40		27	VELO: DB 00H,00H,80H,40H ; VELOCITY
2018 00			
2019 00			
201A 80			
201B 40		28	POSN: DB 00H,00H,00H,00H ; CURRENT POSITION (INTEGER 00 - FFH)
201C 00			
201D 00			
201E 00			
201F 00		29	DESIR: DB 00H,00H,00H,00H ; DESIRED POSITION (INTEGER 00 - FFH)
2020 00			
2021 00			



```

LOC OBJ      LINE      SOURCE STATEMENT
=====
2022 00      =          ;
2023 00      =          ;
2024 00      =          DB      00H,00H,00H,00H ; TEMPORARY STORAGE FOR INTERMEDIATE RESULT
2025 00      =          ;
2026 00      =          ;
2027 00      =          ;
2028 00      =          DB      00H,00H,00H,00H ; MEMORY STORAGE FOR FINAL RESULT
2029 00      =          ;
202A 00      =          ;
202B 00      =          ;
30          =          TEMP:  DB      00H,00H,00H,00H ; TEMPORARY STORAGE FOR INTERMEDIATE RESULT
31          =          RESULT: DB      00H,00H,00H,00H ; MEMORY STORAGE FOR FINAL RESULT
32          =          ;
33          =          ;
34          =          ;
35          =          ;
36          =          ;
37          =          EXTRN PSMTCH,FSET ; STARTING ADDRESSES FOR SWITCH AND FPAL ROUTINES
38          =          PUBLIC FPR      0C000H ; ASSIGN STACK POINTER ADDRESS
39          =          STKPT EQU      0C000H
40          =          $INCLUDE(1;F1;SET.SR1)
41          =          ;
42          =          ; A PROGRAMME SEGMENT FOR TIME TESTING PROGRAMMES
43          =          ; THIS HAS TO BE INSERTED IN THE BEGINNING OF THE
44          =          ; TEST PROGRAMME
45          =          ;
46          =          ; DEFINE STORAGE AREA FOR TIME COUN
47          =          ; ORIGINAL COUNT VALUE
48          =          ; ORIGINAL COUNT VALUE / 2
49          =          CSEG
50          =          START: LXI SP,STKPT
51          =          MVI A,0FFH
52          =          OUT 02CH
53          =          MOV L,A
54          =          MVI A,0BFH
55          =          OUT 02DH
56          =          ANI 03FH
57          =          MOV H,A
58          =          SHLD TIN
59          =          RAR
60          =          MOV H,A
61          =          MOV A,L
62          =          RAR
63          =          MOV L,A
64          =          SHLD HALFT
65          =          LXI B,FPR
66          =          PUSH B
67          =          LXI B,0
68          =          CALL FSET
69          =          CALL PSMTCH
70          =          $INCLUDE(1;F1;SET.SR2)
71          =          ;
72          =          ; A PROGRAMME TO EVALUATE THE TIME-CYCLE COUNT
73          =          ; WRITTEN BY: C W CHUEN
74          =          ; DATE: 09 MAY 1982
75          =          ;
76          =          ; SFT? IN 0?DH ; GET AND STORE HIGH ORDER BYTE OF TIMER VALUE
0000 3100C0
0003 3EFF
0005 D32C
0007 6F
0008 3EBF
000A D32D
000C E63F
000E 67
000F 222E20
0012 1F
0013 67
0014 7D
0015 1F
0016 6F
0017 223020
001A 0100E8
001D C5
001E 010000
0021 C00000
0024 C00000
0027 DB2D
    
```

TIMING TEST FOR FPAL TO EVALUATE SWITCHING FUNCTION

LOC	OBJ	I.LINE	SOURCE STATEMENT
0029	E63F	= 77	ANI 03FH ; MASK OUT THE TIMER MODE BITS 6,7 AND CLEAR CY
002B	1F	= 78	RAR
002C	47	= 79	MOV B,A ; GET AND STORE LOW ORDER BYTE
002D	DB2C	= 80	IN 02CH
002F	1F	= 81	RAR
0030	4F	= 82	MOV C,A
0031	D23A00	C = 83	JNC CONT ; NO ADJUSTMENT IF NO CARRY
0034	2A3020	= 84	LHLD HALFT ; SEE P.6-24 OF MCS 80785
0037	09	= 85	DAD B
0038	44	= 86	MOV B,H
0039	4D	= 87	MOV C,L
003A	7B	= 88	CONT: MOV A,B ; EVALUATE ACTUAL CYCLE COUNT
003B	2F	= 89	CMA
003C	47	= 90	MOV B,A
003D	79	= 91	MOV A,C
003E	2F	= 92	CMA
003F	4F	= 93	MOV C,A
0040	03	= 94	INX B ; TWO'S COMPLEMENT OF B,C
0041	2A2E20	= 95	LHLD TIN ; GET INITIAL COUNT VALUE
0044	09	= 96	DAD B ; ACTUAL CYCLE COUNT EVALUATED AND
0045	222C20	= 97	SHLD TIME ; STORED
004B	CF	= 98	RST 1 ; DISPLAY -8085
E800		= 99	FPR ECU 0B900H ; DEFINE STORAGE FOR FLOATING POINT RECORD
0000		C 100	END START

PUBLIC SYMBOLS

ACCEL	A 2014	COEFF0	A 200C	COEFF1	A 200B	COEFF2	A 2004	COEFF3	A 2000	COEFFD	A 2010	DESIR	A 2020
FPR	A B800	POSN	A 201C	RESUL	A 202B	TEMP	A 2024	VELO	A 201B	TIN	A 202E	CONTC	A 203A

EXTERNAL SYMBOLS

FSET	E 0000	PSWTCH	E 0000	COEFF2	A 2004	COEFF3	A 2000	COEFFD	A 2010	CONTC	A 203A
ACCEL	A 2014	COEFF0	A 200C	COEFF1	A 200B	FSET	E 0000	PSWTCH	E 0000	RESUL	A 202B
DESIR	A 2020	FPR	A B800	STKPT	A C000	TEMP	A 2024	TIME	A 202C	TIN	A 202E
SET2	C 0027	START	C 0000	VELO	A 201B	VELO	A 2018	VELO	A 2018	VELO	A 2018

ASSEMBLY COMPLETE, NO ERRORS

LOC OBJ	LINE	SOURCE STATEMENT
	1	*****
	2	A PROGRAMME TO FIND THE TIME TAKEN TO EVALUATE THE
	3	SWITCHING FUNCTION BY THE MEMORY MAPPED APU - 8231A
	4	SOFTWARE REQUIRED: MAPS2.OBJ
	5	HARDWARE REQUIRED: SDK-BS BOARD, APU BOARD WITH MEMORY
	6	ADDRESSES: 0FC00H - 0FDFFH
	7	WRITTEN BY: C W CHUEN
	8	DATE: 1 JUNE 1982
	9	*****
	10	NAME MMTEST
	11	\$INCLUDE(:F1:COEFF.APU)
	12	*****
	13	SEGMENT TO CONTAIN THE COEFFICIENTS AND
	14	OTHER R A M DEFINITIONS
	15	TO BE USED WITH THE ROUTINE: SWITCH
	16	*****
	17	PUBLIC COEFF3,COEFF2,COEFF1,COEFF0,COEFFD,ACCEL,VELO,POSN
	18	PUBLIC DESIR,TEMP,RESUL
	19	ASEC
2000	20	ORC 02000H ; LOCATE STARTING RAM ADDRESS
2000	21	COEFF3: DB 00H,00H,80H,01H ; COEFFICIENT FOR ACCELERATION TERM
2001	22	COEFF2: DB 00H,00H,80H,01H ; FOR VELOCITY TERM
2002	23	COEFF1: DB 00H,00H,80H,01H ; FOR DISPLACEMENT TERM
2003	24	COEFF0: DB 00H,00H,80H,01H ; CONSTANT TERM
2004	25	COEFFD: DB 00H,00H,80H,01H ; DESIRED POSITION TERM
2005	26	ACCEL: DB 00H,00H,80H,02H ; ACCELERATION
2006	27	VELO: DB 00H,00H,80H,03H ; VELOCITY
2007	28	POSN: DB 00H,00H ; CURRENT POSITION (INTEGER 00 - FFH)
2008	29	DESIR: DB 00H,00H ; DESIRED POSITION (INTEGER 00 - FFH)
2009	30	TEMP: DB 00H,00H,00H,00H ; TEMPORARY STORAGE FOR INTERMEDIATE RESULT
200A		
200B		
200C		
200D		
200E		
200F		
2010		
2011		
2012		
2013		
2014		
2015		
2016		
2017		
2018		
2019		
201A		
201B		
201C		
201D		
201E		
201F		
2020		
2021		

LOC	OBJ	LINE	SOURCE STATEMENT
2022	00	=	
2023	00	=	
2024	00	=	31 RESULT: DB 00H, 00H, 00H, 00H ; MEMORY STORAGE FOR FINAL RESULT
2025	00	=	
2026	00	=	
2027	00	=	
		=	32 ;
		=	33 ;
		=	34 ; END OF SEGMENT
		=	35 ;
		=	36 ;
		=	37 EXTRN SWITCH ; LOCATE SWITCH ROUTINE ADDRESS
		=	38 STKPT EQU 020C2H ; ASSIGN STACK POINTER ADDRESS
		=	39 SET, SR1)
		=	40 ;
		=	41 ; A PROGRAMME SEGMENT FOR TIME TESTING PROGRAMMES
		=	42 ; THIS HAS TO BE INSERTED IN THE BEGINNING OF THE
		=	43 ; TEST PROGRAMME
		=	44 ;
		=	45 TIME! DS 2 ; DEFINE STORAGE AREA FOR TIME COUN
		=	46 TIN: DS 2 ; ORIGINAL COUNT VALUE
		=	47 HALFT: DS 2 ; ORIGINAL COUNT VALUE / 2
		=	48 CSEC
		=	49 START: LXI SP, STKPT ; LOCATE STACK POINTER TO STKPT
		=	50 MVI A, 0FFH ; INITIALIZE COUNTER IN 8155 FOR SINGLE SHOT MODE
		=	51 OUT 02CH ; AND LOAD COUNTER WITH HIGHEST VALUE 5FFFH
		=	52 MOV L, A ; TRANSFER TO H, L-REGISTER PAIR
		=	53 MVI A, 0BFH ;
		=	54 OUT 02DH ;
		=	55 ANI 03FH ; MASK OUT THE MODE BITS (6,7) AND CLEAR CARRY
		=	56 MOV H, A ;
		=	57 SHLD TIN ; STORE THIS INITIAL COUNT VALUE
		=	58 RAR ; HALFCING INITIAL COUNT VALUE
		=	59 MOV H, A ;
		=	60 MOV A, L ;
		=	61 RAR ;
		=	62 MOV L, A ;
		=	63 SHLD HALFT ; STORE THIS VALUE
		=	64 CALL SWITCH ; CALL THE SWITCHING FUNCTION
		=	65 \$INCLUDE(FI:SET,SR2)
		=	66 ;
		=	67 ; A PROGRAMME TO EVALUATE THE TIME-CYCLE COUNT
		=	68 ; WRITTEN BY: C W CHUEN
		=	69 ; DATE: 09 MAY 1982
		=	70 ;
		=	71 SET2: IN 02DH ; GET AND STORE HIGH ORDER BYTE OF TIMER VALUE
		=	72 ANI 03FH ; MASK OUT THE TIMER MODE BITS 6,7 AND CLEAR C
		=	73 RAR ;
		=	74 MOV B, A ; GET AND STORE LOW ORDER BYTE
		=	75 IN 02CH ;
		=	76 RAR ;
		=	77 MOV C, A ;
		=	78 JNC CONT ; NO ADJUSTMENT IF NO CARRY
		=	79 LHLD HALFT ; SEE P.6-24 OF MCS 80/85
		=	80 DAD R

LOC	OBJ	LINE	SOURCE STATEMENT
002E	44	= 81	MOV B,H
002F	4D	= 82	MOV C,L
0030	78	= 83	CONT: MOV A,B
0031	2F	= 84	CMA
0032	47	= 85	MOV B,A
0033	79	= 86	MOV A,C
0034	2F	= 87	CMA
0035	4F	= 88	MOV C,A
0036	03	= 89	INX B
0037	2A2A20	= 90	LHLD TIN
003A	09	= 91	DAD B
003B	222820	= 92	SHLD TIME
003E	CF	= 93	RST 1
0000		C 94	END START

I EVALUATE ACTUAL CYCLE COUNT  
  
 I TWO'S COMPLEMENT OF B, C  
 I GET INITIAL COUNT VALUE  
 I ACTUAL CYCLE COUNT EVALUATED AND  
 I STORED  
 I DISPLAY -8085

PUBLIC SYMBOLS

ACCEL A 2014	COEFF0 A 200C	COEFF1 A 200B	COEFF2 A 2004	COEFF3 A 2000	COEFFD A 2010	DESIR A 201E
POSN A 201C	RESUL A 2024	TEMP A 2020	VELO A 2018			

EXTERNAL SYMBOLS

SWITCH E 0000
---------------

USER SYMBOLS

ACCEL A 2014	COEFF0 A 200C	COEFF1 A 200B	COEFF2 A 2004	COEFF3 A 2000	COEFFD A 2010	CONT C 0030
DESIR A 201E	HALFT A 202C	POSN A 201C	RESUL A 2024	SET2 C 001D	START C 0000	STKPT A 20C2
SWITCH E 0000	TEMP A 2020	TIME A 202B	TIN A 202A	VELO A 201B		

ASSEMBLY COMPLETE, NO ERRORS

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	A ROUTINE TO EVALUATE THE SWITCHING FUNCTION BASED ON
		3	ACCEL,VEL, POSITION AND OTHER COEFFICIENTS USING THE
		4	FLOATING POINT ARITHMETIC LIBRARY (FPAL.LIB)
		5	*****
		6	SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO +
		7	COEFF1*POSN + COEFFD*DESIR + COEFF0
		8	*****
		9	DESIR AND POSN ARE SINGLE PRECISION INTEGERS (2-BYTE)
		10	OTHERS ARE FLOATING POINT NUMBERS (BFP NOS.)
		11	*****
		12	SOFTWARE REQUIRED: FPAL.LIB,PLM80.LIB,SYSTEM.LIB
		13	HARDWARE REQUIRED: SDK-85, EXPANSION ROMs AND RAMs
		14	*****
		15	WRITTEN BY: C W CHUEN
		16	DATE: 01 JUNE 1982
		17	*****
		18	PSWTCH
		19	FPR,FLOAD,FMUL,FADD,FLTDS,FSTOR,FSUB
		20	COEFF3,COEFF2,COEFF1,COEFF0,COEFFD
		21	ACCEL,VELO,POSN,DESIR,TEMP,RESUL
		22	PUBLIC PSWTCH
		23	CSEC
0000	F5	24	PSWTCH: PSW ; SAVE REGISTERS
0001	E5	25	PUSH H
0002	C5	26	PUSH B
0003	3EC0	27	MVI A,0C0H ; START THE COUNT DOWN MODE FOR TIMER
0005	D52B	28	OUT 02BH
0007	010000	29	LXI B,FPR ; LOCATE FLOATING POINT RECORD
000A	110000	30	D,DESIR ; CONVERT DESIR INTO
000D	C00000	31	CALL FLTDS ; BINARY FLOATING POINT
0010	110000	32	LXI D,COEFFD ;
0013	C00000	33	CALL FMUL ; COEFFD*DESIR
0016	110000	34	LXI D,RESUL ;
0019	C00000	35	CALL FSTOR ; <---
001C	110000	36	LXI D,POSN ; CONVERT POSN INTO
001F	C00000	37	CALL FLTDS ; BINARY FLOATING POINT FORMAT
0022	110000	38	LXI D,COEFF1 ;
0025	C00000	39	CALL FMUL ;
0028	110000	40	LXI D,TEMP ; COEFF1*POSN
002B	C00000	41	FSTOR <---
002E	110000	42	LXI D,COEFF3 ; COEFF3
0031	C00000	43	CALL FLOAD ;
0034	110000	44	LXI D,ACCEL ; ACCEL
0037	C00000	45	FMUL ; COEFF3*ACCEL
003A	110000	46	LXI D,TEMP ;
003D	C00000	47	FADD ;
0040	C00000	48	CALL FSTOR ; <---
0043	110000	49	LXI D,COEFF2 ; COEFF2
0046	C00000	50	CALL FLOAD ;
0049	110000	51	LXI D,VELO ; VELO
004C	C00000	52	FMUL ; COEFF2*VELO
004F	110000	53	LXI D,TEMP ;
0052	C00000	54	CALL FADD ;
0055	C00000	55	CALL ;

LOC	OBJ	LINE	SOURCE STATEMENT
0055	110000	E 55	D, COEFF0 ; COEFF0
0058	C00000	E 56	FADD ; +
005B	110000	E 57	D, RESULT
005E	C00000	E 58	FADD ; DESIR ^
0061	C00000	E 59	FSTOR ; <---
0064	3E40	60	A, 040H ; STOP TIMER
0066	D32B	61	OUT ;
0068	C1	62	B ;
0069	E1	63	H ;
006A	F1	64	PSW ;
006B	C9	65	RET ;
		66	;
		67	END

AND RECOVER REGISTERS SAVED  
 BEFORE RETURN TO THE CALLING PROGRAMME

PUBLIC SYMBOLS  
 PSWITCH C 000

EXTERNAL SYMBOLS

ACCEL	E 0000	COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000	DESIR	E 0000
FADD	E 0000	FLOAD	E 0000	FLTDS	E 0000	FMUL	E 0000	FPR	E 0000	FSTOR	E 0000	FSUB	E 0000
POSN	E 0000	RESULT	E 0000	TEMP	E 0000	VELO	E 0000	VELO	E 0000				

USER SYMBOLS

ACCEL	E 0000	COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000	DESIR	E 0000
FADD	E 0000	FLOAD	E 0000	FLTDS	E 0000	FMUL	E 0000	FPR	E 0000	FSTOR	E 0000	FSUB	E 0000
POSN	E 0000	PSWITCH	C 0000	RESULT	E 0000	TEMP	E 0000	VELO	E 0000				

ASSEMBLY COMPLETE, NO ERRORS

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	A ROUTINE TO CALCULATE THE SWITCHING FUNCTION BASED ON ACCEL,
		3	VEL, POSITION AND OTHER COEFFICIENTS USING THE APU B231A WITH
		4	MEMORY MAPPING TECHNIQUE
		5	TEMPORARY STORING OF INTERMEDIATE RESULTS ARE OMITTED
		6	*****
		7	SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO + COEFF1*POSN
		8	+ COEFF0 + COEFFD*DESIR
		9	*****
		10	WRITTEN BY: C W CHUEN
		11	DATE: 1 JUNE 1982
		12	*****
		13	NAME SWITCH
		14	\$INCLUDE(F1:APUCMD.ASM)
		15	\$NOLIST
		66	\$LIST
		67	EXTRN SEND2, SEND4, READ2, READ4
		68	EXTRN COEFF3, COEFF2, COEFF1, COEFF0, COEFFD
		69	EXTRN ACCEL, VELO, POSN, DESIR, TEMP, RESULT
		70	PUBLIC SWITCH
		71	CSEG
0000	F5	72	SWITCH: PUSH PSW ; SAVE REGISTERS
0001	E5	73	PUSH H
0002	C5	74	PUSH B
0003	3E0	75	MVI A, 0C0H ; START THE COUNT DOWN MODE FOR TIMER
0005	D32B	76	OUT 02BH
0007	010000	77	LXI B, DESIR ; CONVERT DESIR FROM 2-BYTE INTEGER INTO
000A	CD0000	78	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
000D	26FD	79	MVI H, APUH ; LOAD H WITH APU COMMAND PORT ADDRESS
000F	361D	80	MVI M, FLTS ; ISSUE THE CONVERSION COMMAND TO APU
0011	010000	81	LXI B, COEFFD ; LOAD APU DATA STACK WITH COEFFD
0014	CD0000	82	CALL SEND4
0017	3612	83	MVI M, FMUL ; COEFFD*DESIR
0019	010000	84	LXI B, POSN ; CONVERT POSN FROM 2-BYTE INTEGER INTO
001C	CD0000	85	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
001F	361D	86	MVI M, FLTS ; RECOGNIZED BY APU
0021	010000	87	LXI B, COEFF1 ; LOAD APU DATA STACK WITH COEFF1
0024	CD0000	88	CALL SEND4
0027	3612	89	MVI M, FMUL ; COEFF1*POSN
0029	3610	90	MVI M, FADD ; COEFFD*DESIR + COEFF1*POSN
002B	010000	91	LXI B, COEFF3 ; COEFF3
002E	CD0000	92	CALL SEND4 ; ^
0031	010000	93	LXI B, ACCEL ; ACCEL
0034	CD0000	94	CALL SEND4
0037	3612	95	MVI M, FMUL ; COEFF3*ACCEL
0039	3610	96	MVI M, FADD ; COEFF3*ACCEL + COEFF1*POSN + COEFFD*DESIR
003B	010000	97	LXI B, COEFF2 ; COEFF2
003E	CD0000	98	CALL SEND4 ; ^
0041	010000	99	LXI B, VELO ; VELO
0044	CD0000	100	CALL SEND4
0047	3612	101	MVI M, FMUL ; COEFF2*VELO
0049	3610	102	MVI M, FADD ; ^
004B	010000	103	LXI B, COEFF0 ; COEFF0
004E	CD0000	104	CALL SEND4 ; ^





LOC	OBJ	LINE	SOURCE STATEMENT
1		1	*****
2		2	A ROUTINE TO CALCULATE THE SWITCHING FUNCTION BASED ON ACCEL,
3		3	VEL, POSITION AND OTHER COEFFICIENTS USING THE APU B231A WITH
4		4	MEMORY MAPPING TECHNIQUE
5		5	*****
6		6	SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO + COEFF1*POSN
7		7	+ COEFF0 + COEFFD*DESIR
8		8	*****
9		9	WRITTEN BY: C W CHUEN
10		10	DATE: 1 JUNE 1982
11		11	*****
12		12	MAPSW
13		13	\$INCLUDE(F1:APUCMD.ASM)
14		14	\$NOLIST
15		15	= 65 \$LIST
66		66	EXTRN SEND2, SEND4, READ2, READ4
67		67	EXTRN COEFF3, COEFF2, COEFF1, COEFF0, COEFFD
68		68	EXTRN ACCEL, VELO, POSN, DESIR, TEMP, RESUL
69		69	PUBLIC SWITCH
70		70	CSEC
0000	F5	71	SWITCH: PUSH PSW ; SAVE REGISTERS
0001	E5	72	PUSH H
0002	C5	73	PUSH B
0003	3E0	74	MVI A, 0C0H ; START THE COUNT DOWN MODE FOR TIMER
0005	D32B	75	OUT 02BH
0007	010000	76	LXI B, DESIR ; CONVERT DESIR FROM 2-BYTE INTEGER INTO
000A	CD0000	77	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
000D	26FD	78	MVI H, APUC ; LOAD H WITH APU COMMAND PORT ADDRESS
000F	361D	79	MVI M, FLTS ; ISSUE THE CONVERSION COMMAND TO APU
0011	010000	80	LXI B, COEFFD ; LOAD APU DATA STACK WITH COEFFD
0014	CD0000	81	CALL SEND4
0017	3612	82	MVI M, FMUL ;
0019	010000	83	LXI B, RESUL ; STORE COEFFD*DESIR
001C	CD0000	84	CALL READ4
001F	010000	85	LXI B, POSN ; CONVERT POSN FROM 2-BYTE INTEGER INTO
0022	CD0000	86	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
0025	361D	87	MVI M, FLTS ; RECOGNIZED BY APU
0027	010000	88	LXI B, COEFF1 ; LOAD APU DATA STACK WITH COEFF1
002A	CD0000	89	CALL SEND4
002D	3612	90	MVI M, FMUL ;
002F	010000	91	LXI B, TEMP ; COEFF1*POSN
0032	CD0000	92	CALL READ4 ; <---
0035	010000	93	LXI B, COEFF3 ; COEFF3
0038	CD0000	94	CALL SEND4 ;
003B	010000	95	LXI B, ACCEL ; ACCEL
003E	CD0000	96	CALL SEND4
0041	3612	97	MVI M, FMUL ; COEFF3*ACCEL
0043	010000	98	LXI B, TEMP ;
0046	CD0000	99	CALL SEND4
0049	3610	100	MVI M, FADD ;
004B	CD0000	101	CALL READ4 ; <---
004E	010000	102	LXI B, COEFF2 ; COEFF2
0051	CD0000	103	CALL SEND4 ;
0054	010000	104	LXI B, VELO ; VELO

1515-II 8080/8085 MACRO ASSEMBLER, V3.0 MAPSM  
 ROUTINE FOR M/M APU TO EVALUATE SWITCHING FUNCTION

LOC	OBJ	LINE	SOURCE STATEMENT
0057	CD0000	E 105	CALL SEND4
005A	3612	E 106	M, FMUL
005C	010000	E 107	B, TEMP
005F	CD0000	E 108	CALL SEND4
0062	3610	E 109	M, FADD
0064	010000	E 110	B, COEFF0
0067	CD0000	E 111	CALL SEND4
006A	3610	E 112	M, FADD
006C	010000	E 113	B, RESUL
006F	CD0000	E 114	CALL SEND4
0072	3610	E 115	M, FADD
0074	CD0000	E 116	CALL SEND4
0077	3E40	E 117	READ4
0079	D328	E 118	A, 040H
007B	C1	E 119	OUT 028H
007C	E1	E 120	B
007D	F1	E 121	H
007E	C9	E 122	PSM
		E 123	RET
		E 124	END

PUBLIC SYMBOLS  
 SWITCH C 0000

EXTERNAL SYMBOLS	COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000	DESIR	E 0000
ACCEL	E 0000		READ4	E 0000	RESUL	E 0000	SEND2	E 0000	SEND4	E 0000	TEMP	E 0000
POSN	E 0000											
VELO	E 0000											

USER SYMBOLS	AC05	A 0006	APUC	A 00FD	APUD	A 00FC	ASIN	A 0005	ATAN	A 0007	CHSD	A 0034
ACCEL	E 0000		COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000
CHSF	A 0015	CHSS	A 0074	DDIV	A 002F	DESIR	DMUL	A 002E	DMUJ	A 0036	DSUB	A 002D
C05	A 0009	DADD	A 002C	FDIV	A 0013	FLXD	FLTS	A 001E	FLTD	A 001C	FLTS	A 001D
EXP	A 000A	FADD	A 0010	LN	A 0009	LOG	POP	A 003B	POFF	A 001B	POPS	A 007B
FMUL	A 0012	FSUB	A 0011	PTOF	A 0017	PTOS	PUF1	A 001A	PUR	A 000B	READ2	E 0000
POSN	E 0000	PTOD	A 0037	SADD	A 006C	SDIV	SEND2	E 0000	SEND4	E 0000	SIN	A 0002
READ4	E 0000	RESUL		SORT	A 0001	SSUB	SWITCH	C 0000	TAN	A 0004	TEMP	E 0000
SMUL	A 006E	SMUJ	A 0076	XCHF	A 0019	XCHS						
VELO	E 0000	XCHD	A 0039									

ASSEMBLY COMPLETE, NO ERRORS

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	A ROUTINE TO CALCULATE THE SWITCHING FUNCTION BASED ON ACCEL,
		3	VEL, POSITION AND OTHER COEFFICIENTS USING THE APU B231A WITH
		4	I/O MAPPING TECHNIQUE
		5	TEMPORARY STORING OF INTERMEDIATE RESULTS ARE OMITTED
		6	*****
		7	SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO + COEFF1*POSN
		8	+ COEFF0 + COEFFD*DESIR
		9	*****
		10	WRITTEN BY: C W CHIDEN
		11	DATE: 1 JUNE 1982
		12	*****
		13	NAME ASWITCH
		14	\$INCLUDE('F1:APUCMD.ASM')
		15	\$MOLIST
		66	\$LIST
		67	EXTRN SEND2, SEND4, READ2, READ4
		68	EXTRN COEFF3, COEFF2, COEFF1, COEFF0, COEFFD
		69	EXTRN ACCEL, VELO, POSN, DESIR, TEMP, RESULT
		70	PUBLIC SWITCH
		71	CSEC
0000	F5	72	SWITCH: PUSH PSW ; SAVE REGISTERS
0001	E5	73	PUSH H
0002	C5	74	PUSH B
0003	3EC0	75	MVI A, 0C0H ; START THE COUNT DOWN MODE FOR TIMER
0005	D328	76	OUT 02BH
0007	010000	77	LXI B, DESIR ; CONVERT DESIR FROM 2-BYTE INTEGER INTO
000A	CD0000	78	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
000D	3E1D	79	MVI A, FLTS ; ISSUE THE CONVERSION COMMAND TO APU
000F	D3FD	80	OUT APUC ; COMMAND PORT
0011	010000	81	LXI B, COEFFD ; LOAD APU DATA STACK WITH COEFFD
0014	CD0000	82	CALL SEND4
0017	3E12	83	MVI A, FMUL ; COEFFD*DESIR
0019	D3FD	84	OUT APUC
001B	010000	85	LXI B, POSN ; CONVERT POSN FROM 2-BYTE INTEGER INTO
001E	CD0000	86	CALL SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
0021	3E1D	87	MVI A, FLTS ; RECOGNIZED BY APU
0023	D3FD	88	OUT APUC
0025	010000	89	LXI B, COEFF1 ; LOAD APU DATA STACK WITH COEFF1
0028	CD0000	90	CALL SEND4
002B	3E12	91	MVI A, FMUL ; COEFF1*POSN
002F	3E10	92	OUT APUC
0031	D3FD	93	MVI A, FADD ; COEFFD*DESIR + COEFF1*POSN
0033	010000	94	OUT APUC
0036	CD0000	95	LXI B, COEFF3 ; COEFF3
0039	010000	96	CALL SEND4 ;
003C	CD0000	97	LXI B, ACCEL ; ACCEL
003F	3E12	98	CALL SEND4
0041	D3FD	99	MVI A, FMUL ; COEFF3*ACCEL
0043	3E10	100	OUT APUC
0045	D3FD	101	MVI A, FADD ; COEFF3*ACCEL + COEFF1*POSN + COEFFD*DESIR
0047	010000	102	OUT APUC
004A	CD0000	103	LXI B, COEFF2 ; COEFF2
		104	CALL SEND4 ;

IS15-11 8080/8085 MACRO ASSEMBLER, V3.0  
 ROUTINE FOR IO APU TO EVALUATE SWITCHING FUNCTION

LOC	OBJ	L	LINE	SOURCE STATEMENT
004D	010000	E	105	B, VELO ; VELO
0050	CD0000	E	106	SEND4 ;
0053	3E12		107	A, FMUL ; COEFF2*VELO
0055	D3FD		108	OUT APUC ;
0057	3E10		109	MVI A, FADD ; +
0059	D3FD		110	OUT APUC ;
005B	010000	E	111	B, COEFF0 ; COEFF0
005E	CD0000	E	112	SEND4 ; ^
0061	3E10		113	MVI A, FADD ; +
0063	D3FD		114	OUT APUC ;
0065	010000	E	115	B, RESULT ; <----
0068	CD0000	E	116	READ4 ;
006B	3E40		117	MVI A, 040H ; STOP TIMER BEFORE RETURN TO
006D	D32B		118	OUT ; THE CALLING PROGRAMME
006F	C1		119	POP B ; RECOVER REGISTERS SAVED
0070	E1		120	H ;
0071	F1		121	POP PSW ;
0072	C9		122	RET ;
			123	;
			124	END

PUBLIC SYMBOLS  
 SWITCH C 0000

EXTERNAL SYMBOLS	COEFF0 E 0000	COEFF1 E 0000	COEFF2 E 0000	COEFF3 E 0000	COEFF4 E 0000	DESIR E 0000
ACCEL E 0000	READ2 E 0000	READ4 E 0000	RESULT E 0000	SEND2 E 0000	SEND4 E 0000	TEMP E 0000
POSN E 0000						
VELO E 0000						
USER SYMBOLS	ACOS A 0006	APUC A 00FD	APUD A 00FC	ASIN A 0005	ATAN A 0007	CHSD A 0034
ACCEL E 0000	CHSS A 0074	COEFF0 E 0000	COEFF1 E 0000	COEFF2 E 0000	COEFF3 E 0000	COEFFD E 0000
CHSF A 0015	DADD A 002C	DDIV A 002F	DESIR E 0000	DMUL A 002E	DMUJ A 0036	DSUB A 002D
COS A 0003	FADD A 0010	FDIV A 0019	FIXD A 001E	FIXS A 001F	FLTD A 001C	FLTS A 001D
EXP A 000A	FSUB A 0011	LN A 0009	LOG A 0008	LOG A 003B	POPF A 001B	POPS A 007B
FMUL E 0000	PTOD A 0037	PTOF A 0017	PTOS A 0077	PUPI A 001A	PWR A 000B	READ2 E 0000
PUSN E 0000	RESUL E 0000	SADD A 006C	SDBV A 006F	SEND2 E 0000	SEND4 E 0000	SIN A 0002
READ4 E 0000	SRUJ A 0076	SGRT A 0001	SSUB A 006D	SWITCH C 0000	TAN A 0004	TEMP E 0000
SRUJ A 006E	XCHD A 0039	XCHF A 0019	XCHS A 0079			
VELO E 0000						

ASSEMBLY COMPLETE, NO ERRORS

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LOC OBJ      LINE      SOURCE STATEMENT
1 1 *****
2 2 A ROUTINE TO CALCULATE THE SWITCHING FUNCTION BASED ON ACCEL,
3 3 VEL, POSITION AND OTHER COEFFICIENTS USING THE APU B231A WITH
4 4 MEMORY MAPPING TECHNIQUE
5 5
6 6 SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO + COEFF1*POSN
7 7 + COEFF0 + COEFFD*DESIR
8 8
9 9 WRITTEN BY: C W CHUEN
10 10 DATE: 1 JUNE 1982
11 11 *****
12 12 NAME APUSM
13 13 $INCLUDE (:F1:APUCMD.ASM)
14 14 $NOLIST
15 15 =
16 16 =
17 17
18 18
19 19
20 20
21 21
22 22
23 23
24 24
25 25
26 26
27 27
28 28
29 29
30 30
31 31
32 32
33 33
34 34
35 35
36 36
37 37
38 38
39 39
40 40
41 41
42 42
43 43
44 44
45 45
46 46
47 47
48 48
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51 51
52 52
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57 57
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61 61
62 62
63 63
64 64
65 65
66 66
67 67
68 68
69 69
70 70
71 71
72 72
73 73
74 74
75 75
76 76
77 77
78 78
79 79
80 80
81 81
82 82
83 83
84 84
85 85
86 86
87 87
88 88
89 89
90 90
91 91
92 92
93 93
94 94
95 95
96 96
97 97
98 98
99 99
100 100
101 101
102 102
103 103
104 104

*****
A ROUTINE TO CALCULATE THE SWITCHING FUNCTION BASED ON ACCEL,
VEL, POSITION AND OTHER COEFFICIENTS USING THE APU B231A WITH
MEMORY MAPPING TECHNIQUE

SWITCHING FUNCTION = COEFF3*ACCEL + COEFF2*VELO + COEFF1*POSN
+ COEFF0 + COEFFD*DESIR

WRITTEN BY: C W CHUEN
DATE: 1 JUNE 1982
*****
NAME APUSM
$INCLUDE (:F1:APUCMD.ASM)
$NOLIST
=
=

*****
PSW ; SAVE REGISTERS
H
B
A,OC0H ; START THE COUNT DOWN MODE FOR TIMER
02BH
B,DESIR ; CONVERT DESIR FROM 2-BYTE INTEGER INTO
SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
A,FLTS ; ISSUE THE CONVERSION COMMAND TO APU
APUC
B,COEFFD ; LOAD APU DATA STACK WITH COEFFD
SEND4
A,FMUL ; *
APUC
B,RESUL ; STORE COEFFD*DESIR
READ4
B,POSN ; CONVERT POSN FROM 2-BYTE INTEGER INTO
SEND2 ; 4-BYTE BINARY FLOATING POINT FORMAT
A,FLTS ; RECOGNIZED BY APU
APUC
B,COEFF1 ; LOAD APU DATA STACK WITH COEFF1
SEND4
A,FMUL ; *
APUC
B,TEMP ; COEFF1*POSN
READ4 ; <---
B,COEFF3 ; COEFF3
SEND4 ; ^
B,ACCEL ; ACCEL
SEND4
A,FMUL ; COEFF3*ACCEL
APUC
B,TEMP ;
SEND4
A,FADD ; +
PVI

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LOC	OBJ	LINE	SOURCE STATEMENT
0053	D3FD	105	APUC
0055	CD0000	E 106	OUT CALL
0058	010000	E 107	LXI B, COEFF2 ; <--- SEND4 ; COEFF2
005B	CD0000	E 108	CALL B, VELO ; ^
005E	010000	E 109	LXI B, VELO ; VELO
0061	CD0000	E 110	CALL A, FMUL ; COEFF2*VELO
0064	3E12	111	MVI APUC
0066	D3FD	112	OUT B, TEMP
0068	010000	E 113	LXI SEND4
006B	CD0000	E 114	CALL A, FADD ; +
006E	3E10	115	MVI APUC
0070	D3FD	116	OUT B, COEFF0 ; COEFF0
0072	010000	E 117	LXI SEND4 ; ^
0075	CD0000	E 118	CALL A, FADD ; +
0078	3E10	119	MVI APUC
007A	D3FD	120	OUT B, RESULT ; DESIR ^
007C	010000	E 121	LXI SEND4 ; +
007F	CD0000	E 122	CALL A, FADD ; +
0082	3E10	123	MVI APUC
0084	D3FD	124	OUT CALL
0086	CD0000	E 125	CALL MVI A, 040H ; <--- 02BH ; STOP TIMER BEFORE RETURN TO B ; THE CALLING PROGRAMME H ; RECOVER REGISTERS SAVED PSW
0089	3E40	126	MVI
008B	D32B	127	OUT
008D	C1	128	POP
008E	E1	129	POP
008F	F1	130	POP
0090	C9	131	RET
132		132	END
133		133	END

PUBLIC SYMBOLS  
 SWITCH C 0000

EXTERNAL SYMBOLS

ACCEL	E 0000	COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000	DESIR	E 0000
POSN	E 0000	READ2	E 0000	READ4	E 0000	RESULT	E 0000	SEND2	E 0000	SEND4	E 0000	TEMP	E 0000
VELO	E 0000												

USER SYMBOLS

ACCEL	E 0000	ACOS	A 0006	APUC	A 00FD	APUD	A 00FC	ASIN	A 0005	ATAN	A 0007	CHSD	A 0034
CHSF	A 0015	CHSS	A 0074	COEFF0	E 0000	COEFF1	E 0000	COEFF2	E 0000	COEFF3	E 0000	COEFFD	E 0000
COS	A 0005	DADD	A 002C	DDIV	A 002F	DESIR	E 0000	DMUL	A 002E	DMUJ	A 0036	DSUB	A 002D
EXP	A 000A	FADD	A 0010	FDIV	A 0015	FIXD	A 001E	FIXS	A 001F	FLTD	A 001D	FLTS	A 001D
FMUL	A 0012	FSUB	A 0011	LN	A 0009	LOC	A 0008	LOP	A 0038	POPF	A 0018	POPS	A 007B
POSN	E 0000	PTOD	A 0037	PTOF	A 0017	PUPI	A 0077	PUPI	A 001A	PWR	A 000B	READ2	E 0000
READ4	E 0000	RESUL	E 0000	SADD	A 006C	SDIV	A 006F	SEND2	E 0000	SEND4	E 0000	SIN	A 0002
SMUL	A 006E	SMUJ	A 0076	SORT	A 0001	SSUB	A 006D	SWITCH	C 0000	TAN	A 0004	TEMP	E 0000
VELO	E 0000	XCHD	A 0059	XCHF	A 0019	XCHS	A 0079						

ASSEMBLY COMPLETE, NO ERRORS

LOC	OBJ	LINE	SOURCE STATEMENT
1		1	ROUTINES TO SEND/READ BYTES TO/FROM
2		2	APU STACK
3		3	I.E. MEMORY <----> APU STACK
4		4	ON ENTRY: REGISTERS B,C CONTAIN
5		5	STARTING MEMORY ADDRESS
6		6	LEAST SIGNIFICANT BYTE FIRST
7		7	
8		8	
9		9	S/R NAME FUNCTION
10		10	-----
11		11	SEND2 SEND 2 BYTES
12		12	SEND4 SEND 4 BYTES
13		13	READ2 READ 2 BYTES
14		14	READ4 READ 4 BYTES
15		15	
16		16	WRITTEN BY : C W CHUEN
17		17	DATE : 08 MAY 1982
18		18	
19		19	NAME RDSND
20		20	*INCLUDE( :F1:APUCMD.ASM )
21		21	*LIST
72		72	*LIST
73		73	PUBLIC SEND2, SEND4, READ2, READ4
74		74	CSEG
75		75	SEND4: PUSH H ; SAVE REGISTERS
76	E5	76	MVI L, 04H ; LOAD COUNTER FOR NUMBER OF BYTES
77	C5	77	L1: PUSH B ; SAVE B,C-REGISTERS
78	F5	78	PUSH PSW ; SAVE PROCESSOR STATUS WORD
79	0B	79	DCX B ; PREPARE MEMORY ADDRESS FOR DATA
80	03	80	L2: INX B ; GET DATA BYTE FROM MEMORY
81	0A	81	LDAX B ; AND SEND TO APU DATA STACK
82	D3FC	82	OUT APUD ; BYTE AND READY FOR TRANSFER TO APU
83	2D	83	DCR L ; LOOP BACK FOR MORE BYTES IF ANY
84	C20600	84	JNZ L2 ; RECOVER REGISTERS BEFORE RETURN
85	F1	85	POP PSW ; RETURN TO CALLING PROGRAMME
86	C1	86	POP B ; SAME AS SEND4 BUT WITH A BYTE COUNT OF 2
87	E1	87	POP H ;
88	C9	88	RET ;
89	E5	89	SEND2: PUSH H ; ROUTINE TO READ AND STORE 4 BYTES FROM APU
90	2E02	90	MVI L, 02H ; LOAD COUNT VALUE FOR BYTE TRANSFER
91	C30300	91	JMP L1 ; SAVE PROCESSOR STATUS WORD
92		92	
93		93	
94		94	
95	E5	95	READ4: PUSH H ; ROUTINE TO READ AND STORE 4 BYTES FROM APU
96	210404	96	LXI H, 0404H ; LOAD COUNT VALUE FOR BYTE TRANSFER
97	F5	97	L3: PUSH PSW ; SAVE PROCESSOR STATUS WORD
98	03	98	L4: INX B ; LOCATE MEMORY ADDRESS TO THE MOST SIGNIFICANT
99	25	99	DCR H ; BYTE AND READY FOR DATA TRANSFER
100	C21D00	100	JNZ L4 ;
101	DBFC	101	IN APUD ; GET DATA BYTE FROM APU DATA STACK
102	08	102	DCX B ; STORE THIS BYTE INTO MEMORY
103	02	103	STAX B ;
104	2D	104	DCR L ; DECREMENT COUNTER



LOC	OBJ	LINE	SOURCE STATEMENT
0027	C22200	C 105	JNZ L5 ; LOOP BACK IF NOT YET ZERO
002A	F1	106	POP PSW ; RECOVER REGISTERS BEFORE RETURNING TO CALLING
002B	E1	107	POP H ; PROGRAMME
002C	C9	108	RET ; RETURN TO CALLING SEGMENT
002D	E5	109	READ2: ; SAME AS READ4 BUT WITH 2 BYTES TO BE
002E	210202	110	LXI M,0202H ; TRANSFERRED
0031	C31C00	C 111	JMP L3
		112	END

PUBLIC SYMBOLS

READ2 C 002D HEAD4 C 001B SEND2 C 0012 SEND4 C 0000

EXTERNAL SYMBOLS

USER SYMBOLS

AC05	A 0006	AFUC	A 00FD	APUD	A 00FC	ASIN	A 0005	ATAN	A 0007	CHSD	A 0034	CHSF	A 0015
CHSS	A 0074	C05	A 0003	DADD	A 002C	DDIV	A 002F	DMUL	A 002E	DMUU	A 0036	DSUB	A 002D
EXP	A 000A	FADD	A 0010	FDIV	A 0013	FIXD	A 001E	FIXS	A 001F	FLTD	A 001C	FLTS	A 001D
FMUL	A 0012	FSUB	A 0011	L1	C 0003	I.2	C 0006	L3	C 001C	L4	C 001D	L5	C 0022
LN	A 0009	LOC	A 0008	LOD	A 000B	POPF	A 0018	POPS	A 0078	PTOD	A 0037	PTOF	A 0017
PT05	A 0077	PUP1	A 001A	PWR	A 000B	READ2	C 002D	READ4	C 0018	SADD	A 006C	SDIV	A 006F
SEND2	C 0012	SEND4	C 0000	SIN	A 0002	SMUL	A 005E	SMUU	A 0076	SORT	A 0001	SSUB	A 006D
TAN	A 0004	XCHD	A 0039	XCHF	A 0019	XCHS	A 0079						

ASSEMBLY COMPLETE, NO ERRORS

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LOC OBJ      LINE      SOURCE STATEMENT
1 *****
2 ROUTINES TO SEND/READ BYTES TO/FROM APU STACK
3 I.E. MEMORY <---> APU STACK
4 ON ENTRY: REGISTERS B,C CONTAIN STARTING MEMORY ADDRESS
5 WITH LEAST SIGNIFICANT BYTE FIRST
6 THE APU 8231 IS MEMORY MAPPED WITH ADDRESSES FC00H - FFFFH
7
8 S/R NAME      FUNCTION
9 -----
10 SEND2        SEND 2 BYTES
11 SEND4        SEND 4 BYTES
12 READ2        READ 2 BYTES
13 READ4        READ 4 BYTES
14
15 WRITTEN BY : C W CHUEN
16 DATE : 1 JUNE 1982
17 *****
18 NAME MRDSND
19 $INCLUDE(F1:APUCMD.ASM)
20 $NOLIST
21 $LIST
22 PUBLIC SEND2,SEND4,READ2,READ4
23 CSEC
24 SEND4:      PUSH -H ; SAVE H,L-REGISTERS
25 LXI H,100H*APUD+04H ; LOAD APU DATA STACK ADDRESS AND COUNT
26 ; VALUE FOR BYTE TRANSFER
27 L1:
28 PUSH B ; SAVE B,C-REGISTERS
29 DCX B ; SAVE PROCESSOR STATUS WORD
30 INX B ; GET MEMORY ADDRESS PREPARED
31 LDAX B ; GET BYTE FROM MEMORY AND
32 MOV M,A ; SEND BYTE TO APU DATA STACK
33 DCR L
34 JNZ L2
35 POP PSW ; RECOVER REGISTERS BEFORE RETURN
36 POP B
37 POP H
38 RET
39 SEND2:      PUSH H ; RETURN TO CALLING PROGRAMME
40 LXI H,100H*APUD+02H ; SAME AS SEND4 BUT WITH A BYTE COUNT OF 2
41 ; LOAD APU DATA STACK ADDRESS AND COUNT
42 ; VALUE FOR BYTE TRANSFER
43 JMP L1
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56 READ4:      PUSH H ; ROUTINE TO READ AND STORE 4 BYTES FROM APU
57 LXI H,0404H ; LOAD COUNT VALUE FOR BYTE TRANSFER
58 PUSH PSW ; SAVE PROCESSOR STATUS WORD
59 INX B ; LOCATE MEMORY ADDRESS TO THE MOST SIGNIFICANT
60 DCR H ; BYTE AND READY FOR DATA TRANSFER
61 JNZ L4
62 JNZ L4
63 MOV MVI H,APUD ; LOAD H-REGISTER WITH APU DATA STACK ADDRESS
64 MOV GET BYTE FROM APU DATA STACK AND
65 DCX B ; STORE DATA INTO MEMORY USING THE B,C-POINTER
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LOC	OBJ	LINE	SOURCE STATEMENT
0027	02	105	STAX B
0028	2D	106	DCR L
0029	C22500	107	JNZ L5
002C	F1	108	POP PSW
002D	E1	109	POP H
002E	C9	110	RET
002F	E5	111	READ2: PUSH H,0202H
0030	210202	112	LXI L3
0033	C31D00	113	JMP
		114	END

! DECREMENT BYTE COUNTER  
 ! LOOP BACK IF NOT YET ZERO  
 ! RECOVER REGISTERS BEFORE RETURNING TO CALLING  
 ! PROGRAMME  
 ! RETURN TO CALLING SEGMENT  
 ! SAME AS READ4 BUT WITH 2 BYTES TO BE TRANSFERRED

PUBLIC SYMBOLS  
 READ2 C 002F READ4 C 0019 SEND2 C 0012 SEND4 C 0000

EXTERNAL SYMBOLS

USER SYMBOLS

ACOS	A 0006	APUC	A 00FD	APUD	A 00FC	ASIN	A 0005	ATAN	A 0007	CHSD	A 0034	CHSF	A 0015
CH55	A 0074	COS	A 0003	DADD	A 002C	DDIV	A 002F	DMUL	A 002E	DMUU	A 0036	DSUB	A 002D
EXP	A 000A	FADD	A 0010	FDIV	A 0013	FIXD	A 001E	FIX5	A 001F	FLTD	A 001C	FLTS	A 001D
FMUL	A 0012	F5UB	A 0011	L1	C 0004	L2	C 0007	L3	C 001D	L4	C 001E	L5	C 0025
LN	A 0009	LOC	A 000B	POPD	A 0038	POPF	A 0018	POPS	A 007B	PTOD	A 0037	PTOF	A 0017
PTOS	A 0077	PUP1	A 001A	PWR	A 000B	READ2	C 002F	READ4	C 0019	SADD	A 006C	SDIV	A 006F
SEND2	C 0012	SEND4	C 0000	SIN	A 0002	SMUL	A 005E	SMUU	A 0076	SORT	A 0001	SSUB	A 006D
TAN	A 0004	XCHD	A 0039	XCHF	A 0019	XCHS	A 0079						

ASSEMBLY COMPLETE, NO ERRORS

APPENDIX F

FUNCTION APPROXIMATIONS USING  
THE CHEBYSHEV SERIES

It is extremely difficult if not impossible for a digital computer to evaluate the exact value of a mathematical function (e.g. sine, cosine), so various methods are used to obtain the approximation to a function. The use of power series is commonly adopted in these approximation methods. The Chebyshev series evaluation, which is developed from a power series expansion, is commonly used in digital computer library routines because it is easy and economical to use this series. The Intel-8231 arithmetic processing unit uses internally the Chebyshev series to evaluate the approximation to a mathematical function. In order to compare the function evaluation times in hardware and software, the Chebyshev series approximations to certain mathematical functions have been developed.

A Chebyshev polynomial, which is orthogonal, can be expressed in one of the two forms below:-

$$\begin{aligned}
 \text{a) } T_n(x) &= \text{Cos}(n\text{Cos}^{-1}x) \\
 n &= 0, 1, 2, 3, \dots & \text{(F.1)} \\
 -1 &\leq x \leq 1
 \end{aligned}$$

and

$$\begin{aligned}
 \text{b) } T_n^*(x) &= T_n(2x - 1) \\
 &= \text{Cos}\{n\text{Cos}^{-1}(2x - 1)\} & \text{(F.2)} \\
 n &= 0, 1, 2, 3, \dots \\
 0 &\leq x \leq 1
 \end{aligned}$$

Any finite range can be transformed to the range  $(-1 \leq x \leq 1)$  by a linear change in the independent variable, therefore (F.1) is normally used. However, if it is easier and more natural in transforming to the range  $(0 \leq x \leq 1)$ , e.g. the function  $\ln(1+x)$ , then (F.2) is used.

It can readily be seen from (F1) that the Chebyshev polynomials have got the recursive property (F3) which is easy to implement in a digital calculating machine.

$$\begin{aligned}
 T_{r+1}(x) &= 2xT_r(x) - T_{r-1}(x) \\
 r &= 1, 2, 3 \dots \dots \quad (F3) \\
 T_0(x) &= 1 \\
 T_1(x) &= x
 \end{aligned}$$

The first ten explicit expressions in x for these polynomials are listed in Table F1.

TABLE F1 THE FIRST TEN CHEBYSHEV  
POLYNOMIALS  $T_n(x)$

n	Term	Explicit Expression in x
0	$T_0(x)$	1
1	$T_1(x)$	x
2	$T_2(x)$	$2x^2 - 1$
3	$T_3(x)$	$4x^3 - 3x$
4	$T_4(x)$	$8x^4 - 8x^2 + 1$
5	$T_5(x)$	$16x^5 - 20x^3 + 5x$
6	$T_6(x)$	$32x^6 - 48x^4 + 18x^2 - 1$
7	$T_7(x)$	$64x^7 - 112x^5 + 56x^3 - 7x$
8	$T_8(x)$	$128x^8 - 256x^6 + 160x^4 - 32x^2 - 1$
9	$T_9(x)$	$256x^9 - 576x^7 + 432x^5 - 120x^3 - 9x$

A continuous function in the region  $(-1 \leq x \leq 1)$  can be expressed in a Chebyshev polynomial:

$$\begin{aligned}
 f(x) &= a_0 + a_1T_1(x) + a_2T_2(x) + \dots \\
 &= \sum_{r=0}^{\infty} a_r T_r(x) \quad (F4)
 \end{aligned}$$

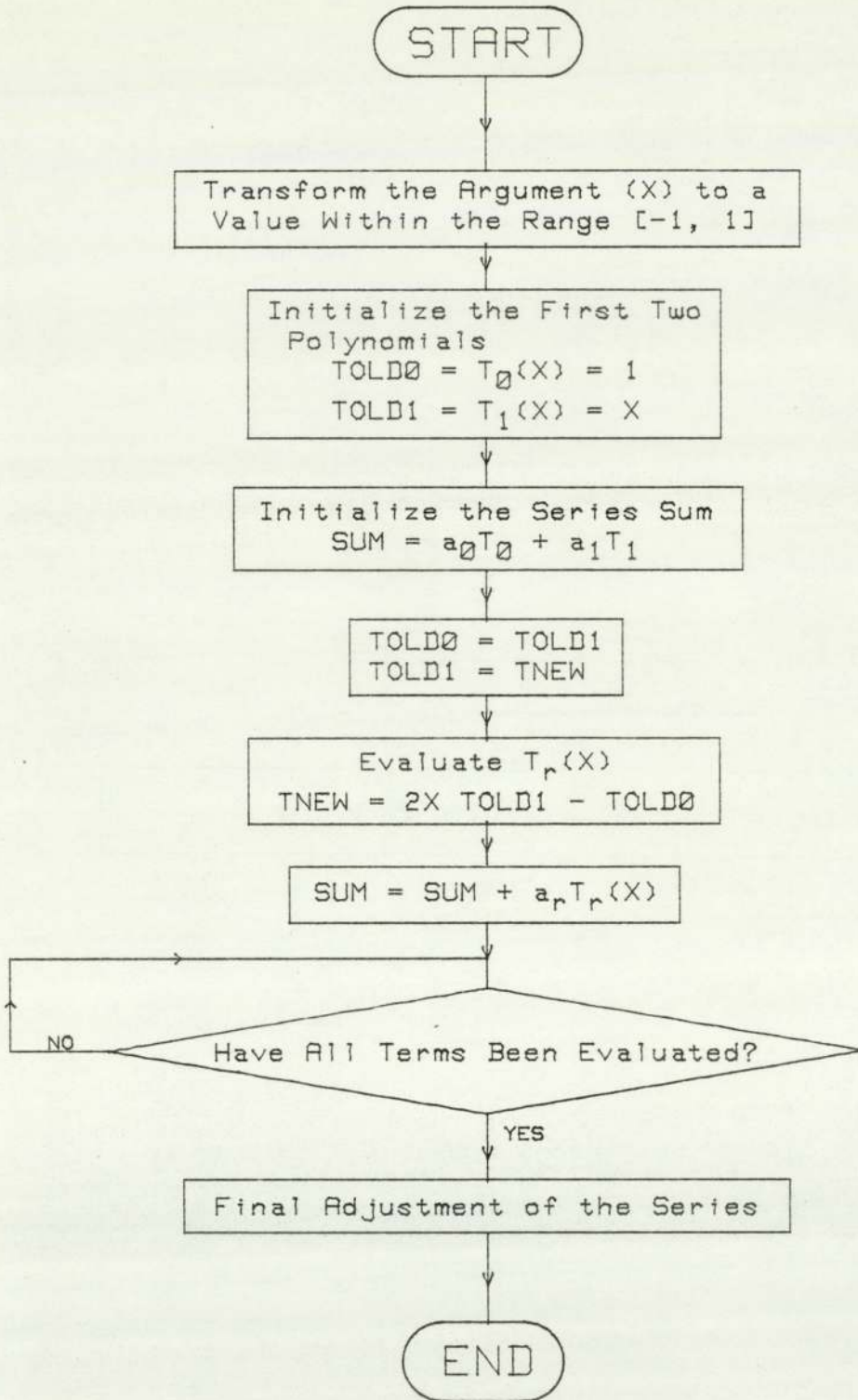


FIG F.1 Flow Diagram for the Algorithm in Evaluating the Chebyshev Series

In fact, (F4) represents the Fourier cosine series expansion of the function  $f(\cos \theta)$  and is periodic. This periodic function converges rapidly, the coefficients are in general readily evaluated and the approximations of any specified accuracy can be obtained by mere truncation of the series. An algorithm in evaluating the Chebyshev series can be obtained from (F3), the flow diagram for this algorithm can be represented in Figure F1. The coefficients,  $a_r$ , of the Chebyshev series are taken from the Mathematical Tables by Clenshaw [46].

The sine function is used as an example to illustrate how this algorithm works. This series evaluation uses the Intel floating point arithmetic library (FPAL) and works in binary floating point numbers.

The following FPAL routines are involved:

<u>Name</u>	<u>Function</u>
FSET	Set up the arithmetic library routines.
FLOAD	Load the binary number from the memory to the floating point accumulator (FAC).
FSTOR	Store the binary number in the FAC to the memory.
FMUL	Multiply the binary number in the FAC to that in the memory and put the result in the FAC.
FDIV	Divide the binary number in the FAC by that in the memory and put the result in the FAC.
FADD	Add the binary number in the FAC by that in the memory and put the result in the FAC.
FSUB	Subtract the binary number in the FAC by that in the memory and put the result in the FAC.
FZTST	Check whether the number in the FAC is less than, equal to or greater than zero.



FNEG      Change the sign of the number in the FAC.  
 FCMPR     Compare the number in the FAC to that in the  
 memory.

The following sub-routines are constructed in evaluating  
 the Chebyshev series,

<u>Name</u>	<u>Function</u>	<u>Code Lines in CHEBY</u>
INCHEB	Initialise the first two terms in the Chebyshev series.	20 - 43
NEWX	Evaluate the new term by F3. $T_{r+1}(x) = 2xT_r(x) - T_{r-1}(x)$ .	44 - 59
MOVE	Move $T_r$ to $T_{r-1}$ . $T_{r+1}$ to $T_r$ .	60 - 73
SUMCH	Sum the Chebyshev series. $SUM = SUM + FAC$ .	74 - 79

The program for evaluating the function,  $\sin(x)$ , is also listed. The argument is first reduced to a value within the range  $[-2\pi, 0]$  and then transformed to a value within the range  $[-1, 1]$ . The approximation is then carried out by the algorithm in Figure F1, with the following Chebyshev series:

$$\sin \frac{1}{2}\pi x = x \sum_{r=0}^{\infty} a_{2r} T_{2r}(x) \quad \text{where } -1 \leq x \leq 1$$

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	ROUTINE TO CALCULATE CHEBYSHEV SERIES *****
		3	ON ENTRY: B,C POINT TO FPR
		4	PSW HAS BEEN PUSHED
		5	ONTO STACK BEFORE CALL-
		6	ING THIS ROUTINE *****
		7	*****
		8	CHEBY
		9	PUBLIC INCHEB,NEWX,MOVE,SUMCH
		10	EXTRN ONE,FSTOR,FLOAD,FMUL,FSUB,FADD
		11	*INCLUDE(:F1:CHEB.STK)
0018		= 12	XSTOR EQU 24D ; DEFINE XSTOR TO BE 24D
0014		= 13	TWOX EQU XSTOR - 4
0010		= 14	TOLD0 EQU TWOX - 4
000C		= 15	TOLD1 EQU TOLD0 - 4
0008		= 16	TNEW EQU TOLD1 - 4
0004		= 17	SUM EQU TNEW - 4
		= 18	*LIST
		19	CSEC
0000	211800	20	INCHEB: LXI H,XSTOR ; ROUTINE TO INITIALIZE FIRST
0003	39	21	DAD SP ; THREE TERMS IN THE CHEBYSHEV
0004	EB	22	XCHG ; SERIES
0005	CD0000	23	FSTOR FSTOR ; ARGUMENT (X) IS IN FAC WHEN
0008	210C00	24	LXI H,TOLD1 ; INCHEB IS CALLED
000B	39	25	DAD SP
000C	EB	26	XCHG
000D	CD0000	27	FSTOR ; T(1) = X
0010	211000	28	LXI H,16D ; FORM 2*ARGU BY ADDING ONE TO
0013	09	29	DAD B ; THE EXPONENT BYTE OF FAC
0014	7E	30	MOV A,M ; THIS BYTE IS 16D BYTE
0015	3C	31	INR A ; THE LOW BYTE OF FPR
0016	77	32	MOV M,A ; STORE THIS INTO TWOX
0017	211400	33	LXI H,TWOX ; STORE THIS INTO TWOX
001A	39	34	DAD SP
001B	EB	35	XCHG
001C	CD0000	36	CALL FSTOR
001F	110000	37	LXI D,ONE ; MAKE T(0) = 1
0022	CD0000	38	CALL FLOAD
0025	211000	39	LXI H,TOLD0
0028	39	40	DAD SP
0029	EB	41	XCHG
002A	CD0000	42	CALL FSTOR
002D	C9	43	RET
002E	211400	44	LXI H,TWOX
0031	39	45	DAD SP
0032	EB	46	XCHG
0033	CD0000	47	CALL FLOAD ; LOAD 2*ARGU INTO FAC
0036	210C00	48	LXI H,TOLD1 ; PERFORM 2*X*(N)
0039	39	49	DAD SP
003A	EB	50	XCHG
003B	CD0000	51	CALL FMUL
003E	211000	52	LXI H,TOLD0 ; PERFORM 2*X*(N) - T(N-1)
0041	39	53	DAD SP
0042	EB	54	XCHG

LOC	OBJ	I.LINE	SOURCE STATEMENT
0043	CD0000	E 55	CALL FSUB
0046	210800	E 56	LXI H, TNEW ; STORE THIS T(N+1) INTO TNEW
0049	39	E 57	DAD SP
004A	EB	E 58	XCHG
004B	CD0000	E 59	CALL FSTOR
004E	210C00	E 60	LXI H, TOLD1 ; MOVE TOLD1 TO TOLD0
0051	39	E 61	DAD SP
0052	EB	E 62	XCHG
0053	CD0000	E 63	CALL FLOAD
0056	211000	E 64	LXI H, TOLD0
0059	39	E 65	DAD SP
005A	EB	E 66	XCHG
005B	CD0000	E 67	CALL FSTOR
005E	210800	E 68	LXI H, TNEW ; MOVE TNEW TO TOLD1
0061	39	E 69	DAD SP
0062	EB	E 70	XCHG
0063	CD0000	E 71	CALL FLOAD
0066	210C00	E 72	LXI H, TOLD1
0069	C32800	C 73	JMPL LIN1
006C	210400	C 74	JUMPL: LXI H, SUM ; PERFORM SUMMATION FOR SERIES
006F	39	E 75	DAD SP ; AND STORE SUM IN FAC AND SUM
0070	EB	E 76	XCHG
0071	CD0000	E 77	CALL FADD
0074	CD0000	E 78	CALL FSTOR
0077	C9	E 79	RET
		E 80	END

PUBLIC SYMBOLS		MOVE	C 004E	NEWX	C 002E	SUMCH	C 006C		
INCHEB	C 0000							ONE	E 0000
EXTERNAL SYMBOLS		FLOAD	E 0000	FMUL	E 0000	FSTOR	E 0000	FSUB	E 0000
FADD	E 0000								
USER SYMBOLS		FLOAD	E 0000	FMUL	E 0000	FSTOR	E 0000	FSUB	E 0000
FADD	E 0000								
LIN1	C 002B	MOVE	C 004E	NEWX	C 002E	ONE	E 0000	SUM	A 0004
TOLD0	A 0010	TOLD1	A 000C	TWOX	A 0014	XSTOR	A 001B		
ASSEMBLY COMPLETE, NO ERRORS									

JUMPL C 0069  
 TNEW A 000B

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	!# SEGMENTS CONTAIN CONSTANTS :
		3	!# ONE, PI, 2PI, PI/2 ,ZERO
		4	!#*****
		5	NAME CONST
		6	CSEC
		7	PUBLIC ONE,ZERO,PI,TWOPI,PI2
0000	00	8	ZERO: DB 000H,000H,000H,000H ; BFP FOR 0
0001	00		
0002	00		
0003	00		
0004	00	9	ONE: DB 000H,000H,080H,03FH ; BFP FOR 1
0005	00		
0006	80		
0007	3F		
0008	DB		
0009	0F	10	PI: DB 0DBH,00FH,049H,040H ; BFP FOR PI
000A	49		
000B	40		
000C	DB		
000D	0F	11	TWOPI: DB 0DBH,00FH,0C9H,040H ; BFP FOR 2PI
000E	C9		
000F	40		
0010	DB		
0011	0F		
0012	C9	12	PI2: DB 0DBH,00FH,0C9H,03FH ; BFP FOR PI/2
0013	3F		
		13	END

PUBLIC SYMBOLS  
 ONE C 0004 PI C 0008 PI2 C 0010 TWOPI C 000C ZERO C 0000

EXTERNAL SYMBOLS

USER SYMBOLS  
 ONE C 0004 PI C 0008 PI2 C 0010 TWOPI C 000C ZERO C 0000

ASSEMBLY COMPLETE, NO ERRORS

1515-11 8080/8085 MACRO ASSEMBLER, V3.0 SIN PAGE 1  
 SINE ROUTINE

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	A ROUTINE FOR THE SINE FUNCTION
		3	THIS REQUIRES: FPAL.LIB
		4	MYLIB
		5	CHEBY.OBJ
		6	ON ENTRY: B,C CONTAIN FPR ADDRESS
		7	D,C CONTAIN ARGUMENT ADDRESS
		8	ON EXIT: FAC CONTAINS THE VALUE OF
		9	SINE(ARGUMENT)
		10	THIS REQUIRES AN EXTRA 24D-BYTE STACK
		11	FOR TEMPORARY STORAGE OF RESULTS
		12	*****
		13	SIN
		14	EXTRN FLOAD,FSTOR,FMUL,FCLR,FNEG,FZTST,FADD
		15	EXTRN FSUB,FDIV,ONE,PI,TWOPI,INCHEB,NEWX,SUMCH
		16	PUBLIC SIN,SINE1
		17	\$INCLUDE(1:CHEB.STK)
001B		= 18	XSTOR EQU 24D ; DEFINE XSTOR TO BE 24D
0014		= 19	TWOX EQU XSTOR - 4
0010		= 20	TOLD0 EQU TWOX - 4
000C		= 21	TOLD1 EQU TOLD0 - 4
0008		= 22	TNEW EQU TOLD1 - 4
0004		= 23	SUM EQU TNEW - 4
		= 24	\$LIST
		25	CSEG
0000	F5	26	SIN: PUSH PSW ; SAVE REGISTERS ON ENTRY
0001	C5	27	B PUSH B
0002	D5	28	D PUSH D
0003	E5	29	H PUSH H
0004	21EBFF	30	LX1 H,-24D ; LOCATE STACK POINTER TO 24D
0007	39	31	DAD SP ; BYTE BELOW THE CURRENT
0008	F9	32	SPHL ; LOCATION
0009	CD0000	E 33	FLOAD ; LOAD ARGUMENT INTO FAC
000C	CD0000	E 34	FZTST ; CHECK IF ARGUMENT IS -VE, 0 OR +VE
000F	F5	35	PUSH PSW ; SAVE STATUS WORD FOR LATER USE
0010	17	36	RAL X150 ; JUMP IF ARGUMENT IS ZERO
0011	DACA00	C 37	JC XGT0 ; JUMP IF ARGUMENT IS POSITIVE
0014	17	38	RAL FNEG ; TAKE ABSOLUTE VALUE OF ARGUMENT
0015	DA1500	C 39	JC D,TWOPI ; REDUCE TO RANGE (-2PI, 0)
0018	CD0000	E 40	CALL FSUB
001B	100000	E 41	LX1 FZTST
001E	CD0000	E 42	MINUS: CALL
0021	CD0000	E 43	CALL
0024	17	44	RAL X150 ; JUMP IF ARGUMENT IS MULTIPLE OF 2PI
0025	DACA00	C 45	JC MINUS ; NEEDS FURTHER REDUCTION
0028	17	46	RAL D,PI ; FORM -(Y + PI)/PI
0029	DA1E00	C 47	JC FADD
002C	100000	E 48	LX1 FZTST ; CHECK IF ARGUMENT IS MULTIPLE OF PI
002F	CD0000	E 49	CALL
0032	CD0000	E 50	CALL
0035	17	51	RAL X150
0036	DACA00	C 52	JC FDIV
0039	CD0000	E 53	CALL
003C	CD0000	E 54	CALL FNEG

LOC	OBJ	LINE	SOURCE STATEMENT
003F	CD0000	E 55	CALL INCHEB ; INITIALIZE THE CHEBYSHEV SERIES
0042	11D800	C 56	LXI D,A0 ; INITIALIZE THE SUM
0045	CD0000	E 57	FLOAD
0048	210200	E 58	LXI H,SUM-2
004B	39	59	DAD SP
004C	EB	60	XCHG
004D	CD0000	E 61	CALL FSTOR
0050	CD0000	E 62	CALL NEWX
0053	11DC00	C 63	LXI D,A2
0056	CD0000	E 64	CALL FMUL
0059	CD0000	E 65	CALL SUMCH
-	-	66	IRP X,<A4,A6,AB,A10,A12,A14>
-	-	67	CALL NEWX
-	-	68	CALL NEWX
-	-	69	LXI D,X
-	-	70	CALL FMUL
-	-	71	CALL SUMCH
-	-	72	ENDM
005C	CD0000	E 73+	CALL NEWX
005F	CD0000	E 74+	CALL NEWX
0062	11E000	C 75+	LXI D,A4
0065	CD0000	E 76+	CALL FMUL
0068	CD0000	E 77+	CALL SUMCH
006B	CD0000	E 78+	CALL NEWX
006E	CD0000	E 79+	CALL NEWX
0071	11E400	C 80+	LXI D,A6
0074	CD0000	E 81+	CALL FMUL
0077	CD0000	E 82+	CALL SUMCH
007A	CD0000	E 83+	CALL NEWX
007D	CD0000	E 84+	CALL NEWX
0080	11E800	C 85+	LXI D,AB
0083	CD0000	E 86+	CALL FMUL
0086	CD0000	E 87+	CALL SUMCH
0089	CD0000	E 88+	CALL NEWX
008C	CD0000	E 89+	CALL NEWX
008F	11EC00	C 90+	LXI D,A10
0092	CD0000	E 91+	CALL FMUL
0095	CD0000	E 92+	CALL SUMCH
0098	CD0000	E 93+	CALL NEWX
009B	CD0000	E 94+	CALL NEWX
009E	11F000	C 95+	LXI D,A12
00A1	CD0000	E 96+	CALL FMUL
00A4	CD0000	E 97+	CALL SUMCH
00A7	CD0000	E 98+	CALL NEWX
00AA	CD0000	E 99+	CALL NEWX
00AD	11F400	C 100+	LXI D,A14
00B0	CD0000	E 101+	CALL FMUL
00B3	CD0000	E 102+	CALL SUMCH
00B6	211600	E 103	LXI H,XSTOR-2
00B9	39	104	DAD SP
00BA	EB	105	XCHG
00BB	CD0000	E 106	CALL FMUL
00BE	F1	107	CALL PSW ; CHECK IF ADJUSTMENT IS REQUIRED
00BF	17	108	POP RAL
00C0	17	109	RAL

LOC	OBJ	LINE	SOURCE STATEMENT
00C1	DACE00	C 110	NOADJ JC
00C4	CD0000	E 111	FNEG CALL ; SINCE SIN(-X) = -SIN(X)
00C7	C3CE00	C 112	NOADJ JMP
00CA	CD0000	E 113	FCLR CALL ; CLEAR FAC SINCE SIN(0) = 0
00CD	F1	114	PSW H,24D ; RECOVER STACK POINTER LOCATION
00CE	211800	115	NOADJ: LXI ; RECOVER REGISTERS BEFORE RETURNING
00D1	S9	116	DAD SP
00D2	F9	117	SPHL
00D3	E1	118	POP POP
00D4	D1	119	POP POP
00D5	C1	120	POP POP
00D6	F1	121	POP POP
00D7	C9	122	RET
00D8	BD	123	DB ; RETURN TO CALLING PROGRAMME
00D9	7B		0BDH, 07BH, 0ACH, 03FH ; 2.69505262934798034246/2
00DA	AC		
00DB	3F		
00DC	61	124	A2: DB 061H, 03EH, 0C7H, 0BFH ; -1.55659125662896931308
00DD	3E		
00DE	C7		
00DF	BF		
00E0	A6	125	A4: DB 0A6H, 01AH, 064H, 03EH ; 0.22275791181701117152
00E1	1A		
00E2	64		
00E3	3E		
00E4	7E	126	A6: DB 07EH, 0BAH, 068H, 08CH ; -0.01419317414853727256
00E5	8A		
00E6	68		
00E7	EC		
00E8	B4	127	AB: DB 084H, 031H, 006H, 03AH ; 0.00051190727455411454
00E9	31		
00EA	06		
00EB	9A		
00EC	37	128	A10: DB 037H, 0BAH, 047H, 0E7H ; -0.00001189350465334208
00ED	BA		
00EE	47		
00EF	B7		
00F0	A5	129	A12: DB 0A5H, 03DH, 04FH, 034H ; 0.00000019300803606380
00F1	3D		
00F2	4F		
00F3	34		
00F4	5C	130	A14: DB 05CH, 0EBH, 01EH, 0B1H ; -0.0000000231258105781
00F5	EB		
00F6	1E		
00F7	B1		
00F8	BC	131	A16: DB 0ECH, 064H, 0BBH, 02DH ; 0.00000000002130417439
00F9	64		
00FA	BB		
00FB	2D	132	END

PUBLIC SYMBOLS  
SIN C 0000 SINEI C 000C

SINE ROUTINE

EXTERNAL SYMBOLS

FADD E 0000 FCLR E 0000 FDIV E 0000 FLOAD E 0000 FMUL E 0000 FSTOR E 0000  
 FSUB E 0000 FZTST E 0000 INCHEB E 0000 NEWX E 0000 ONE E 0000 FNEG E 0000 FSTOR E 0000  
 TWOP1 E 0000

USER SYMBOLS

A0 C 00DB A10 C 00EC A12 C 00F0 A14 C 00F4 A16 C 00FB A2 C 00DC A4 C 00E0  
 A6 C 00E4 A8 C 00E8 FADD E 0000 FCLR E 0000 FZTST E 0000 FLOAD E 0000 FDIV E 0000 FMUL E 0000  
 FNEG E 0000 FSTOR E 0000 FSUB E 0000 PI E 0000 SIN C 0000 TOLD1 A 0010 TOLD0 A 0010 TOLD1 A 000C  
 NOADJ C 00CE ONE E 0000 PI E 0000 SIN C 0000 TWOP1 E 0000 TWOX A 0014 XCT0 C 001B  
 TNEW A 0008 TOLD0 A 0010 TOLD1 A 000C TWOP1 E 0000 TWOX A 0014 XCT0 C 001B X150 C 00CA  
 XSTOR A 0018

ASSEMBLY COMPLETE, NO ERRORS



APPENDIX G

LISTING OF THE ASSEMBLY LANGUAGE PROGRAMS  
FOR THE BOOM POSITIONAL CONTROL BY SIMPLE  
COMPARISON WITH SLOW SPEED LIMITATION

LOC	OBJ	LINE	SOURCE STATEMENT
		1	*****
		2	THE INITIALIZATION PROGRAM FOR BOOM POSITION CONTROL
		3	*****
		4	HARDWARE INTERRUPTS ARE USED IN THE FOLLOWING WAYS:
		5	*****
		6	INTERRUPT
		7	-----
		8	TRAP
		9	THE TUNNEL PROFILE HAS BEEN
		10	DEFINED BY THE OPERATOR
		11	TO CUT THE TUNNEL PROFILE
		12	TO EXCAVATE THE CAVITY
		13	TO MOVE THE BOOM
		14	*****
		15	NAME CNTRL
		16	PUBLIC STKPTR,CUTSTP,SORTEL,INTMSK,SIGNAL
		17	*****
		18	THE INITIALIZATION
		19	*****
		20	ASEC
0000		21	ORG 00000H
		22	START:
		23	*****
0000	310000	24	LXI SP,STKPTR
0003	3E1F	25	MVI A,01FH
0005	6F	26	MOV L,A
0006	2608	27	MVI H,08H
0008	30	28	COLDST: SIM
0009	220000	29	SHLD INTMSK
		30	*****
		31	DECLARE THE ABOVE INFORMATION
		32	THE INTERRUPT MASK AT INTMSK
000C	3EFF	33	MVI A,OFFH
000E	D300	34	OUT 00H
		35	*****
0010	D302	36	OUT 02H
		37	*****
		38	DECLARE ALL THE 8 BITS OF PORT 0 TO BE OUTPUTS
		39	BY SENDING OFFH TO ITS D.D.R. (PORT 2)
		40	PORT 0 IS USED FOR CONTROLLING THE DIRECTION VALVES
		41	CLOSE A VALVE BY SENDING AN ONE TO THE ASSOCIATE PORT
0012	3E67	42	MVI A,067H
0014	D301	43	OUT 01H
		44	*****
		45	THIS IS 0110 0111 B

LOC	OBJ	LINE	SOURCE STATEMENT
0016	3E7F	44	RVI A,07FH
0018	D303	45	OUT 03H
001A	321800	46	STA DDR01
		47	PORT 1 IS USED IN THE FOLLOWING WAY:
		48	PB0, PB1 --- FAST/SLOW CONTROL (DEFAULTED TO 1)
		49	--- 1 FOR SLOW MOVE
		50	--- 0 FOR FAST MOVE
		51	PB2 --- START CUTTER (DEFAULTED TO 1)
		52	PB3 --- STOP CUTTER CUTTER (DEFAULTED TO 0)
		53	PB4 --- RESET THE APU (DEFAULTED TO 0)
		54	THIS IS HIGH ACTIVE
		55	PB5 --- 0 TO HOLD THE OTHER SYSTEM
		56	--- 1 NORMAL OPERATION (DEFAULT CONDITION)
		57	PB6 --- 0 TO ENABLE THE INERFACE BUFFERS
		58	--- 1 NORMAL OPERATION (DEFAULT CONDITION)
		59	PB7 --- HOLDA FROM THE OTHER SYSTEM (INPUT)
		60	PATTERN: 1000000 --- I-INPUT, 0-OUTPUT
		61	SET THE INITIAL PATTERN FOR PORT 1: 01100111B
		62	ROOM SLOW MOVE MODE IS THE DEFAULT SETTING
		63	DATA ARE NOT REQUIRED FROM THE OTHER SYSTEM
		64	THE APU IS NOT RESETTED
		65	THE CUTTER IS STOPPED
		66	SET THE INITIAL SORT PROFILE FLAG TO ZERO
001D	97	67	SUB A
001E	320200	67	STA SORTFL
0021	75	68	HLT
		69	WAIT FOR TRAP INTERRUPT TO INDICATE THAT THE TUNNEL PROFILE HAS BEEN DEFINED BY THE OPERATOR
		70	HARDWARE TRAP INTERRUPT ENTRY POINT
0024	C52B03	71	TRAP: ORC 00024H
002C		72	JMP RST45
002C	C3B500	72	ORC 0002CH
0034		73	JMP PROFIL
0034	C31802	74	ORC 00094H
0038		75	JMP CAVITY
0038	C35500	76	ORC 00058H
003C		77	JMP RST5B
003C	C3DC02	78	ORC 0003CH
		79	JMP MOVE
		80	Hardware RST 7.5 INTERRUPT ENTRY POINT
			Hardware RST 5.5 INTERRUPT ENTRY POINT
			Hardware RST 6.5 INTERRUPT ENTRY POINT
			Hardware INTR INTERRUPT ENTRY POINT
			Hardware RST 7.5 INTERRUPT ENTRY POINT

LOC	OBJ	LINE	SOURCE STATEMENT
81		81	*****
82		82	ROUTINES TO SET UP AND DIS-ENGAGE THE DATA LINK BETWEEN
83		83	THE MAIN CONTROL SYSTEM AND THE DATA CAPTURING SYSTEM
84		84	NAME
85		85	FUNCTION
86		86	-----
87		87	SETLNK SET UP THE DATA LINK BETWEEN THE TWO SYSTEMS
88		88	OFFLNK DIS-ENGAGE THE DATA LINK
89		89	BIT 7 OF PORT 1 IS USED FOR THE HOLD ACKNOWLEDGE SIGNAL
90		90	BIT 6 OF PORT 1 IS USED FOR ENABLING THE BUFFERS
91		91	BIT 5 OF PORT 1 IS USED FOR HOLDING THE DATA SYSTEM
92		92	THEREFORE BIT 7 OF PORT 1 IS AN INPUT
93		93	BITS 5 AND 6 OF PORT 1 ARE OUTPUTS
94		94	THE PROCESSOR STATUS WORD WILL BE DESTROYED
95		95	*****
96		96	PUBLIC SETLNK,OFFLNK
97		97	ORG 00040H
98		98	SETLNK:
99		99	
100		100	! A ROUTINE TO SET UP THE INTERFACE CONDITION FOR DATA TRANSFER
101	DE01	101	! FROM THE DATA CAPTURING SYSTEM
102	E6DF	102	! THE PROCESSOR STATUS WORD WILL BE DESTROYED
103		103	! GET CURRENT SETTING OF PORT 1
104		104	! SET THE APPROPRIATE BIT VALUES FOR HOLDING THE ADDRESS AND DATA
105	D301	105	! BUSES OF THE SECOND SYSTEM BY ANDING IT WITH 11011111B
106	D801	106	! THE PATTERN IS: XX0XXXX B
107	17	107	! OUTPUT THIS PATTERN TO PORT 1
108	D24600	108	! READ THE PORT 1 SIGNAL
109	1F	109	! CHECK WHETHER THE SECOND SYSTEM HAS ACKNOWLEDGED THE HOLD SIGNAL
110	E6BF	110	! KEEP CHECKING IF NOT YET ACKNOWLEDGED
111	D301	111	! RECOVER THE ORIGINAL PORT 1 READING
112	C9	112	! MAKE BIT 6 ZERO BY ANDING IT WITH 10111111 B
113		113	! SEND IT TO PORT 01
114		114	! RETURN TO THE CALLING PROGRAM
115	DE01	115	! A ROUTINE TO DIS-ENGAGE THE SECOND SYSTEM
116	F660	116	! THE PROCESSOR STATUS WORD WILL BE DESTROYED
117	D301	117	! GET CURRENT PORT 1 BYTE VALUE
118	D301	118	! SET THE 'DIS-ENGAGE' BIT-PATTERN: X11XXXXX B
119	C9	119	! BY ORING IT WITH 01100000 B
120		120	! OUTPUT THIS PATTERN
121		121	! MODE CHANGE IS REQUIRED
122	DB01	122	! SET THE SPEED TO SLOW MODE
123	F603	123	! PATTERN: XXXX XX11
124	D301	124	! CLOSE ALL THE DIRECTIONAL VALVES
125	3EFF	125	! CHECK IF THE PROFILE DATA HAVE BEEN SORTED
126	D300	126	! SORT AND EXPAND THEM IF NOT
127	3A0200	127	! CHECK IF THE PROFILE DATA HAVE BEEN SORTED
128	17	128	! SORT AND EXPAND THEM IF NOT
129	D22B03	129	! OTHERWISE, RESET THE INTERRUPT MASK
130	3E1B	130	! RE-LOCATE THE STACK POINTER
131	30	131	! WAIT FOR THE SIGNAL TO INITIATE AN OPERATION
132	320000	132	
133	310000	133	
134	F6	134	
135	4 76	135	

LOC	OBJ	LINE	SOURCE STATEMENT
		156	*INCLUDE(1F1:TRNPF1, SRC)
		137	*****
		138	A ROUTINE TO TRANSFER PROFILE DATA FROM THE SOURCE TO
		139	THE DESTINATION
		140	THE DATA MUST BE STORED IN CONTIGUOUS MEMORIES FOLLOWING
		141	THE TWO BYTES SPECIFYING THE LAST ADDRESS OF DATA
		142	I.E. LAST DATA ADDRESS, DATA
		143	ON ENTRY: REGISTERS B,C POINT TO THE SOURCE DATA BLOCK
		144	REGISTERS D,E POINT TO THE DESTINATION BLOCK
		145	TRANSFER COMPLETED WITH THE LAST DATA ADDRESS
		146	AT THE DESTINATION BLOCK STORED IN THE TWO
		147	BYTES BELOW THE DATA AS IN THE SOURCE
		148	*****
		149	PUBLIC TRNPF1
0075	F5	150	TRNPF1: PUSH PSW ; SAVE THE REGISTERS ON ENTRY
0076	E5	151	PUSH H
0077	C5	152	PUSH B
0078	D5	153	PUSH D
0079	69	154	MOV L,C ; GET THE LAST ADDRESS OF THE SOURCE DATA
007A	60	155	MOV H,B ; AND PUT IT IN THE REGISTERS B,C
007B	4E	156	MOV C,M
007C	23	157	INX H
007D	46	158	MOV B,M
007E	13	159	INX D
007F	23	160	INX H
0080	13	161	INX D
0081	7E	162	MOV A,M ; ADJUST THE ADDRESSES OF BOTH THE DESTINATION AND SOURCE DATA
0082	12	163	STAX D ; MOVE THE SOURCE DATA TO THE ACCUMULATOR
0083	7C	164	MOV A,H ; THEN STORE IT IN THE DESTINATION MEMORY
0084	B8	165	CMP B ; CHECK IF THE LAST DATA HAS BEEN TRANSFERRED
0085	C2F00	166	JNZ LOOP1 ; LOOP BACK IF NOT
0086	7D	167	MOV A,L
0089	B9	168	CMP C
008A	DA700	169	JC LOOP1 ; LOOP BACK IF NOT
008D	E1	170	POP H ; GET THE POINTER FOR STORING THE LAST DATA ADDRESS AT DESTINATION
008E	E5	171	PUSH H
008F	73	172	MOV M,E
0090	23	173	INX H
0091	72	174	MOV M,D
0092	D1	175	POP D
0093	C1	176	POP B
0094	E1	177	POP H
0095	F1	178	POP PSW
0096	C9	179	RET ; RECOVER THE REGISTERS
		180	*EJECT

LOC	OBJ	LINE	SOURCE STATEMENT
		181	* INCLUDE (:F1:TRANSF.SRC)
		182	*****
		183	***** A ROUTINE TO TRANSFER DATA FROM SOURCE TO DESTINATION *****
		184	***** FOR X BYTES *****
		185	***** ON ENTRY: REGISTERS B,C --- STARTING SOURCE ADDRESS *****
		186	***** REGISTERS D,E --- STARTING DESTINATION ADDRESS *****
		187	***** ON EXIT: REGISTERS HAVE BEEN PRESERVED *****
		188	*****
		189	PUBLIC TRANSF *****
0097	F5	191	TRANSF: PUSH PSW ; PUSH THE BYTE COUNTER ONTO THE STACK BEFORE CALLING THIS ROUTINE
		192	***** ; MAXIMUM NUMBER OF BYTES TRANSFERRED: 0FFFFH (64K) *****
		193	***** ; SAVE THE REGISTERS *****
0098	E5	194	PUSH H
0099	C5	195	PUSH B
009A	D5	196	PUSH D
009B	210A00	197	LXI H,10D ; ADJUST THE STACK POINTER TO GET THE BYTE COUNT
009E	39	198	DAD SP ; THIS BYTE COUNTER IS 10D BYTE ABOVE THE CURRENT STACK POINTER
009F	5E	199	MOV E,M ; TRANSFER THIS BYTE VALUE (LOW ORDER) TO REGISTER E
00A0	23	200	INX H ; GET THE HIGH ORDER BYTE AND TRANSFER TO REGISTER D
00A1	56	201	MOV D,M ; GET THE STARTING DESTINATION ADDRESS
00A2	E1	202	POP H
00A3	E5	203	PUSH H
00A4	EB	204	XCHC ; PUT THIS ADDRESS BACK TO REGISTERS D,E AND THE BYTE COUNTER TO H,L
00A5	0A	205	LDAX B ; GET THE BYTE VALUE FROM SOURCE
00A6	12	206	STAX D ; STORE THIS TO THE DESTINATION
00A7	03	207	INX B ; INCREMENT THE ADDRESSES
00A8	15	208	INX D ; INCREMENT THE COUNTER
00A9	2B	209	DCX H ; DECREMENT THE COUNTER
00AA	97	210	SUB A ; CLEAR THE ACCUMULATOR
00AB	B4	211	ORA H
00AC	B5	212	ORA L
00AD	C2A500	213	JNZ LOOP3 ; LOOP BACK IF THE TRANSFER HAS NOT BEEN COMPLETED
00B0	D1	214	POP D ; RECOVER THE REGISTERS
00B1	C1	215	POP B
00E2	E1	216	POP H
00B3	F1	217	POP PSW
00B4	C9	218	RET
			***** \$EJECT *****

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LOC OBJ      LINE      SOURCE STATEMENT
0085 SEFF          219 *INCLUDE(:FI:PROFIL,SRC)
0087 D300          220 ;
0089 DB01          221 ; A ROUTINE TO CUT THE TUNNEL PROFILE
0091 E6F7          222 ; THIS ROUTINE IS INVOKED BY THE RST 5.5 INTERRUPT
0093 F604          223 ;
0095 D301          224 PUBLIC PROFIL
0097          225 ;
0099          226 PROFIL: MVI A,0FFH ; CLOSE THE DIRECTIONAL VALVES BY SENDING A 'FF'
0101          227 OUT 00H ; TO PORT 0
0103          228 IN 01H ;
0105          229 ANI 0F7H ; STOP THE CUTTER
0107          230 ORI 004H ; PATTERN: XXXX 01XX
0109          231 OUT 01H ;
0111          232 LXI SP,STKPTR ; LOCATE THE STACK POINTER TO MAKE THE WHOLE STACK AVAILABLE
0113          233 LDA INTMSK ; SET THE INTERRUPT MASK TO DISABLE THE 'CUT PROFIL' AND
0115          234 ORI 01DH ; THE 'MOVE' CONTROLS. PATTERN IS XXX111X1
0117          235 STA INTNSK ;
0119          236 SIM ; CHECK IF THE PROFILE DATA HAVE BEEN SORTED
0121          237 LDA SORTFL ;
0123          238 RAL ;
0125          239 JNC NOTSRT ; JUMP TO THE NO SORT ROUTINE IF NOT
0127          240 LHLD STARC ; GET THE FIRST SET OF DATA
0129          241 MOV B,H ;
0131          242 MOV C,L ;
0133          243 LDAX B ; FIRST THE RAISE DATA
0135          244 STA RSDESR ; INCREMENT THE B,C TO POINT AT THE MIN. SLEW BOUND
0137          245 INX B ; GET THE SLEW VALUE
0139          246 LDAX B ;
0141          247 STA FIRSL ; STORE THIS DESIRED SLEW VALUE
0143          248 STA SLDESR ; RECOVER THE REGISTER B,C PAIR
0145          249 DCX B ; FAST MOVE TO THIS STARTING POINT
0147          250 FASTS: CALL FMOVE ; CHECK IF THE DIRECTIONAL VALVES HAVE BEEN CLOSED
0149          251 IN 00H ;
0151          252 CMA ; LOOP BACK IF NOT
0153          253 ORA A ; CHECK IF THE DESIRED POSITION HAS BEEN REACHED
0155          254 JNZ FASTS ;
0157          255 CALL CHKPOS ;
0159          256 RAL ;
0161          257 JNC FASTS ; LOOP BACK IF NOT YET REACHED
0163          258 FASTS1: IN 01H ; OTHERWISE, ISSUE THE START CUTTER SIGNAL
0165          259 ANI 0FEH ;
0167          260 ORI 00BH ;
0169          261 OUT 01H ;
0171          262 MVI A,0FFH ; PATTERN IS: XXXX 10XX
0173          263 STA CUTTER ; STORE THE FLAG INDICATING THAT THE CUTTER HAS BEEN STARTED
0175          264 LHLD STARC ;
0177          265 MOV B,H ;
0179          266 MOV C,L ;
0181          267 CTR01: LHLD LASTCY
0183          268 MOV A,B ;
0185          269 CMP H ;
0187          270 JC NOTUP ;
0189          271 JNZ OVER1 ;
0191          272 MOV A,C ;
0193          273 CMP L ;

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LOC	OBJ	LINE	SOURCE STATEMENT
0116	D23301	= 274	JNC OVER1
0119	03	= 275	NOTUP: INX B
011A	0A	= 276	LDAX B
011B	321000	D = 277	STA FIRSL
011E	06	= 278	DCX B
011F	5A1000	D = 279	LDA FIRSL
0122	321A00	D = 280	STA SLDESR
0125	0A	= 281	LDAX B
0126	321800	D = 282	STA RSDESR
0129	CD4007	= 283	CALL SMOVE1
012C	03	= 284	INX B
012D	03	= 285	INX B
012E	03	= 286	INX B
012F	03	= 287	INX B
0130	C30901	= 288	JMP CTPRO1
0133	2A0600	D = 289	LHLD LASTCV
0136	44	= 290	MOV B,H
0137	4D	= 291	MOV C,L
0138	CD7805	= 292	CALL SHORT
013B	0A	= 293	LDAX B
013C	321800	D = 294	STA RSDESR
013F	03	= 295	INX B
0140	0A	= 296	LDAX B
0141	321000	D = 297	STA FIRSL
0144	03	= 298	INX B
0145	0A	= 299	LDAX B
0146	321100	D = 300	STA FIRSL+1
0149	0B	= 301	DCX B
014A	0B	= 302	DCX B
014B	5A1000	D = 303	LDA FIRSL
014E	321A00	D = 304	STA SLDESR
0151	CD4007	= 305	CALL SMOVE1
0154	5A1100	D = 306	LDA FIRSL+1
0157	321A00	D = 307	STA SLDESR
015A	CD4007	= 308	CALL SMOVE1
015D	2A0400	D = 309	LHLD STARCX
0160	78	= 310	MOV A,B
0161	BC	= 311	CMP H
0162	DAB201	= 312	JC UNDER1
0165	C27001	= 313	JNZ NOTLOW
0168	79	= 314	MOV A,C
0169	BD	= 315	CMP L
016A	DAB201	= 316	JC UNDER1
016D	CAB201	= 317	JZ UNDER1
0170	0B	= 318	DCX B
0171	0B	= 319	DCX B
0172	0A	= 320	LDAX B
0173	321000	D = 321	STA FIRSL
0176	0B	= 322	DCX B
0177	0B	= 323	DCX B
0178	0A	= 324	LDAX B
0179	321800	D = 325	STA RSDESR
017C	5A1000	D = 326	LDA FIRSL
017F	C35701	= 327	JMP CTPRO2
0182	2A0400	D = 328	UNDER1: LHLD STARCX

! IF NOT, GET THE NEXT SLEW BOUND AND RAISE LEVEL DATA

! CUT TO THIS POINT (MOVE ALONG THE LEFT BOUNDARY)

! CUT THE UPPER MOST LEVEL

! MOVE ALONG THE RIGHT BOUNDARY

! CHECK IF THE LOWEST LEVEL HAS BEEN CUT

! GET THE SLEW BOUND

! AND THE RAISE LEVEL

! LOOP BACK TO CUT THE PROFILE SIDE BOUNDARY

! CUT THE LOWEST LEVEL



LOC	OBJ	LINE	SOURCE STATEMENT
0185	44	= 329	MOV B,H
0186	4D	= 330	MOV C,L
0187	0A	= 331	LDAX B
0188	321B00	D = 332	STA R5DESR
018B	CD7B05	D = 333	CALL SHORT
018E	3A1000	D = 334	LDA FIRSL
0191	321A00	D = 335	STA SLDESR
0194	CD4007	D = 336	CALL SMOVE1
0197	3A1100	D = 337	LDA FIRSL+1
019A	321A00	D = 338	STA SLDESR
019D	CD4007	D = 339	CALL SMOVE1
01A0	DB01	= 340	IN 01H
01A2	E6F7	= 341	ANI 0F7H
01A4	F604	= 342	ORI 004H
01A6	D301	= 343	OUT 01H
01AB	3E1B	= 344	MVI A,018H
01AA	30	= 345	SIM
01AB	320000	D = 346	STA INTMSK
01AE	FB	= 347	EI
01AF	76	= 348	HLT
		= 349	*EJECT

; OUTPUT THE 'STOP CUTTER' SIGNAL  
 ; RESET THE INTERRUPT MASK  
 ; WAIT FOR THE SYSTEM INTERRUPT SIGNAL TO START AN OPERATION

LOC	OBJ	L.I.N.E	SOURCE STATEMENT
		= 350	NOTSRT:
01B0	01FE1B	= 351	LXI B, PFLEND
01B3	11FE8B	= 352	LXI D, LASTAD
01B6	CD4000	= 353	CALL SETLNK
01B9	CD7500	= 354	CALL TRNFFL
01BC	CD5200	= 355	CALL OFFLNK
01BF	2100BC	= 356	FAST1: LXI H, STADU
01C2	56	= 357	MOV D, M
01C3	23	= 358	INX H
01C4	5E	= 359	MOV E, M
01C5	EB	= 360	XCHC
01C6	221A00	D = 361	SHLD SLDES R
01C9	EB	= 362	XCHC
01CA	CDAE80	= 363	CALL FMOVE
01CD	DB00	= 364	IN 00H
01CF	2F	= 365	CMA
01D0	B7	= 366	ORA A
01D1	C2BF01	= 367	JNZ FAST1
01D4	CD6F07	= 368	CALL CHKPOS
01D7	17	= 369	RAL
01D8	D2BF01	= 370	JNC FAST1
		= 371	IN 01H
		= 372	
		= 373	
		= 374	
		= 375	
		= 376	
01DD	E6FB	= 377	ANI 0FBH
01DF	F60B	= 378	ORI 00BH
01E1	D301	= 379	OUT 01H
01E3	3EFF	= 380	MVI A, 0FFH
01E5	521900	D = 381	STA CUTTER
01E8	E5	= 382	PUSH H
01E9	2AFEBB	= 383	LHLD LASTAD
01EC	4D	= 384	MOV C, L
01ED	44	= 385	MOV B, H
01EE	E1	= 386	POP H
01EF	03	= 387	INX B
01F0	23	= 388	INX H
01F1	7C	= 389	MOV A, H
01F2	B8	= 390	CMR B
01F3	DAFB01	= 391	JC CONTP1
01F6	7D	= 392	MOV A, L
01F7	B9	= 393	CMR C
01FB	CA0902	= 394	JZ STOFF1
01FB	56	= 395	CONTP1: MOV D, M
01FC	23	= 396	INX H
01FD	5E	= 397	MOV E, M
01FE	EB	= 398	XCHC
01FF	221A00	D = 399	SHLD SLDES R
0202	EB	= 400	XCHC
0203	CD4007	= 401	CALL SMOVE1
0206	C3F001	= 402	JMP LOOP
0209	DB01	= 403	STOFF1: IN 01H
020B	E6F7	= 404	ANI 0F7H

; THE PROFILE DATA HAVE NOT BEEN SORTED AND EXPANDED  
 ; LOCATE THE REGISTERS E,C POINTING TO THE SOURCE BLOCK  
 ; LOCATE THE REGISTERS D,E POINTING TO THE DESTINATION BLOCK  
 ; SET UP THE LINK BETWEEN THE TWO SYSTEMS  
 ; GET THE PROFILE DATA FROM THE DATA CAPTURING SYSTEM  
 ; RELEASE THE LINK BETWEEN THE TWO SYSTEMS  
 ; LOAD THE STARTING PROFILE DATA ADDRESS TO H,L  
 ; LOAD THE DESIRED 'ELEVATION' VALUE TO THE REGISTER D  
 ; LOAD THE DESIRED 'SLEW' VALUE TO THE REGISTER E  
 ; EXCHANGE D,E WITH H,L  
 ; STORE THESE INFORMATIONS INTO THE APPROPRIATE LOCATIONS  
 ; RECOVERS THE REGISTERS D, E, H, AND L  
 ; CALL FAST MOVE  
 ; GET THE CURRENT VALVE SETTING  
 ; CHECK IF ALL VALVES ARE CLOSED  
 ; LOOP BACK IF THIS POSITION HAS NOT YET BEEN REACHED  
 ; CHECK IF THE DESIRED POSITION HAS BEEN REACHED  
 ; LOOP BACK IF NOT  
 ; OTHERWISE, GET THE CURRENT PATTERN OF PORT 1 AND THEN  
 ; ISSUE THE START CUTTER COMMAND  
 ; ( BIT 2 = 0 TO START THE CUTTER  
 ; BIT 3 = 0 TO STOP THE CUTTER)  
 ; AND SET THE SLOW MOVE MODE  
 ; THE PATTERN IS XXXX1011B AND IS OBTAINED  
 ; BY FIRST ANDING IT WITH 11111011B  
 ; AND THEN ORING IT WITH 00001011B  
 ; SEND THE SIGNAL TO THE PORT  
 ; STORE THE CUTTER STATUS  
 ; SAVE H,L  
 ; LOAD THE REGISTERS B,C WITH THE LAST ADDRESS FOR THE PROFILE DATA  
 ; RECOVER H,L  
 ; INCREMENT THE FINAL ADDRESS BY 1  
 ; INCREMENT PROFILE DATA ADDRESS  
 ; CHECK IF LAST DATA HAS BEEN REACHED  
 ; CONTINUE OPERATION IF NOT  
 ; CHECK THE LOW ORDER BYTE  
 ; STOP IF THE COMPLETE PROFILE HAS BEEN TRIMMED  
 ; GET THE RAISE DATA  
 ; GET THE SLEW DATA  
 ; STORE THESE DATA IN THE APPROPRIATE MEMORIES  
 ; MOVE SLOWLY TO THIS POSITION  
 ; LOOP BACK FOR THE NEXT PROFILE POSITION  
 ; ISSUE THE STOP CUTTER COMMAND: XXXX01XXB  
 ; BY ANDING IT WITH 11110111B

LOC	OBJ	LINE	SOURCE STATEMENT
020D	F604	= 405	ORI 004H ; THEN ORING IT WITH 00000100B
020F	D301	= 406	OUT 01H ;
0211	97	= 407	SUB A ; CLEAR THE ACCUMULATOR
0212	321900	D = 408	STA CUTTER ; STORE THIS CUTTER STATUS
0215	C32B03	= 409	JMP RST45 ; JUMP TO SORT THE PROFILE DATA OUT
		= 410	*EJECT

LOC	OBJ	LINE	SOURCE STATEMENT
		411	INCLUDE(:FI:CAVITY.SRC)
		412	*****
		413	THE CUT CAVITY PROGRAM
		414	A PROGRAM TO EXCAVATE THE CAVITY OF THE TUNNEL
		415	THIS PROGRAM IS INVOKED BY THE RST 6.5 INTERRUPT
		416	*****
		417	PUBLIC CAVITY
		418	*****
0218	3EFF	419	CAVITY: MVI A,0FFH ; CLOSE ALL THE DIRECTIONAL VALVES
021A	D300	420	OUT 00H
021C	310000	421	LXI SP,STKPTR ; LOCATE THE STACK POINTER TO THE INITIAL VALUE
021F	3E1E	422	MVI A,01EH ; SET THE CURRENT INTERRUPT MASK TO MASK OUT THE RST 7.5
		423	AND THE RST 6.5 INTERRUPTS WHILE UNMASK THE RST 5.5 INTERRUPT
		424	RST 5.5 --- TRIM PROFILE' BUTTON
		425	RST 6.5 --- 'CUT CAVITY' BUTTON
		426	RST 7.5 --- 'MOVE' CONTROL LEVER
		427	01EH = 0001 1110 B
0221	320000	428	STA INTMSK ; STORE THIS NEW INTERRUPT MASK SETTING
0224	30	429	SIM ; SET THE INTERRUPT MASK
0225	2A0100	430	LHLD CUTSTP ; GET THE CUTTER STEP SIZE
0228	7D	431	MOV A,L ; REDUCE THE CUTSTP BY HALF
0229	FE02	432	CPI 02H ; CHECK WHETHER THE CUTSTP IS GREATER THAN 2
022B	DA2F02	433	JC \$+4 ; MARGIN WILL BE EQUAL TO THE STEP SIZE IF LESS
022E	1F	434	RAR
022F	321600	435	STA HALFST ; AND STORE THIS IN HALFST
0232	2600	436	MVI H,00H ; MULTIPLY THIS BY 4
0234	29	437	DAD H
0235	29	438	DAD H
0236	221200	439	SHLD FOURST
0239	FB	440	EI ; STORE THIS RESULT IN FOURST
023A	2A0400	441	LHLD STARCVC ; ENABLE THE SYSTEM INTERRUPT
023D	23	442	INX H ; LOAD THE REGISTERS B,C WITH THE STARTING DATA ADDRESS
023E	23	443	INX H ; INCREASE THE START CAVITY DATA ADDRESS
023F	23	444	INX H
0240	23	445	INX H
0241	3D	446	DCR A ; BY HALF BLOCKS
0242	C23D02	447	JNZ REDUST ; COPY THIS ADDRESS TO B,C
0245	44	448	MOV B,H
0246	4D	449	MOV C,L
0247	2A0600	450	LHLD LASTCV ; LOAD THE REGISTERS H,L WITH THE END ADDRESS FOR THE ENVELOPE
024A	3A1600	451	LDA HALFST ; GET THE HALF STEP SIZE
024D	2B	452	DCX H ; DECREMENT THE LAST CAVITY DATA ADDRESS
024E	2B	453	DCX H
024F	2B	454	DCX H
0250	2B	455	DCX H
0251	30	456	DCR A
0252	C24D02	457	JNZ REDULS
0255	221400	458	SHLD LASTCT ; STORE THIS ADDRESS
0258	EB	459	XCHG ; THEN PUT IT ONTO D,E
0259	CD4605	460	CALL REDUCE ; GET THE FIRST SLEW BOUND
025C	0A	461	LDAX B ; GET THE DESIRED RAISE DATA
025D	321B00	462	STA RSDISR ; STORE IT
0260	3A1000	463	LDA FIRSL ; GET THE SLEW BOUND
0263	321A00	464	STA SLDSLR ; AND STORE IT
0266	CD4E80	465	CALL FMOVE

LOC	OBJ	L.LINE	SOURCE STATEMENT
0269	0B00	= 466	IN 00H
026B	2F	= 467	CMA
026C	B7	= 468	ORA A
026D	C26602	= 469	JNZ CALSHV
0270	CD6F07	= 470	CALL CHKPOS
0273	17	= 471	RAL
0274	D26602	= 472	JNC CALSHV
0277	0B01	= 473	IN 01H
0279	F508	= 474	ORI 00BH
027B	E6FB	= 475	ANI 0FBH
027D	D301	= 476	OUT 01H
027F	3A1100	D = 477	CONTC5: LDA FIRSL+1
0282	321A00	D = 478	STA SLDESR
0285	CD4007	D = 479	CALL SMOVE1
0288	2A1200	D = 480	LHLD FOUKST
028B	09	= 481	DAD B
028C	7C	= 482	MOV A,H
028D	8A	= 483	CMP D
028E	DA9902	= 484	JC CONTC0
0291	C2AE02	= 485	JNZ CONTC2
0294	7D	= 486	MOV A,L
0295	8B	= 487	CMP E
0296	D2AE02	= 488	JNC CONTC2
0299	44	= 489	CONTC0: MOV B,H
029A	4D	= 490	MOV C,L
029B	CD8605	= 491	CALL REDUCE
029E	0A	= 492	LDAX B
029F	321E00	D = 493	STA REDESR
02A2	3A1000	D = 494	LDA FIRSL
02A5	321A00	D = 495	STA SLDESR
02AB	CD4007	= 496	CALL SMOVE1
02AB	C37F02	= 497	JMP CONTC5
02AE	2A1400	D = 498	CONTC2: LHLD LASTCT
02B1	7E	= 499	MOV A,M
02B2	321E00	D = 500	STA REDESR
02B5	44	= 501	MOV B,H
02B6	4D	= 502	MOV C,L
02B7	CD8605	= 503	CALL REDUCE
02BA	3A1000	D = 504	LDA FIRSL
02BD	321A00	D = 505	STA SLDESR
02C0	CD4007	= 506	CALL SMOVE1
02C3	3A1100	D = 507	LDA FIRSL+1
02C5	321A00	D = 508	STA SLDESR
02C9	CD4007	= 509	CALL SMOVE1
02CC	DB01	= 510	IN 01H
02CE	EGF7	= 511	ANI 0F7H
02D0	F604	= 512	ORI 00HH
02D2	D301	= 513	OUT 01H
02D4	3E1B	= 514	MVI A,01BH
02D6	320000	D = 515	STA INTMSK
02D9	30	= 516	SIM
02DA	FB	= 517	EI
02DB	76	= 518	HLT
		= 519	\$EJECT

```

; GET THE DIRECTIONAL VALVES SETTING
; CHECK IF THEY ARE CLOSED
; LOOP BACK IF THE POSITION HAS NOT BEEN REACHED
; CHECK IF THE DESIRED POSITION HAS BEEN REACHED
; LOOP BACK IF NOT
; START THE CUTTER IF YES
; PATTERN XXXX 10XX B
; GET THE NEXT SLEW BOUND DATA
; MOVE AND CUT TO THIS BOUND
; GET THE STEP IN THE TABLE
; THEN ADDED TO CURRENT ADDRESS FOR 'RAISE' DATA
; CHECK IF THE LASTCV HAS BEEN REACHED
; CONTINUE IF NOT
; ASSUME LASTCV IF CONE PASS
; THEN THE LOW ORDER BYTE
; ASSUME LASTCV IF CONE PASS
; GET THIS RAISE DATA
; STOP THE CUTTER
; PATTERN XXXX 01XX B
; RESET THE INTERRUPT MASK TO ENABLE ALL INTERRUPTS
; PATTERN: 0001 1000 B
; WAIT FOR ANOTHER SYSTEM INTERRUPTTAKING SOURCE ADDRESS

```

LOC	OBJ	LINE	SOURCE STATEMENT
02DC	F3	= 520	\$INCLUDE (1:MOVE.SRC)
02DD	3E1C	= 521	*****
02DF	30	= 522	PROGRAM SEGMENT TO SERVICE THE RST 7.5 INTERRUPT
02E0	320000	= 523	*****
02E1	E3	= 524	PUBLIC MOVE
02E2	01DA1B	= 525	*****
02E3	11A00	= 526	SYMBOLS FOR DATA STORAGE IN THE DATA CAPTURING SYSTEM
02E4	CD4000	= 527	SYMBOL
02E5		= 528	-----
02E6	E3	= 529	CRTSL CURRENT SLEW POSITION (1 BYTE)
02E7	01DA1B	= 530	MOVEST CURRENT MOVE LEVER STATUS (1 BYTE)
02E8	11A00	= 531	DESRSL DESIRED SLEW POSITION (1 BYTE)
02E9	CD4000	= 532	-----
02EA		= 533	SYMBOLS USED FOR DATA STORAGE IN THE CURRENT SYSTEM
02EB		= 534	SYMBOL
02EC		= 535	-----
02ED	3E1C	= 536	SLDESR DESIRED SLEW POSITION (1 BYTE)
02EE	30	= 537	SLCRRT CURRENT SLEW POSITION (1 BYTE)
02EF	01DA1B	= 538	INTMSK INTERRUPT MASK SETTING (1 BYTE)
02F0	CD5200	= 539	*****
02F1	F1	= 540	MOVE: D1
02F2	FE10	= 541	MV1 A, 01CH
02F3	3AA21B	= 542	SIM
02F4	32300	= 543	STA INTMSK
02F5	3AD1B	= 544	LXI H, 9D
02F6	32300	= 545	XTHL
02F7	01DA1B	= 546	LXI B, DESRSL
02F8	3AD1B	= 547	LXI D, SLDESR
02F9	32300	= 548	CALL SETLNK
02FA	32300	= 549	CALL TRANSF
02FB	3AA21B	= 550	LDA CRTSL
02FC	32300	= 551	STA SLCRRT
02FD	3AA21B	= 552	LDA CRTRTS
02FE	32300	= 553	STA RSCRRT
02FF	3AD1B	= 554	LDA MOVEST
0300	F5	= 555	PUSH PSW
0301	FE10	= 556	CALL OFFLNK
0302	F1	= 557	POP PSW
0303	CD5200	= 558	CPI 010H
0304	FE10	= 559	JC NOMOVE
0305	DA1C03	= 560	JC
0306	17	= 561	RAL
0307	DA1603	= 562	JC MOVE1
0308	CDEA05	= 563	CALL SMOVE
0309	C3E702	= 564	JMP MOVE2
0310	CDAE80	= 565	CALL FMOVE
0311	C3E702	= 566	JMP MOVE2
0312	3EFF	= 567	NOMOVE: MV1 A, OFFH
0313	D300	= 568	OUT 00H
0314	3E1B	= 569	MV1 A, 018H
0315	30	= 570	SIM
0316	320000	= 571	STA INTMSK
0317	310000	= 572	LXI SP, STKPTR
0318	FB	= 573	EI
0319	76	= 574	HLT

LOC	OBJ	LINE	SOURCE STATEMENT
575			*****
576			THE ROUTINE ON RECOGNIZING THE TRAP INTERRUPT AFTER THE
577			PROFILE DATA HAS BEEN ENTERED
578			BOTH THE 'STOP CUTTER' AND 'START CUTTER' LEDs ARE ON
579			INDICATING THAT ONLY THE 'CUT PROFILE' IS OPERATIVE
580			*****
581			ON EXIT: ONLY THE 'STOP CUTTER' LED IS ON
582			ALL INTERRUPTS ARE ENABLED
583			*****
584			FUNCTION: TO SORT THE PROFILE ENVELOPE
585			INTERPOLATE THESE DATA AND EXPAND TO COVER EVERY
586			BIT CHANGE IN ALTITUDE IF NECESSARY
587			*****
588			PUBLIC RST45, STARCV, LASTCV, LAST50, DMINI, LEVEL
589			*****
590			RST45: DI ENSURE THAT THE SYSTEM IS DISABLED
591			LXI SP, STKPTR RE-LOCATE THE STACK POINTER
592			MVI A, OFFH CLOSE ALL THE DIRECTIONAL VALVES
593			OUT 00H
594			IN 01H
595			ANI 0F3H
596			OUT 01H
597			MVI A, 01EH
598			SIM
599			STA INTMSK
600			LXI B, PFLEND
601			LXI D, LASTAD
602			*****
603			CALL SETLNK
604			CALL TRNPF
605			CALL OFFLNK
606			*****
607			SUB A
608			STA SORTFL
609			EI
610			LXI B, STADD
611			LHLD LASTAD
612			XCHC
613			CALL SORT
614			XCHC
615			DCX H
616			SHLD LASTAD
617			MOV B, H
618			MOV C, L
619			LHLD STADD-2
620			PUSH H
621			DCX B
622			DCX B
623			PUSH B
624			LXI H, STADD
625			SHLD STARCV
626			MOV B, H
627			MOV C, L
628			DCX B
629			INX H
032B	F3		*****
032C	310000	D	THE ROUTINE ON RECOGNIZING THE TRAP INTERRUPT AFTER THE
032F	3EFF		PROFILE DATA HAS BEEN ENTERED
0331	0300		BOTH THE 'STOP CUTTER' AND 'START CUTTER' LEDs ARE ON
0333	0B01		INDICATING THAT ONLY THE 'CUT PROFILE' IS OPERATIVE
0335	E6F3		*****
0337	1501		ON EXIT: ONLY THE 'STOP CUTTER' LED IS ON
0339	3E1E		ALL INTERRUPTS ARE ENABLED
033B	30		*****
033C	320000	D	FUNCTION: TO SORT THE PROFILE ENVELOPE
033F	01FE1B		INTERPOLATE THESE DATA AND EXPAND TO COVER EVERY
0342	11FE3B		BIT CHANGE IN ALTITUDE IF NECESSARY
0345	CD4000		*****
0348	CD7500		PUBLIC RST45, STARCV, LASTCV, LAST50, DMINI, LEVEL
034B	CD5200		*****
034E	97		RST45: DI ENSURE THAT THE SYSTEM IS DISABLED
034F	320200	D	LXI SP, STKPTR RE-LOCATE THE STACK POINTER
0352	F8		MVI A, OFFH CLOSE ALL THE DIRECTIONAL VALVES
0353	0100BC		OUT 00H
0356	2AFEBB		IN 01H
0359	EB		ANI 0F3H
035A	CD0003		OUT 01H
035D	EB		MVI A, 01EH
035E	2B		SIM
035F	22FEBB		STA INTMSK
0362	44		LXI B, PFLEND
0363	1D		LXI D, LASTAD
0364	2AFEBB		*****
0367	E5		CALL SETLNK
0368	0B		CALL TRNPF
0369	0B		CALL OFFLNK
036A	C5		*****
036B	2100BC		SUB A
036E	220400	D	STA SORTFL
0371	44		EI
0372	4B		LXI B, STADD
0373	0B		LHLD LASTAD
0374	23		XCHC

LOC	OBJ	LINE	SOURCE STATEMENT
0375 23		630	INX H
0376 7E		631	MOV A,M
0377 02		632	STAX B
0378 2B		633	DCX H
0379 0B		634	DCX B
037A 7E		635	MOV A,M
037B 02		636	STAX B
037C 2B		637	DCX B
037D 110C00	D	638	LXI D,DMINI
0380 C1		639	CONTR1: POP B
0381 C5		640	PUSH B
0382 7C		641	MOV A,H
0383 B8		642	CMP B
0384 DABC03		643	JC CONTR2
0387 7D		644	MOV A,L
0388 B9		645	CMP C
0389 D2C003		646	JNC COMPL
038C 44		647	CONTR2: MOV B,H
038D 4D		648	MOV C,L
038E 0B		649	DCX B
039F 0B		650	DCX B
0390 23		651	INX H
0391 7E		652	MOV A,M
0392 12		653	STAX D
0393 23		654	INX H
0394 13		655	INX D
0395 7E		656	MOV A,M
0396 12		657	STAX D
0397 23		658	INX H
0398 1B		659	DCX D
0399 23		660	INX H
039A 0A		661	LDAX B
039B B6		662	ADD M
039C 1F		663	RAR
039D EB		664	XCHG
039E BE		665	CMP M
039F D2A303		666	JNC $\Phi+4$
03A2 77		667	MOV M,A
03A3 E8		668	XCHG
03A4 23		669	INX H
03A5 13		670	INX D
03A6 03		671	INX B
03A7 0A		672	LDAX B
03A8 96		673	ADD M
03A9 1F		674	RAR
03AA EB		675	XCHG
03AB BE		676	CMP M
03AC DAE003		677	JC $\Phi+4$
03AF 77		678	MOV M,A
03B0 EB		679	XCHG
03B1 1B		680	DCX D
03B2 03		681	INX B
03B3 03		682	INX B
03E4 1A		683	LDAX D
03B5 02		684	STAX B

```

; NOW H,L IS POINTING AT THE MAX.
; STORE THE MAX.
; NOW H,L IS POINTING AT THE MIN.
; GET THIS AND
; STORE IT
; NOW H,L POINT AT THE 'RAISE' DATA
; LOAD D,E WITH THE ADDRESS DMINI
; GET THE LAST DATA ADDRESS FOR CHECKING
; PUT IT BACK ONTO THE STACK
; CHECK IF THE LAST DATA SET HAS BEEN SORTED
; FIRST THE HIGH ORDER BYTE
; CONTINUE IF NOT YET FINISHED
; CHECK FOR THE LOW ORDER BYTE IF THE HIGH BYTE ARE EQUAL
; JUMP OUT IF THE INTERPOLATING HAS BEEN COMPLETED
; COPY THE CURRENT 'RAISE' DATA ADDRESS TO B,C
; THEN DECREMENT IT TO POINT AT THE MIN. DATA
; GET THE CURRENT MIN. DATA AND STORE IT IN DMINI
; D,E POINT AT THE LOCATION DMINI
; GET THE CURRENT MAX. DATA AND STORE IT IN ONE LOCATION
; ABOVE DMINI
; POINT H,L AT THE 'RAISE' OF THE NEXT SET OF DATA
; POINT D,E AT THE DMINI
; POINT H,L AT THE MIN. OF NEXT DATA SET OF DATA
; GET THE MIN. BELOW
; AND ADD TO THE MIN. ABOVE
; DIVIDE THE RESULT BY TWO
; H,L <---> D,E
; COMPARE WITH THE MIN. SETTING
; RESET THIS MIN. IF THE ACCUMULATOR < M
; H,L <---> D,E
; POINT H,L AT THE MAX. OF NEXT SET OF DATA
; POINT D,E AT THE LOCATION ABOVE DMINI
; POINT B,C AT THE MAX. BELOW THE CURRENT DATA SET
; GET THIS DATA
; AND ADD TO THE NEXT MAX.
; FIND THE AVERAGE
; H,L <---> D,E
; COMPARE WITH THE CURRENT MAX. SETTING
; RESET THE MAX. IF THE ACCUMULATOR > M
; H,L <---> D,E
; POINT D,E AT DMINI AGAIN
; B,C POINT AT THE CURRENT 'RAISE' DATA
; B,C POINT AT THE CURRENT MIN SLEW DATA
; GET DMINI
; AND STORE IT AT THE MIN SLEW POSITION

```



LOC	OBJ	LINE	SOURCE STATEMENT
0386	03	685	INX B
0387	13	686	INX D
0388	1A	687	LDAX D
0389	02	688	STAX B
038A	1B	689	DCX D
038B	2B	690	DCX H
038C	2B	691	DCX H
038D	C98003	692	JMP CONTR1
03C0	C1	693	COMPL1
03C1	E1	694	POP B
03C2	22FEBB	695	POP H
03C5	C00905	696	SHLD STADD-2
03C8	3EFF	697	CALL EXPAND
03CA	320200	698	MVI A,0FFH
03CD	3E18	699	STA SORTFL
03CF	320000	700	MVI A,018H
03D2	30	701	STA INTRSK
03D3	D801	702	SIM
03D5	E6F7	703	IN 01H
03D7	F604	704	ANI 0F7H
03D9	D301	705	ORI 004H
03DB	FB	706	OUT 01H
03DC	75	707	EI
		708	HLT

; GO ON FOR THE MAX, SLEW  
 ; GET DMINI + 1 (I.E. THE MAX)  
 ; AND STORE IT IN THE CURRENT MAX, SLEW DATA POSITION  
 ; POINT D,E AT DMINI AGAIN  
 ; POINT H,L AT THE 'RAISE' DATA ADDRESS  
 ; LOOP BACK FOR FURTHER SORTING AND CHECKING  
 ; RECOVER THE TWO BYTE DATA BELOW STADD  
 ; CALL THE TABLE EXPANSION ROUTINE  
 ; RESET THE SORTFL TO OFFH  
 ; RESET THE INTERRUPT MASK TO ENABE ALL THE FUNCTIONS  
 ; STORE THIS NEW PATTERN  
 ; PATTERN IS: 0001 1000  
 ; RESET THE LED LIGHT PATTERN TO 'STOP CUTTER'  
 ; PATTERN: XXXX 01XX  
 ; ENABLE THE SYSTEM INTERRUPT AND  
 ; WAIT

LOC	OBJ	LINE	SOURCE STATEMENT
709			*****
710			NAME OF ROUTINE: SORT
711			'THIS IS THE ROUTINE SECMET' TO SORT THE PROFILE DATA
712			*****
713			ON ENTRY: REGISTERS B,C --- START PROFILE DATA ADDRESS
714			REGISTERS D,E --- LAST PROFILE DATA ADDRESS
715			ON EXIT : REGISTERS B,C --- START ADDRESS OF SORTED DATA
716			REGISTERS D,E --- LAST ADDRESS OF SORTED DATA + 1
717			*****
718			PROFILE DATA ARE STORED IN THE ORDER:
719			ELEVATION, MIN. SLEW, MAX. SLEW, ELEVATION, ...
720			*****
721			PUBLIC SORT
722			ROUTINE OPEN TO OTHER SEGMENTS
723			*****
723			SORT: PUSH PSW ; SAVE THE REGISTERS ON ENTRY
724			PUSH H
725			PUSH B
726			INX D
727			PUSH D
728			PUSH B
729			PUSH B
730			LXI H,-3
731			DAD SP
732			PUSH H
733			MVI M,0
734			MOV A,B
735			CMP D
736			JNC CONT11
737			POP H
738			LDAX B
739			CMP M
740			JNZ CONT21
741			INX B
742			LDAX B
743			MOV E,A
744			DCX B
745			LDAX B
746			MOV D,A
747			POP H
748			POP H
749			MOV A,M
750			STAX B
751			MOV M,D
752			INX H
753			INX B
754			MOV A,M
755			STAX B
756			MOV M,E
757			INX H
758			INX B
759			POP D
760			POP D
761			PUSH H
762			PUSH B
763			DCX SP
03DD	F5		
03DE	E5		
03DF	C5		
03E0	13		
03E1	D5		
03E2	C5		
03E3	C5		
03E4	21FDFF		
03E7	39		
03E8	E5		
03E9	9600		
03EB	75		
03EC	BA		
03ED	D21A04		
03F0	E1		
03F1	0A		
03F2	BE		
03F3	C21104		
03F6	03		
03F7	0A		
03F8	5F		
03F9	0B		
03FA	0A		
03FB	57		
03FC	E1		
03FD	E1		
03FE	7E		
03FF	02		
0400	72		
0401	23		
0402	03		
0403	7E		
0404	02		
0405	73		
0406	23		
0407	03		
0408	D1		
0409	D5		
040A	E5		
040B	C5		
040C	5B		

LOC	OBJ	LINE	SOURCE STATEMENT
0400	5B	764	DCX SP
040E	C3EB03	765	LOOP11
0411	C1	766	JMP B
0412	03	767	INX B
0413	03	768	INX B
0414	C5	769	PUSH B
0415	3B	770	DCX SP
0416	3B	771	DCX SP
0417	C3EB03	772	JMP LOOP11
041A	C22204	773	JNZ CONT31
041D	79	774	MOV A,C
041E	BB	775	CMF E
041F	DAF003	776	JC CONT41
0422	E1	777	POP H
0423	3A	778	INR M
0424	CA3404	779	JZ STOP1
0427	E5	780	PUSH H
0428	E1	781	POP H
0429	E1	782	POP H
042A	E1	783	POP H
042B	E5	784	PUSH H
042C	E5	785	PUSH H
042D	4D	786	MOV C,L
042E	44	787	MOV B,H
042F	3B	788	DCX SP
0430	3B	789	DCX SP
0431	C3EB03	790	JMP LOOP11
0434	D1	791	POP D
0435	D1	792	POP D
0436	D1	793	POP D
0437	C1	794	POP B
0438	C5	795	PUSH B
0439	D5	796	PUSH D
043A	C5	797	PUSH B
043B	C5	798	PUSH B
043C	21F7F	799	LXI H,-9
043F	39	800	DAD SP
0440	E5	801	PUSH H
0441	0A	802	LDAX B
0442	77	803	MOV M,A
0443	2B	804	DCX H
0444	36FF	805	MVI M,0FFH
0446	2B	806	DCX H
0447	3600	807	MVI M,000H
0449	E1	808	POP H
044A	C1	809	POP B
044B	0A	810	LDAX B
044C	5E	811	CMF M
044D	C2B604	812	JNZ CONT12
0450	2B	813	DCX H
0451	03	814	INX B
0452	0A	815	LDAX B
0453	5E	816	CMF M
0454	D25804	817	JNC CONT22
0457	77	818	MOV M,A

; KEY ADDRESS  
 ; LOOP BACK FOR OTHER CHECKS  
 ; INCREMENT THE CURRENT ADDRESS FOR NEXT PROFILE DATA  
  
 ; STORE THIS NEW VALUE ONTO STACK  
 ; ADJUST THE STACK POINTER  
  
 ; LOOP BACK FOR OTHER CHECKS  
 ; JUMP IF ADDRESS > LAST ADDRESS  
 ; CHECK IF THE LOW ORDER BYTE OF THE CURRENT DATA POINTER  
 ; IS LESS THAN THAT OF THE LAST DATA ADDRESS  
 ; JUMP TO CHECK FOR KEY IF YES  
 ; OTHERWISE INCREMENT THE KEY VALUE FOR COMPARISON  
  
 ; RE-STORE THE SORTED ENVELOPE IF 0256D STEPS HAVE BEEN CHECKED  
 ; OTHERWISE PUT THE ADDRESS FOR KEY ONTO THE STACK  
  
 ; GET THE 'DESTINATION' ADDRESS AND  
  
 ; DUPLICATE FOR CURRENT ADDRESS USED IN CHECKING  
 ; AND PUT THIS ADDRESS TO B,C  
  
 ; ADJUST THE STACK POINTER  
  
 ; LOOP BACK FOR CHECKING  
 ; SORTING COMPLETED  
  
 ; RECOVER REGISTERS TO CONTINUE  
 ; GET THE START ADDRESS OF THE PROFILE DATA LIST  
 ; RE-SAVE THE REGISTERS B,C  
 ; LAST ADDRESS OF THE PROFILE DATA + 1  
 ; ADDRESS FOR DESTINATION OF TRANSFER  
 ; ADDRESS FOR CURRENT PROFILE TO BE CHECKED  
 ; INITIALIZE THE COMPARING KEY AND STROKE  
 ; IT TO THE ADDRESS BELOW THE STACK  
 ; SAVE THIS KEY ADDRESS  
 ; GET DATA FOR ELEVATION  
 ; PUT IT INTO THE WORKING STORE  
 ; INITIALIZE THE MINIMUM AND MAXIMUM DATA  
 ; CURRENT MINIMUM  
  
 ; CURRENT MAXIMUM  
 ; SECRET TO ADJUST MINIMUM AND MAXIMUM  
  
 ; COMPARE ELEVATION DATA  
 ; JUMP IF DATA HAS CHANGED  
  
 ; GET CURRENT MINIMUM VALUE  
 ; JUMP IF NOT LESS  
 ; SET THIS NEW MINIMUM

L.OC	OBJ	LINE	SOURCE STATEMENT
0458	2B	819	CONT22: DCX H
0459	BE	820	COMP M
045A	DA5E04	821	JC CONT32
045D	77	822	MOV M,A
045E	03	823	CONT32: INX B
045F	C5	824	PUSH B
0460	3E	825	DCX SP
0461	3B	826	DCX SP
0462	78	827	MOV A,B
0463	BA	828	COMP D
0464	DA4904	829	JC CONT42
0467	C26F04	830	JNZ STOP2
046A	79	831	MOV A,C
046B	BB	832	COMP E
046C	DA4904	833	JC CONT42
046F	E1	834	STOP2: POP H
0470	C1	835	POP B
0471	C1	836	POP B
0472	3E	837	DCX SP
0473	3E	838	DCX SP
0474	3E	839	DCX SP
0475	3E	840	DCX SP
0476	3E	841	DCX SP
0477	3E	842	DCX SP
0478	CDB104	843	CALL CSTORE
047B	50	844	MOV D,B
047C	59	845	MOV E,C
047D	210800	846	LXI H,B
0480	39	847	DAD SP
0481	F9	848	SPHL
0482	C1	849	POP B
0483	E1	850	POP H
0484	F1	851	POP PSM
0485	C9	852	REI
		853	\$EJECT

; COMPARE WITH CURRENT MAXIMUM  
 ; JUMP IF LESS  
 ; SET THIS NEW MAXIMUM  
 ; INCREMENT CURRENT ADDRESS FOR CHECK  
 ; STORE THIS CURRENT ADDRESS ON STACK  
 ; ADJUST STACK POINTER  
  
 ; SKIP IF LESS  
 ; STOP IF GREATER  
 ; COMPARE LOW ORDER BYTE ADDRESS  
  
 ; SKIP IF LESS  
 ; GET 'SOURCE' ADDRESS  
  
 ; GET 'DESTINATION' ADDRESS  
 ; ADJUST THE STACK POINTER  
  
 ; STORE THE 'CAVITY PROFILE' DATA  
 ; ASSIGN END ADDRESS TO D,E  
 ; ADJUST THE STACK POINTER  
  
 ; RECOVER REGISTERS BEFORE RETURNING

LOC	OBJ	LINE	SOURCE STATEMENT
0486	C1	854	*****
0487	3B	855	DATA FOR ELEVATION HAS CHANGED
0488	3B	856	*****
0489	3B	857	CONT12: POP B ; GET CAVITY PROFILE ADDRESS
048A	3B		; ADJUST THE STACK POINTER
048B	3B		
048C	3B		
048D	C3+104		
0490	7B	865	MOV A,B ; STORE THE CAVITY PROFILE DATA
0491	BA	866	MOV A,B ; CHECK IF THE LAST PROFILE DATA HAS BEEN STORED BY FIRST
0492	DA9D04	867	JC ; CHECKING AT THE HIGH ORDER BYTE
0493	C27B04		CONT52 ; SKIP IF THE HIGH ORDER BYTE IS LESS
0498	79	869	JNZ ; STOP IF GREATER
0499	BB	870	MOV A,C ; CHECK IF THE LOW ORDER BYTE IS LESS
049A	D27B04		CONT62 ; STOP IF NOT
049D	210600	872	LXI H,6 ; ADJUST THE STACK POINTER
04A0	39	873	DAD SP
04A1	F9	874	SPHL
04A2	C5	875	PUSH B ; THEN PUSH THE NEW 'DESTINATION' ADDRESS ONTO THE STACK
04A3	21FCFF	876	LXI H,-4 ; ADJUST THE STACK POINTER
04A6	39	877	DAD SP
04A7	F9	878	SPHL
04AB	E1	879	POP H ; RECOVER THE KEY ADDRESS
04A9	C1	880	POP B ; GET THE CURRENT ADDRESS
04AA	3B	881	DCX SP ; ADJUST THE STACK POINTER
04AB	3B	882	DCX SP
04AC	3B	883	DCX SP
04AD	3B	884	DCX SP
04AE	C3+104	885	JMP LOOP12 ; LOOP BACK FOR THIS NEW ELEVATION VALUE
		886	\$EJECT

LOC	OBJ	LINE	SOURCE STATEMENT
887		*****	*****
888		ROUTINE TO STORE THE SORTED CAVITY PROFILE DATA IN THE	*****
889		FORM OF: ELEVATION, MIN. SLEW, MAX. SLEW,	*****
890		ON ENTRY: REGISTERS B,C --- DESTINATION ADDRESS FOR THE	*****
891		ELEVATION DATA	*****
892		REGISTERS H,L --- ADDRESS OF THE ELEVATION TO	*****
893		BE STORED	*****
894		ON EXIT: REGISTERS B,C --- NEXT AVAILABLE DESTINATION	*****
895		ADDRESS FOR ELEVATION DATA	*****
896		*****	*****
897		CSTORE: PUSH PSW	*****
898	04B1 F5	LXI H,6	*****
899	04B2 210600	DAD SP	*****
900	04B5 39	SPHL	*****
901	04B6 F9	POP H	*****
902	04B7 E1	INX B	*****
903	04B8 03	INX B	*****
904	04B9 03	MOV A,B	*****
905	04BA 78	MOV A,B	*****
906	04BB 8C	CMR H	*****
907	04BC DA6E04	JC NODUP	*****
908	04BF C2C704	JNZ DUPL1	*****
909	04C2 79	MOV A,C	*****
910	04C3 8D	CMR L	*****
911	04C4 DA6E04	JC NODUP	*****
912	04C7 33	INX SP	*****
913	04C8 33	INX SP	*****
914	04C9 33	INX SP	*****
915	04CA 33	INX SP	*****
916	04CB 13	INX D	*****
917	04CC D5	PUSH D	*****
918	04CD 1B	DCX D	*****
919	04CE 1A	LDAX D	*****
920	04CF 13	INX D	*****
921	04D0 12	STAX D	*****
922	04D1 1B	DCX D	*****
923	04D2 1B	DCX D	*****
924	04D3 7C	MOV A,H	*****
925	04D4 BA	CMR D	*****
926	04D5 DACE04	JC DUPL11	*****
927	04D8 C2E004	JNZ DUPL12	*****
928	04DB 7D	MOV A,L	*****
929	04DD DACE04	CMR E	*****
930	04E0 23	JC DUPL11	*****
931	04E1 3B	INX H	*****
932	04E2 3B	DCX SP	*****
933	04E3 E5	DCX SP	*****
934	04E4 53	PUSH H	*****
935	04E5 33	INX SP	*****
936	04E6 3B	INX SP	*****
937	04E7 3B	DCX SP	*****
938	04E8 3B	DCX SP	*****
939	04E9 3B	DCX SP	*****
940	04EA E1	POP H	*****
941	04EB C1	POP B	*****

LOC	OBJ	LINE	SOURCE STATEMENT
04EC	C1	942	POP B
04ED	D1	943	POP D
		944	REPT 12D
		945	DCX SP
		946	ENDM
04EE	3B	947+	DCX SP
04EF	3B	948+	DCX SP
04F0	3B	949+	DCX SP
04F1	3B	950+	DCX SP
04F2	3B	951+	DCX SP
04F3	3B	952+	DCX SP
04F4	3B	953+	DCX SP
04F5	3B	954+	DCX SP
04F6	3B	955+	DCX SP
04F7	3B	956+	DCX SP
04F8	3B	957+	DCX SP
04F9	3B	958+	DCX SP
04FA	7E	959	MOV A, M
04FB	02	960	STAX B
04FC	2H	961	DCX H
04FD	03	962	INX B
04FE	7E	963	MOV A, M
04FF	02	964	STAX B
0500	2B	965	DCX H
0501	03	966	INX B
0502	7E	967	MOV A, M
0503	02	968	STAX B
0504	03	969	INX B
0505	23	970	INX H
0506	23	971	INX H
0507	F1	972	PDP P5W
0508	C9	973	RET
		974	*EJECT

! GET THE STARTING DESTINATION ADDRESS  
! GET THE LAST PROFILE DATA ADDRESS

!! RE-ADJUST THE STACK POINTER BY 12 DECREMENTS

! GET THE FIRST SOURCE DATA (ELEVATION)  
! STORE THIS DATA INTO THE DESTINATION  
! DECREMENT THE SOURCE DATA POINTER  
! INCREMENT THE DESTINATION DATA POINTER  
! TRANSFER DATA

! MOVE SOURCE DATA POINTER TO THE MAXIMUM SLEW  
! INCREMENT THE CAVITY PROFILE POINTER

! DATA TRANSFER COMPLETED  
! INCREMENT THE DESTINATION ADDRESS  
! RECOVER H, L TO POINT AT THE 'RAISE' DATA KEY  
! RECOVER THE PROCESSOR STATUS WORD ON EXIT

LOC	OBJ	LINE	SOURCE STATEMENT
0509	F5	975	*****
050A	C5	976	! A ROUTINE TO EXPAND THE TABLE OF SORTED PROFILE ENVELOPE
050B	D5	977	! ON ENTRY: LASTAD CONTAINS THE LAST ADDRESS FOR THE DATA
050C	E5	978	! STADD IS THE STARTING ADDRESS FOR THE DATA
050D	ZAFEBB	979	! LASTCV CONTAINS THE LAST ADDRESS OF THE TABLE
0510	2B	980	! STARCVC CONTAINS THE START ADDRESS OF THE TABLE
0511	2B	981	! SORTFL IS SET TO OFFH
0512	7E	982	*****
0513	6F	983	! PUBLIC EXPAND
0514	2600	984	! EXPAND: PUSH PSW
0516	29	985	! PUSH B
0517	29	986	! PUSH D
0518	0100BC	987	! PUSH H
051B	09	988	! LHL D LASTAD
051C	220600	989	! DCX H
0520	23	990	! DCX H
0521	EB	991	! MOV A, M
0522	ZAFEBB	992	! MOV L, A
0525	7E	993	! MVI H, 000H
0526	12	994	! DAD H
0527	2B	995	! DAD H
0528	1B	996	! LXI B, STADD
0529	7E	997	! DAD B
052A	12	998	! SHLD LASTCV
052B	2B	999	! INX H
052C	1B	1000	! INX H
052D	7E	1001	! XCHG
052E	12	1002	! LHL D LASTAD
052F	F5	1003	! MOV A, M
0530	0100BC	1004	! STAX D
0533	7C	1005	! DCX H
0534	6B	1006	! DCX D
0535	C23D05	1007	! MOV A, M
0538	7D	1008	! STAX D
0539	E9	1009	! DCX H
053A	CABE05	1010	! DCX D
053B	F1	1011	! MOV A, M
053C	2B	1012	! STAX D
053D	2B	1013	! PUSH PSW
053E	2B	1014	! LXI B, STADD
0540	2B	1015	! MOV A, H
0541	96	1016	! CMP B
0542	FE01	1017	! JNZ EXPA2
0544	C24E05	1018	! MOV A, L
0547	23	1019	! CMP C
0548	2B	1020	! JZ EXPAF
		1021	! EXPA2: POP PSW
		1022	! DCX H
		1023	! DCX H
		1024	! DCX H
		1025	! EXPA3: SUB M
		1026	! CPI 01H
		1027	! JNZ EXPA4
		1028	! INX H
		1029	! INX H



LOC	OBJ	LINE	SOURCE STATEMENT
0549	1B	1030	DCX D
054A	1B	1031	DCX D
054B	C32505	1032	JMP EXP41
054E	E5	1033	EXPA41: PUSH H
054F	C5	1034	PUSH B
0550	23	1035	INX H
0551	13	1036	INX D
0552	1A	1037	LDAX D
0553	86	1038	ADD M
0554	1F	1039	RAR
0555	4F	1040	MOV C, A
0556	23	1041	INX H
0557	13	1042	INX D
0558	1A	1043	LDAX D
0559	86	1044	ADD M
055A	1F	1045	RAR
055B	47	1046	MOV B, A
055C	1B	1047	DCX D
055D	1B	1048	DCX D
055E	1A	1049	LDAX D
055F	1B	1050	DCX D
0560	1B	1051	DCX D
0561	EB	1052	XCHG
0562	70	1053	MOV M, B
0563	2B	1054	DCX H
0564	71	1055	MOV M, C
0565	2B	1056	DCX H
0566	30	1057	DCR A
0567	77	1058	MOV M, A
0568	EB	1059	XCHG
0569	C1	1060	POP B
056A	E1	1061	POP H
056B	C3+105	1062	JMP EXP43
056E	F1	1063	EXPAF: POP PSW
056F	EB	1064	XCHG
0570	220400	D	SHLD STARCX
0573	E1	1066	POP H
0574	D1	1067	POP D
0575	C1	1068	POP B
0576	F1	1069	POP PSW
0577	C9	1070	RET
		1071	#EJECT

! SAVE THE REGISTERS H, L  
! AND B, C

! POINT H, L AT THE MIN.  
! POINT D, E AT THE MIN.

! GET THIS VALUE  
! AND ADD TO THE MIN. OF THE SET BELOW

! AVERAGING  
! STORE THIS IN C

! DO THE SAME FOR THE MAX.

! ADD THE MAXIMUMS TOGETHER  
! THEN AVERAGING

! STORE THIS IN B  
! POINT D, E AT THE NEXT DATA SET BELOW

! GET THIS 'RAISE' DATA SETTING

! NOW AT THE MAX.  
! H, L <---> D, E

! SET THE MAX.  
! SET THE MIN.

! SET THE 'RAISE' DATA  
! H, L <---> D, E

! RECOVER THE REGISTERS

! RECOVER THE REGISTERS

! RECOVER THE REGISTERS

! RECOVER THE REGISTERS

! RECOVER THE REGISTERS

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! RECOVER THE REGISTERS

! RECOVER THE REGISTERS

LOC	OBJ	LINE	SOURCE STATEMENT
1072		1072	*****
1073		1073	NAME OF ROUTINE: SHORT
1074		1074	A SUB-ROUTINE TO FIND THE NEAREST STARTING POINT FROM THE
1075		1075	CURRENT CUTTER POSITION AS FAR AS THE SLEW IS CONCERNED
1076		1076	ON ENTRY: B,C POINT TO THE RAISE DATA OF THE BLOCK
1077		1077	ON EXIT: THE REQUIRED SLEW POSITIONS ARE SORTED AND STORED
1078		1078	IN FIRSL WITH THE FIRST OCCUPYING THE LOW POSITION
1079		1079	*****
1080		1080	PUBLIC SHORT, FIRSL, FOURST
1081		1081	*****
1082		1082	SHORT: PUSH PSW ; SAVE THE REGISTERS ON ENTRY
1083		1083	PUSH H
1084		1084	PUSH D
1085		1085	MOV L,C
1086		1086	MOV H,B
1087		1087	DI ; COPY REGISTERS B,C TO H,L
1088		1088	CALL SETLNK ; DISABLE THE SYSTEM INTERRUPT DURING DATA TRANSFER
1089		1089	LJA CRTSL ; GET THE CURRENT SLEW DATA
1090		1090	MOV E,A ; AND STORE IT IN THE REGISTER E
1091		1091	CALL OFFLNK
1092		1092	EI ; ENABLE THE SYSTEM INTERRUPT AFTER THE DATA TRANSFER
1093		1093	MOV A,E ; GET THE CURRENT SLEW POSITION
1094		1094	STA SLCRT ; AND STORE IT IN THE MEMORY LOCATION 'SLCRT'
1095		1095	INX H ; FIND THE ABSOLUTE DISTANCE BETWEEN CURRENT AND THE NEXT MIN.
1096		1096	SUB M ; EVALUATE THE DISTANCE (DIFFERENCE IN SLEW)
1097		1097	JNC \$+5 ; EVALUATE THE ABSOLUTE BY TWO'S COMPLEMENT
1098		1098	CMA
1099		1099	INR A ; STORE THE ABSOLUTE DISTANCE IN REGISTER D
1100		1100	MOV D,A ; FIND THE ABSOLUTE DISTANCE BETWEEN CURRENT AND THE NEXT MAX.
1101		1101	INX H ; GET THE CURRENT SLEW POSITION FROM E
1102		1102	MOV A,E ; EVALUATE THE DISTANCE (DIFFERENCE IN SLEW)
1103		1103	SUB M
1104		1104	JNC \$+5
1105		1105	CMA
1106		1106	INR A
1107		1107	CMP D
1108		1108	LXI D, FIRSL ; COMPARE TO CHECK WHICH IS THE SHORTEST
1109		1109	JC CONT5 ; GET THE ADDRESS FOR STORING THE FIRST SLEW DATA
1110		1110	DCX H ; THE PREVIOUS ONE IS THE SHORTEST
1111		1111	MOV A,M ; FIRST REQUIRED SLEW IS FOLLOWED BY THE SECOND ONE
1112		1112	STAX D
1113		1113	INX H
1114		1114	INX D
1115		1115	JMP CONT6 ; THE CURRENT ONE IS THE NEAREST TO THE CUTTER
1116		1116	CONT5: MOV A,M
1117		1117	STAX D
1118		1118	INX D
1119		1119	DCX H
1120		1120	CONT6: MOV A,M
1121		1121	STAX D
1122		1122	POP D
1123		1123	POP H
1124		1124	POP PSW
1125		1125	RET ; RECOVER THE REGISTERS BEFORE RETURNING
1126		1126	;

L0C	0BJ	LINE	SOURCE STATEMENT
05B6	F5	1127	*****
05D7	E5	1128	***** A ROUTINE TO REDUCE THE ENVELOPE TO A SMALLER SIZE BY HALF *****
05B8	05	1129	***** THE CUTTER ADVANCE STEP (I.E. HALF OF CUTSTP) *****
05B9	CD,705	1130	*****
05BC	2A1000	1131	REDUCE: PUSH PSW ; SAVE THE REGISTERS ON ENTRY
05BF	EB	1132	PUSH H
05C0	7A	1133	PUSH D
05C1	93	1134	CALL SHORT ; ON ENTRY: B,C POINT TO THE 'RAISE' DATA
05C2	D2C705	1135	LHLD FIRSL ; GET THE SLEW BOUNDS
05C5	2F	1136	XCHG ; H,L <---> D,E
05C6	9C	1137	MOV A,D ; CHECK IF REDUCTION IS REQUIRED
05C7	210100	1138	SUB E ; B: COMPARING THE ABSOLUTE DIFFERENCE BETWEEN BOUNDS AND
05CA	6E	1139	JNC \$+5 ; THE CUTTER ADVANCE STEP (CUTSTP)
05CB	DAE205	1140	CMA
05CE	211600	1141	INR A ; POINT H, L TO THE CUTSTP FOR COMPARISON
05D1	7A	1142	LXI H,CUTSTP
05D2	8B	1143	CNP M
05D3	DADD05	1144	JC NORED ; SKIP THE REDUCTION SINCE THE SEPERATION IS LESS
05D6	96	1145	LXI H,HALFST ; OTHERWISE REDUCE THE BOUNDARY DATA
05D7	57	1146	MOV A,D ; CHECK THE WAY OF REDUCING THE BOUNDS
05D8	7B	1147	CMP E
05D9	86	1148	JC REDU1 ; JUMP TO REDUCE E AND INCREASE D
05DA	C3E105	1149	SUB M ; OTHERWISE REDUCE D AND INCREASE E BY HALFST
05DB	86	1150	MOV D,A ; NO CHECKING FOR OVERFLOW/UNDERFLOW SINCE DATA ARE AUTOMATICALLY
05DE	57	1151	MOV A,E ; WITHIN THE BOUNDS
05DF	7B	1152	ADD M
05E0	96	1153	JMP REDU2 ; JUMP TO CONTINUE
05E1	5F	1154	ADD M ; INCREASE D AND REDUCE E
05E2	EB	1155	MOV D,A
05E3	221000	1156	MOV A,E
05E4	D	1157	SUB M
05E5	D1	1158	MOV E,A
05E6	D1	1159	XCHG ; XCHG
05E7	E1	1160	SHLD FIRSL ; STORE THEM BACK TO THE LOCATIONS
05E8	F1	1161	POP D
05E9	C9	1162	POP H
		1163	POP PSW
		1164	RET
		1165	*EJECT

LOC	OBJ	LINE	SOURCE STATEMENT
		1166	RC)
		=1167	*****
		=1168	***** A ROUTINE TO MOVE THE BOOM SLOWLY TO THE DESIRED POSITION *****
		=1169	*****
		=1170	PUBLIC SMOVE, SMOVEI, SMOVEF
		=1171	*****
		=1172	SMOVE: PUSH PSW ; SAVE THE REGISTERS
		=1173	PUSH H
		=1174	PUSH D
		=1175	PUSH B
		=1176	FMOVES: IN 01H
		=1177	ORI 003H
		D =1178	SMOVEF: STA MDSIGN
		D =1179	LDA FORCE
		=1180	CPI 01AH
		=1181	JC ANYMOD
		D =1182	LDA MDSIGN
		=1183	OKI 003H
		D =1184	STA MDSIGN
		D =1185	LDA FORCE
		=1186	CPI 0E5H
		=1187	JNC STOPMD
		D =1188	ANYMOD: LHLD SLDESR
		=1189	
		=1190	XCHG
		D =1191	LHLD SLCRRT
		=1192	
		0614 7D	MOV A,L
		0615 93	SUB E
		0616 17	RAL
		0617 5F	MOV E,A
		0618 1F	RAR
		0619 D21E06	JNC \$+5
		061C 2F	CMA
		061D 3C	INR A
		061E FE02	CPI 02H
		0620 E5	PUSH H
		0621 210300	LXI H,SIGNAL
		0624 DA3606	JC CLOSL
		0627 7B	MOV A,E
		0628 1F	RAR
		0629 DA3106	JC
		062C 366F	MVI M,06FH
		062E C33806	RSCMP
		0631 369F	JMP
		0633 C33806	MVI M,09FH
		0636 36FF	JMP RSCMP
		063B 3A0200	MVI M,0FFH
		063E 17	LDA SORTFL
		063C D23106	RAL
		063F 2A0400	JNC MTSORT
		0642 7E	LHLD STARCV
		0643 BA	MOV A,M
		0644 DA4806	CMP D
		=1219	JC \$+4
		=1220	MOV D,A

LOC	OBJ	LINE	SOURCE STATEMENT
064B	2A0600	D = 1221	LHLD LASTCV ; GET THE UPPER LIMIT FOR THE ROOM RAISE
064B	7E	= 1222	MOV A, M
064C	BA	= 1223	CMR D ; COMPARE THIS VALUE WITH THE DESIRED VALUE
064D	D25106	= 1224	JNC \$+4
0650	57	= 1225	MOV D, A ; RESET THIS VALUE IF DESIRED > HIGHEST SOUND
0651	E1	= 1226	NTSORT: POP H ; RECOVER THE H, L REGISTERS
0652	7C	= 1227	MOV A, H ; CHECK FOR DIRECTION OF VERTICAL MOVEMENT REQUIRED
0653	92	= 1228	SUB D ; BY COMPARING THE CURRENT AND DESIRED RAISE VALUES
0654	17	= 1229	RAL ; SAVE THE CARRY FLAG IN THE
0655	57	= 1230	MOV D, A ; LSB OF THE REGISTER D
0656	1F	= 1231	RAK
0657	D25C06	= 1232	JNC \$+5
065A	2F	= 1233	CMR A
065B	3C	= 1234	INR A ; 0/3H
065C	FE02	= 1235	CPI 0/3H
065E	DA7E06	= 1236	JC CLOS
0661	7A	= 1237	MOV A, D
0662	1F	= 1238	RAR
0663	110300	D = 1239	LXI D, SIGNAL ; LOAD D, E WITH SIGNAL ADDRESS
0665	1A	= 1240	LDAX D ; GET THE SIGNAL
0667	DA7406	= 1241	JC UP1
066A	E6F6	= 1242	ANI 0F6H ; SET THE MOVE DOWN SIGNAL
066C	12	= 1243	STAX D ; STORE THIS NEW SIGNAL
066D	3A2400	D = 1244	LDA RSCRR ; GET THE CURRENT RAISE DATA
0670	5D	= 1245	DCR A
0671	C58106	= 1246	JMP CHKBND
0674	E6F9	= 1247	ANI 0F9H ; SET THE MOVE UP SIGNAL
0676	12	= 1248	STAX D ; STORE THIS NEW SIGNAL
067A	3C	= 1249	LDA RSCRR ; GET THE CURRENT RAISE DATA
067B	C58106	= 1250	INR A
067E	3A2400	D = 1252	CHKBND: LDA RSCRR ; GET THE CURRENT RAISE DATA
0681	F5	= 1253	PUSH PSM ; SAVE THE CURRENT PROCESSOR STATUS WORD
0682	3A0200	D = 1254	LDA SORTFL ; CHECK IF THE PROFILE DATA HAVE BEEN SORTED
0685	17	= 1255	RAL
0686	D22807	= 1256	JNC NTSRT1 ; SKIP IF THE PROFILE DATA HAVE NOT BEEN SORTED
0689	F1	= 1257	POP PSM ; RECOVER THE PROCESSOR STATUS WORD
068A	6F	= 1258	MOV L, A ; FIND THE ADDRESS IN THE SORTED TABLE
068B	2600	= 1259	MVI H, 000H ; FOR THIS 'RAISE' LEVEL
068D	25	= 1260	DAD H ; THIS ADDRESS = STADD + 4*RAISE
068E	29	= 1261	DAD H
068F	0100BC	= 1262	LXI B, STADD
0692	09	= 1263	DAD B
0693	EB	= 1264	XCHG ; TEMPORARILY STORE IN D, E
0694	2A0600	D = 1265	LHLD LASTCV ; LIMIT THE MAXIMUM ADDRESS TO LASTCV
0697	7A	= 1266	MOV A, D
0698	BC	= 1267	CMR H
0699	DAAS06	= 1268	JC CHKB1 ; NO NEW SETTING IF LESS THAN (<)
069C	C2A406	= 1269	JNZ CHKB2 ; SET NEW ADDRESS IF GREATER (>)
069F	7B	= 1270	MOV A, E ; CHECK FOR THE LOW ORDER BYTE IF EQUAL (=)
06A0	BD	= 1271	CMR L
06A1	DAAS06	= 1272	JC CHKB1 ; NO NEW SETTING IF LESS (<)
06A4	EB	= 1273	XCHG ; OTHERWISE SET THIS MAXIMUM ADDRESS (>=)
06A5	2A0400	D = 1274	LHLD STARC ; LIMIT THE MINIMUM ADDRESS TO THE STARC BLOCK
06A8	7A	= 1275	MOV A, D

LOC	OBJ	LINE	SOURCE STATEMENT
06A9	BC	=1276	CMP H
06AA	DAB506	=1277	JC CHKB3
06AD	C2B606	=1278	JNZ CHKB4
06B0	7B	=1279	MOV A,E
06B1	8D	=1280	CMP L
06B2	D2B606	=1281	JNC CHKB4
06B5	EB	=1282	XCHC
06B6	EB	=1283	XCHC
06B7	23	=1284	INX H
06E8	3A1700	D =1285	LDA MDSIGN
06BB	1F	=1286	RAR
06BC	D2C306	=1287	JNC FASTSL
06BF	1F	=1288	RAR
06C0	DAEA06	=1289	JC NTFAST
06C3	C5	=1290	PUSH B
06C4	7E	=1291	MOV A,M
06C5	C618	=1292	ADI 018H
06C7	D2CB06	=1293	JNC #+4
06CA	9F	=1294	SBB A
06CB	4F	=1295	MOV C,A
06CC	23	=1296	INX H
06CD	7E	=1297	MOV A,M
06CE	D617	=1298	SUI 017H
06D0	D2D406	=1299	JNC #+4
06D3	97	=1300	SUB A
06D4	47	=1301	MOV B,A
06D5	2B	=1302	DCX H
06D6	3A2300	D =1303	LDA SLRRT
06D9	B9	=1304	CMP C
06DA	DAE106	=1305	JC FASTM2
06DD	B8	=1306	CMP B
06DE	DAE906	=1307	JC FASTM3
06E1	3A1700	D =1308	LDA MDSIGN
06E4	F603	=1309	ORI 03H
06E6	321700	D =1310	STA MDSIGN
06E9	C1	=1311	FASTM3: POP B
06EA	3A2300	D =1312	LDA SLRRT
06ED	BE	=1313	CMP M
06EE	DAFC06	=1314	JC NOVER1
06F1	C20407	=1315	JNZ VERT2
06F4	3A0300	D =1316	LDA SIGNAL
06F7	F690	=1317	ORI 090H
06F9	C50107	=1318	JMP VERT3
06FC	3A0300	D =1320	LDA SIGNAL
06FF	F69F	=1321	ORI 09FH
0701	320500	D =1322	VERT3: STA SIGNAL
0704	23	=1323	VERT2: INX H
0705	3A2300	D =1324	LDA SLRRT
0708	BE	=1325	CMP M
0709	DA2207	=1326	JC VERT4
070C	CA1707	=1327	JZ ONVER2
070F	3A0300	D =1328	LDA SIGNAL
0712	F65F	=1329	ORI 06FH
		=1330	

LOC	OBJ	LINE	SOURCE STATEMENT
0714	C31C07	=1331	JMP VERTS
0717	3A0300	D =1332	ONVER2: LDA SIGNAL
071A	F660	=1333	ORI 060H
		=1334	
071C	320300	D =1335	VERTS: STA SIGNAL
071F	C32C07	=1336	JMP OUTRS
0722	3A0300	D =1337	VERT4: LDA SIGNAL
0725	C32C07	=1338	JMP OUTRS
072B	F1	=1339	NTSKRT1: POP PSW
0729	3A0300	D =1340	LDA SIGNAL
072C	D300	=1341	OUTRS: OUT 00H
072E	3A1700	D =1342	LDA RDSIGN
0731	D301	=1343	OUT 01H
0733	C1	=1344	POP B
0734	D1	=1345	POP D
0735	E1	=1346	POP H
0736	F1	=1347	POP PSW
0737	C9	=1348	RET
0738	3EFF	=1349	STOPMD: MVI A,0FFH
073A	320300	D =1350	STA SIGNAL
073D	C32C07	=1351	JMP OUTRS
		=1352	*****
0740	F5	=1353	SMOVE1: PUSH PSW
0741	E5	=1354	PUSH H
0742	F3	=1355	SMOVE2: DI
0743	CD4000	=1356	CALL SETLNK
0746	3AA21B	=1357	LDA CRTSL
0749	6F	=1358	MOV L,A
074A	3AAD1B	=1359	LDA CRTTRS
074D	67	=1360	MOV H,A
074E	222300	D =1361	SHLD SLCRRT
0751	3ADD1B	=1362	LDA CUTFOR
0754	321D00	D =1363	STA FORCE
0757	CD5200	=1364	CALL OFFLNK
075A	FB	=1365	EI
075B	CD6A05	=1366	CALL SMOVE
075E	DB00	=1367	IN 00H
0760	2F	=1368	CMA
0761	E7	=1369	ORA A
0762	C24207	=1370	JNZ SMOVE2
0765	CD6F07	=1371	CALL CHKPOS
076B	17	=1372	RAL
0769	D24207	=1373	JNC SMOVE2
076C	E1	=1374	POP H
076D	F1	=1375	POP PSW
076E	C9	=1376	RET
076F	3A1D00	D =1377	CHKPOS: LDA FORCE
0772	FE05	=1378	CPI 0ESH
0774	9F	=1379	SBB A
		=1380	
		=1381	00 --- IF IT IS STOPPED DUE TO HIGH CUTTING FORCE
		=1382	FF --- IF IT IS STOPPED DUE TO POSITION ACHIEVED
0775	C9	=1383	RET

LOC	OBJ	LINE	SOURCE STATEMENT
		1384	*INCLUDE(F1:FMOVE1.SRC)
		=1385	*****
		=1386	** A ROUTINE TO FAST MOVE THE BOOM TO THE DESIRED POSITION *****
		=1387	*****
		=1388	PUBLIC FMOVE
		=1389	*****
		=1390	ASEC
		=1391	ORG 080AEH ; DECLARE ABSOLUTE PROGRAM CODE SEGMENT
		=1392	FMOVE: ; FMOVE STARTING ADDRESS: 80AE H
		=1393	PUSH PSW ; SAVE THE REGISTERS
		=1394	PUSH H
		=1395	PUSH D
		=1396	PUSH B
		=1397	DI ; DISABLE THE INTERRUPT
		=1398	LXI H, 16D ; SET THE BYTE COUNT FOR DATA TRANSFER
		=1399	PUSH H
		=1400	CALL SETLNK ; SET UP THE DATA LINK BETWEEN THE TWO SYSTEMS
		=1401	LXI B, SLVEL ; SET THE STARTING ADDRESS FOR SOURCE DATA BLOCK
		=1402	LXI D, VELSL ; SET THE STARTING ADDRESS FOR DESTINATION DATA BLOCK
		=1403	CALL TRANSF ; TRANSFER THE DATA
		=1404	LDA CRTSL ; GET AND STORE THE CURRENT BOOM POSITION
		=1405	MOV L, A
		=1406	LDA CRRTRS
		=1407	MOV H, A
		=1408	SHLD SLCRRT
		=1409	LDA CUTFOR ; GET THE CUTTER MOTOR CURRENT READING
		=1410	STA FORCE ; STORE THIS VALUE
		=1411	CALL OFFLNK ; RELEASE THE DATA LINK BETWEEN THE TWO SYSTEMS
		=1412	PUP H ; POP THE STACK TO KEEP THE BALANCE
		=1413	EI
		=1414	IN 01H ; GET THE CURRENT VALVE STATUS
		=1415	ANI 0FCH ; SET THE SPEED CONTROL VALVE TO FAST MODE BY ANDING IT WITH 11111005
		=1416	PUSH PSW
		=1417	LHLD SLDESR
		=1418	XCHG
		=1419	LHLD SLCRRT
		=1420	
		=1421	MOV A, L ; LOAD THE REGISTER L WITH THE DESIRED SLEW VALUE
		=1422	SUB E ; AND THE REGISTER H WITH THE DESIRED RAISE VALUE
		=1423	JNC \$+5 ; EXCHANGE REGISTERS D, E WITH REGISTERS H, L
		=1424	CMA
		=1425	INR A ; LOAD THE REGISTER L WITH THE CURRENT SLEW VALUE
		=1426	CPI 018H ; AND THE REGISTER H WITH THE CURRENT RAISE VALUE
		=1427	JNC \$+7 ; LOAD THE ACCUMULATOR WITH THE CURRENT SLEW VALUE
		=1428	POP PSW
		=1429	ORI 01H ; LOAD THE REGISTER H WITH THE CURRENT RAISE VALUE
		=1430	PUSH PSW ; SUBTRACT THE DESIRED SLEW VALUE FROM THIS CURRENT VALVE
		=1431	MOV A, H
		=1432	SUB D
		=1433	JNC \$+5
		=1434	CMA
		=1435	INR A ; SET SLEW TO SLOW MODE
		=1436	CPI 018H ; SUBTRACT THE DESIRED RAISE VALUE FROM THE CURRENT VALVE
		=1437	JNC \$+7
		=1438	POP PSW



LOC	OBJ	LINE	SOURCE STATEMENT
8102	F602	=1439	ORI 003H ; SET THE RAISE TO SLOW MODE
8104	F5	=1440	PUSH PSW ; PUT THIS SIGNAL ONTO THE STACK
8105	F1	=1441	POP PSW ; GET BACK THE MOVE-MODE SIGNAL
8106	C3F205	=1442	JMP 5MOVEF ; JUMP TO CONTINUE THE OPERATION
		1443	INCLUDE(F1;DATA,SRC)
		=1444	*****
		=1445	DATA SEGMENT FOR THE BOOM POSITIONAL CONTROL PROGRAM
		=1446	*****
		=1447	PUBLIC CRTSL, CRTRS, MOVESL, DESRSL, DESRRS, PFLEND, PFLST, CUTTER
		=1448	PUBLIC SLEVL, SLACC, RSVEL, RSACC, STADD, LASTAD, DDR01, TEMP, TEMPI, TEMPRI
		=1449	PUBLIC SLCRT, RSCRRT, SLDES, RSDES, VELSL, VELRS, ACCSL, ACCRS, PREVSL, PREVRS
		=1450	PUBLIC SIMUX, SIMUY, SIMUAX, SIMUAY, SIMUAZ
		=1451	ASEC
		=1452	ORC 01BA2H
		=1453	*****
		=1454	SYMBOLS TO BE USED IN THE DATA CAPTURING SYSTEM
		=1455	*****
		=1456	SYMBOL
		=1457	*****
		=1458	CRTSL: DS 1 ; CURRENT SLEW POSITION (1 BYTE)
		=1459	ORC 01BADH
		=1460	CRTRS: DS 1 ; CURRENT RAISE POSITION (1 BYTE)
		=1461	ORC 01B0AH
		=1462	DESKSL: DS 1 ; DESIRED SLEW POSITION (1 BYTE)
		=1463	DESRSL: DS 1 ; DESIRED RAISE POSITION (1 BYTE)
		=1464	MOVEST: DS 1 ; CURRENT MOVE LEVER STATUS (1 BYTE)
		=1465	CUTFOR: DS 1 ; CUTTER MOTOR CURRENT (1 BYTE)
		=1466	ORC 01BE3H
		=1467	PREVSL: DS 1 ; CURRENT SLEW POSITION (1 BYTE)
		=1468	PREVRS: DS 1 ; CURRENT RAISE POSITION (1 BYTE)
		=1469	SLEVL: DS 4 ; CURRENT SLEW VELOCITY (4 BYTES)
		=1470	SLACC: DS 4 ; CURRENT SLEW ACCELERATION (4 BYTES)
		=1471	RSVEL: DS 4 ; CURRENT RAISE VELOCITY (4 BYTES)
		=1472	RSACC: DS 4 ; CURRENT RAISE ACCELERATION (4 BYTES)
		=1473	ORC 1BFEH
		=1474	PFLEND: DS 2 ; LAST ADDRESS FOR PROFILE DATA (2 BYTES)
		=1475	PFLST: DS 400H ; STARTING ADDRESS FOR PROFILE DATA (MAX. 400H BYTES)
		=1476	ORC 0B8FEH
		=1477	*****
		=1478	SYMBOLS TO BE USED IN THE MAIN CONTROL SYSTEM PROGRAM
		=1479	*****
		=1480	SYMBOL
		=1481	*****
		=1482	LASTAD: DS 2 ; LAST ADDRESS FOR THE PROFILE (2 BYTES)
		=1483	STADD: DS 400H ; DATA STORAGE FOR THE PROFILE (MAX. 400H BYTES)
		=1484	DSEC
		=1485	STKPTR: DS 0 ; DEFINE DATA SEGMENT
		=1486	*****
		=1487	INTMSK: DS 1 ; DEFINE THE HIGHEST STACK BYTE TO BE THE ONE BELOW
		=1488	CUTSTP: DS 1 ; STORAGE LOCATION FOR THE INTERRUPT MASK
		=1489	SORTFL: DS 1 ; LOCATION FOR THE CUTTER STEP SIZE IN ADVANCING
		=1490	*****
		=1491	00 ; 00 --- NOT YET SORTED
		=1492	SIGNAL: DS 1 ; SIGNAL FOR THE DIRECTIONAL VALVES
		=1493	STARCV: DS 2 ; DEFINE THE STORAGE FOR STARTING CAVITY 'RAISE' LEVEL

LOC	OBJ	LINE	SOURCE STATEMENT
0006		1494	LASTCV: DS 2 ; DEFINE THE STORAGE FOR THE LAST 'CAVITY' LEVEL
0008		1495	LASTSO: DS 2 ; DEFINE THE STORAGE FOR THE LAST SORTED CAVITY ADDRESS
000A		1496	LEVEL: DS 2 ; DEFINE THE STORAGE FOR THE ADDRESS FOR 'X'
000C		1497	DMINI: DS 2 ; DEFINE THE STORAGE FOR TEMPORARY MIN
000E		1498	DMAXI: DS 2 ; DEFINE THE STORAGE FOR TEMPORARY MAX
0010		1499	FIRSL: DS 2 ; TEMPORARY STORAGE FOR THE NEXT TWO DESIRED SLEW VALUES
0012		1500	FOURST: DS 2 ; TEMPORARY STORAGE FOR 4*(CUTTER STEP SIZE)
0014		1501	LASTCT: DS 2 ; STORAGE FOR THE ADDRESS OF THE LAST DATA BLOCK IN CAVITY CUTTING
0016		1502	HALFST: DS 1 ; STORAGE FOR HALF CUTTER STEP SIZE
0017		1503	MDSIGN: DS 1 ; STORAGE FOR THE BOOM MOVING SPEED MODE
0018		1504	DDR01: DS 1 ; DATA DIRECTION REGISTER FOR PORT 1
0019		1505	CUTTER: DS 1 ; CUTTER STATUS --- 00=STOPPED, OFFH=STARTED
001A		1506	SLDESR: DS 1 ; DESIRED SLEW POSITION (1 BYTE)
001B		1507	RSDESR: DS 1 ; DESIRED RAISE POSITION (1 BYTE)
001C		1508	STMOVE: DS 1 ; MOVE CONTROL STATUS (1 BYTE)
001D		1509	FORCE: DS 1 ; CUTTER MOTOR CURRENT (1 BYTE)
001E		1510	SIMUX: DS 1 ; SIMULATED X DISPLACEMENT (1 BYTE)
001F		1511	SIMUY: DS 1 ; SIMULATED Y DISPLACEMENT (1 BYTE)
0020		1512	SIMUAX: DS 1 ; SIMULATED ALPHA X (1 BYTE)
0021		1513	SIMUAY: DS 1 ; SIMULATED ALPHA Y (1 BYTE)
0022		1514	SIMUAZ: DS 1 ; SIMULATED ALPHA Z (1 BYTE)
0023		1515	SLCRR: DS 1 ; CURRENT SLEW POSITION (1 BYTE)
0024		1516	RSCRR: DS 1 ; CURRENT RAISE POSITION (1 BYTE)
0025		1517	VELSL: DS 4 ; CURRENT SLEW VELOCITY (4 BYTES)
0029		1518	ACCSL: DS 4 ; CURRENT SLEW ACCELERATION (4 BYTES)
002D		1519	VELRS: DS 4 ; CURRENT RAISE VELOCITY (4 BYTES)
0031		1520	ACCRS: DS 4 ; CURRENT RAISE ACCELERATION (4 BYTES)
0035		1521	TEMP: DS 4 ; TEMPORARY STORAGE LOCATION (4 BYTES)
0039		1522	TEMP1: DS 4 ; TEMPORARY STORAGE FOR RESULT (4 BYTES)
003D		1523	TEMPR: DS 4 ; TEMPORARY STORAGE FOR RESULT (4 BYTES)
0041		1524	TEMPRI: DS 4 ; TEMPORARY STORAGE FOR RESULT (4 BYTES)
0045		1525	DIFF1: DS 4 ; DIFFERENCE BETWEEN CURRENT AND DESIRED (4 BYTES)
0049		1526	SWFN: DS 4 ; SWITCHING FUNCTION VALUE (4 BYTES)
004D		1527	DUMMY1: DS 4 ; SWITCHING FUNCTION VALUE (4 BYTES)
0051		1528	FNSIGN: DS 1 ; EXPECTED SIGN FOR SWITCHING (1 BYTE)
		1529	*****
		1530	*****
		1531	*****
		1532	#\$EJECT

LOC OBJ L.I.NE SOURCE STATEMENT

-1539 END

PUBLIC SYMBOLS

ACCRS D 0031	ACCSL D 0029	CAVITY A 021B	CRRTRK A 1B4D	CRR'TSL A 1B2A	CUTSTP D 0001	CUTTER D 0019
DDR01 D 0018	DESRRS A 1BDB	DESRSL A 1BDA	DMINI D 000C	EXPAND A 0509	FIRSL D 0010	FMOVE A 80AE
FOURST D 0012	INTMSK D 0000	LASTAD A 1BFE	PFLST A 1C00	LAST50 D 0008	LEVEL D 000A	MOVE A 02DC
MOVEST A 1BDC	OFFLNK A 0052	PFLND A 1BFE	R5T45 A 032B	PREVRS A 1BE4	PREVSL A 1BE3	PROFIL A 00B5
KSALC A 1BF1	RSCRRT D 0024	R5DESR D 001B	SIMUJZ A 0022	R5VEL A 1BED	SETLNK A 0040	SHORT A 057B
SIGNAL D 0003	SIMUAX D 0020	SIMUAY D 0021	SMOVE A 05EA	SIMUX D 001E	SIMUY D 001F	SLACC A 1BE9
SLCRRT D 0023	SLDESR D 001A	SLVEL A 1BE5	STKPTR D 0000	SMOVE1 A 0740	SMOVEF A 05F2	SMOVT A 030D
SORTFL D 0002	STADD A 1C00	STARCV D 0004	VELRS D 002D	TEMP D 0035	TEMP1 D 0039	TEMPR D 003D
TEMPRI D 0041	TRANSF A 0097	TRNPF1 A 0075		VELSL D 0025		

EXTERNAL SYMBOLS

ACCRS D 0031	ACCSL D 0029	ANYMOD A 060D	CALSMV A 0266	CAVITY A 021B	CHKBI A 06A5	CHK'E2 A 06A4
CHK'B3 A 06B5	CHK'POS A 0681	CHK'ND A 0681	CHK'POS A 076F	CLOS1 A 0636	CLOS2 A 067E	COLDST A 0009
COMPL A 03C0	CNT11 A 041A	CNT12 A 04B6	CNT21 A 0411	CNT22 A 045B	CNT31 A 0422	CONT32 A 045E
CNT41 A 03F0	CNT42 A 0449	CNT5 A 05AC	CNT52 A 049D	CNT6 A 05B0	CNT62 A 047B	CONTC0 A 0299
CNTC2 A 02AE	CNTC5 A 027F	CTPRO1 A 01FB	CONTR1 A 0380	CONTR2 A 038C	CRRTRK A 1B4D	CRR'TSL A 1A2A
CSTORE A 04B1	CTPRO1 A 0109	CUTFOR A 1BDD	CUTFOR A 1BDD	CUTSTP D 0001	CUTTER D 0019	DDR01 D 0018
DESRRS A 1B0K	DESRSL A 1BDA	DIFF1 D 0045	DMAX1 D 000E	DMINI D 000C	DOWN1 A 066A	DUMMY1 D 004D
DUP1 A 04C7	DUP11 A 04CE	DUP12 A 04E0	EXPA1 A 0525	EXPA2 A 053D	EXPA3 A 0541	EXPA4 A 054E
EXPAN A 056E	EXPAN A 0509	FAS11 A 01BF	FASTM2 A 06E1	FASTM3 A 06E9	FAS1 A 00E6	FAS1 A 00F7
FASTSL A 06C3	FIRSL D 0010	FMOVE A 80AE	FMOVES A 05EE	FNSICN D 0051	FORCE D 001D	FOURST D 0012
HALFST D 0016	INTMSK D 0000	LASTAD A 1BFE	LASTCT D 0014	LAS1CV D 0006	LASTSD D 0008	LEFT1 A 062C
LEVEL D 000A	LOOP A 01F0	LOOP1 A 007F	LOOP11 A 03E8	LOOP12 A 0441	LOOP3 A 00A5	MDSIGN D 0017
MUVE A 02DC	MOVE1 A 0316	MOVE2 A 02E7	MOVEST A 1BDC	NDUP A 04E6	NOHVE A 031C	NORED A 05E2
NOTLOW A 0170	NOTSRT A 0160	NOTUP A 0119	NOVER1 A 06FC	NTFAST A 06EA	NTSORT A 0651	NTSRT1 A 072B
OFFLNK A 0052	ONVER2 A 0717	OUTRS A 072C	OVER1 A 0193	PFLND A 1BFE	PFLST A 1C00	PREVRS A 1BE4
RIGHT1 A 0631	RSACC A 1BF1	REDU1 A 05DD	R5DU2 A 05E1	REDUCE A 05E6	REDULS A 024D	REDUST A 023D
RST55 A 002C	RST65 A 0094	RST75 A 003C	R5VEL A 1BED	R5DESR D 001B	RST3B A 0059	RST45 A 032B
SIMUAX D 0020	SIMUAY D 0021	SIMUJZ A 0022	SIMUX D 001E	SETLNK A 0040	SHORT A 057B	SIGNAL D 0003
SLDESR D 001A	SLVEL A 1BE5	SMOVE A 05EA	SMOVE1 A 0740	SIMUY D 001F	SLACC A 1BE9	SLCRRT D 0023
SORTFL D 0002	STADD A 1C00	STARCV D 0004	STAR1 A 0600	STKPTR D 0000	SMOVEF A 05F2	SORT A 030D
TEMPRI D 0041	TRANSF A 0097	TRNPF1 A 0075	TRAP A 0024	TEMP D 0035	TEMP1 D 0039	TEMPR D 003D
VELBL D 0025	VENT2 A 0704	VENT3 A 0701	VERT4 A 0722	VERT5 A 071C	UP1 A 0674	VELKS D 002D

ASSEMBLY COMPLETE, NO ERRORS

APPENDIX H

LISTING OF THE ASSEMBLY LANGUAGE PROGRAMS  
FOR THE 'FMOVE' ROUTINE WITH SWITCHING  
FUNCTION EVALUATION

LOC	OBJ	LINE	SOURCE STATEMENT
		1385	\$INCLUDE(:F1:FMOVE.SRC)
		=1386	*****
		=1387	*****
		=1388	*****
		=1389	*****
			ASEC
			ORC 08000H
8000		=1391	\$INCLUDE(:F1:CONAPU.SRC)
		1-1392	*****
		1-1393	ROUTINES TO SEND/READ BYTES TO/FROM APU STACK
		1-1394	I.E. MEMORY (<---> APU STACK
		1-1395	ON ENTRY: REGISTERS B,C CONTAIN STARTING MEMORY ADDRESS
		1-1396	WITH LEAST SIGNIFICANT BYTE FIRST
		1-1397	THE APU 8231 IS MEMORY MAPPED WITH ADDRESSES FC00H - FDFEH
		1-1398	DATA WITH ADDRESSES: 0FCXX H
		1-1399	COMMAND WITH ADDRESSES: 0FDXX H
		1-1400	*****
		1-1401	*****
		1-1402	*****
		1-1403	*****
		1-1404	*****
		1-1405	*****
		1-1406	*****
		1-1407	*****
		1-1408	*****
		1-1409	*****
		1-1410	*****
		1-1411	ASEC
		1-1412	PUBLIC SEND2, SEND4, READ2, READ4, SEND30, SEND10, CHKBSY
8000 E5		1-1413	SEND4: PUSH H ; SAVE H, L-REGISTERS
8001 2104FC		1-1414	LXI H, 100H*APUD+04H ; LOAD APU DATA STACK ADDRESS AND COUNT
		1-1415	; VALUE FOR BYTE TRANSFER
8004 C5		1-1416	SENDL1: PUSH B ; SAVE B, C-REGISTERS
8005 F5		1-1417	PUSH PSW ; SAVE PROCESSOR STATUS WORD
8006 0B		1-1418	DCX B ; GET MEMORY ADDRESS PREPARED
8007 03		1-1419	SENDL2: INX B
8008 0A		1-1420	LDAX B ; GET BYTE FROM MEMORY AND
8009 77		1-1421	MOV M, A ; SEND BYTE TO APU DATA STACK
800A 2D		1-1422	DCR L
800B C20780		1-1423	JNZ SENDL2 ; RECOVER REGISTERS BEFORE RETURN
800E F1		1-1424	POP PSW
800F C1		1-1425	POP B
8010 E1		1-1426	POP H
8011 C9		1-1427	RET ; RETURN TO CALLING PROGRAMME
8012 E5		1-1428	SEND2: PUSH H ; SAME AS SEND4 BUT WITH A BYTE COUNT OF 2
8013 2102FC		1-1429	LXI H, 100H*APUD+02H ; LOAD APU DATA STACK ADDRESS AND COUNT
		1-1430	; VALUE FOR BYTE TRANSFER
8016 C30480		1-1431	JMP SENDL1
		1-1432	*****
		1-1433	*****
		1-1434	*****
		1-1435	ROUTINE TO READ AND STORE 4 BYTES FROM THE APU DATA STACK
		1-1436	*****
8019 E5		1-1435	READ4: PUSH H ; LOAD COUNT VALUE FOR BYTE TRANSFER
801A 210404		1-1436	LXI H, 0404H ; SAVE PROCESSOR STATUS WORD
801D F5		1-1437	READL3: PUSH PSW ; LOCATE MEMORY ADDRESS TO THE MOST SIGNIFICANT
801E 03		1-1438	READL4: INX B ; LOCATE MEMORY ADDRESS TO THE MOST SIGNIFICANT
801F 25		1-1439	DCR H ; BYTE AND READY FOR DATA TRANSFER

LOC	OBJ	LINE	SOURCE STATEMENT
8020	C21E80	1-1440	JNZ READL4
8023	DEFC	1-1441	IN APUD
8025	0B	1-1442	DCX B
8026	02	1-1443	STAX B
8027	20	1-1444	DCR L
802B	C22380	1-1445	JNZ READL5
802B	F1	1-1446	POP PSW
802C	E1	1-1447	POP H
802D	C9	1-1448	RET
802E	E5	1-1449	PUSH H
802F	210202	1-1450	LXI H, 0202H
8032	C31D80	1-1451	JMP READL3
8035	F5	1-1452	*****
8036	E5	1-1453	ROUTINE TO SEND DATA BYTES WITH ZERO LSBs
8037	26FC	1-1454	*****
8039	0A	1-1455	SEND30: PUSH PSW
803A	77	1-1456	PUSH H
803B	97	1-1457	MVI H, APUD
803C	77	1-1458	LDAX B
803D	77	1-1459	MOV M, A
803E	77	1-1460	SUB A
803F	E1	1-1461	MOV M, A
8040	F1	1-1462	MOV M, A
8041	C9	1-1463	MOV M, A
8042	F5	1-1464	POP H
8043	E5	1-1465	POP PSW
8044	26FC	1-1466	RET
8045	0A	1-1467	PUSH PSW
8046	0A	1-1468	PUSH H
8047	77	1-1469	MVI H, APUD
8048	97	1-1470	LDAX B
8049	C33E80	1-1471	MOV M, A
		1-1472	SUB A
		1-1473	JMP SEND31
		1-1474	*****
		1-1475	A ROUTINE TO CONVERT THE FLOATING POINT FORMAT RECOGNIZED
		1-1476	BY THE FLOATING POINT LIBRARY (FPAL) TO THAT RECOGNIZED
		1-1477	BY THE ARITHMETIC PROCESSING UNIT 8231A
		1-1478	ON ENTRY: REGISTERS B,C --- ADDRESS OF THE LOWEST BYTE OF
		1-1479	THE 4-BYTE BFP NUMBER
		1-1480	ON EXIT: REGISTERS B,C --- ADDRESS OF THE LOWEST BYTE OF
		1-1481	THE 4-BYTE BFP NUMBER
		1-1482	*****
		1-1483	PUBLIC CONVER
		1-1484	*****
		1-1485	CONVER: PUSH PSW
		1-1486	PUSH D
		1-1487	PUSH B
		1-1488	INX B
		1-1489	INX B
		1-1490	LDAX B
		1-1491	RAL
		1-1492	INX B
		1-1493	LDAX B
		1-1494	RAL
804C	F5		*****
804D	D5		ROUTINE TO SEND 2 BYTES TO THE APU DATA STACK WITH 1 ZERO MSB
804E	C5		THE ADDRESS OF THE MSB IS SPECIFIED IN REGISTERS B,C
804F	03		*****
8051	0A		GET THE LSB
8052	17		SEND IT TO THE APU DATA STACK
8053	03		*****
8054	0A		JUMP TO SEND THE DATA TO THE APU STACK
8055	17		*****

LOC	OBJ	LINE	SOURCE STATEMENT
8056	F600	1-1495	ORI 00H ; CHECK IF THE EXPONENT IS ZERO
8059	CA9480	1-1496	JZ RET10 ; IF YES THEN NO CONVERSION IS NEEDED
805B	1E7E	1-1497	MVI E,07EH ; TAKE AWAY THE OFFSET OF THE EXPONENT FIELD
805D	93	1-1498	SUB E ; EXPONENT IS OFFSET BY 07EH
805E	57	1-1499	MOV D,A ; TEMPORARILY STORE RESULT IN D
805F	DA7890	1-1500	JC NEGAT ; EXPONENT IS NEGATIVE
8062	FE40	1-1501	POSIT ; CHECK IF OVERFLOW THE EXPONENT BYTE
8064	DAB880	1-1502	JC CONT10 ; NO ADJUSTMENT IF WITHIN THE RANGE
8067	163F	1-1503	MVI D,03FH ; ASSUME THE LARGEST NUMBER IF OVERFLOW
8069	3EFF	1-1504	MVI A,0FFH ; THE LARGEST NUMBER IS: 3F FF FF FF H
806B	0B	1-1505	DCX B
806C	02	1-1506	STAX B
806D	0B	1-1507	DCX B
806E	02	1-1508	STAX B
806F	0B	1-1509	DCX B
8070	02	1-1510	STAX B
8071	03	1-1511	INX B
8072	03	1-1512	INX B
8073	03	1-1513	INX B
8074	1C	1-1514	INR E
8075	C8B880	1-1515	JMP CONT20 ; JUMP TO CONTINUE
8078	FECO	1-1516	NEGAT ; CHECK IF UNDERFLOW
807A	D2B880	1-1517	JNC CONT10 ; NO ADJUSTMENT IF WITHIN THE RANGE
807D	97	1-1518	SUB A ; ASSUME ZERO IF UNDERFLOW
807E	02	1-1519	STAX B ; ZERO IS REPRESENTED BY: 00 00 00 00 H
807F	0B	1-1520	DCX B
8080	02	1-1521	STAX B
8081	0B	1-1522	DCX B
8082	02	1-1523	STAX B
8083	0B	1-1524	DCX B
8084	02	1-1525	STAX B
8085	C39480	1-1526	JMP RET10
8088	1C	1-1527	CONT10: INR E ; MASK OUT THE M.S.B.
8089	A3	1-1528	ANA E
808A	57	1-1529	MOV D,A ; AND TEMPORARILY STORE THIS IN D
808B	0A	1-1530	LDAX B ; EXTRACT THE SIGN BIT FROM THE LOWEST BYTE
808C	1C	1-1531	INR E ; MASK OUT THE 7 L.S.B.
808D	A3	1-1532	ANA E
808E	B2	1-1533	ORA D ; COUPLE THIS TO THE EXPONENT BYTE
808F	02	1-1534	STAX B ; STORE THIS RESULTING SIGN-EXPONENT BYTE
8090	0B	1-1535	DCX B ; GET THE FIRST FRACTION BYTE (SECOND BYTE OF THE 4)
8091	0A	1-1536	LDAX B
8092	E3	1-1537	ORA E ; PUT A 'ONE' IN THE M.S.B.
8093	02	1-1538	STAX B ; PUT THE RESULTING BYTE BACK TO THE MEMORY
8094	C1	1-1539	POP B ; RECOVER REGISTERS BEFORE RETURN
8095	D1	1-1540	POP D
8096	F1	1-1541	POP PSW
8097	C9	1-1542	RET
		1-1543	*****
		1-1544	RESET THE APU *****
		1-1545	*****
		1-1546	PUBLIC RSTAPU *****
8098	F5	1-1547	RSTAPU: PUSH PSW
8099	D801	1-1548	IN 01H ; GET THE PORT 01 SETTING
809B	F610	1-1549	ORI 010H ; SET THE HIGH ACTIVE BIT 4 TO RESET THE APU

LOC	OBJ	LINE	SOURCE STATEMENT
809D	D301	I=1550	OUT 01H ; SEND THIS RESET SIGNAL TO THE APU RESET PIN
809F	00	I=1551	NOP ; KEEP THE RESET SIGNAL ACTIVE FOR THE TIME REQUIRED
80A0	00	I=1552	NOP
80A1	E6EF	I=1553	ANI 0EFH ; REMOVE THE RESET SIGNAL FOR THE APU
80A3	D301	I=1554	OUT 01H
80A5	F1	I=1555	POP PSW
80A6	C9	I=1556	RET
80A7	D8FD	I=1557	
80A9	17	I=1558	CHKBSY: IN APUC ; GET THE APU STATUS WORD
80AA	DA4780	I=1559	RAL ; CHECK IF THE APU IS BUSY
80AD	C9	I=1560	JC \$-3 ; KEEP LOOPING IF IT IS BUSY
		I=1561	RET ; OTHERWISE RETURN TO THE CALLING SEGMENT
		I=1562	
		I=1563	*INCLUDE(FI:APUCMD.SRC)
		I=1564	*NOLIST
		I=1615	*LIST
		-1616	*****
		-1617	A ROUTINE FOR FAST MOVE OF BOOM CUTTER
		-1618	WITH THE SWITCHING FUNCTION EVALUATION
		-1619	VERTICAL MOVEMENTS HAVE PREFERENCE OVER
		-1620	HORIZONTAL MOVEMENTS
		-1621	*****
		-1622	PUBLIC FMOVE
		-1623	ASEC
80AE	F5	-1624	FMOVE: PSW ; SAVE THE REGISTERS ON ENTRY
80AF	E5	-1625	PUSH H ;
80B0	D5	-1626	PUSH D ;
80B1	C5	-1627	PUSH B ;
80B2	F3	-1628	DI ;
80B3	211000	-1629	LXI H, 16D ; DISABLE SYSTEM INTERRUPT
80B6	E5	-1630	PUSH H ; SET BYTE TRANSFER COUNTER
80B7	CD4000	-1631	CALL SETLINK ; SET THE LINK BETWEEN THE TWO SYSTEMS
80BA	01E31B	-1632	LXI B, SLVEL. ; SET THE STARTING ADDRESS FOR SOURCE DATA
80BD	112500	-1633	LXI D, VELSL. ; SET THE STARTING ADDRESS FOR DESTINATION
80C0	CD9700	-1634	CALL TRANSF ; TRANSFER BYTE
80C3	3ADD1B	-1635	LDA CUTFOR ; GET THE CUTTER FORCE
80C5	321D00	-1636	STA FORCE ; AND STORE IT
80C9	3AA21B	-1637	LDA CRRTSL ; GET THE CURRENT BOOM POSITION
80CC	6F	-1638	MOV L, A ;
80CD	3AAD1B	-1639	LDA CRRTS ;
80D0	67	-1640	MOV H, A ;
80D1	22300	-1641	SHLD SLCRRT ; AND STORE THIS INFORMATION
80D4	CD5200	-1642	CALL OFFLINK ; RELEASE THE LINK
80D7	E1	-1643	POP H ; SCRATCH THE BYTE COUNTER FOR TRANSFER
80D8	FB	-1644	EI ; ENABLE INTERRUPTS
80D9	5A1D00	-1645	LDA FORCE ; CHECK FOR THE FORCE SETTING
80DC	FE18	-1646	CPI 01BH ; LOWER BOUND = 01BH
80DE	D2EE05	-1647	JNC FMOVES ; JUMP TO SLOW MOVE ROUTINE IF LARGER
80E1	2A2500	-1648	LHLD LHL ; OTHERWISE, CHECK IF FAST MOVEMENT IS ALLOWED
80E4	EB	-1649	XCHC ; DUE TO THE POSSIBILITY OF OVERTHROTTLE
80E5	2A1A00	-1650	LHLD LHL ;
80E8	D801	-1651	IN ;
80EA	E6FC	-1652	ANI OFCH ;
80EC	321700	-1653	STA MUSIGN ;
80EF	7D	-1654	MOV A, L ; FIRSTLY, FOR SLEW



LOC	OBJ	LINE	SOURCE STATEMENT	
80F0	93	=1655	SUB E	
80F1	D2F680	=1656	JNC \$+5	
80F4	2F	=1657	CMA A	
80F5	3C	=1658	INR A	
80F6	FE04	=1659	CPI 04H	
80F8	D20381	=1660	JNC FCONT1	‡ TOLERANCE IS SET TO 4 BITS
80FB	3A1700	D =1661	LDA MDSIGN	‡ CONTINUE IF DIFFERENCE IS LARGER
80FE	F601	=1662	ORI 01H	‡ SET SLOW SLEW
8100	321700	D =1663	STA MDSIGN	‡ THEN CHECK FOR RAISE
8103	7C	=1664	MOV A,H	
8104	92	=1665	SUB D	
8105	D20AB1	=1666	JNC \$+5	
8108	2F	=1667	CMA A	
8109	3C	=1668	INR A	
810A	FE04	=1669	CPI 04H	
810C	D21781	=1670	JNC FCONT	
810F	3A1700	D =1671	LDA MDSIGN	‡ LIMIT LIFTING TO SLOW MODE
8112	F602	=1672	ORI 02H	
8114	321700	D =1673	STA MDSIGN	
8117	3A1700	D =1674	LDA MDSIGN	‡ CHECK IF THE SLOW MODE HAS BEEN SET
811A	F6FC	=1675	ORI 0FCH	
811C	FEFF	=1676	CPI 0FFH	‡ JUMP TO SLOW MODE IF YES
811E	CAEE05	=1677	JZ FMOVES	
8121	3A0200	D =1678	LDA SORTFL	‡ CHECK IF THE PROFILE DATA HAS BEEN SORTED AND EXPANDED
8124	17	=1679	RAL	
8125	D26683	=1680	JNC NOSORT	‡ JUMP IF THE PROFILE DATA HAS NOT BEEN SORTED
8128	2A0600	D =1681	LHLD LASTCV	‡ OTHERWISE, LIMIT THE DESIRED POSITION TO THE BOUNDS
812B	EB	=1682	XCHG	
812C	2A0400	D =1683	LHLD STARC	
812F	3A1B00	D =1684	LDA KSDCSR	
8132	BE	=1685	CMP M	
8133	D23781	=1686	JNC \$+4	‡ FIRST FOR THE RAISE DATA
8135	7E	=1687	MOV A,M	‡ FOR LOWEST LIMIT
8137	EB	=1688	XCHG	
8138	BE	=1689	CMP M	
8139	DA3081	=1690	JC \$+4	‡ FOR UPPER MOST LIMIT
813C	7E	=1691	MOV A,M	
813D	321B00	D =1692	STA RSDCSR	‡ STORE THIS ADJUSTED SETTING
8140	6F	=1693	MOV L,A	‡ NOW CHECK FOR THE SLEW BOUNDS
8141	2600	=1694	MVI H,000H	
8143	29	=1695	DAD H	
8144	29	=1696	DAD H	
8145	1100EC	=1697	LXI D,STADD	
8148	19	=1698	DAD D	
8149	29	=1699	INX H	
814A	3A1A00	D =1700	LDA SLDESR	‡ POINT TO THE LEFTMOST SLEW BOUND
814D	BE	=1701	CMP M	‡ GET THE DESIRED SLEW POSITION
814E	D25281	=1702	JNC \$+4	‡ SKIP IF GREATER
8151	7E	=1703	MOV A,M	
8152	29	=1704	INX H	
8153	BE	=1705	CMP M	‡ POINT TO THE RIGHTMOST SLEW BOUND
8154	DA5881	=1706	JC \$+4	‡ SKIP IF SMALLER
8157	7E	=1707	MOV A,M	
8158	321A00	D =1708	STA SLDESR	‡ STORE THIS ADJUSTED SETTING
815B	3A2400	D =1709	LDA RSCRRT	‡ CHECK IF CURRENT BOOM POSITION IS IN THE NEIGHBOURHOOD

LOC	OBJ	LINE	SOURCE STATEMENT
815E	D610	=1710	SUI 010H
8160	DAB81	=1711	JC \$+10D
8163	2A0+00	D =1712	LHLD STARCVC
8166	BE	=1713	CMP M
8167	D27481	=1714	JNC \$+13D
816A	F5	=1715	PUSH PSW
816B	3A1700	D =1716	LDA MDSIGN
816E	F602	=1717	ORI 02H
8170	521700	D =1718	STA MDSIGN
8173	F1	=1719	POP PSW
8174	C61F	=1720	ADI \$+10D
8176	DAB081	=1721	JC LASTCV
8179	2A0600	D =1722	LHLD M
817C	BE	=1723	CMP \$+11D
817D	DAB981	=1724	JC MDSIGN
8180	3A1700	D =1725	LDA 02H
8183	F602	=1726	ORI MDSIGN
8185	321700	D =1727	STA MDSIGN
8188	3A2400	D =1728	LDA RSCRRT
818B	6F	=1729	MOV L,A
818C	2600	=1730	MVI H,00H
818E	29	=1731	H H
818F	29	=1732	DAD D
8190	11006C	=1733	LXI D,STADD
8193	19	=1734	DAD D
8194	23	=1735	INX H
8195	3A2300	D =1736	LDA SLCRRT
8198	D610	=1737	SUI 010H
819A	BE	=1738	CMP M
819B	DAEE05	=1739	JC FMOVES
819E	C61F	=1740	ADI 01FH
81A0	DAEE05	=1741	JC FMOVES
81A3	23	=1742	H H
81A4	BE	=1743	CMP M
81A5	D2EE05	=1744	FMOVES
81A8	3A1700	D =1745	LDA MDSIGN
81AB	D301	=1746	OUT 01H
81AD	211000	=1747	LXI H,16D
81B0	F5	=1749	PUSH H
81B1	CD+000	=1750	CALL SETLINK
81B4	01ES1B	=1751	LXI B,SLVEL
81B7	112500	D =1752	LXI D,VELSL
81BA	CD9700	=1753	CALL TRANSF
81BD	3AA21B	=1754	LDA CRTSL
81C0	322300	D =1755	STA SLCRRT
81C3	3AAD1B	=1756	LDA CRTTRS
81C6	322400	D =1757	STA RSCRRT
81C9	CD5200	=1758	CALL OFFLINK
81CC	E1	=1759	H H
81CD	CD980	=1760	CALL RSTAPU
81D0	011B00	D =1761	LXI B,RSEDES
81D3	CD+280	=1762	CALL SEND10
81D6	2bFD	=1763	MVI H,APUC
81D8	361D	=1764	MVI M,FLTS

; OF THE PROFILE BOUNDARY  
 ; CHECK IF IT IS CLOSE TO THE LOWEST BOUND  
 ; IF YES, SET SLOW LIFT MODE  
 ; OTHERWISE, CHECK FOR THE UPPER MOST BOUND  
 ; IF HIGHER, THEN SET SLOW LIFT MODE  
 ; RAISE IS AWAY FROM THE ALTITUDE LIMITS  
 ; CHECK FOR THE SLEW BOUNDS  
 ; POINT H,L TO THE LEFTMOST SLEW BOUND  
 ; GET THE CURRENT SLEW DATA  
 ; COMPARE WITH THE LEFTMOST SLEW BOUND  
 ; JUMP TO SLOW MOVE IF WITHIN  
 ; CHECK THE RIGHTMOST BOUND  
 ; JUMP TO SLOW MOVE IF WITHIN  
 ; NOW THE CURRENT BOOM POSITION IS WITHIN THE 'FREE' AREA  
 ; GET THE CURRENT SPEED MODE SETTING  
 ; SET THE SPEED MODE TO HIGH  
 ; LOAD THE COUNTER FOR BYTE TRANSFER  
 ; SET UP THE INTERFACE LINK  
 ; SET THE STARTING SOURCE ADDRESS  
 ; SET THE STARTING DESTINATION ADDRESS  
 ; CALL THE BYTE TRANSFER ROUTINE  
 ; RELEASE THE LINK  
 ; RESET THE APU  
 ; GET THE DESIRED RAISE POSITION  
 ; SEND THIS TO THE APU DATA STACK  
 ; LOAD THE AUP COMMAND PORT ADDRESS  
 ; FLOAT THIS INTEGER

LOC	OBJ	LINE	SOURCE STATEMENT
81DA	3A1E00	D =1765	LDA RSDSR
81DD	47	=1766	MOV B,A
81DE	3A2400	D =1767	LDA RSCRRT
81E1	90	=1768	B B
81E2	9F	=1769	SBB A
		=1770	
81E3	325100	D =1771	STA FNSIGN
81E6	014D00	D =1772	LX1 B,DUMMY1
81E9	CDA780	=1773	CALL CHKBSY
81EC	CD1980	=1774	CALL READ4
81EF	012400	D =1775	LX1 B,RSCRRT
81F2	CD4280	=1776	CALL SEND10
81F5	361D	=1777	MVI M,FLTS
81F7	014D00	D =1778	LX1 B,DUMMY1
81FA	CJA780	=1779	CALL CHKBSY
81FD	CD0080	=1780	CALL SEND4
8200	3611	=1781	MVI M,FSUB
8202	26FD	=1782	MVI H,APUC
8204	012D00	D =1783	LX1 B,VELRS
8207	CD4C80	=1784	CALL CONVER
820A	014500	D =1785	LX1 B,DIFF1
820D	CDA780	=1786	CALL CHKBSY
8210	CD1980	=1787	CALL READ4
8213	012D00	D =1788	LX1 B,VELRS
8216	CD0080	=1789	CALL SEND4
8219	01D583	=1790	LX1 B,TIMECV
821C	CD0080	=1791	CALL SEND4
821F	3612	=1792	MVI M,FMUL
8221	015100	D =1793	LX1 B,ACCRS
8224	CD4C80	=1794	CALL CONVER
8227	014500	D =1795	LX1 B,DIFF1
822A	CDA780	=1796	CALL CHKBSY
822D	CD0080	=1797	CALL SEND4
8230	3610	=1798	MVI M,FADDD
8232	01D983	=1799	LX1 B,CONSTV
8235	CDA780	=1800	CALL CHKBSY
8238	CD0080	=1801	CALL SEND4
823B	3610	=1802	MVI M,FADD
823D	014800	D =1803	LX1 B,SWFN
8240	CDA780	=1804	CALL CHKBSY
8243	CD1980	=1805	CALL READ4
8246	3A4C00	D =1806	LDA SWFN+3
8249	17	=1807	RAL
824A	9F	=1808	SBB A
		=1809	
824B	215100	D =1810	LX1 H,FNSIGN
824E	BE	=1811	CMPL M
824F	C26082	=1812	JNZ STOPRS
8252	B7	=1813	ORA A
8253	C25B82	=1814	JNZ \$+8
8256	3EF6	=1815	MVI A,0F6H
8258	C36282	=1816	JMP \$+10D
825B	3EF9	=1817	MVI A,0F9H
825D	C36282	=1818	JMP \$+5
8260	3EFF	=1819	MVI A,0FFH

STOPRS:

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; B <--- RSDSR
; A <--- RSCRRT
; RSCRRT - RSDSR
; A = 000H IF RSCRRT >= RSDSR
; A = 0FFH IF RSCRRT < RSDSR
; STORE THIS FUNCTION SIGN
; LOAD ADDRESS FOR THE RESULT
; FORM THE DIFFERENCE BETWEEN THE CURRENT AND 040H
;
; PERFORM (CURRENT SLEW - DESIRED SLEW)
; LOAD THE APU COMMAND PORT ADDRESS
; CONVERT THE VELOCITY INTO APU BFP FORMAT
; STORE (CURRENT RAISE - DESIRED RAISE)
;
; SEND THE VELOCITY TO THE APU DATA STACK
; SEND THE COEFFICIENT (TIMECV) TO APU STACK
; PERFORM (VELRS * TIMECO)
; CONVERT THE ACCELERATION INTO APU BFP
;
; SEND (CURRENT - DESIRED) TO APU
; FIND (VELRS*TIMECV + [CURRENT - DESIRED])
;
; FIND (VELRS*TIMECV + [CURRENT - DESIRED] + CONSTV)
;
; READ AND STORE THIS
;
; A = 0FFH IF EVALUATED FUNCTION IS -VE
; A = 000H IF EVALUATED FUNCTION IS +VE
; POINT H,L TO FNSIGN FOR COMPARISON
; STOP IF NOT EQUAL
; CHECK WHAT SIGNAL IS REQUIRED
;
; SET THE LOWER DOWN SIGNAL
; SET THE RAISE UP SIGNAL
; SET THE NO-LIFT SIGNAL

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LOC	06:J	LINE	SOURCE STATEMENT	
8262	320300	D =1820	SIGNAL	↑ STORE THIS SIGNAL
8265	3A2400	D =1821	RSCRRT	↑ GET THE CURRENT RAISE
8268	4F	=1822	C, A	
8269	3A1800	D =1823	RSDESR	↑ GET THE DESIRED RAISE
826C	91	=1824	C	↑ FIND THE DIFFERENCE
826D	D27282	=1825	*+5	
8270	2F	=1826	CMA	↑ FORM THE ABSOLUTE DIFFERENCE
8271	5C	=1827	INR	
8272	FE02	=1828	CPI	
8274	D27B82	=1829	JNC	↑ ENSURE NO-LIFT IF WITHIN THE TOLERANCE
8277	9F	=1830	SBB	
827B	320300	D =1831	SIGNAL	↑ GET THE DESIRED SLEW POSITION
827B	011A00	D =1832	B, SLDESR	↑ SEND THIS TO THE APU DATA STACK
827E	CD4280	=1833	CALL	↑ LOAD THE AUP COMMAND PORT ADDRESS
8281	26FD	=1834	MVI	↑ FLOAT THIS INTEGER
8283	361D	=1835	M, FLTS	↑ B <--- SLDESR
8285	3A1A00	D =1836	LDA	
8288	47	=1837	MOV	
8289	3A2300	D =1838	SBCRRT	↑ A <--- SLCRRT
828C	90	=1839	SUB	↑ SLCRRT - SLDESR
828D	9F	=1840	SBB	↑ A = 000H IF SLCRRT >= SLDESR
		=1841	A	↑ A = 0FFH IF SLCRRT < SLDESR
828E	325100	D =1842	FNSIGN	↑ STORE THIS FUNCTION SIGN
8291	014D00	D =1843	B, DUMMY1	↑ LOAD ADDRESS FOR THE RESULT
8294	CD4780	=1844	CALL	
8297	CD1980	=1845	CALL	
829A	012300	D =1846	B, SLCRRT	↑ FORM THE DIFFERENCE BETWEEN THE CURRENT AND 040H
829D	CD4280	=1847	CALL	
82A0	361D	=1848	MVI	
82A2	014D00	D =1849	LXI	
82A5	CD4780	=1850	CALL	
82A8	CD0080	=1851	CALL	
82AB	3611	=1852	SEND4	
82AD	012500	D =1853	M, FSVB	↑ PERFORM (CURRENT SLEW - DESIRED SLEW)
82B0	CD4C80	=1854	B, VELSL	↑ CONVERT THE VELOCITY INTO APU BFP FORMAT
82B3	014500	D =1855	CONVER	
82B6	CD4780	=1856	LXI	↑ STORE (CURRENT SLEW - DESIRED SLEW)
82B9	CD1980	=1857	B, DIFF1	
82BC	012500	D =1858	CALL	
82BF	CD0080	=1859	CALL	
82C2	01CDB3	=1860	B, TIMECH	↑ SEND THE VELOCITY TO THE APU DATA STACK
82C5	CD0080	=1861	CALL	↑ SEND THE COEFFICIENT (TIMECH) TO APU STACK
82C8	3612	=1862	SEND4	↑ PERFORM (VELSL * TIMECH)
82CA	01D183	=1863	M, FMUL	
82CD	CD4780	=1864	LXI	
82D0	CD0080	=1865	B, CONSTH	
82D3	3610	=1866	CALL	
82D5	012500	D =1867	M, FADD	↑ CONVERT THE ACCELERATION INTO APU BFP
82D8	CD4C80	=1868	B, ACCSL	
82DB	014500	D =1869	CONVER	
82DE	CD4780	=1870	LXI	
82E1	CD0080	=1871	B, DIFF1	
82E4	3610	=1872	CALL	↑ SEND (CURRENT - DESIRED) TO APU
82E6	014900	D =1873	M, FADD	↑ FIND (VELSL*TIMECO + (CURRENT - DESIRED))
82E9	CD4780	=1874	LXI	
			CALL	

LOC	OBJ	LINE	SOURCE STATEMENT
82EC	CD1980	=1875	CALL READ4
82EF	3A4C00	D =1876	LDA SWFN+3
82F2	17	=1877	RAL
82F3	9F	=1878	SEB A
		=1879	
82F4	215100	D =1880	LXI H, FNSIGN
82F7	BE	=1881	M H, SIGNAL
82F8	210300	D =1882	LXI H, SIGNAL
82FB	C20C83	=1883	JNZ STOPSL
82FE	B7	=1884	ORA A
82FF	C207B9	=1885	JNZ \$+8D
8302	3E6F	=1886	MVI A, 06FH
8304	C30E83	=1887	JMP \$+10D
8307	3E9F	=1888	MVI A, 09FH
8309	C30E89	=1889	JMP \$+5D
830C	3EFF	=1890	MVI A, 0FFH
830E	A6	=1891	M A
830F	77	=1892	M A
8310	3A2300	D =1893	LDA SLCRRT
8313	4F	=1894	MOV C, A
8314	3A1A00	D =1895	LDA SLDES R
8317	91	=1896	SUB C
8318	D21D83	=1897	JNC \$+5
831B	2F	=1898	CMA
831C	5C	=1899	INR
831D	FE02	=1900	CPI A
831F	D22683	=1901	JNC DEBUG2
8322	3EF0	=1902	MVI A, 0F0H
8324	86	=1903	ORA M
8329	77	=1904	MOV M, A
8326	7E	=1905	MOV A, M
8327	6BF0	=1906	ORI 0F0H
8329	5C	=1907	INR A
832A	CA3383	=1908	JZ \$+9D
832D	3D	=1909	DCR A
832E	D300	=1910	OUT 00H
8330	C33689	=1911	JMP STOFF1
8333	7E	=1912	MOV A, M
8334	D300	=1913	OUT 00H
8336	FEFF	=1914	CPI 0FFH
8338	CA4083	=1915	JZ STOPMV
833B	C1	=1916	POP B
833C	D1	=1917	POP D
833D	E1	=1918	POP H
833E	F1	=1919	POP PSW
833F	C9	=1920	RET
8340	DE01	=1921	STOPMV: IN 001H
8342	FE03	=1922	ORI 003H
8344	321700	D =1923	STA MDSIGN
8347	D301	=1924	OUT 01H
8349	21FFFF	=1925	LXI H, 0FFFFH
834C	97	=1926	SUB A
834D	2B	=1927	DCX H
834E	B4	=1928	ORA H
834F	B5	=1929	ORA L

```

; READ AND STORE THIS
;
; A = 0FFH IF EVALUATED FUNCTION IS -VE
; A = 000H IF EVALUATED FUNCTION IS +VE
; POINT H, L TO FNSIGN FOR COMPARISON
; COMPARE TO SEE IF SLEW MOVEMENT IS REQUIRED
; POINT H, L TO THE PREVIOUS LIFTING SIGNAL SET
; STOP IF NOT EQUAL
; CHECK WHAT SIGNAL IS REQUIRED
;
; SET THE SLEW LEFT SIGNAL
;
; SET THE SLEW RIGHT SIGNAL.
;
; SET THE NO-SLEW SIGNAL
; FORM THE RESULTANT SIGNAL.
;
; GET THE CURRENT SLEW
;
; GET THE DESIRED SLEW
; FORM THE DIFFERENCE
;
; FORM THE ABSOLUTE DIFFERENCE
;
; COMPARE WITH THE TOLERANCE
; ENSURE NO-SLEW IF WITHIN THE BAND
; FORM THE COMBINED SIGNAL 1111 XXXX
;
; CHECK IF LIFT SIGNAL HAS BEEN SET
; (I.E. PATTERN NOT XXXX 1111)
; SEND THE SIGNAL IF LIFT IS NOT REQUIRED
; OTHERWISE, SEND LIFT SIGNAL ONLY
;
; JUMP IF THE DIRECTIONAL VALVES ARE CLOSED
; OTHERWISE, RECOVER THE REGISTERS AND
;
; RETURN
; SET THE SLOW SPEED MODE
;
; STORE THE SPEED MODE SETTING
; SEND THIS TO THE VALVES
; INTRODUCE TIME DELAY FOR VALVE TO RESPOND

```

LOC	OBJ	LINE	SOURCE STATEMENT
8350	C24CB3	=1930	JNZ \$-4
8353	CD4000	=1931	SETLNK
8355	3AA21B	=1932	CRRTSL
8359	6F	=1933	LDA L,A
835A	3AAD1B	=1934	LDA CRTRH
835D	67	=1935	MOV H,A
835E	222500	D =1936	SLCRT
8361	3ADD1B	=1937	LDA SHLD
8364	321D00	D =1938	LDA CUTFOR
8367	CD5200	=1939	STA FORCE
836A	3A1D00	D =1940	CALL OFFLNK
836D	FE5	=1941	LDA FORCE
836F	D2AFB9	=1942	CPI OESH
8372	2A2500	D =1943	JNC STOPF2
8375	EB	=1944	LHLD SLCRT
8376	2A1A00	D =1945	LHLD SLDESR
8379	7B	=1946	MOV A,E
837A	95	=1947	L
837B	DAB3B3	=1948	JC \$+8
837E	0E6F	=1949	MVI C,06FH
8380	C3B7B3	=1950	JMP \$+7
8383	2F	=1951	CMA
8384	3C	=1952	INR
8385	0E9F	=1953	MVI C,09FH
8387	FE02	=1954	CPI 02H
8389	D28EB3	=1955	JNC \$+5
838C	0EFF	=1956	MVI C,0FFH
839E	7A	=1957	MOV A,D
83BF	94	=1958	SUB H
8390	DA98B3	=1959	JC \$+8
8393	06F6	=1960	MVI B,0F6H
8395	C39CB3	=1961	JMP \$+7
8398	2F	=1962	CMA
8399	3C	=1963	INR
839A	06F9	=1964	MVI B,0F9H
839C	FE02	=1965	CPI 02H
839E	D2A3B3	=1966	JNC \$+5
83A1	06FF	=1967	MVI B,0FFH
83A3	78	=1968	MOV A,B
83A4	A1	=1969	ANA C
83A5	D300	=1970	OUT 00H
83A7	FEFF	=1971	CPI 0FFH
83A9	C253B3	=1972	JNZ ADJUST
83AC	C398B3	=1973	JMP STOPF1
83AF	3EFF	=1974	MVI A,0FFH
83B1	D300	=1975	OUT 00H
83B3	C353B3	=1976	JMP ADJUST
83B6	DB01	=1977	IN 01H
83B9	E6FC	=1978	ANI 0FCH
83BA	C3F205	=1979	JMP SNOVEF
83BD	3EFF	=1980	MVI A,0FFH
83BF	D300	=1981	OUT 00H
83C1	DB01	=1982	IN 01H
83C3	F603	=1983	ORI 03H
83C5	321700	D =1984	STA MDSICN

ADJUST THE FINAL BOOM POSITION IN SLOW MODE  
GET THE CURRENT BOOM POSITION

STORE THIS IN MEMORY OF THIS SYSTEM  
GET THE CUTTER FORCE  
AND STORE THIS INFORMATION

CHECK IF THE CUTTER FORCE EXCEEDS THE UPPER LIMIT  
STOP IF THE FORCE IS TOO HIGH  
SET THE APPROPRIATE SIGNALS FOR THE DIRECTIONAL VALVES  
PUT THE CURRENT BOOM POSITION IN D,E  
PUT THE DESIRED BOOM POSITION IN H,L  
DETERMINE THE SLEW SIGNAL

SET THE SLEW LEFT SIGNAL

FIND THE ABSOLUTE DIFFERENCE  
SET THE SLEW RIGHT SIGNAL  
COMPARE WITH THE ADC TOLERANCE

SET THE NO-SLEW SIGNAL IF WITHIN THIS RANGE  
DETERMINE THE RAISE SIGNAL

SET THE LOWER DOWN SIGNAL

FIND THE ABSOLUTE DISTANCE BETWEEN THE CURRENT  
AND THE DESIRED RAISE POSITIONS  
SET THE RAISE UP SIGNAL  
COMPARE WITH THE ADC TOLERANCE

SET THE NO-RAISE SIGNAL IF WITHIN THE RANGE

FORM THE COMBINED SIGNAL  
SEND THIS SIGNAL TO OPERATE THE DIRECTIONAL VALVES  
CHECK IF THE DESIRED POSITION HAS BEEN REACHED  
LOOP BACK IF NOT  
JUMP TO RETURN IF YES  
SET NO-MOVE SIGNAL DUE TO HIGH CUTTING FORCE

LOOP BACK TO ADJUST THE FINAL POSITION  
THE PROFILE DATA HAS NOT BEEN SORTED  
SET THE FAST SPEED MODE  
JUMP TO CONTINUE

CLOSE ALL THE DIRECTIONAL VALVES

SET SLOW SPEED MODE

LOC	OBJ	LINE	SOURCE STATEMENT
83C8	D301	=1985	OUT 01H
83CA	C33E83	=1986	JMP STOPF1
83CD	99	=1987	TIMECH: DB 099H, 0F9H, 0E7H, 00FH
83CE	F9	=	
83CF	B7	=	
83D0	0F	=	
83D1	44	=1988	CONSTH: DB 044H, 01FH, 0EBH, 0FFH
83D2	1F	=	
83D3	EB	=	
83D4	FF	=	
83D5	00	=	
83D6	F6	=1989	TIMECV: DB 000H, 0F6H, 0E3H, 00FH
83D7	E3	=	
83D8	0F	=	
83D9	00	=	
83DA	00	=1990	CONSTV: DB 000H, 000H, 090H, 080H
83DB	80	=	
83DC	80	=	
18A2		=1991	#INCLUDE(:F1:DATA.SRC)
		=1992	*****
		=1993	DATA SEGMENT FOR THE BOOM POSITIONAL CONTROL PROGRAM *****
		=1994	*****
		=1995	PUBLIC CRTSL, CRRTS, MOVEST, DESRSL, DESRRS, PFLSND, PFLST, CUTTER
		=1996	PUBLIC SLVEL, SLACC, RSVEL, RSACC, STADD, LASTAD, DDRO1, TEMP, TEMP1, TEMPRI
		=1997	PUBLIC SLCRRT, RSCRRT, SLDESR, RSDESR, VELSL, VELRS, ACCSL, ACCRS, PREVSL, PREVRS
		=1998	PUBLIC SIMUX, SIMUY, SIMUAX, SIMUAY, SIMUAZ
		=1999	ASEC
		=2000	ORG 01BA2H
		=2001	*****
		=2002	SYMBOLS TO BE USED IN THE DATA CAPTURING SYSTEM *****
		=2003	*****
		=2004	SYMBOL
		=2005	*****
		=2006	CRRTSL: DS 1 ; CURRENT SLEW POSITION (1 BYTE)
		=2007	ORG 01BADH
		=2008	CRRTS: DS 1 ; CURRENT RAISE POSITION (1 BYTE)
		=2009	ORG 01BDAH
		=2010	DESRSL: DS 1 ; DESIRED SLEW POSITION (1 BYTE)
		=2011	DESRRS: DS 1 ; DESIRED RAISE POSITION (1 BYTE)
		=2012	MOVEST: DS 1 ; CURRENT MOVE LEVER STATUS (1 BYTE)
		=2013	CUTFOR: DS 1 ; CUTTER MOTOR CURRENT (1 BYTE)
		=2014	ORG 01BE3H
		=2015	PREVSL: DS 1 ; CURRENT SLEW POSITION (1 BYTE)
		=2016	PREVRS: DS 1 ; CURRENT RAISE POSITION (1 BYTE)
		=2017	SLVEL: DS 4 ; CURRENT SLEW VELOCITY (4 BYTES)
		=2018	SLACC: DS 4 ; CURRENT SLEW ACCELERATION (4 BYTES)
		=2019	RSVEL: DS 4 ; CURRENT RAISE VELOCITY (4 BYTES)
		=2020	RSACC: DS 4 ; CURRENT RAISE ACCELERATION (4 BYTES)
		=2021	ORG 1BFEH
		=2022	PFLSND: DS 2 ; LAST ADDRESS FOR PROFILE DATA (2 BYTES)
		=2023	PFLST: DS 400H ; STARTING ADDRESS FOR PROFILE DATA (MAX. 400H BYTES)
		=2024	ORG 0BBFEH
		=2025	*****
		=2026	SYMBOLS TO BE USED IN THE MAIN CONTROL SYSTEM PROGRAM *****
		=2027	*****
18A2			
18A2			
18A3			
18A4			
18A5			
18A6			
18A7			
18A8			
18A9			
18AA			
18AB			
18AC			
18AD			
18AE			
18AF			
18B0			
18B1			
18B2			
18B3			
18B4			
18B5			
18B6			
18B7			
18B8			
18B9			
18BA			
18BB			
18BC			
18BD			
18BE			
18BF			
18C0			
8BFE			

LOC	OBJ	LINE	SOURCE STATEMENT	DEFINITION
		-2028	SYMBOL	-----
BFE		-2029	I-----	
BC00		-2030	LASTAD: DS 2	LAST ADDRESS FOR THE PROFILE (2 BYTES)
		-2031	STADD: DS 400H	DATA STORAGE FOR THE PROFILE (MAX. 400H BYTES)
		-2032	DSEC	DEFINE DATA SEGMENT
0000		-2033	STKPTR: DS 0	DEFINE THE HIGHEST STACK BYTE TO BE THE ONE BELOW
		-2034		THE LOWEST DATA MEMORY BYTE
0000		-2035	INTMSK: DS 1	STORAGE LOCATION FOR THE INTERRUPT MASK
0001		-2036	CUTSTP: DS 1	LOCATION FOR THE CUTTER STEP SIZE IN ADVANCING
0002		-2037	SORTFL: DS 1	FLAG FOR SORTED PROFILE DATA
		-2038		00 --- NOT YET SORTED
		-2039		FF --- SORTING COMPLETED
0003		-2040	SIGNAL: DS 1	SIGNAL FOR THE DIRECTIONAL VALUES
0004		-2041	STARCV: DS 2	DEFINE THE STORAGE FOR STARTING CAVITY 'RAISE' LEVEL
0006		-2042	LASTCV: DS 2	DEFINE THE STORAGE FOR THE LAST 'CAVITY' LEVEL
0008		-2043	LASTSO: DS 2	DEFINE THE STORAGE FOR THE LAST SORTED CAVITY ADDRESS
000A		-2044	LEVEL: DS 2	DEFINE THE STORAGE FOR THE ADDRESS FOR 'X'
000C		-2045	DMIN1: DS 2	DEFINE THE STORAGE FOR TEMPORARY MIN
000E		-2046	DMAX1: DS 2	DEFINE THE STORAGE FOR TEMPORARY MAX
0010		-2047	FIRSL: DS 2	TEMPORARY STORAGE FOR THE NEXT TWO DESIRED SLEW VALUES
0012		-2048	FOURST: DS 2	TEMPORARY STORAGE FOR 4*(CUTTER STEP SIZE)
0014		-2049	LASTCT: DS 2	STORAGE FOR THE ADDRESS OF THE LAST DATA BLOCK IN CAVITY CUTTING
0016		-2050	HALFST: DS 1	STORAGE FOR HALF CUTTER STEP SIZE
0017		-2051	MDSIGN: DS 1	STORAGE FOR THE BOOM MOVING SPEED MODE
0018		-2052	DDR01: DS 1	DATA DIRECTION REGISTER FOR PORT 1
0019		-2053	CUTTER: DS 1	CUTTER STATUS --- 00=STOPPED, OFFH=STARTED
001A		-2054	SLDESR: DS 1	DESIRED SLEW POSITION (1 BYTE)
001B		-2055	RSDESR: DS 1	DESIRED RAISE POSITION (1 BYTE)
001C		-2056	STMOVE: DS 1	'MOVE' CONTROL STATUS (1 BYTE)
001D		-2057	FORCE: DS 1	CUTTER MOTOR CURRENT (1 BYTE)
001E		-2058	SIMUX: DS 1	SIMULATED X DISPLACEMENT (1 BYTE)
001F		-2059	SIMUY: DS 1	SIMULATED Y DISPLACEMENT (1 BYTE)
0020		-2060	SIMUAX: DS 1	SIMULATED ALPHA X (1 BYTE)
0022		-2062	SIMUAZ: DS 1	SIMULATED ALPHA Z (1 BYTE)
0023		-2063	SLCRRT: DS 1	SIMULATED ALPHA Z (1 BYTE)
0024		-2064	RSCRRT: DS 1	CURRENT SLEW POSITION (1 BYTE)
0025		-2065	VELSL: DS 4	CURRENT RAISE POSITION (1 BYTE)
0029		-2066	ACCSL: DS 4	CURRENT SLEW VELOCITY (4 BYTES)
002D		-2067	VELRS: DS 4	CURRENT SLEW ACCELERATION (4 BYTES)
0031		-2068	ACCRS: DS 4	CURRENT RAISE VELOCITY (4 BYTES)
0035		-2069	TEMP: DS 4	CURRENT RAISE ACCELERATION (4 BYTES)
0039		-2070	TEMP1: DS 4	TEMPORARY STORAGE LOCATION (4 BYTES)
003D		-2071	TEMPR: DS 4	TEMPORARY STORAGE FOR RESULT (4 BYTES)
0041		-2072	TEMPRI: DS 4	
0045		-2073	DIFF1: DS 4	DIFFERENCE BETWEEN CURRENT AND DESIRED (4 BYTES)
0049		-2074	SWFN: DS 4	SWITCHING FUNCTION VALUE (4 BYTES)
004D		-2075	DUMY1: DS 4	
0051		-2076	FNSIGN: DS 1	EXPECTED SIGN FOR SWITCHING (1 BYTE)
		-2077	*****	*****
		-2078	*****	*****
		-2079	*****	*****
		-2080	\$EJECT	*****



LOC OBJ LINE SOURCE STATEMENT  
 =2091 END

PUBLIC SYMBOLS  
 ACCRS D 0031 ACCSL D 0029 CAVITY A 0218 CHKBSY A 80A7 CONVER A 804C CRRTSL A 1BAD  
 CUTSTP D 0001 CUTTER D 0019 DDR01 D 0018 DESRSL A 1BDA LASTAD A 1BDA CRRTS A 1BAD  
 FIRSL D 0010 FMOVE A 00AE FURST D 0012 INTMSK D 0000 LASTAD A 1BDA LASTCV D 000C  
 LEVEL D 000A MOVE A 02DC MOVEST A 1BDC OFFLANK A 1BFE PFLST A 1C00 PREVRS A 1BE4  
 PREVSL A 1BE3 PROFIL A 00B5 READ2 A 802E READ4 A 8019 RSACC A 1BF1 RSCRT D 001B  
 RST45 A 032B RSTAPU A 8098 RSVEL A 1BED RYVEL A 1BED SEMD10 A 8042 SEND2 A 8012 SEND30 A 8035 SEND4 A 8000  
 SETLNK A 0040 SHORT A 0578 SIGNAL D 0003 SIMUAX D 0020 SIMUAX D 0022 SIMUAX D 001E SIMUAX D 001E  
 SIMUY D 001F SLACC A 1BE9 SLCRT D 0023 SLDESR D 001A SLVEL A 1BE5 SMOVE A 05EA SMOVE1 A 0740  
 SMOVEF A 05F2 SORT A 03DD SORTFL D 0002 STADD A 8C00 STKPTR D 0000 STKPTR D 0000 TEMP D 0035  
 TEMPI D 0039 TEMPR D 003D TEMPRI D 0041 TRANSF A 0097 TRNPFEL A 0075 TRNPFEL A 0004 VELSL D 0025

EXTERNAL SYMBOLS  
 USER SYMBOLS  
 ACCRS D 0031 ACCSL D 0029 AC05 A 0006 ADJUST A 8359 ANYM0D A 060D APUC A 00FD APUD A 00FC  
 ASIN A 0005 CALEXP A 03C5 CALSMV A 0266 CAVITY A 0218 CCKB1 A 06A5  
 ASIN A 0005 CALEXP A 03C5 CALSMV A 0266 CAVITY A 0218 CCKB1 A 06A5  
 CHKB2 A 06A4 CHKB4 A 06B6 CLOS1 A 0696 CLOS2 A 067E CLOS3 A 067E CLOS4 A 067E CLOS5 A 067E  
 CHKB3 A 06B5 CLOS4 A 067E CLOS5 A 067E CLOS6 A 067E CLOS7 A 067E CLOS8 A 067E  
 CHSS A 0074 CLOS1 A 0696 CLOS2 A 067E CLOS3 A 067E CLOS4 A 067E CLOS5 A 067E  
 CHSVA A 83D9 CONT11 A 041A CONT12 A 0486 CONT13 A 0486 CONT14 A 0486 CONT15 A 0486  
 CONT31 A 0422 CONT32 A 045E CONT33 A 045E CONT34 A 045E CONT35 A 045E CONT36 A 045E  
 CONT62 A 047B CONT63 A 047B CONT64 A 047B CONT65 A 047B CONT66 A 047B CONT67 A 047B  
 CONVER A 804C CUTTER D 0019 CUTTER D 0019 CUTTER D 0019 CUTTER D 0019 CUTTER D 0019  
 CUTFOK A 1BDD DIFF1 D 0045 DIFF2 D 0045 DIFF3 D 0045 DIFF4 D 0045 DIFF5 D 0045  
 DESRSL A 1BDA DUMY1 D 004D DUMY2 D 004D DUMY3 D 004D DUMY4 D 004D DUMY5 D 004D  
 DOWN1 A 066A EXPA2 A 059D EXPA3 A 0541 EXPA4 A 054E EXPA5 A 054E EXPA6 A 054E  
 FAST1 A 01EF FAST2 A 06E1 FAST3 A 06E9 FAST4 A 06E9 FAST5 A 06E9 FAST6 A 06E9  
 FC0NT1 A 8103 FIRSL D 0010 FIRSL D 0010 FIRSL D 0010 FIRSL D 0010 FIRSL D 0010  
 FMOVE A 80AE FMUL A 0012 FMUL A 0012 FMUL A 0012 FMUL A 0012 FMUL A 0012  
 HALFST D 0016 FMUL A 0012 FMUL A 0012 FMUL A 0012 FMUL A 0012 FMUL A 0012  
 LEVEL D 000A LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 LOOP3 A 00A5 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 NOHUP A 04E6 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 NOVER1 A 06FC LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 OVER1 A 0133 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 PREVRS A 1BE4 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 PWR A 000B LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 REDU2 A 05E1 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 RSCMP A 0638 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 RST5 A 003C LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 SEND2 A 8012 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 SHORT A 0578 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 SIN A 0002 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 SMOVE2 A 0742 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 SORT A 0001 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 STOP1 A 0209 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 STOP1V A 8340 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 TEMPR D 003D LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 UNDER1 A 0182 LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA  
 VERT5 A 071C LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA LASTAD A 1BDA

APPENDIX I

Brief Description of the Hydraulic Circuit  
for the Boom Ripper under Test

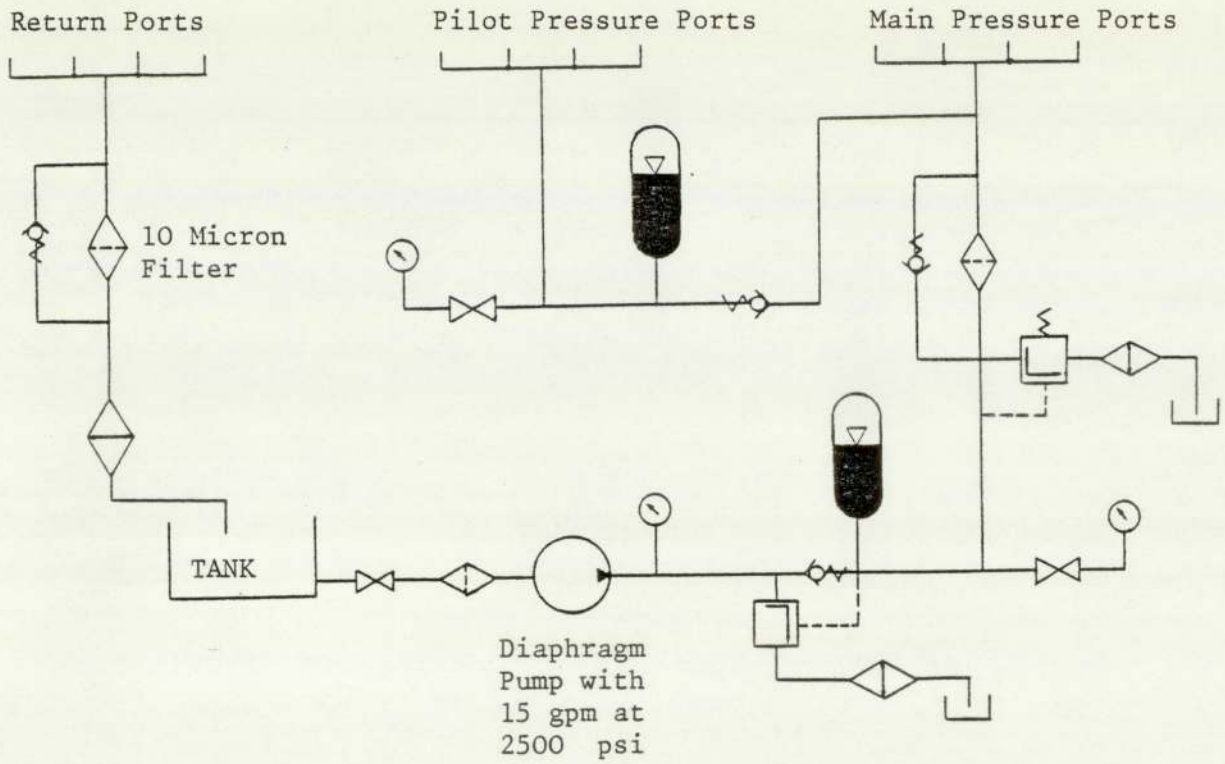


Figure I.1 Schematic Diagram for the Hydraulic Power Supply Circuit

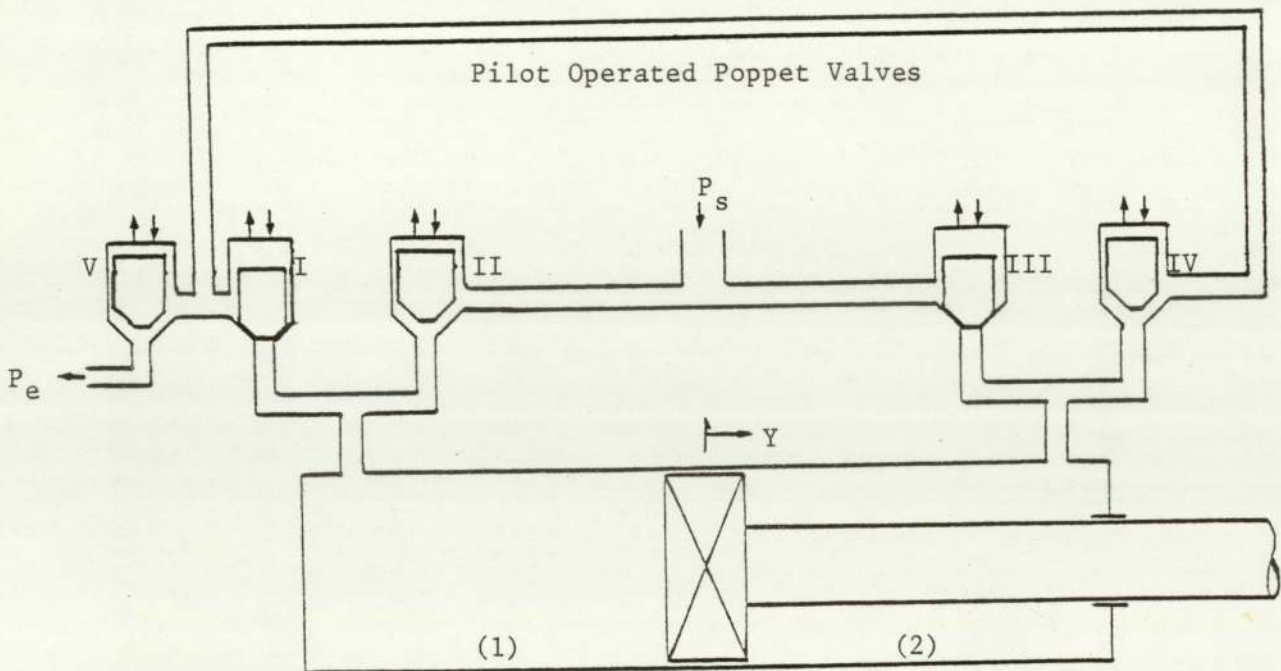


Figure I.2 Arrangement for One of the Hydraulic Actuators  
in the Boom Ripper under Test

The boom ripper under test is connected to a hydraulic power pack which delivers the 5/95 diluted emulsion at a flow rate of 1.135 litres/sec with a system supply pressure of 172.4 bars using a diaphragm pump. Fig I.1 shows the schematic diagram for this hydraulic power supply circuit.

Figure I.2 shows the arrangement of one of the hydraulic actuators used in the test rig. Only the main poppets are shown in this diagram, they are driven by their pilot poppet valves, which are operated on the amplified electrical signals from the microprocessor control system (please refer to Fig 6.2).

Four of these valves are the directional control valves: this actuator is moved towards right by opening the valve-pair (II) and (IV), while it will be forced to go left by opening the poppet-pair (I) and (III). The poppet (V) is the metering out control valve, which is also pilot-valve operated. When the directional control valves are opened, the poppet (V) can be operated in one of the following two modes: (i) it can be fully opened to provide the maximum flow area so that the actuator will move at the high speed, or (ii) this can be closed to leave a small opening to reduce the fluid flow rate by this metering out technique for moving the hydraulic actuator at a slow speed.

All the four hydraulic actuators are connected in the same way as described in the above paragraph. The paired actuators for the boom arcing or lifting share the same

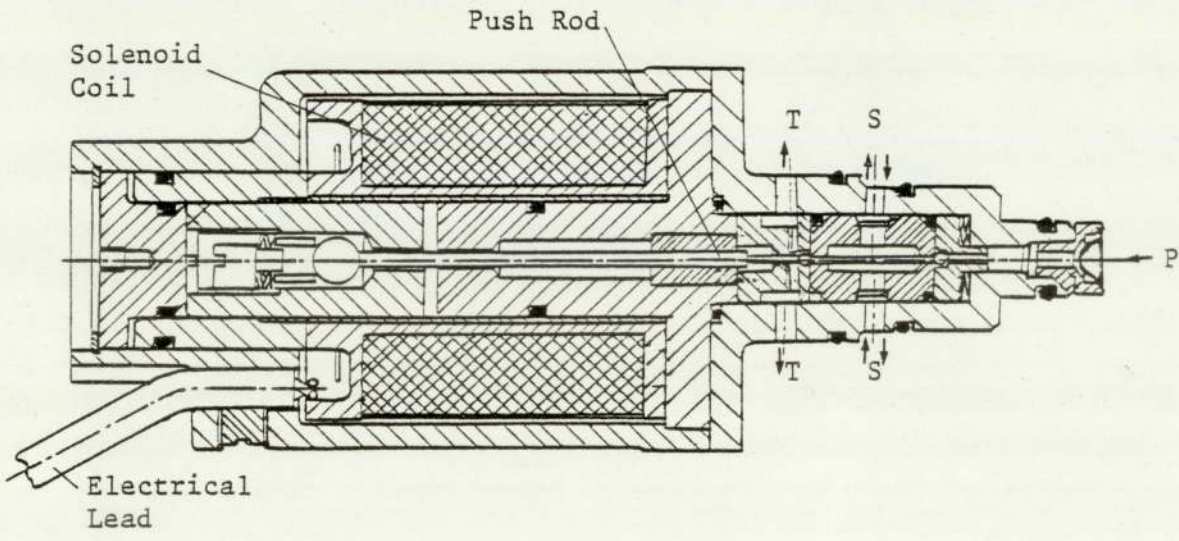


Figure I.3 Cross-section of the Solenoid Operated Pilot Valve used in the System under Test

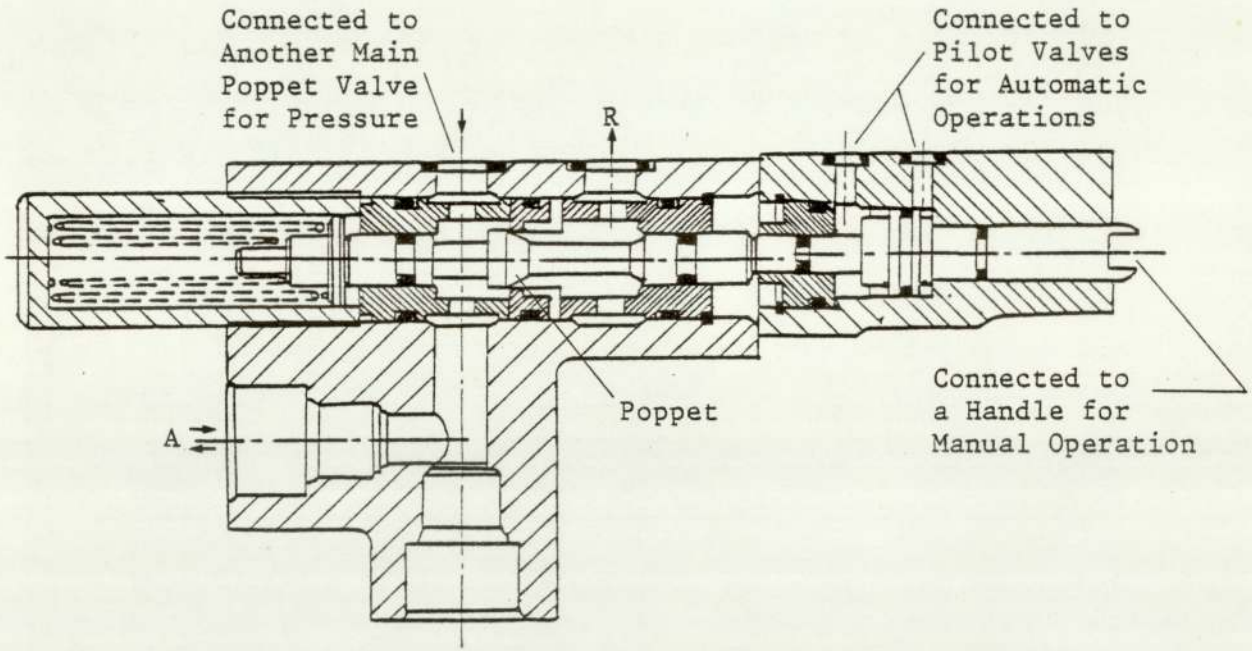


Figure I.4 Cross-section of the Main Poppet Valve used in the System under Test

set of directional control and metering out valves, so that only five pilots are operated to provide the desired boom movement in either slewing or lifting.

Figure I.3 shows the cross-section of one of the solenoid operated pilot valves used in the system under test, while Figure I.4 shows the cross-section of one of the main poppets used. When the solenoid of the pilot valve is not operated, the push rod is brought to its leftmost position by the spring, the ball valve between P and S is forced to rest on its seat by the supply pressure at P, but the ball valve between S and T is opened, therefore the fluid will flow from S to T. If this solenoid is energised, the push rod will be pulled to the right to reduce the gap by the electro-magnetic force generated. This action causes the ball valve between S and T to close with that between P and S to open. Therefore, the fluid will then flow from P to S. The S of this pilot is connected to operate the main poppet. If the pilot solenoid is de-energised, the main poppet will be forced to rest on its seat by its spring, therefore the fluid will flow between A and the paired poppet. When the pilot solenoid is energised, the poppet will be lifted off its seat by the pilot pressure and the fluid flows between A and R. The R in one of the paired poppets is connected to the system supply with R in the other one connected to the system exhaust.

The performance of these valves is being studied in a parallel project by J H Knight at Aston University. The Solenoid takes time to establish the magnetic flux when

energised and to remove the residual electro-magnetism when it is de-energised. Therefore there is a time delay in the pilot operation. This time delay can be shortened by reducing the mass of the moving parts in the solenoid. It is found that the time delay in the main poppet operation is mainly due to that in the pilot stage.